



UNIVERSITY OF CALIFORNIA
OBSERVATORIES

The Mechanical Design of HIRES

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Chapter 1

The Mechanical Design of HIRES

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1.1 Abstract

The Spectrograph design was begun in 1987 and published as "HIRES Phase C: Core Proposal" in 1988 (UCO/L.O. Technical Report No. 57). The design has continuously evolved: the latest change was to accommodate the curved CCD from Tektronics (Epps' design #4052 of 6/3/92: 65" convex radius CCD). The instrument will be shipped to Hawaii by mid-1993.

1.2 Introduction

The mechanical design was done by Jack Osborne, Bruce Bigelow and Steve Vogt. Bigelow's work included the slit area items, the corrector lenses, the camera mirror and the instrument cooling system. The "slit area" includes the television guide system and the comparison source system. Also described here will be the collimators; echelle mosaic; cross-disperser mosaic; CCD dewar; main housing; main structure; re-aluminizing and re-alignment; and miscellaneous tooling. Bruce left in September 1992 and Osborne and Vogt finished the alignment, integration and testing. A report detailing the optical components has been prepared by Dave Hilyard. Two separate reports by Terry Ricketts describe the instrument control system and the CCD control system.

The instrument was designed and drawn using AutoCAD software resulting in a complete 2-D representation and a partial set of 3-D drawings. There are 1380 mechanical drawings (25" by 42"). Many assembly drawings are included in this report. We have anticipated changes and additions to the instrument. HIRES was fabricated by the UCO/Lick Observatory shops at UCSC working from full size blueprints. These were prepared from pen plots. Vellums are stored at UCSC as well as the digital files on 1/4" magnetic tape cassettes (8

cassettes). 11" x 17" laserwriter drawings will accompany the instrument to Hawaii. The British Engineering system of units (as opposed to the metric system) was used in this project.

A brief general overview of the optical design and scientific philosophy behind the HIRES instrument was presented by Vogt²¹. The optical design for the HIRES camera is described by Epps and Vogt¹. The general assembly of the Keck Telescope and HIRES optics are shown on drawing H0010. Drawing H0102 shows the optical layout and H0148 details just the camera. A schematic is shown on H0200. An additional schematic is also in the Appendix.

All drawings and figures are in the Appendices at the rear of this report. Some drawings are used in more than one place. When updating this report please change all copies of the same drawing.

Chapter 2

DESIGN OF THE COLLIMATOR MIRRORS

2.1 Drawing List:

There are 85 E-size drawings which are listed in the locating tree, H2005.

2.2 Mirror Cell:

A single cell supports both the red and blue collimator mirrors. These are 17" diameter Pyrex® mirrors. The 2.7" thickness was set by the manufacturer and agrees with the 6:1 rule of thumb for diameter-to-thickness ratio for glass disks. The radial supports are located at 120° but should have been 90° for the lower two supports. This error should not present a problem. The radial supports are designed to be temperature-compensating. This technique is described by Nelson in TMT Note 28². Details of the design are shown in Drawing H2209. The 3 axial supports are front-defining. Three swivel-foot pads are used on the front of each mirror with spring loaded plungers directly opposite on the rear. The axial supports are also located at 120° around the mirror. The usable light beam was roughly square and so this support geometry cleared the beam. See Drawing H2206 for details of the spring plungers and Drawing H2000 for the cell assembly. The total force on the rear of the mirror should equal 1.5 times the weight of the mirror (90 lbs = 1.5 x 60). The general shape of the cell and shuttle mechanism is narrow along the optical axis of the spectrograph. This is because the original design had the slit 20" further back to allow more room at the slit area. The collimator cell was thus up against the rear wall of the housing. It wasn't until later that more room became available in the area behind the cell. (The slit was moved to improve image quality.)

The two mirrors have the same spherical shape: 327" radius. The difference between them is 0.02". This makes the cell easier to design since the mirror support and cell need only accommodate 0.02" of motion along the mirror axes in locating the two mirrors in their cell.

The mirrors will be removed for re-coating by removing the front faceplate of the double cell. Each mirror will be removed manually after removing the 3 front-defining pad assemblies. The adjustment will not be lost if the collimating

screws are not turned. The upper radial support is loosened and a special clip is installed to retain the Delrin® pad. The mirror can be replaced by noting the orientation of the outer edge index mark. Re-installation is the reverse of this. The upper radial support screw is tightened to 5 in-lbs.

The mirror covers are air-operated with magnetic reed switches mounted onto the cylinders to indicate open or closed condition. Speed control valves are mounted to either end of the air cylinders. The covers are not interlocked with the shuttle motion.

2.3 Mirror Shuttle (Red to Blue):

The motion from red collimator to blue collimator is controlled by 2 double-acting air-cylinders. An air switch causes an air-operated 4-bar linkage locking mechanism to hold one of the two upper kinematic mounts tight and an air-operated caliper brake holds the lower kinematic mount tight. The 3rd kinematic mount is loaded by the weight of the cell and focus mechanism (about 300 lbs). The electrical signal to shift from red to blue puts pressure (75 psi) on the release side of the lock which overcomes the 60 psi locking pressure. When the stage moves about 1/2" the mechanical limit switch opens to make the locking pressure zero. The switch is then ready for the blue-to-red motion. The opposite switch causes the opposite lock to engage when the shuttle is within 1/2" of touchdown. This locking motion also helps seat the ball-in-groove kinematic mount since we found that friction was high enough that seating at the ball-in-cone mount was a problem. We added check valves to prevent exhaust air from operating the lock cylinders prematurely. See Drawing H2415 for details of the pneumatic control logic.

The speed control valves on the main motion cylinders are set to bleed slowly so that the shuttle motion is very gentle at the touchdown portion of the travel. Valves were added to bleed the remaining air after touchdown to unload the drive cylinders and speed up the cycle time from red to blue and back. Time delay circuits were designed to open the valves for 4 seconds after arriving at the kinematic mounts. All motion is prohibited during the time these valves are open.

Before the collimator mirrors are aligned to the rest of the spectrograph, the shuttle should be adjusted so that the mirror cell is tilted 1.75° (half of 3.5°) at each end of travel. There are 2 adjustments, one at each of the lower (ball-on-flat) kinematic mounts.

The air exhaust runs through a long, large diameter pipe. A 1/2" tube was tried but the back pressure caused the clamp cylinders to engage the stage while it was moving.

Red or blue position status is via magnetic reed switches mounted to the main drive cylinders.

Drawing H2401 shows the parallel linkage of the shuttle system; Drawing H2415 shows the air system schematically; and Drawing H2460 shows one of the locking mechanisms and one kinematic (ball-in-cone) mount.

There is a middle position for the shuttle: this was designed after assembly. Moving to the middle and staying there is rather crude. Two split, plastic cylinders attach to the main drive cylinder rods to hold the stage in the center position. To center manually, halt the shuttle and release the brake button in the middle of its travel. This should be done with 2 people and lots of light. The rear-mounted autocollimator can now be used without removing either mirror or opening the covers.

2.4 Focus Control:

The focus control was added after the project was begun and the cell was finished. The total travel is about 3/4" along an axis halfway between the incoming and outgoing beams. That is, along the mirror normal axis. The focus control stage is 3 flex plates arranged in a triangular pattern. There is no sideplay in the stage. The stage is driven by 3 THK ball screws bearing on 3 hardened steel pads. Drawing H2600 shows the focus housing and shuttle stage. The mirror cell, shuttle housing and 3 flex plates are all made of aluminum, thus avoiding thermal expansion and contraction problems. Coil springs are used to keep the cell in contact with the 3 screws. The 3 screws are tied together with 3 timing belt sprockets and a long timing belt (337 teeth). A DC servo motor is mounted directly to one of the 3 screws. The motor is mounted to a THK linear slide so that it follows the screw (and cell) relative to the housing of the focus stage. Four limit switches locate the travel ends and an optical interrupter locates the center of travel and which side of center you start from. Encoding is via the motor-mounted incremental encoder. There is no absolute encoder for this stage. The motor is used throughout HIRES but has a 5:1 gear box mounted here for more torque. There are 3 dial indicators which serve as rough alignment devices in case the belt must be removed from the 3 drive screws. Drawing

H2610 is a schematic of the drive. Drawing H2620 is an assembly drawing of the focus stage and shows the triangular arrangement of the 3 flex arms. The motor/encoder is shown in Drawing H2623 and the gearbox is shown in Drawing H2660. The limit switches are shown in Drawing H2630. During the testing we discovered that one of the electric brakes failed causing the belt to come off. We then removed both brakes and saw no drift in the focus. We are still watching for this. The 5:1 gearbox is able to hold the stage from back-driving.

Chapter 3

DESIGN OF THE ECHELLE MOSAIC

3.1 List of Drawings:

There are 93 E-size drawings which are shown on locating tree H3005.

3.2 Grating Support and Subplate:

A 1 x 3 mosaic of gratings was used with overall dimensions of 12.6" by 49.6". The gratings were replicated by Milton Roy from a master grating which they ruled. The substrates are Zerodur® from Schott Glass. Each Zerodur grating is 12.6" by 16.5" by 3" thick. (see Drawing H3454) The rulings run along the 12.6" direction. The two gaps are 0.04" wide. The gratings are stepped by 0.09" to hide the un-used portion in the shadow of each upper grating (about 3/8" wide). See H3150. The pivot axis is arranged to be on the surface of the central grating, parallel to the rulings. See assembly Drawings H3600 and H3100. Drawing H3425 shows the blaze detail. All 3 gratings are held against a 4" thick granite subplate. The grating back support detail is shown in Drawing H3500 and side support details are shown in Drawings H3510, H3511 and H3520. The subplate is held in an aluminum cell. The subplate is black gabro, a type of granite with an elastic modulus of 12×10^6 psi. Note: standard tool room granite plates are 30% quartz for better wear resistance but have 1/3 the stiffness ($E = 4 \times 10^6$). The subplate is seen in Drawing H3470. Details of the three subplate back-supports are shown in Drawing H3300. The three side supports are shown in Drawings H3340 and H3345. The finite element analysis results are shown in Drawing H3801: The predicted deflection for 180° gravity change is 0.16 arc-sec for the worst case. This is the relative angular change between the worst 2 grating blanks. The software package used to model this flexure was ANSYS. The location of the 3 back supports was selected to minimize this flexure.

3.3 Grating Supports:

The mosaic concept was to be a passive support. The initial alignment would then be done and no adjusting would be needed forever. The Zerodur gratings

would be located by three Zerodur washers against the granite subplate. The details of the support can be seen in Drawings H3500 (back support), H3510 (side support, type I) and H3520 (side support, type II). The washers were polished on one side and fine ground on the other. The polished side was optically contacted to the polished grating surface. The three supports in each grating pull the grating against the subplate with 1.5 times the grating weight (90 lbs = 1.5×60). The washers were machined by fine grinding after measurements of the mosaic were made by interferometer. The washers could be measured by Anorad profilometry in 6 places around the perimeter, ground to remove as little as 0.000,02", re-measured and re-installed for another interferometric test in one day. It took one day to evaluate the test results and predict the next cut and it took typically 4 cuts to get one pair of gratings aligned to each other. This was about 8 cuts for the echelle mosaic (and 4 cuts for the cross-disperser mosaic). We did this with bare, polished blanks and then again after replicating. The error in replication turned out to be less than 3 arc-sec. The gratings were aligned in the plane of the gratings by pushing on the sides or on Zerodur blocks which were glued to the sides. These adjustments and supports were aluminum and stainless steel. We found during cold testing that the side supports are not stable enough and so invar parts are being built to replace these components.

3.4 Gluing:

The first experience with mosaics is documented in UCO/Lick Technical Report #49 (Reference 9). This 2 x 2 mosaic used 6" x 8" Zerodur blanks. Each blank was supported with three invar inserts epoxied into the back of each blank. We used Devcon 5 minute epoxy. We scaled up the design for the HIRES mosaics. The under-cut pockets in the back were larger and we used Eccobond 285 with catalyst 11 by Emerson and Cuming to embed the invar threaded inserts. This epoxy cures at an elevated temperature (80°C). We had failures in the glass after this process. The side pads were glued with Eccobond 285 and catalyst 24LV. This is a black adhesive and comes apart at boiling water temperature. The plan was to remove the side blocks for replication after our initial alignment testing, by placing the blanks in boiling water. The rear inserts would then stay in place and not come out. The 285 glue with this catalyst is rated to take up to 177°C. They would not interfere with the replication process at Milton Roy. The side pads were 2" long invar blocks and we had glass failures here, too.

Though the expansion coefficient for invar is small, it is not zero and so we suspect the motion of the invar pads from day to night in the lab caused the glass to be stressed to failure.

The solution to both these problems was to use a different epoxy. We were referred to Hysol EA 9313 by A. DuBois (See Reference 18). Drawing H3040 details the gluing procedure and shows a test block. The side support block is now Zerodur instead of invar and we have had no more problems there. The Hysol epoxy is softened with Methylene Chloride and by using a thin saw blade, the side blocks were removed before replicating. This was a time-consuming process. The blocks were re-attached after replication. The invar inserts were bonded into new holes bored into the grating blanks by Zygo. The old pockets were cleaned out with Eccostrip and then we decided to acid etch the cracks with Ammonium Bisulfate. This caused more problems than it solved because the acid seemed to migrate to all the surfaces of the grating blanks and left stains on every surface. We sent the tarnished blanks to Zygo to polish again.

3.5 Echelle Mosaic Testing:

This topic is covered in greater detail in the optical report. Drawing H3061 shows one of the grating tests using a folding flat mirror and Zygo interferometer. Drawing H3603 shows the final test with the mosaic at 10.5° to the interferometer. An isolation platform using Newport air-mounts was built to test this mosaic. We have tested the mosaic with the interferometer while it is mounted inside HIRES.

The blaze angle turned out to be slightly different from specified and so the drive was modified to move the center position by 1.5° (70.5° instead of 69°). The grating was enough oversize to accommodate this change and the tilted grating and cell did not interfere with the incoming light from the slit. Final alignment of the three gratings in the plane formed by the mosaic is less than 1 arc-sec ("hinge" or dispersion direction). Alignment in the cross-dispersion direction is also sub arc-sec. Alignment of the gratings about the center grating normal axis is less than 5 arc-sec.

We tried to make grating blanks here (UCSC) but found it was cheaper and faster to have Zygo make the blanks. The front and back surfaces have to be flat to 20 waves. Zygo can do this easily. We had tighter tolerances since we were first testing the concept to 1 arc-sec and we needed 1 wave surfaces. This will not be necessary in the future. Zygo bored the mounting pockets in the back.

Zygo also machined the bevels which are very important for proper separation during the replication process. Milton Roy returned our first set of blanks because the bevels were out of tolerance. This "take-apart" process is proprietary and so we never learned exactly what the bevel minimum could be. The success of replication lies in their ability to split apart a 12" by 16" glue joint without damaging the glass or the thin (0.001") layer of epoxy.

3.6 Echelle Drive:

The drive is shown in Drawing H3200 and the bearings in Drawing H3240. Echelle drive and encoder sub-assembly are shown in Drawing H7300 and limit switches are shown in Drawing H7330. A 6" linear stage from Daedal and Heidenhain linear encoder are used here. This encoder was not rated below 0°C but we have tested it to -20°C. We found that the motor encoder works as well as the (expensive) linear encoder. After the echelle drive was finished and tested we decided not to design a linear encoder into the CCD focus stage.

The echelle pivots by two stub axles bolted to the cell. These are carried in two SKF double row spherical roller bearings. One stub axle has a contingency stub protruding from the bearing block. This is for locating a rotary encoder if the linear drive and encoder did not work out. The opposite side of the A-frame support structure has a tab which angles down to the third kinematic mount of the main frame. This was not used but was intended for a stiffening element if the echelle ended up with a low natural frequency. Drawing H3244 shows the alignment tooling for locating the echelle bearings.

3.7 Gluing Notes:

Keck Observatory Technical Notes 194⁴ and 160⁵ derive practical guidelines for glue joint design. We have used Hysol EA9313 epoxy resin with a nominal thickness of 0.012". A description of the glue methods used in the Keck Telescope was written by A. DuBois as "Keck Observatory PMSS, Epoxy Bonds for Segment Inserts," November 26, 1990. HIRES Drawing H3040 lists our cleaning and preparation methods and shows a 3" x 3" x 3" test block which has been loaded to twice the maximum insert load. The three small side blocks which are arranged around the test block, are loaded to 1 times the maximum expected load. This test assembly was glued and loaded in July, 1991. It was then placed

in a freezer at -20°C . It has been stored at that temperature for almost 2 years.
It will be kept at UCSC and inspected periodically.

The solvent for this epoxy is Methylene Chloride.

3.8 Cover

The air-operated cover is shown in Drawing H3700 and the pneumatic control is diagramed in Drawing H3703.

Chapter 4

DESIGN OF THE CROSS-DISPERSER MOSAIC

4.1 List of Drawings:

There are 126 E-size drawings which are shown in the locating tree, H4005. 21 of these are for the replacement turntable.

4.2 Grating Support and Subplate:

A 2 x 1 mosaic of 12.5" x 16.5" blanks was designed. The gap is 0.04". The overall size is 16.5" x 25". This is very similar to the echelle support and common parts were used wherever possible. Drawing H4470 shows the subplate. Drawing H4000 shows the cell assembly. Drawing H4700 shows the air-operated doors and Drawing H4751 shows the pneumatic details.

4.3 Mosaic Testing:

The final alignment accuracy yielded a pair of gratings with 0.77 arc-sec error in the direction perpendicular to the echelle dispersion ("cross" dispersion) and less than 0.02 arcsec error in the echelle dispersion plane ("hinge" error at the joint). This corresponds to mismatches of less than 5.4 microns of cross-dispersion error and 0.14 microns of echelle dispersion error at the CCD. We were not sure what kind of specification we had to meet, but are very happy with this result.

4.4 Rotary Table and Drive:

An initial purchase of an Anorad precision rotary table was a disaster. We then found ourselves in the position of designing a precision turntable to fit in a very narrow package to replace the Anorad table. We were successful. The drive uses one of the servo motors used throughout HIRES and a Heidenhain incremental rotary encoder is mounted directly on axis. This encoder has 720,000

pulses per revolution and a single pulse per revolution index track. The resolution is 2 arc-sec. Drawing H4251 shows the encoder. A brake was designed and installed. This is shown on Drawing H4534. It was discontinued because we observed the stage to creep over long (8 hour) time periods. An air cylinder provides a counter balance to the offset weight of the mosaic (450 lbs). See Drawing H4511. Turntable details can be seen in Drawings H4264 and H4530. Drive schematic is on Drawing H4281.

4.5 Lifting Equipment and a Second Cross-Disperser:

The lifting bridle is detailed in Drawing H4740. The manual hydraulic crane is used to exchange grating mosaics. A second mosaic will be built soon.

Chapter 5

CCD DEWAR AND SUPPORT

5.1 List of Drawings:

There are 216 E-size drawings which are shown in the locating tree, H7005.

5.2 Focus Stage and Drive:

Drawing H7250 shows the dewar frame and stage. The stage is 3 flex links mounted with disconnect pins. This is the point where a second dewar will be attached. The dewar frame is then defined by 3 hardened steel pads resting on 3 steel buttons. These are driven through a 20:1 lever system by 3 micrometer screw drives. The micrometer screws are coupled to a single motor by timing belts. See the H73xx series of drawings. Drawing H7350 is a kinematic drawing of the drive. One pulse of the motor encoder represents a theoretical motion of 6.94×10^{-8} inch (1.76 nano-meters). Total focus travel is $\pm 0.012"$. (The target was $\pm 0.5"/20 = \pm 0.025"$, but the double set of limit switches and the required overtravel for each one, limited the focus travel.) There are three manually-adjusted bronze screws in the focus system which allow for a coarse adjustment of $\pm 0.5"$.

5.3 Vacuum Chamber and Liquid Nitrogen Fill System:

Drawing H7130 shows the CCD dewar and support frame. Drawing H7131 shows the detail of the liquid nitrogen can, Drawing H7180 shows the central vacuum housing with field flattener window and Drawing H7500 shows the pre-amp box which hides in the shadow of the center housing. The liquid nitrogen automatic fill system is shown in Drawing H7600. The design for the level-sensing tri-axial capacitors was borrowed from Lawrence Berkeley Lab⁶. See Drawing H7601. The dewar transport and lifting systems are shown in Drawing H7800.

5.4 Dewar Performance:

The LN₂ reservoir holds 3.5 liters. An ion pump keeps the vacuum at 10⁻⁸ torr. Details of the dewar design and construction can be found in a UCO/Lick Observatory Technical Report in progress by Osborne. Hold time is approximately 15 hours and so transfer happens twice daily. A 50 liter supply dewar rests on the Nasmyth platform. The utility board in the CCD controller monitors the level sensors and begins the transfer of cryogen and stops it when full. The venting of nitrogen gas is carefully done so that the housing does not fill up with nitrogen which is deadly. The 50 liter supply dewar is pressurized to 2.5 psi with clean, dry compressed air until the transfer is completed.

The Varian ion pump (2 liter/second) power supply has been modified. The schematic is shown in the Appendix. This modification prevents over-heating of the pump unit after power loss and during start up at a pressure higher than about 10⁻⁴ torr. The pump could get hot enough to cause the o-ring in the valve to fail. Or the pump could get hot enough to start a fire.

Chapter 6

HOUSING and STRUCTURE

6.1 List of Drawings:

There are 225 E-size drawings. These are shown on H6000, the locating tree for Housing and Structure.

6.2 Bally Enclosures:

Three pre-fab modular rooms were bought for this instrument: The small, 6 foot by 6 foot "Vault" contains the electronics and operates above 5°C. It is cooled by Observatory coolant until the temperature drops below 5°C. The heat generated by the electronics component serves to keep the temperature above 5°C. The components are not rated to go below 5°C. The large room was 15'5" by 17'4" when ordered from Bally. It was modified with a large cylindrical portion which fits inside the elevation axle. The design clearance is 1" in radius or 2" in diameter. A 5' by 10' ante-room was added later. These enclosures use 4" thick poured-in-place polyurethane foam for insulation. They come with self-closing sealed doors and insulated floors. The locking latches have been removed (for safety). The incandescent lights are dim. We added 8-foot fluorescent lights which are rated for very low temperatures (-40°F). These fluorescent lights have persistent glow for many minutes after being turned off and also cause the paint of the interior walls to fluoresce. Thus, we recommend that the bulbs be removed during routine operation of the spectrometer and only installed and used during extended periods of maintenance. We added an overhead raceway for wires and hoses. The main enclosure is shown in Drawing H6400. The Vault is shown in Drawing H6425. H6410 is a 3-D drawing of the main housing and vault. It shows the conical transition section and cylindrical "nose." This drawing can only be viewed in the drawing editor in AutoCAD. The single view shown in the Appendix here is one of an infinite number of views.

6.3 Light Trap Ante-room:

An ante-room was added to help keep the main enclosure dark and provide a location for donning clean-room apparel. This room is shown on Drawings H6460 and H6461. It is about 5' by 10'.

6.4 Kinematic Mounting of the Structure on the Nasmyth Platform:

The instrument is located on the RIGHT Nasmyth Platform. The decision to move from the Left platform was made in May, 1993, during packing and shipping. There are benefits in occupying the Right platform: 1. access to the elevation encoder is preserved, 2. access from the fixed Nasmyth Deck structure is easier while the telescope is in the stored position, and 3. no walkways need modifications. Drawing H0010.H shows the new configuration. Drawing H0010.G shows the old configuration. The mounting holes are different between the two platforms. We intend to match the slots in the 3 support feet to the mounting holes (12 mm threaded holes) on the Right Platform by using transfer screws and aligning the feet, then removing the feet and using a milling machine at the Summit to locate the slots.

The attachment to the telescope is through 9 threaded holes (12 mm). The mainframe of HIRES attaches to 3 feet which bolt to the 9 holes in the platform to distribute the load into the telescope. The foot closest to the slit is fitted with a spherical bearing. This bearing is shown in Drawing H6177. A typical foot assembly is shown in Drawing H6120. An adjustment in the up/down direction is provided by a large acme screw (1.5" Dia x 5 TPI). Adjustment in the slit jaw direction (telescope-defined y-axis, that is horizontal direction) is via slots in this foot assembly. The foot near the collimator mirrors has a spherical pad resting on a flat surface. This simulates a ball-on-flat joint. An acme screw here (5 TPI) allows vertical adjustment and a tilt adjustment to align with the telescope elevation axis. The 3rd foot has a simulated ball-in-groove joint. The detail is shown in Drawing H6174. This last foot allows the Nasmyth deck to change shape without introducing forces into the HIRES optical bench structure. This preserves the original alignment of the spectrograph optics. There are slots in this 3rd foot which run in the telescope-defined z-axis (along the instrument axis). This allows the instrument frame to be steered in the plane of the deck to align with the elevation axis. There is no vertical adjustment in this foot. If

one is required, then we will use shim plates. It is thus possible the slit jaws will not be perfectly horizontal.

A series of drawings beginning with Drawing H6020 show the day-to-day assembly of HIRES onto the Nasmyth deck. The mounting feet are shown clearly.

6.5 Optical Layout:

The spacing of the optical elements is detailed in Drawing H0102. The Camera layout is shown in Drawing H0148.

6.6 Cooling Tests at UCSC:

Drawing H6470 shows the Bally freezer unit located in the main enclosure. The freezer is not to be shipped to Hawaii. During the initial cold testing in 1989, the temperature was -22°C . The final optical testing in February of 1993 was done at -7°C . The electronics vault also had a Bally freezer installed. Its temperature was kept at 5°C .

Chapter 7

MISCELLANEOUS (SHIPPING AND INITIAL ALIGNMENT)

7.1 Handling Equipment:

The Ruger Crane is a manually operated 1/2 ton crane. It has a 2 speed feature which is not straightforward to initiate: When lifting very heavy items (750 lbs), the relief valve must be set to 1/4 to 1/2 turns open. This is not an obvious thing to do. The crane has 2 mounting sockets. There are 2 lifting arms to reach different items. See Drawing H9000. The main use for the crane will be changing Cross-Dispersers. The socket on the optical centerline will be used for this. Note- be sure to remove the crane after using at this station or it will interfere with the optical path of HIRES! See Drawing H9005 for the correct arm location. Other items to be lifted include the Collimator Mirror Cell (used with the white lift arm) and the Echelle mosaic cell. The Echelle can be lifted from either socket (See Drawing H9006). These will not be routine operations.

The pneumatic-tired dollies are for moving the grating mosaics (See Drawing H3075 for the Echelle and Drawing H4075 for the Cross-Disperser cart). This keeps the loads on the fragile subplate supports to less than 1.2 g's. Only manual lifting devices should be used to lift grating cells. A lifting bridle is provided for the Cross-Disperser. A commercial swivel eye is located in the top of the Echelle cell.

The roll-around hand-cranked lifter is for corrector lens cells and the CCD dewar. See Drawing H7800.

The camera mirror and mount are removed as a single unit using the overhead I-beam trolley and manual chain hoist. Be sure to release the hold-down clamps before attempting to lift the camera mirror, or you will lift the entire spectrograph! The mount has 4 casters for rolling once it is lowered to the floor. In the future, there will be both a red and a blue camera mirror. The passing arrangement is shown in Drawing H6936.

Three large steel casters mount onto the mainframe and are for moving during dis-assembly, packing and re-assembly. They will have no other use after

commissioning. See Drawing H6605. The mainframe dis-assembles for shipping in 8 foot wide containers. See Drawing H6200.

7.2 Initial Alignment Scenario:

Drawing H6500 shows some of the alignment tooling. The permanent tripod at the rear of the mainframe has been bolted, pinned and welded. It is not intended to be removed or adjusted. An autocollimating telescope is fitted to the tripod and used to align the spectrograph. There are no adjustments to this mounting. The philosophy here was to open the slit jaws and use this same autocollimator to find the telescope optical axis. This will be used in the future for modifications, additions or to look for problems. The Davidson Autocollimator will remain with the HIRES instrument.

Drawing H6510 shows the rear tripod. A moveable tripod assembly which mounts ahead of the slit is shown in Drawing H6530. This device has an aluminum storage plate for protection and a wooden storage crate when not in use. The tripod is fitted with adjustments but should never need adjusting. Should this tripod be needed, it can either be installed through the forward panel which has the entrance hatch (use the "man-lift") or by carefully 'snaking' it into position from the inside of the HIRES housing. Whenever this tripod is re-installed, its alignment should first be verified by checking against the fixed autocollimator mounted behind the collimator cell.

Alignment of the slit area is detailed in Section 18. No re-alignment is intended; all of the joints have been pinned. The only alignment which may need looking after is the tilt of the comparison lamp feed mirror and the centering of the pupil mask in the comparison lamp optical system. The strategy is to ensure that the apparent pupil of the comparison source system coincides precisely with that from the telescope.

Chapter 8

RE-ALUMINIZING AND RE-ALIGNMENT

8.1 Camera mirror (or "Hextek" mirror); about 44" diameter:

This mirror will be coated at Mauna Kea using an adapter which is shown in Drawing H5420. The adapter assembly has been engraved to help identify it. The mirror support hardware is left in place during cleaning and aluminizing. The old coating will be stripped off using chemicals. A strip of 1/4" wide non-hardening strip caulking is placed in the joint just behind the front plate. This will keep any liquid from going into the cavities in the interior of the mirror. This strip of material must be removed before placing in the vacuum chamber. The original coating was done at Mt. Hamilton hanging vertically. The tank at Mauna Kea has the mirror looking face down. The red and blue mirrors will require different coatings. We are assuming that the coating technology will be present at the Summit. This will keep travel to a minimum for these fragile mirrors.

8.2 Sol-gel re-coating:

The 2 large corrector lenses (about 32" diameter) have been Sol-gel coated at the Lawrence Livermore National Lab. They have lifting slings and carrying fixtures which have been reviewed by the technicians there. Refer to UCO/Lick Observatory Technical Report No. 62 dated June 1992 by Bigelow entitled: "Mechanical Engineering Safety Note - Lifting Slings for the HIRES Spectrograph Corrector Optics." (Reference 8). There are 2 clean cases for shipping the lenses.

The field flattener lens (6.5" diameter) has a case but no special sling. This optic will be coated before HIRES arrives in Hawaii. LLNL has slings for this size lens.

The Sol-gel is extremely fragile and scratches easily if even lightly wiped. The Sol-gel also has 35 times the surface area of the part it is on, so is a virtual sponge for dust and other contamination. Great care must thus be taken to guard against dirt or touching of these coatings.

8.3 Collimator Mirrors (17" diameter) red and blue:

These mirrors have wooden cases for storage and shipping. They will be hand-carried to Z, C & R in Los Angeles for red coating and blue coating. These mirrors weigh 60 lbs each and so can be installed or removed manually.

8.4 Re-aligning the collimator mirrors:

This is done with the Davidson autocollimator mounted in the slit area tripod socket (refer to Drawing H6530). Before removal of one or both of the mirrors, the autocollimator will be installed and the target mounted onto the central pedestal using the kinematic mount and the 3 steel balls (refer to Drawings H6560 thru H6565). We have used this method and achieved 10 arc-sec re-installation accuracy. If a focus drive belt should break, then refer to the 3 dial indicators on the rear of the collimator housing. These will be recorded after initial instrument assembly and checkout. Drawing H2640 would be a good place to record these readings.

8.5 Pumping:

A clean, oil-less vacuum pump must be used inside the HIRES enclosure after the field flattener lens is re-installed onto the dewar since this is the vacuum sealing window. We expect to have such a pump in place after arrival in Hawaii. This pump uses a liquid nitrogen cold trap and so the room must be ventilated while pumping the dewar or while operating the pumping station.

8.6 Mosaic testing:

There is no provision for mosaic (either echelle or cross disperser) re-alignment. This was intended to be a passive support and so far we have seen nothing to change our original intent. We do have hardware for mounting the Zygo interferometer on the main frame to look at either of the 2 echelle mosaic grating joints or the single cross disperser mosaic grating joint. These setups are shown in Drawings H6220 and H6222. We are not leaving the interferometer in Hawaii. We have used it in our shop at UCSC and found it to give stable enough results that an air isolation table was not needed. We hope that the

telescope environment is at least this stable. The mounting stand replaces the center hydraulic lift mount socket.

The gratings cannot be re-aluminized, of course. There is no cleaning procedure. The spectrograph was designed with the echelle facing down to minimize dust collection. There is one thumb print on the echelle mosaic and it has been documented on H3640. It is fortunately outside of the beam footprint.

8.7 Re-aligning the corrector lenses:

The first step before removing anything is to align the autocollimator with the corrector lenses: Lift the cross-disperser cell and store it on its cart. The autocollimator adapter fits onto the turntable and the autocollimator can then be rotated to line up with the camera axis. Drawing H4210 shows the adapter. Drawing H6500 shows the autocollimator in place. The inter-lens spacing should be documented. There is no need to locate the lenses along the optical axis to better than 0.01" and the elements will easily go back into their cells to that tolerance.

To re-align the entire camera, the dewar frame and camera mirror must be removed and the second camera mirror mount installed with its autocollimator. (Two autocollimators are required here for efficient aligning) Note that when the second mirror has been installed onto this mount, an additional fixture will be needed for this alignment procedure. It will be designed and built at the same time the mirror support hardware is fabricated. This fixture allows the cross-disperser table to be rotated to align with the camera mirror axis. The camera mirror (and camera mirror mount #1) can now be re-installed to check its alignment using the cross-disperser autocollimator. The corrector lenses are re-installed and checked. Finally, the dewar frame is replaced. If you suspect it has shifted or want to check it, the camera mirror is removed again and the camera mirror mount autocollimator is installed to look at the field flattener. The spacing between the camera mirror and the field flattener cannot be measured directly anymore, unless the field flattener can be recoated since any touching with an inside micrometer will damage the Sol-gel. The spacing can however be measured indirectly by measuring between the mirror and front lip of the dewar body, and then subtracting the known distance from dewar body to lens vertex. See Drawing H0148. The CCD is aligned by taking many spectra at many focus settings. This has been detailed by Vogt in his notes.

Chapter 9

OTHER COMMENTS

Hooks were left for future expansion. The large side door in the housing was located so that in the future a Cross-Disperser might be rotated to look at a camera axis on the other side of the optical axis. See the layout on Drawing H6100. There are 4 stub flanges attached to the main frame to add additional supporting structure.

This project has been reviewed by a committee chaired by Dr. Bob Tull at the University of Texas. Also on that committee was Dr. Frank Melsheimer of DFM Engineering Inc, Longmont, Colorado. Progress was also monitored by 17 Quarterly Reports submitted to the Science Steering Committee (SSC). Software reviews were included with the mechanical reviews and supplemented with reviews of the Low Resolution Imaging Spectrograph (LRIS) together with HIRES.

Chapter 10

The HIRES Slit Area

The HIRES spectrograph slit area is composed of a large steel supporting structure and twelve motorized or air-powered stages for remote control of slit area functions. The intent of this document is to provide a brief overview of the standards and design philosophy used throughout the slit area, and to give a general description of the layout, operation and adjustment of each of the various stages (see Drawing P1001). This document will also serve as a roadmap for any journey in search of engineering information concerning the slit area.

10.1 Slit Area Structure:

Previous experience with slit area instrumentation at Lick Observatory suggested three important features for the HIRES slit area. First, the slit area is subject to improvement and modification throughout the life of the instrument, and should be designed for easy modification from the start. Second, the slit area is typically the most crowded region in the spectrograph, so all structures and components should strive for compactness. Finally, certain accessories will tend to preclude the use of others, so stages should be able to be readily removed or interchanged. These three features of easy modification, compactness and interchangeability will be discussed in relation to the slit area and its motion stages.

The central backbone of the HIRES slit area is the “slit area structure” and several constraints drove its design. One aspect dictated by the telescope was the requirement that the slit area structure be of a cantilever design. The location of the nominal Nasmyth focus at the center of the elevation axis journal required that the slit, decker and autoguider all be located inside the bearing. Another important objective was to leave a maximum amount of space for accessories and relatively easy access to them. Another issue was to design the structure such that its first natural frequency was not likely to be excited by motion of the telescope or the active mirror control system. All of these points and others lead to the design of the cantilevered space frame for the slit area (see Drawings H6710 and H6711). The 2” x 4” structural steel tubes are liberally peppered with 1/4–20 threaded inserts on regular spacings to provide easy installation of

new components. Note that the threaded inserts are stainless steel and will gall with stainless fasteners unless they are properly lubricated (i.e. "Never-seez") on assembly. The structure is primarily a rectangular weldment, with several bolt-on diagonals to increase stiffness, but which are removable for access if needed.

10.2 Common Motion Stage Features:

From the outset of the HIRES project, it was clear that there might be as many as twenty (first light) remotely controlled functions, which strongly suggested standardization on a minimum assortment of components. To that end, DC servo systems were selected for all areas requiring precision motion control and air cylinders for all toggling and non-precision motion applications.

We chose to standardize on DC servo systems for all of the precision motion functions for a variety of reasons including freedom from annoying (stepper motor) resonant frequencies, low power and high torque, compact motor/encoder units, and prepackaged servo drive and optimizing software. We selected Galil 50/1000 systems including Galil motor controllers, Pittman 50 oz-in DC servo motors with integral Hewlett-Packard 4000-count per revolution shaft encoders, and the Galil software package. For fiducial location sensing, a TRW OP8970T55 optical interrupter switch was chosen. To provide software limit sensing, Microswitch BZ-2RW822-A2 limit switches were used. A second switch, Microswitch DT-2RV22-A7, was used to cut power to the servo motor in the event of a software or hardware limit switch failure.

For the roughly twenty non-precision situations, we selected a variety of different sizes of air cylinders manufactured by Bimba. Although we had already decided to use the Microswitch limit switches for the precision stages, we chose to use the Bimba magnetic reed switches for sensing limits on all of the air powered stages. These switches are designed for use with the air cylinders and helped avoid a fair amount of bracket design and fabrication. Along with the cylinders, we standardized on Skinner Magnalatch 4-way solenoid valves, which have the benefit of no power consumption after latching. Wilkerson R16-02-000 air pressure regulators were used throughout. Camozzi "super rapid" quick assembly fittings were used for many areas. These developed trouble during the cold test and are being removed and replaced with Swagelok tube connectors. The nature of the trouble was that these finger-operated fittings couldn't be operated by human fingers at cold temperatures. The Swagelok

connectors dis-assemble with small hand tools (wrenches). The Bimba cylinders should always be supplied with clean dry air. We experienced many cold-related problems: moisture freezing caused cylinders to hang up or operate un-reliably; the factory-supplied lubricant became very viscous and caused the speed controls to work slowly. We also had trouble with the latching solenoids sometimes not shuttling from one state to the other. Much of our testing will be done in Hawaii where the temperature is cold while the air is dry. Our testing problems were made worse because we had to keep the instrument full of dry nitrogen during the cold testing to prevent condensation. This required oxygen masks and made testing impossible. We have cleaned all of the air cylinders with the solvent (petroleum distillate) and purged all air lines with dry gas. During shipment all air ports have been sealed to prevent moisture from entering.

There were a variety of linear and rotary bearing requirements in the slit area, and we chose a different manufacturer for each, based on previous experience. For linear bearings and ball screws, THK products were used exclusively. SKF bearings with integral seals are used throughout.

For small mechanical components such as clutches, timing gears and belts, and worm gears, Berg products were used. All worm gearing meshes are lubricated with Nyogel 788 from W. F. Nye Co.

10.3 Slit Area Stages:

The HIRES Phase C Proposal (UCO/Lick Observatory Technical Report No. 57, January 1991, Vogt) briefly listed the remote control functions to be provided in the slit area. All of these were completed in addition to several others. The following sections will give a general description of the design and operation of each stage, as well as a short list of appropriate adjustments for each. The stages are discussed as follows:

1. Slit
2. Slit Accessory Server (Decker)
3. Instrument Filterwheels
4. Shutter
5. TV guider
 - a. CCD camera and cooler
 - b. Lens Drives (Focus and Aperture Control)

- c. Filterwheels
- 6. Comparison Source System
 - a. Light Source Motion Stage
 - b. Thermal Enclosure and cooler
 - c. Filterwheel
 - d. Projection Optics
- 7. Iodine Cell System (Stage)

Chapter 11

THE SLIT

11.1 General Description:

The slit controls the vertical dimension of the aperture to the spectrograph by opening or closing a bilateral pair (both jaws move about their centerline) of reflective jaws. The slit jaws are opened by a ball-screw driven wedge which separates the flex-pivot mounted slit plates (see H1324). The slit jaws are roughly 2" x 4", made of 420 stainless steel, and polished flat to about 5 waves. Return springs force the slit jaws closed as the wedge is withdrawn. The shaft encoder attached to the motor provides for a theoretical resolution of about 0.5 microns at the slit jaws. An optical interrupter provides the fiducial location, and travel is protected by software and mechanical limit switches (see dig H1326). The slit is mounted at an angle (7.5° nominal) to reflect most of the field back to a TV camera for autoguiding.

11.2 Adjustments:

The slit has individual adjustments for tip, tilt, rotation, and piston on one jaw, while the other jaw remains fixed. The adjustments are made by tightening or loosening the screws supporting the jaw (see dig H1322). The jaws are readily adjusted within 10 arc-sec using an autocollimator and a reference flat for parallelism. The minimum slit opening can be adjusted using the travel limit screws, which can be reached through the access holes, found near the return springs (see dig H1324). The travel limit switches may be adjusted by loosening the mounting screws and sliding the switches in the slots.

The slit assembly as a whole can be adjusted for up/down, left/right, focus, and tilt, using the slotted mounting screws attaching the slit to its support (see dig H6760). In addition the adjustments on the stage, the slit and filterwheel supporting beam can be adjusted for left/right, up/down, and focus, using its slotted mounting screws (see dig H6758).

Chapter 12

THE SLIT ACCESSORY SERVER (Decker)

12.1 General Description:

The Slit Accessory Server (SAS) can provide both vertical and horizontal control over the aperture into the spectrograph. The SAS can be used in conjunction with the slit, or it can be used alone to define the aperture. The SAS carriage has positions for four, 4" square aperture trays, and sufficient travel to see all of the trays, plus a stowed position giving a full unvignetted view of the slit (see Drawing H1359). The aperture plates are EDM machined, and are optically polished flat to a few waves (see Drawings H1384, H1389, H1390 and H1391). The carriage is attached to a timing belt driven by a motor and gearbox. An optical interrupter provides the fiducial location, and travel is protected by software and mechanical limit switches (see Drawing H1378). The shaft encoder attached to the motor provides a theoretical resolution of about 0.8 microns at the aperture plate (see Drawing H1387). Like the slit, the SAS is mounted at a 7.5° angle, in order to reflect most of the field back to a TV camera for autoguiding.

12.2 Adjustments:

No adjustments are provided for the SAS carriage itself. All alignment of the SAS is controlled by adjustment of the upper and lower stage mounts. By translating the upper and lower mounts, left/right, up/down, rotation, and focus can be accomplished (see Drawings H6759 and H6760). The travel limit switches can be adjusted by loosening the mounting screws and sliding the switches in the slots. The belt tension for the carriage can be adjusted by loosening or tightening the idler pulley adjusting screw. An autocollimator mounted in the front of the slit area is used to align the SAS with the slit on the optical axis, and to adjust the reflected beam to the TV camera. The TV camera, once operational, can be used to verify that the Slit and SAS are both properly aimed.

Chapter 13

FILTERWHEELS

13.1 General Description:

Two 12 position filterwheels provide a wide range of filter options for the spectrograph. Each filterwheel has 12 numbered locations for mounting a HIRES-standard filter holder, which can accommodate a 2" round or 2" square filter, up to 3/8" thick. The number 1 position has an extended opening which allows a 1" x 4" light beam to pass thru unfiltered if desired (see dig H1104). The wheels themselves are interchangeable, so that an observer can have a "personal" set of filters optimized for a given task (see dig H1115). A worm gear drive rotates the filterwheels and precludes the need for a brake to hold position with power off. An optical interrupter provides the fiducial location. A shaft encoder mounted to the motor provides a theoretical resolution of 10.8 arc-seconds (see dig H1114).

13.2 Adjustments:

The worm gear mesh can be adjusted by pivoting the worm gear carrier (see dig H1110) around its mounting shoulder screw. No other adjustments are provided on the filter stage.

The filterwheel assemblies can be aligned left/right, up/down, and focused by adjusting the slit and filterwheel support beam (see Drawings H6758 and H6760). Note that this will also affect the alignment of the slit. The filters are generally oversized for a given aperture, and typically should not require adjustment after initial collimation.

Chapter 14

SHUTTER

14.1 General Description:

A Melles Griot / Ilex (p/n 04 IES 005) 2.5" aperture shutter controls the exposure time for the spectrograph. The shutter is mounted to the back of the second instrument filterwheel, and defines the rear light boundary of the slit area (see dig H1203). The shutter is controlled by the CCD detector controller mounted above the dewar. The shutter was modified at the Lick Instrument Lab to use a reed switch rather than the stock limit switch, in order to improve reliability.

14.2 Adjustments:

The shutter is hard-mounted to the filterwheel via a threaded ring and has no adjustments. With the shutter removed there is a 3.2" diameter clearance hole.

Chapter 15

TV AUTOGUIDER

15.1 General Description:

The TV guider stage is composed of a high quality commercial CCD camera system, a commercial camera lens, and two filterwheels, all mounted together in a compact package. There are two motorized drives for the lens, one for focus and the another for aperture adjustment. Between the CCD camera head and the lens are a pair of 8 position filterwheels for color and brightness control. Due to the variety of features and adjustments to be covered, the CCD camera and the motor driven functions will be discussed individually in the following order: CCD camera system, lens drives, and filterwheels. Finally, the entire stage and its adjustment as a unit will be described.

15.2 CCD Camera System Description:

The TV guider uses a Photometrics 200 series CCD camera system for direct and offset guiding from the slit and/or decker. This system includes a CH250 liquid cooled camera head with a Thomson TH7883 frame transfer CCD, a CE200 camera electronics unit individually tuned for the CCD, a special order 10' cable from the camera head to the electronics box and a LC200 liquid recirculator for cooling the CCD. The camera electronics unit and the liquid recirculator are mounted in a thermal enclosure mounted inside of HIRES but outside of the slit area (see Drawing H1401). The TV thermal enclosure is in turn cooled by the instrument cooling system (see Cooling System Drawing H1406). The last item required in the TV system is the CC200, a controller unit which is provided by the telescope.

15.3 CCD Camera Adjustments:

The only mechanical adjustments for the camera head are by way of the slotted mounting screws which allow some travel in the up/down and focus directions (see Drawing H8105).

15.4 Lens Focus Drive Description:

The TV guider system uses a Canon 200mm focal length, f/1.8 lens to image the slit onto the camera detector. The plate scale at the slit is 1.379 arc-sec/mm, and 0.26 arc-sec/mm at the TV CCD. The Thomson CCD has a $23\mu\text{m}$ pixel size, giving a detector scale of 0.156 arc-sec/pixel (see Drawing P1003).

The lens focus is driven by a timing belt gear reduction attached to a 60:1 worm gear speed reducer, which is in turn driven by the standard DC servo motor. An optical interrupter provides the fiducial location, and travel is protected by software and hardware limit switches, as well as by a clutch between the worm gearbox and the lens gear. A large split clamp attached to the focus ring of the lens carries the drive gear, limit switch triggers and the optical interrupter blade (see Drawings H8105 and H8119).

15.5 Lens Focus Drive Adjustments:

There are several adjustments for the focus drive which will be discussed in the following order: fiducial, limits, belt tension and clutch.

1) Fiducial:

The blade attached to the clamp ring is adjusted by loosening the clamp ring and rotating the blade until its edge is in the center of the optical switch at the middle of the lens focus travel. This will place the lens focus at its nominal position on power-up.

2) Limits:

The limit switch mounts have oversized holes for attaching the switches, as well as slotted holes for attaching to the mounting plate. By loosening all the mounting screws for the switch, it can be translated and rotated as required to trip sometime before the mechanical stop in the lens is reached. The software limit switch should be adjusted to trip before the mechanical limit switch cuts power to the motor.

3) Belt Tension:

The belt tension is adjusted by loosening the screws which mount the motor and gearbox assembly to the baseplate.

4) Clutch:

The clutch is adjusted by loosening the locknut on the clutch and tightening or loosening the adjusting nut as required. The intent of the clutch is to prevent the motor from overdriving the lens in the case of both limit switches failing.

15.6 Lens Aperture Drive Description:

The Canon lens aperture setting is driven by a bell-crank linkage attached to a 60:1 worm gear speed reducer, and powered by a DC servo motor. An optical interrupter defines the fiducial location. Because the bell-crank rotates freely without any mechanical stop no limit switches are required. One half revolution of the bell crank drives the aperture ring through its full range of travel. A split clamp attached to the aperture ring carries the bell crank link and the fiducial blade (see Drawings H8105 and H8119).

15.7 Lens Aperture Adjustments:

The stroke of the drive wheel is fixed by the radius to the link end. The stroke at the aperture ring can be adjusted by moving the motor-gearbox assembly on its slotted mounting screws, and by varying the position split clamp on the aperture ring.

15.8 TV Filterwheel Description:

Two 8 position filterwheels provide a variety of filter options for the TV guider camera. Each filterwheel has 8 locations for mounting a HIRES-standard filter holder, which can accommodate 2" diameter or 2"x 2" square filters up to 3/8" thick (see Drawing H8326). The wheels themselves are readily removed and are interchangeable, so that an observer can have a "personal" set of guiding filters optimized for a given task (see Drawing H8325). A worm gear drive, identical to the instrument filterwheel drive, rotates the filterwheels. An optical interrupter indicates the fiducial position and the motor encoder provides a theoretical resolution of 10.8 arc-sec. The filterwheels are removed by withdrawing the center mounting pin, rotating the filterwheel assembly out of the guider assembly, depressing the button in the wheel hub, and pulling off the wheel.

15.9 TV Filterwheel Adjustments:

The worm gear mesh can be adjusted by pivoting the worm gear carrier (see Drawing H1110) around its mounting shoulder screw. The filterwheel assemblies can be adjusted left/right using the slotted mounting blocks.

15.10 TV Autoguider Stage Description:

The camera head, lens, and filterwheels are mounted together on the Autoguider stage (see Drawing H8001). The upper and lower pin mounts for the stage provide support and allow the stage to be manually rotated up into a stowed position (see Drawing H8002). By removing all four pins, and disconnecting the electrical and cooling connections, the stage may be removed from the slit area as a unit.

15.11 TV Guider Stage Adjustments:

The four main components of the TV guider; the camera head, lens and filterwheels; may be adjusted relative to each other using the slotted mountings for each part. The camera head mount has slots for adjusting up/down, left/right, and focus. In order to collimate the stage with the reflected slit optical axis, its mounts can be translated to provide left/right, up/down, focus and rotation of the stage.

Chapter 16

COMPARISON SOURCE SYSTEM

16.1 General Description:

The Comparison Source System provides a variety of light sources for calibration of the spectrograph. The initial complement of light sources includes two Thorium-Argon hollow cathode lamps, a diode laser, a 3450°K color temperature quartz lamp and a Deuterium lamp. The latter two lamps are mounted in two 8" Labsphere integrating spheres. Wiring is installed to support up to eight sources (low voltage and current). The motion stage is mounted in a cooled thermal enclosure to prevent thermally induced turbulence above the slit area. The four main parts of the system are the light source motion stage, the thermal enclosure and cooler, the filterwheel and the projection optics.

16.2 Motion Stage Description:

The light source motion stage carries and positions the various lamps which are used for aligning or calibrating the spectrograph (see Drawing H1620). The carriage nominally has 6 positions, each providing 6 square inches for mounting a source. The moving stage is a 8" x 36" long platform with a mounting hole pattern for mounting any number and size of sources as required (see Drawing H1618). The stage has 34" of travel, which is more than sufficient to bring any of the mounting locations on to the optical axis of the slit area. The carriage is driven by the same arrangement designed for the decker, using a timing belt to position the stage, and servo motor and worm gear speed reducer to drive the belt. A pair of negator springs preload the stage to eliminate backlash. An optical interrupter defines the fiducial location and software and mechanical limit switches protect the range of travel. The theoretical resolution of the stage is the same as the decker, about 0.8 microns at the stage. The actual resolution has been measured mechanically to be better than 0.0005". Drawing H1622 gives encoder resolution.

16.3 Motion Stage Adjustments:

The main adjustment for the motion stage is the drive belt tension, which is accomplished by loosening or tightening the idler pulley screw. The limit switches can be adjusted by loosening the screws and sliding the switches in the mounting slots, but will require that the stage be removed from the enclosure. The entire assembly has four pin joint supports for left/right, up/down, focus, and rotation adjustments. By translating the mounting points in their slots, the full range of adjustments can be made. For collimation and alignment, the mounts are designed to be adjusted while the stage is installed in its housing (see Drawing H1688).

16.4 Thermal Enclosure Description:

The comparison sources operate inside of a sealed enclosure in order to avoid thermal convection currents which could disrupt the seeing in the slit area. The enclosure construction consists of a sandwich of extruded fiberglass ribs, polyethylene foam insulation, and aluminum or CPVC skins. The CPVC skins are used internally in locations where the aluminum would create a thermal short to the main instrument enclosure (see dig H1680). The panels and ribs are epoxied and riveted together to form a light, rigid unit. The top and end panels are latched and/or hinged to allow for removal when servicing the calibration lamps. A small heat exchanger and two muffin fans are installed in the top of the enclosure directly above the optical axis. Coolant flows continuously through the system, while a sensor cycles the fans as required to maintain the temperature at the same level as the main housing.

16.5 Thermal Enclosure Adjustments:

Once the motion stage has been aligned, the housing can be realigned with the stage to avoid any interference. Care should be taken to insure that the lamps will always have clearance throughout their travel. This is especially important for the hollow cathodes, which move in a trough on the inside of the housing. Four slotted stainless steel supports carry the weight of the housing and locate it relative to the motion stage. The housing may be moved by loosening the mounting screws and sliding the unit to the desired position. The weight of the

housing must be supported while the mounts are being adjusted. The latches for the top and end covers can be adjusted for tightness by turning the threaded catch in or out as required. The hinges are made of delrin and cannot be adjusted. The lamps can be replaced by either removing an end or top panel for access. Poron foam is used as a gasket material to seal the panels. (Boyd Corp., Ceres, CA)

16.6 Filterwheel Description:

The comparison source system is equipped with a filterwheel for controlling source bandpass or brightness. The filterwheel and drive assembly are virtually identical to the main instrument filterwheels, except for their mountings (see "Filterwheels" and Drawings H1701 & H1706).

16.7 Filterwheel Adjustments:

The filterwheel locates a filter between the doublet and the aperture stop of the comparison source optical system. Adjustments are provided to insure adequate clearance between the filters and the optics (see Drawing H1706). Slots in the filterwheel mounts allow for up/down, left/right, and focus adjustments.

16.8 Comparison Source Optics Description:

The design of the comparison source optics attempts to mimic the exit pupil of the telescope, in order to more accurately calibrate the spectrograph. The optical system consists of a 2" doublet, a filter, a fixed aperture stop, a 6" triplet, and a 6" folding flat (see Drawings H1655 and H1692). The comparison source apertures tend to be small compared to the slit width, so the projection optics also provide a 2:1 magnification at the slit. If the entire 4" width of the slit requires illumination, the source can be scanned across the slit using the motion stage. The folding flat reflects the comparison source beam into the spectrograph and stows to one side when not in use. A pupil plate is located between the 2" doublet and the 6" triplet in order to create a circular image with a central obstruction which simulates the telescope and the secondary mirror baffle.

16.9 Comparison Source Optics Adjustments:

There are three main assemblies for the comparison source optics, including the 2" doublet, the 6" triplet, and the 6" flat. The doublet and the triplet both have slotted mounts and tilt adjustments to accommodate up/down, left/right, focus, and rotation (see Drawings H1812 and H1816). The 6" folding flat is mounted in a commercial mirror mount made by Janos (model no. 2017-0003). The mount has two rotational adjustments and a slotted support for focus. Left and right adjustment is controlled by hard stops on the air powered stage that moves the flat in and out of the optical path. Vertical adjustment is accomplished by raising or lowering the folding flat stage (see Drawing H1814).

Chapter 17

IODINE CELL STAGE

17.1 General Description:

The iodine cell stage is an air-powered translation stage mounted to the slit accessory server. The iodine cell can be used to impress a field of iodine spectra on top of an object of interest. The iodine cell stage consists of the iodine cell, a mount to attach the cell to a linear bearing, and an air cylinder to extend or retract the cell in and out of the optical path (see Drawing H1955). Electrical connections to the cell include a 120VAC strip heater, two temperature sensors and limit switches to report the location of the stage to the instrument computer. Travel of the stage is limited by hard stops attached to the linear bearing support. Limit switches attached to the air cylinder indicate the position of the stage. The cell is mounted on a simple platform which could be used to support other accessories if required in the future.

Note that the iodine cell is designed to operate at 122° F, approximately 90° F warmer than the surroundings. No provisions other than insulation have been made to prevent thermal gradients and air motion in the slit area due to heat dissipation by the cell. It remains to be seen if this will be a problem.

17.2 Adjustments:

The stage mounting platform has mounting slots for adjusting focus of the cell. Travel of the stage can be adjusted by moving the stops in their mounting slots. No other adjustments are provided because the stage is designed to be coaxial with the slit accessory server. If the SAS is properly collimated, the iodine cell should be as well.

Chapter 18

SLIT AREA COLLIMATION

The purpose of this section is to describe the order and procedure of collimating the slit area stages. Detailed notes about the collimation procedure can be found throughout Vogt's test/integration lab notes. The procedure begins after the establishment of the instrument optical axis, using alignment telescopes located in the front of the slit area and behind the 17" collimating mirror stage. Once this axis has been set, the stages can be collimated successively in the order listed here.

1) Slit and Filterwheels:

The slit and instrument filterwheels are mounted together on a beam attached to the slit area structure (see Drawings H6758 and H6760). The slit and filterwheels should be adjusted to be at the same height and left/right location relative to each other on the supporting beam. The whole assembly can then be adjusted on to the optical axis by sliding the support beam in the mounting bracket slots. Finally, the slit should be individually collimated for tip, tilt and rotation, taking care not to translate the slit off of the optical axis.

2) Slit Accessory Server (Decker):

The decker should be adjusted to place the back side of the decker plates as close as possible to the slit jaws (about 0.060") to minimize reflection pathlength differences for offset guiding. The decker is supported by upper and lower mounts which provide for up/down, left/right, and two rotations (see Drawings H6758, H6759 and H6760). The decker housing has a cut-out to accommodate the slit housing, and the cut-out should be adjusted to center over the slit. Once the slit and decker have been collimated, their alignment relative to each other should be verified by viewing the slit and decker with the guide camera.

3) Guider TV and filterwheels:

The four main components of the guider system are the lens, the two filterwheels, and the CCD camera head. The lens is attached to the main platform (see Drawing H8105), and the lens height relative to the platform is fixed. Consequently, the lens should be collimated first, by adjusting the four mountings for the platform. Once the lens is collimated, the CCD camera head can be adjusted in its mounting slots to be coaxial with the lens. The filterwheels are

not adjustable individually, but are fixed on the platform at the same elevation as the lens, so that if the lens is collimated, the filterwheels should be as well.

4) Comparison Source System:

The comparison source system consists of a motion stage, a thermal enclosure, a filterwheel, and an optical projection system which doubles the size of image of the source at the slit. All parts of the system have a range of adjustments which will allow the various parts to be collimated relative to each other, as well as to the spectrograph. However, the enclosure is probably the limiting component in terms of travel adjustment and distance from the slit, so it will be most practical to establish the optical axis based on the window in the enclosure. Once the optical axis is set, the stage, lens groups and filterwheel should be aligned in that order.

5) Comparison Source Housing:

The comparison source stage is mounted in an insulated enclosure to prevent thermal air disturbances in the slit area. The enclosure has adjustments to allow it to be moved relative to the stage during collimation. The housing mounting brackets have slots for fore/aft and up/down adjustment (see Drawing H1688).

6) Comparison Source Stage:

The comparison source stage is mounted on a steel weldment attached to the slit area frame (see Drawings H1688 and H1692). The four degrees of freedom to be adjusted are focus and the three rotations (tip and tilt relative to the optical axis, and rotation around the direction of travel.) The left/right alignment is controlled by motion of the stage, and the height of the sources is adjusted at the individual source on the stage. All of the stage adjustments are accomplished by sliding mounting brackets in their slots, which are found in the following locations:

1. slots in the individual source mountings on the stage;
2. up/down slots in the pivot brackets attached to the bottom of the stage;
3. fore/aft slots and set-screws in the brackets attaching the stage to the support weldment;
4. fore/aft slots in the support weldment flanges where they attach to the slit area structure;

7) Comparison Source Optics

The comparison source stage (CSS) optical system has a 6" diameter folding flat, a 6" triplet, a 2" aperture stop and a 2" doublet. The optical system is

defined to be in the plane of the instrument optical path and plumb to the slit area. This plane is established by hanging a plumb bob through the slit area such that the plumb line is visible at the slit and at the window into the comparison source stage housing. Using the plumb line for a vertical reference, and alignment telescopes for the instrument optical axis, the CSS folding flat, triplet, and doublet can be collimated in that order.

1. **Folding Flat:** The flat height adjustment is by way of the slots in the supporting beam brackets. The slots can also be used to adjust the angle of beam tilt (see Drawing H1814).
2. **Triplet:** The triplet has adjustments for focus, left/right, up/down, and two rotations. The adjustments are either by screws in slots or by set-screw (see Drawing H1816).
3. **Doublet:** The doublet has adjustments for focus, left/right, up/down, and two rotations. The adjustments are either by screws in slots or by set-screw (see Drawing H1812).

8) **Comparison Source Filterwheel:**

The comparison source system has a 12 position filterwheel which places a filter between the aperture stop and the doublet in the projection optics. The filterwheel and mounting brackets have slotted adjustments for focus, up/down, and left/right (see Drawing H1706). Additional adjustments are by slots in the filterwheel support beam and mounting brackets.

9) **Iodine Cell Stage:**

The iodine cell is mounted on an air-driven linear bearing which allows the cell to be quickly inserted or removed from the optical path. The iodine cell is essentially a pair of windows with iodine gas between them, with limited adjustments for collimation. The height of the cell is fixed by attachment of the stage to the decker housing. The left/right position of the stage is set by adjusting the travel limit brackets on the stage. The primary collimating adjustment is focus, being the distance from the cell to the decker, and is set by sliding the cell support in its slots (see Drawing H1955). The tip and tilt of the cell relative to the optical axis can be changed by adjusting and shimming the cell in the supporting cradle.

Chapter 19

INSTRUMENT COOLING SYSTEM

19.1 General Description:

A liquid cooling system is provided to reduce the buildup of heat in the spectrograph, and avoid the resulting thermally induced air motion which could degrade seeing in the instrument. Coolant at about 3°C below the ambient air temperature is provided by “the telescope system” at a nominal flow rate of 2.6 gallon per minute (gpm) and 80 psi. The HIRES cooling system is designed to remove all the heat generated in the instrument with the 1500 W of cooling provided. A control panel for the cooling system is located near the large door into HIRES and houses a flow meter and flow control valve, as well as gauges for the inlet and outlet pressures.

The design of the cooling system faced a number of challenges. First, the cooling provided by the telescope was provided over a very narrow temperature difference (5°C), and a relatively low flow rate, both of which made heat removal more difficult. The main variables to balance for an exchanger are the size and the flow rates of the coolant and the air. In most cases in HIRES, the heat exchangers were limited in size by packaging constraints. With the ΔT of the coolant, the exchanger size, and the flow rate all fixed, the main choices left were the coolers themselves and the air flow rates. A second consideration was the cooling circuit through the instrument, and whether to make it a serial or parallel system. Finally, the type and design of the coolers was considered to maximize the efficiency of the system. There are four sources of heat, in decreasing heat load; the electronics vault (600 W), the CCD TV electronics and liquid recirculator (130 W), the CCD controller (100 W) and the Comparison source stage (50 W). This adds up to 880 Watts. The CCD controller chassis is mounted close to the CCD dewar and pre-amp. This thermally insulated box has a cooler mounted inside. See Drawings H1462 and H1463 for details.

The final design of the HIRES cooling system uses a serial fluid circuit and Hayden and LYTRON all-aluminum heat exchangers. The coolant flows first through the comparison source enclosure, which has the smallest heat load (50 W) and the tightest packaging constraints. Because of the size limit in

this case, a small Hayden (Thinline TU10416G5-14JE2) cooler is used with a custom plenum and two 12VDC muffin fans (see Drawing H1423). This choice allows the smallest cooler to see the greatest initial temperature difference (ITD) between the inlet coolant and the ambient air in the enclosure, for maximum heat removal. The HIRES instrument computer reads two temperature sensors in the enclosure and cycles the fans to maintain the temperature at the ambient level. The coolant then flows to the CCD TV enclosure (130 W), where it passes through a LYTRON (5221G10) cooler with integral fans (see Drawing H1406). This exchanger was chosen to fit the space constraints and to simplify the mounting of the cooler, shrouding and fans. The temperature is controlled in the same manner as in the Comparison source box. The HIRES electronics enclosure (600 W) has plenty of room for large exchangers and consequently is the last area cooled. In this case, two large LYTRON coolers (5321G10), with two 9" fans each, operate with the smallest ITD (see Drawing H1411). The long coolant path and the high air flow rates allow the exchangers to remove the heat given the worst ITD and the highest heat load in the system. The electronic enclosure must be maintained at 5°C, rather than ambient, but the computer again uses sensors to monitor the temperature and cycles the fans as required.

The design objectives for the rest of the cooling system were to minimize danger to the instrument from coolant leaks and other failures, to simplify assembly and maintenance, and to standardize on plumbing fittings.

Most of the coolant runs are through insulated copper tubing attached to the outside of the instrument housing, which provides several benefits (see Drawing H1450). First, in the event of a leak, the coolant is more likely to run down the sides of the HIRES enclosure rather than onto optical surfaces or electronics. Second, the coolant lines are more easily accessible for removal and reinstallation. Finally, long straight runs of copper tubing are the most economical way to move the coolant over the distances between the various thermal enclosures. In locations where flexibility or disconnection are required, Goodyear "Gorilla" 3/4" ID hose is used. This hose is inert to a variety of solvents, resistant to ozone and remains flexible at low temperatures.

For shipment and re-assembly, there are several logical places to provide quick disconnects in the plumbing circuit. The most obvious break is at the instrument and electronics enclosures where hoses and Parker 60 Series quick disconnects attach to a feedthru to the heat exchangers. Quick disconnects

are also found at the comparison source, CCD controller box, and CCD TV enclosures to simplify assembly and removal.

Unfortunately, a wide variety of brass and stainless steel plumbing fittings were required to join the various parts of the cooling system. For brass parts, pipe thread and SAE 45° flare fittings were used whenever possible, with a bias towards the SAE standard. The LYTRON heat exchangers, fabricated with 37° flare ends, were the only exception. Stainless steel tubing and Swagelok fittings were used inside the electronics enclosure because the heat exchangers are hard-mounted to the roof of the housing. The control panel is plumbed with brass fittings and bronze pipe, along with the flow valve, meter and pressure gauges (see Drawing H1432).

Chapter 20

DESIGN OF THE CAMERA

MIRROR SUPPORT

The design, analysis and testing of a large mirror mount are described. The optic is a 44" diameter, f/ 0.75 gas-fusion structured mirror manufactured by Hextek (Tucson, AZ). The mirror is the primary reflector for a split-corrector camera system for the Keck Telescope High Resolution Echelle Spectrograph (HIRES). The spectrograph is mounted on the Nasmyth platform of the telescope, with the camera mirror optical axis downward-looking 10.3° below horizontal. This paper describes the finite element analysis of the mirror, conceptual and detail design of the mount, and interferometric testing of the mirror figure before and after installation in the support. This design chapter was extracted from Reference 3 by Bigelow.

20.1 Conceptual Design of the Support:

The Hextek mirror, at 44 inches in diameter, is the largest of its type ever fabricated. The construction of the mirror is a two step process. First, the front and rear face sheets, 0.47" thick Schott Tempax, are fusion bonded to a center section of Tempax tubes. The sandwich of face sheets and tubes is heated to molten temperature, the tubes are inflated at low pressure through holes in the rear face. The face sheets and tubes fuse together. The second step involves reheating the blank as it sits on a convex mold. When the glass softens, it slumps to conform to the shape of the mold.

A variety of design considerations were evaluated for supporting the mirror. The support system was simplified by the fact the mirror would be stationary, and so would always have the same orientation to gravity. The positioning requirements for the mirror were stringent, but once met, would not require repositioning or focusing. The mirror would actually be one of two mirrors, each with a coating optimized for red or blue wavelengths, so the support would need to be as light-weight as possible and readily interchanged. The end result of the two diffraction grating dispersions is that the beam through the camera system is roughly rectangular. Consequently, only about 60% of the clear aperture of the

mirror, centered with the long axis vertical with respect to the mirror, must meet the optical specifications. The thermal environment for the mirror would be cold but stable on the order of 4° F per day, with a median temperature of about 32° F. The small daily temperature variation was especially favorable considering the 2 to 3 hour time constant for the 25% weight density mirror. On Hextek's recommendation, it was decided to exercise the internal mounting boss option (for \$7000), which provided six 3-inch-square mounting bosses in the plane of the center of gravity, located symmetrically around the mirror at the 0.7 R locations. Hextek had previous experience providing both radial and axial support of smaller mirrors through similar mounting bosses, using a six-link kinematic connection to a sub-cell. The mirror was to be re-coated with its mounting boss hardware intact, which required vacuum rated materials and components. Extensive mirror mounting research and development work conducted by others for the 36 Keck primary mirror segments was reviewed for possible adaptation to the Hextek mirror. The Keck segment design uses 36 individual axial supports bonded to the back of the segments, a torsional link for stiffness about the normal axis, and a stainless steel diaphragm radial support bonded into a pocket in the back of the segment and passing through its center of gravity.

After much consideration and debate, it was decided to use a hybrid of the Keck and Hextek approaches, using the mid-plane mounting bosses for the axial support, a torsional link and a diaphragm bonded to the back of the mirror for the radial support. The six axial supports would be reduced to three mounting points on three whiffletree balance beams. The diaphragm would be designed such that it was very stiff in the radial (in plane) direction and very compliant in the axial direction where it would be in conflict with the axial supports.

Several concepts dictated this choice. First, for collimation and initial focus the mirror support would have to be readily adjustable. The proposed Hextek design would have required a sub-cell for kinematic support of the mirror, and a second cell for carrying and adjusting the mirror by way of the sub-cell. This duplication would be more complicated (and expensive) than a single support cell. The six-link kinematic support uses three pairs of crossed links to constrain the mirror, and these links would be long and relatively more complicated in order to reach into the submerged mounting bosses. In order to reach the sub-cell in the required locations and at the proper angles, the forces in the links would be much larger than required to simply support the mirror at the six locations. The first benefit of the hybrid support was that the kinematic support and the

axial tilt adjustability would be combined in a single cell. The radial support minimized the forces input through the mounting bosses, allowed for differential thermal expansion between the radial and axial supports, and simplified the assembly and adjustment of the cell. Separating and isolating the radial and axial supports reduced the danger that the imperfectly realized kinematic supports would excessively strain the optic.

20.2 Finite Element Analysis of the Mirror:

The objective of the analysis was to determine whether or not the proposed support system would acceptably carry the mirror while maintaining its figure to a $\lambda/2$ specification. It was clear from the outset that the radial support, attached several inches away from the CG plane, would add a bending moment into the mirror, which would be counter-acted by the axial supports. The FEA would allow the testing of a variety of connection schemes and would provide insight into the deflections and stresses as a function of the number and location of the radial support points.

Hextek provided a 2-D AutoCAD file containing the mirror's exterior geometry and interior tube structure. A 1/12TH symmetric section of this file was translated into ANSYS, using the ANSYS/AutoCAD DXF file translator. Inside the solid modeling preprocessor of ANSYS, the "pie-slice" of tube and facesheet detail was projected onto a spherical surface of areas at the correct radius of curvature, and then spherically offset to create the rear surface of the mirror. Given the front and rear areas of the mirror and the correct tube mesh from the translator, the tube wall areas were created by connecting the corresponding front and rear surface edges. Once the 1/12 section was complete, it was reflected three times to create an accurate half-geometry model. The mounting boss areas were added to the model after the main structure was finished. With solid model plane areas now in place for all the plate geometry of the mirror, 4 node, 3-D plate bending (no shear deformation) elements with the correct material constants and thicknesses were assigned to each area. In order to model the effects of the axial support whiffletrees, 3-D beam elements were defined to connect one pair of support points, while the third remained a simple constraint (see figure 1 in the Appendix). The beam elements were defined by the material constants to be very stiff, so that deformation of the whiffletree would be negligible compared to the displacements of the mirror. The finished model contained 1345 nodes,

2106 elements, and 7959 degrees of freedom (see figure 2 in the Appendix). The weight of the various support components was neglected in the analyses.

A total of about twenty configurations were run with ANSYS several of which were used to determine the sensitivity of the model to parameters such as the modulus of elasticity and plate element thicknesses. The final version of the model showed 0.8λ deformation across the whole front surface, and about 0.5λ across the required aperture (see figure 3 in the Appendix). This residual deformation was primarily astigmatic, and judged to be acceptable for the requirements of the camera system. The analyses also indicated that the mirror was relatively insensitive to small variations in material constants, face sheet and cell wall thicknesses, and small loads such as the weight of the torsional link. The satisfactory results from the FEA gave us the confidence to continue on the detail design of the mirror support.

20.3 Detail Design of the Mountings:

The conceptual design of the mirror support specified a kinematic arrangement of three mutually exclusive constraints for the radial, axial and torsional degrees of freedom. The detail design process was then simply a matter of approximating the idealized supports with reliable hardware that could be easily manufactured and assembled. Stainless steel blade flexures were used in situations where small translations, high stiffness and zero backlash were desired. Simple pin joints with stainless steel pins in aluminum bores were used where stiff, small angle pivots were required. Commercial (THK) spherical ball-joints were used for one-degree-of-freedom constraints. The design features of the radial, axial and torsional supports will be discussed in that order.

20.3.1 Radial Support:

Keck Observatory Technical Note 142¹¹ derives several useful formulae for designing diaphragms for radial supports. The design of the Keck segment radial support has several well considered features which were adapted for the support of the Hextek mirror. The primary requirement of the diaphragm is that it safely carry the radial loads of the mirror under static (installed), as well as dynamic (transport and installation) loadings. It was anticipated that the mirror might see loads during handling perhaps as high as 6 G's. It was also required that the diaphragm be sufficiently compliant in its axial direction that it not adversely

affect the figure of the mirror by over-constraining the six axial supports. Keck Observatory Technical Note 189¹² documents the testing of a 8" diameter, 0.010" thick stainless steel diaphragm and found the small displacement radial and axial spring rates to be 200,000 lbs/in and 894 lbs/in respectively, for a stiffness ratio of 224:1. The results of the finite element analyses had indicated that a diaphragm of the same diameter and thickness as the Keck diaphragm would provide the necessary radial stiffness and axial compliance, even though the Hextek mirror is only 1/6 the weight of a Keck segment (180 lbs vs. 1200 lbs). The in-plane bending stiffness of the diaphragm was found to be the most important consideration because excessive stiffness would overconstrain the axial supports. Testing verified that the diaphragm was compliant enough to avoid deforming the mirror through a range of axial adjustments.

There is a significant difference in the coefficient of thermal expansion for Tempax (borosilicate, 1.8 ppm/ $^{\circ}$ F) and 304 stainless steel (9.6 ppm/ $^{\circ}$ F). For a given temperature change, the diaphragm and its supporting ring would expand or contract 5–6 times as much as the mirror. This difference would lead to an axial strain in the mirror if it were not compensated. The Keck approach was to mount the diaphragm in a pocket at the center of gravity of the segment, with a ring of axial flex springs which would deflect to allow for the temperature induced size variations. The flex springs for the Hextek mirror were designed such that a 40 $^{\circ}$ F change in temperature will create a force of 2–3 ounces at each of the six mounting points. The Keck design was modified for the camera mirror by attaching the flex springs to Pyrex blocks epoxied to the back of the Hextek mirror (see Drawing H5530). A fixture was built to test the radial support system, using a Pyrex plate to substitute for the mirror (see Drawing H5514). This set-up was used to test the mounting blocks and diaphragm assembly under varying temperatures and loads. The assembly was tested to failure in the case of the mounting blocks, two of which showed small fractures at the bond-line at about 1200 lbs, more than six times the weight of mirror. This failure was disappointing, but reassuring in several ways. First, the fractures were in the mounting blocks, which could conceivably be removed and replaced. Second, there was no apparent damage to the mirror. Finally, the failures were in no way catastrophic, and the support continued to carry the load after the fractures appeared.

20.3.2 Axial Support:

Hextek provided six bosses evenly spaced around the mirror approximately

two inches inside the back surface, which were to be used for the axial support connecting points. The bosses were 0.47" thick and roughly 3" square, with a 3/4" diameter hole bored through the center of the boss. These holes and the clearance holes in the back sheet, were bored out in the Lick Optical Lab to 1.25" and 2" respectively. Several pieces of 0.47" thick Tempax were provided by Hextek for axial support testing. The first connection design used 2 stainless steel flanges to clamp to the mounting boss. The flanges indexed loosely on the hole, to axially locate the supporting link. Unfortunately, when the two flanges were clamped together, the indexing ridge managed to jam in the hole, and neatly cleaved the test mounting boss in half during tightening. An identical clamp made of Delrin was also able to fracture the mounting boss. With these enlightening experiences fresh in mind, all components expected to contact the glass were redesigned using plastic; Delrin in the case of the axial supports, and CPVC in the case of the radial support block clamps. Additionally, the clamps were redesigned to assure that only compressive stress could be generated in the mounting boss. Drawing H5537 shows the final design of the Delrin 3-point contacting mounts. The Delrin mounting flanges accommodate variations in the thickness of the mounting bosses, and the Delrin contact points are compliant enough to provide low contact stresses and avoid introducing moment loads into the mounting boss. Drawing H5536 shows the remainder of the axial support, showing the THK spherical ball-joints, links and whiffletree balance beams. The threaded connection between the spider and the balance beam allows for collimation.

20.3.3 Torsional Support:

The torsional support is largely redundant but insures that under all conditions, the radial support is not subjected to large torque loads. The weight of the torsion link was a concern initially, but was not found to cause a problem. The weight of the torque link could be counterweighted or supported independently if necessary. Drawing H5539 shows the torque link and its connections to the supporting cell and the mirror. The Pyrex mounting block is identical to the radial support blocks. Note: as of May 1993, this support element has been eliminated. The glass block is still glued to the back of the mirror.

20.4 Mirror Testing:

The mirror was tested in two different positions in order to confirm the finite

element modeling and to insure that the figure as polished was still acceptable once installed on the mount. Zenith and 10° below horizontal tests were performed on a large vibration isolation stand which was designed for testing the Keck secondary mirror (see Drawings H5824 and H5825). After several inconclusive tests it was determined that the only good time to test the mirror was early in the morning, before small temperature variations ($1^\circ\text{F}/\text{hr}$) in the test tunnel began to change the mirror's figure. The Hextek blank is believed to thermalize in 2 to 3 hours and is unstable under even small temperature changes.

20.4.1 Mirror Figure Tests

The initial tests were performed on the mirror without its supporting cell, mounted instead on a layer of foam which closely matched the back radius of curvature. All mirror figure tests were conducted in the zenith-looking position. Interferograms were taken after a 12 to 18 hour soak in the test tunnel at 67° F . The mirror was then rotated 90° , allowed to soak again and more interferograms taken. The figures found in the samples are very consistent, with amplitudes ranging from 1 to 1.5λ , mostly astigmatic. The amplitude of the astigmatism varied slightly with rotation of the part, suggesting that the foam support might not be as neutral as we thought, but the topography of the figure was very consistent.

20.4.2 Mirror/Cell Figure Tests

After the horizontal testing was completed the mirror was installed in its supporting cell, mounted onto the camera sub-structure and tested again. The finite element analyses had indicated that the predominant aberrations induced by the cell would be astigmatism and coma, with a vertical axis of symmetry. The interferogram and fringe analysis data (WYCO WISP®) shows the predicted astigmatism and coma, with amplitudes roughly three times the expected values. It is not entirely clear how much of the amplitude discrepancy is due to testing and data analysis uncertainty or the FEA model predictions. The 1.28λ P-V and 0.25λ RMS values exceed the initial $\lambda/2$ P-V specification but were judged to be acceptable based on analysis of the worst slope errors. The worst-case slope error of 4.4 micro-radians was found to cause a $6.7\mu\text{m}$ deviation at the focal plane. The optical design of the camera system (i.e. perfect optics) predicted a $12.6\mu\text{m}$ RMS image diameter. The actual degraded image diameter can be estimated by adding the predicted image size and worst case ray deviation in quadrature, which yields

a $13.9\mu\text{m}$ RMS image size. This was considered to be a negligible degradation of the ideal image and consequently an acceptable figure for the camera mirror.

20.5 Conclusions:

A stationary support was designed to carry a very large and fragile gas-fusion structured mirror. Finite element analysis was successfully used to analyze deformation of the mirror under a variety of conditions and orientations. Extensive testing of the mirror confirmed the results of the analyses and although the amplitudes of the deflections were greater than expected, it was not entirely clear how much of the difference was due to the models, the final figure on the mirror, testing conditions or unexpected effects from the cell. In terms of topography, the finite element model did an exceptional job of predicting the flexural behavior of a large and complicated structured mirror. Finally, optical testing and slope error analysis confirmed the acceptability of the mirror, its figure, and the hybrid kinematic mounting.

Thanks to Mark Rodamaker of MCR Associates, Sunnyvale, California for help with the creation of the ANSYS® FEA model and to Robert Parks and Richard Whortley of Hextek Corporation, Tucson, Arizona for assistance concerning the handling, figuring and support of the mirror.

and assemble. A previous paper discussed the initial finite element analyses of the lenses, and a more complicated plan of attack for static figure correction¹⁰. The earlier work was completed without a good understanding of the required clear apertures, and consequently did not take advantage of the areas on the lenses which were outside the required aperture. In fact, the echelle and cross-dispersed beam is roughly rectangular, with the long axis rotated about 8° from vertical. The clear aperture only uses about 60% of the total aperture of the lenses. Once the beam shape was known, it was clear that the forces could be applied closer to the center of the lenses and that two force points would be sufficient. With a conceptual layout of the support and constraining points, a second round of finite element analysis was undertaken to determine the optimum locations and values for the reforming forces.

21.2 Finite Element Analyses of the Lenses:

The initial finite element analyses for the lenses were described in the previous paper. The same general input routine was used again, with an added subroutine for applying the reforming forces. The ANSYS® finite element models typically used about 1450 nodes, 1050 3-D solid elements, and 4200 degrees of freedom. More than thirty trials were completed for each lens, manually varying the force or location with each run. Given enough computing power, this process could be automated for a more complete optimization. Still, a variety of solutions showing a $\lambda/4$ P-V optical path difference (OPD) were found, and the location requiring the least force was used in each case (see ANSYS input files in the Appendix). Figure 1 shows a typical finite element model of the meniscus lens showing the defining points, symmetric boundary conditions and the locations of the reforming forces. Figures 2 and 3 show the finite element predictions for deformation of the meniscus lens before and after the application of the forces. Figures 4 and 5 show the same plots for the biconvex lens. Note that although the overall P-V deformations are more than $\lambda/4$, the clear apertures (bold lined) are $\lambda/4$ P-V. It is important to note that the reforming forces induce stresses in the fused silica lenses and consequently contribute birefringence effects to the optical path length errors. However, because the stresses never exceeded 50 psi, well below the accepted 500 psi limit for birefringence¹⁷, the OPD errors were considered negligible.

21.3 Detail Design of the Cells:

As mentioned before, a variety of considerations drove the design process for the corrector cells. The corrector lenses would ultimately be anti-reflection coated using a Sol-Gel process, developed and applied at Lawrence Livermore National Lab for the Nova Laser Program. This coating was expected to be stripped and reapplied occasionally, which required that the lenses be completely and readily separable from their cells, and precluded “potting”, or elastomeric mounting. The cells would be manufactured in the Lick Observatory Instrument Labs, which required that conventional fabricating and testing processes be used. Reducing cell weight would benefit assembly and handling. Corrosion resistant materials and surface treatments would be used throughout. Detail design of the radial supports, axial supports, force application assemblies and cells will be discussed in that order.

21.3.1 Radial Supports

The justifications for the basic design of the radial supports is discussed in the previous paper. To summarize, the radial support contacts are split twice in the theta direction and twice again in the z-direction (see Detail A in Drawing H5324). The balance beams operate in both directions in order to reduce the bearing stresses in the optic. Two sets of two radial supports were selected to reduce contact stress and minimized the deformation of the lenses. Stainless steel flex pivots were designed to minimize the axial forces transferred to the optic, while allowing minor focus adjustments without lifting the lens off of the support. The flex pivots reduce (but do not eliminate) the risk of axially overconstraining the lens. The Delrin contact pads are mounted to the second flex pivot, to divide the load across the lens center of gravity, again reducing stress and avoiding local deformation of the optic. A single threaded post at the top of the cell constrains the lens in the cell during handling.

21.3.2 Axial Supports

The axial supports provide two functions in the corrector lens cells: First, the axial constraints define the location of the optical surface and provide a way to align the optical axis of the part. Second, the axial contacts provide a location for the force points to react against. So in addition to locating the lens, the axial constraints help to reform the figure. The design of the defining points and

backside constraints are identical for both lens cells. The defining surface axial contacts are composed of fine threaded stainless steel posts, with commercial swivel-feet and Delrin caps to follow and protect the surface of the lens (see Drawing H5328). The backside axial constraints are primarily “earthquake clips”, to prevent the lens from falling out of its cell during handling. The backside contacts are also threaded stainless steel shafts, with spherical Delrin tips (see Drawing H5328). In the case of the meniscus, the forces applied to correct the figure actually transfer the lower pair of axial loads to the back of the lens, causing the backside (or “uphill”) constraints to become the defining points.

21.3.3 Force Application Assemblies

The force application assemblies (force points) provide a means for applying an adjustable force on the optics. The force point consists of a stainless steel shaft, a swivel-foot/Delrin cap, a threaded brass adjusting body, a spring, and a stainless steel end cap (see Drawing H5328). The springs are standard commercial parts that were calibrated for use in the force points. Although the forces applied to the two lenses are different by a factor of two, the force points are identical except for the stiffness of the springs. The length and spring constants for the springs were selected so that both would require about 0.5” of compression to apply the desired force (11.5 lbs for the meniscus, 6.0 lbs. for the biconvex lens). The springs can be compressed from each end, either by tightening the end cap, or by tightening the brass adjusting body. The two adjustments allow for very fine tuning of the force applied and allow the force point to compensate for axial adjustments of the lens.

21.3.4 Lens Cells

The cells house the lenses and provide a base for attaching the defining and forcing hardware. The housing is composed of two stiffened face plates which are screwed to a ring weldment. This construction is relatively light weight and allows the cell to be dismantled for installing or removing the lens. The remainder of the cell includes a door for dust protection and a hoisting ring for handling. Drawings H5324 and H5349 show the complete lens and cell assemblies. For reference, the corrector lenses both weigh about 120 lbs., and the meniscus and biconvex cells weigh 130 lbs. and 110 lbs. respectively.

21.4 Lens Testing:

The preliminary optical specification for the corrector lenses was to achieve a P-V OPD of $\lambda/4$ over the specified clear apertures. This is probably the easiest value to work towards during finite element analysis as well as for figuring in the Optical Lab. However, later analysis of the optical system indicated that local slope errors were much more important than the overall P-V figure on a given surface.

21.4.1 Biconvex Lens Testing

Due to convex surfaces and long radii of curvature, it was not practical to perform interferometric testing on the biconvex lens. Instead, all testing of the biconvex lens was completed using 10" diameter test plates. Although the test plates cannot qualify the peak-to-valley specification for the full aperture, they do give an excellent representation of local slope errors. Based on test plate measurements, the biconvex lens was found to be better than $\lambda/4$ over any given 10" aperture. The following section will focus entirely on the testing of the concave surface of the meniscus lens, which was the only surface readily tested with the interferometer.

21.4.2 Meniscus Lens Testing

The meniscus lens was tested in two different positions to confirm the results of the finite element analysis and to verify that the figure as polished was still acceptable after installation and figure-correction in the cell. In each case, interferograms were taken and the Wyco WISP® program was used for fringe analysis. Zenith-pointing testing was conducted with the lens mounted on a foam-lined support which was in turn carried on a large vibration isolation structure (see Drawing H5811). Interferograms were taken for several rotations of the lens relative to the foam support. The figures were consistent at about 1.5λ P-V, mostly astigmatism, and 0.25λ RMS.

The next set of tests were run in the final, tilted orientation. The lens cell and interferometer were assembled on a Newport Research Series vibration isolation table down-looking (see Drawing H5826). The first set of tests was to establish the deformed figure without any correcting forces applied. The finite element model predicted about 2 waves of deformation across the full aperture (Figure

2). The measured figure was a little more than 4 waves P-V (Figures 9 and 11), roughly twice the expected amount.

The next set of tests involved applying the correcting forces and taking more interferograms. One of the goals at this stage was to determine if the predicted correcting forces were in fact the optimum. Tests were run at 1 lb. intervals above and below the predicted optimum of 11.5 lbs. The sign of the deformation was seen to reverse between 10.5 lbs. and 12.5 lbs., suggesting that 11.5 lbs. really was the best value. By applying the optimum 11.5 lb. force at each location, the overall figure improved from 4.34 waves to 1.47 waves, a reduction of almost 3 waves of astigmatism (See Figure 14). The 1.47λ P-V was still much worse than the 0.3λ P-V (see Figure 3) prediction from the finite element model, but nevertheless was a dramatic improvement realized by a rather simple mechanical correction. The FEA predictions and interferograms do agree on the topography of the deformed (astigmatic) shape, but the FEA model underestimates the amplitude of the deformation by almost 5 times. Finite element models are typically reliable to 10-20%, not 500%, so there is clearly a problem. Possible contributors include errors in the FEA model, the figure on the lens, errors in the location or assembly of the lens in its cell or other testing error.

The last interferogram shows the figure over the required clear aperture (Figure 17). For this area, the P-V OPD is 0.68λ and 0.10 RMS. Again, the P-V number is worse than predicted, but the RMS value is actually quite good. The fringe analysis program indicated that the worst slope error was $2.1\lambda/14.15''$ ($14.15''$ is the pupil radius) for a maximum slope of 3.7×10^{-6} radians. The slope error can then be multiplied by the focal length of the lens to determine the worst case typical ray deviation at the focal plane. In this case, the maximum deviation was $1.4\mu\text{m}$ and the RMS image diameter was $1.2\mu\text{m}$. The diffraction limited image diameter was $1.64\mu\text{m}$ suggesting that the concave surface was essentially diffraction limited. This conclusion was also indicated by the 0.66 Strehl ratio calculated by WISP (Figure 18).

21.5 Conclusions:

Two novel figure-correcting lens cells were analyzed, designed and tested. Finite element models were successfully used to predict the optimum forces and locations for applying corrections to the lenses. Although the FEA predictions seriously underestimated the amplitude of the deformations, the models closely

matched the measured topography of the figures. This result suggests that there is room for improvement in the modeling process, although it is not entirely clear whether the disagreement lies in the FEA, the design or assembly of the cell, or the testing methods. Although much work remains to be done in improving the accuracy of the figure correcting cell, the concept has been tested and proven able to remove as much as 3 waves of elastic deformation.

21.6 Lens Installation and Removal

The lenses are shown in Drawings H5384 and H5385. The removal tool is detailed in Drawing H5363. The lens carrying frames are used during the installation and removal procedures. They are shown in Drawings H5247 and H5254.

21.7 Lens Cell Covers and Doors

Drawing H5376 shows the bi-folding door for the first lens cell. This door replaced the original sliding door which did not provide an adequate seal. The door is air-operated. The air logic diagram is shown in H5375 and the air control station for both lenses and the camera mirror is shown in Drawing H5380. Drawing H5342 shows the meniscus lens cell air-operated door.

Between the two lens cells is a 6" gap. Instead of building doors for these interior cell faces, a flexible cover has been installed around the cells in this gap. It is held on using Velcro strips glued to the cells and cell mounting base structure.

21.8 Lens Lifting Slings

Bigelow⁸ describes the Sol-Gel coating procedure at Lawrence Livermore National Lab. Drawings H5266 and H5267 show the lens lifting slings assembled with the lenses.

Appendix A REFERENCES

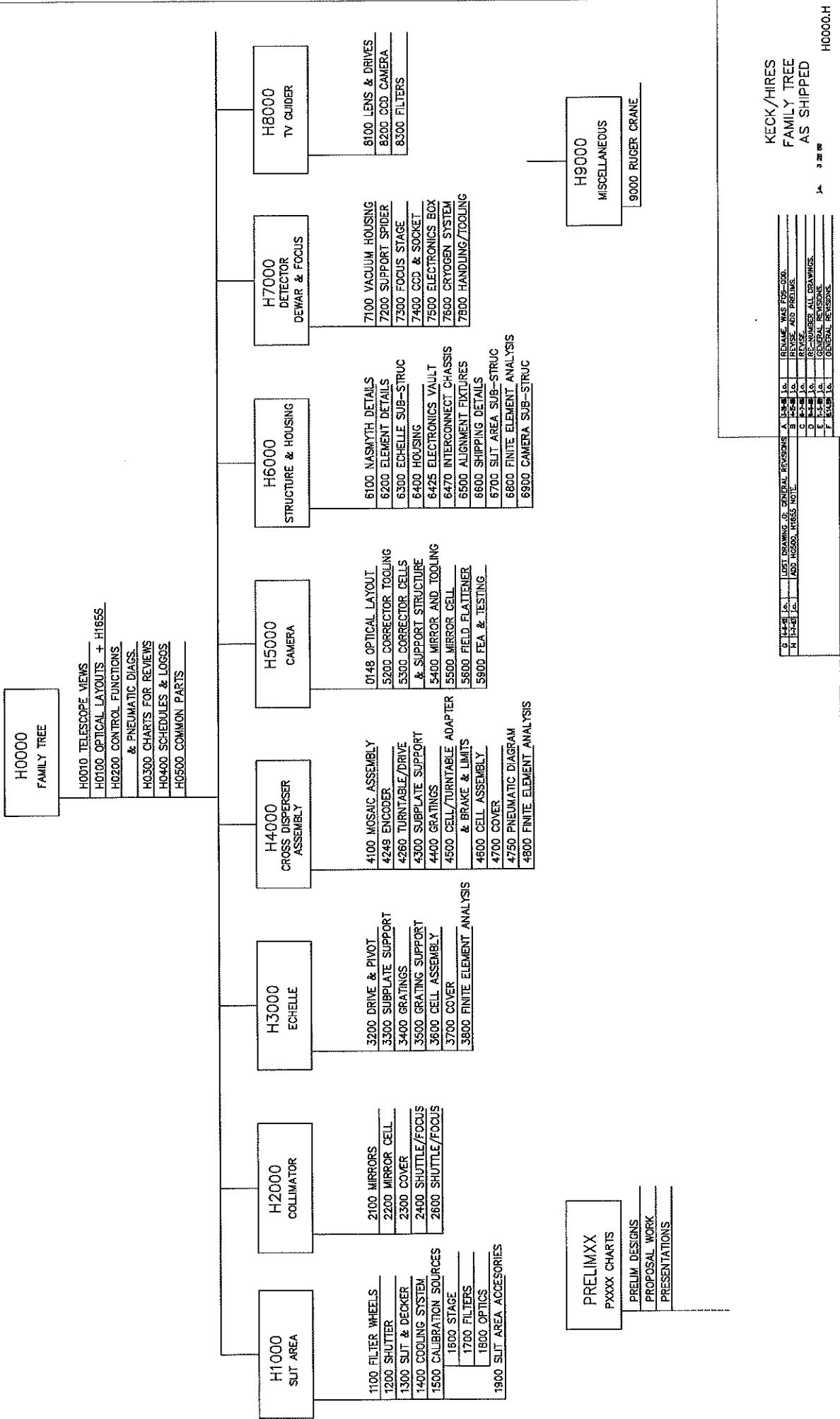
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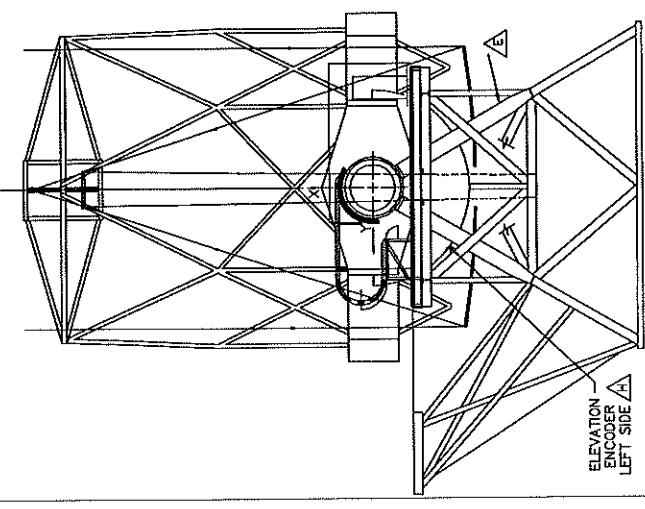
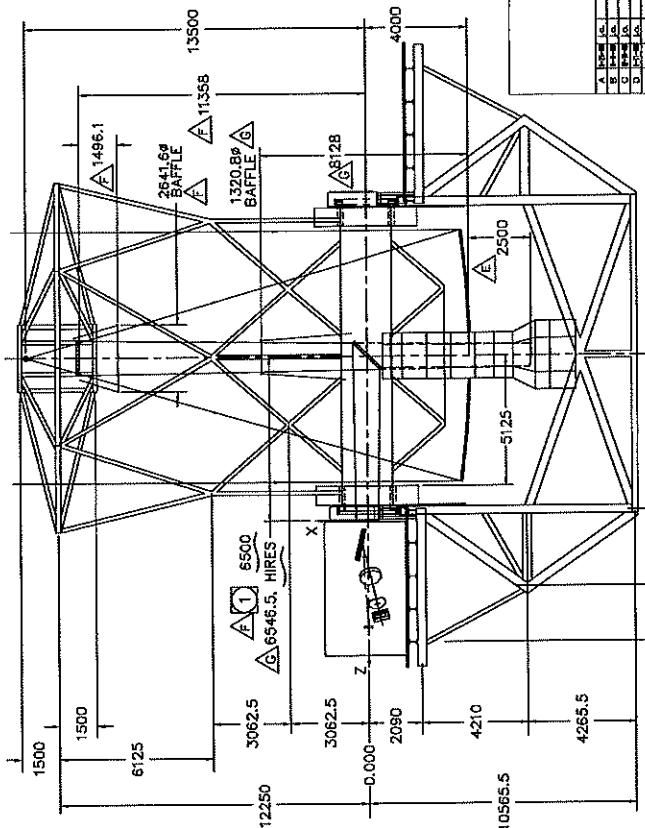
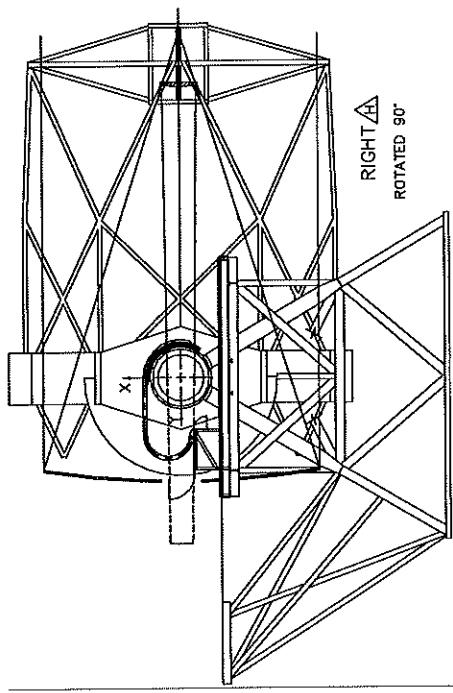
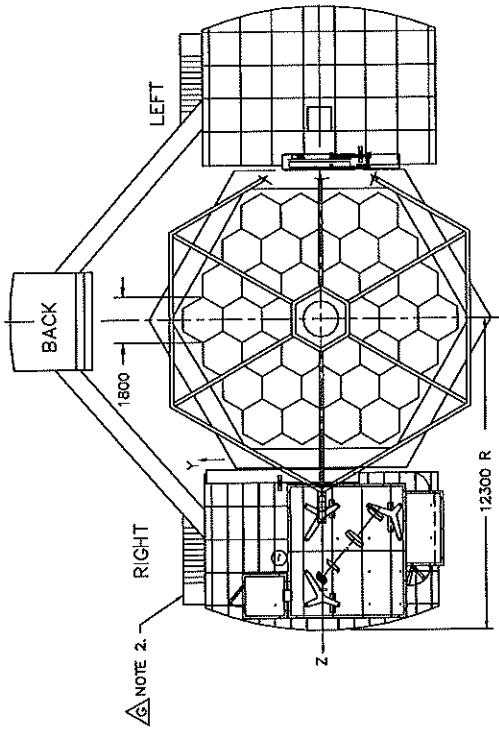
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Appendix B List of Drawings — Design

1. H0000 Family Tree, Entire
2. H0001 General Design Tree
3. H0010 General Telescope Assembly
4. H0102 Final Optical Layout
5. H0148 Super-Duper Camera Layout
6. H0200 Control Functions, Schematic
7. Un-named schematic





(-1) GENERAL ASSEMBLY

NOTES:

- [1] 6500 IS NOMINAL FOCUS.
[2] LANDING AND STAIR AS NOTED BY
STEVE VOGT, 3-21-92.

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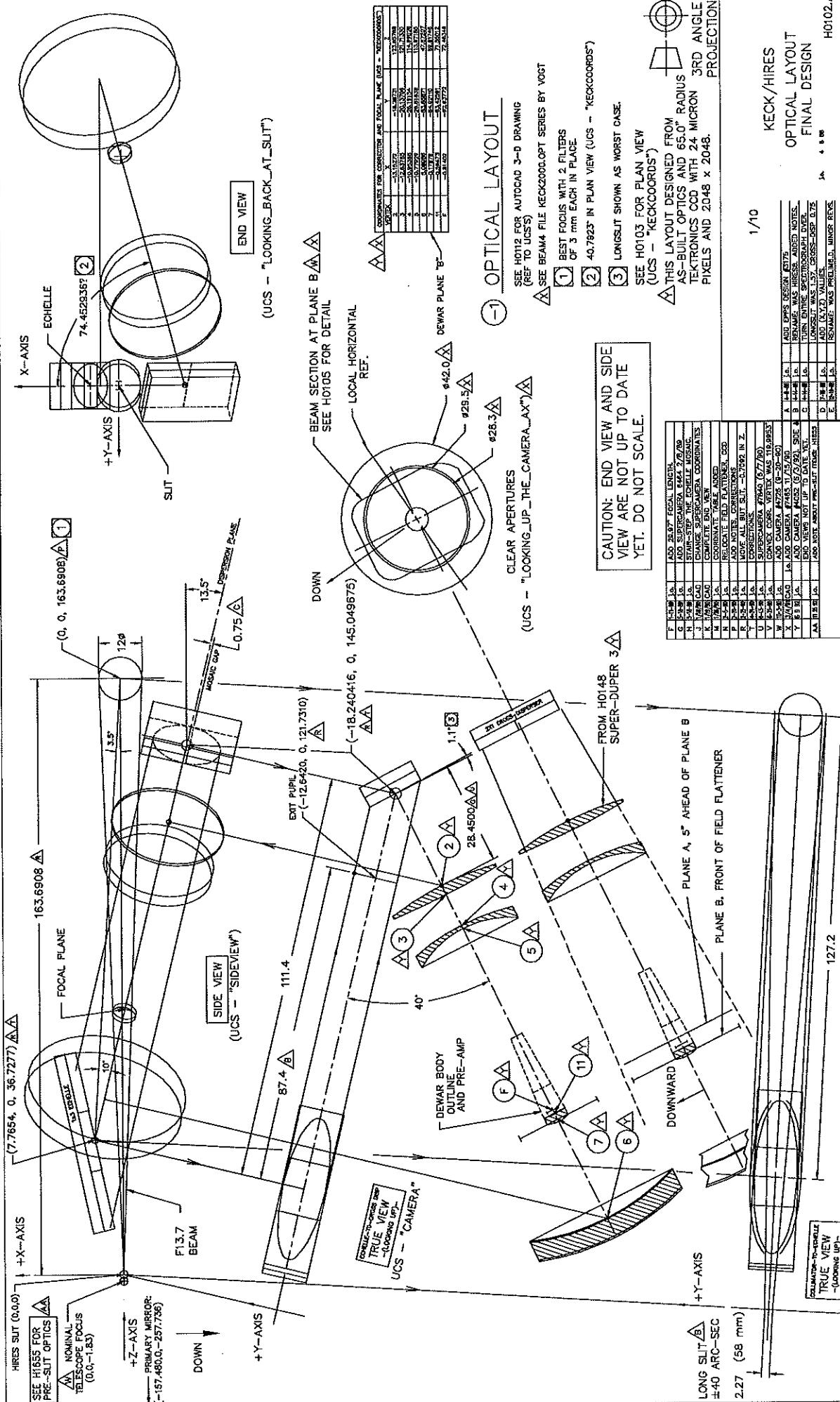
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GENERAL ASSEMBLY

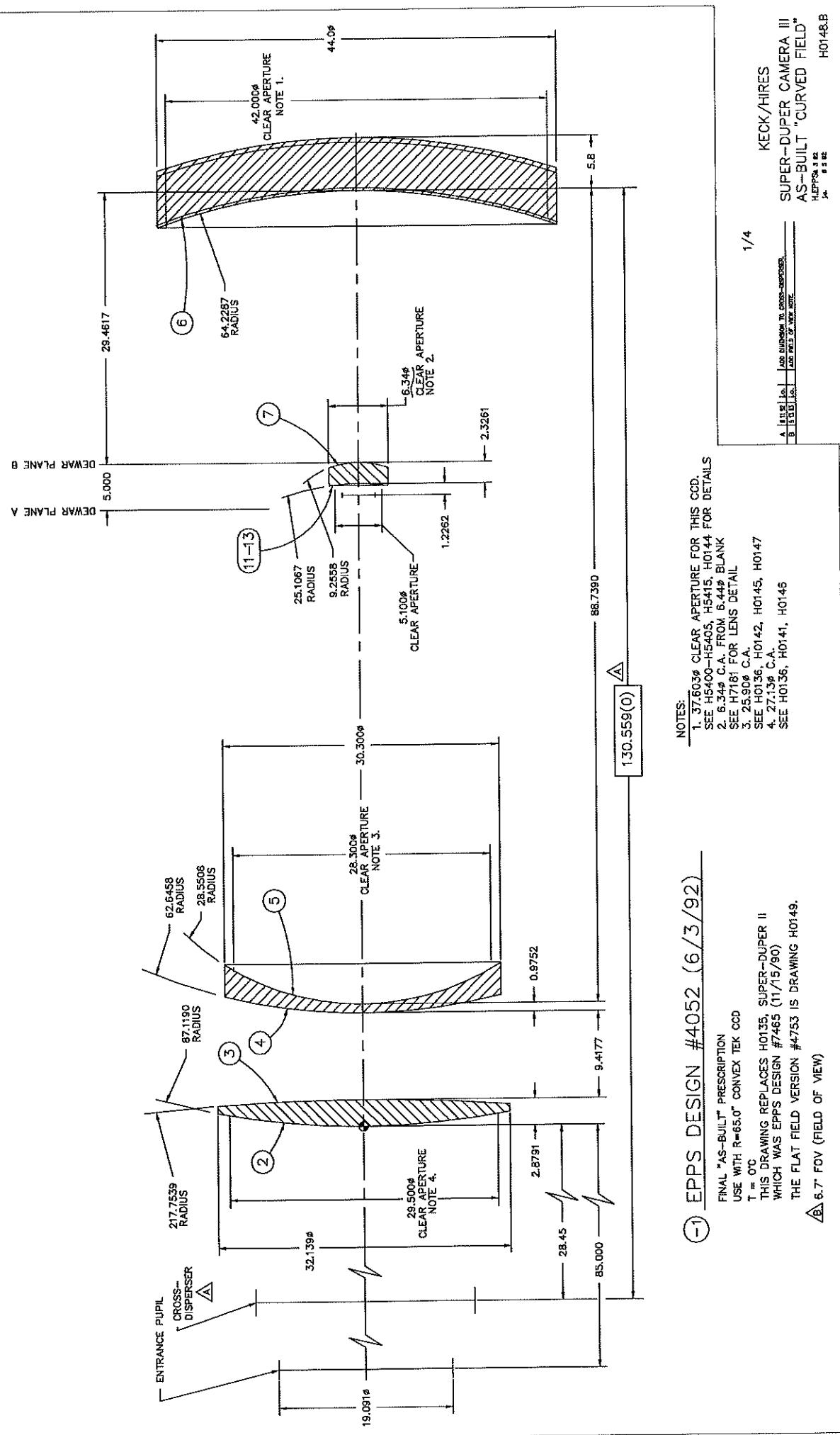
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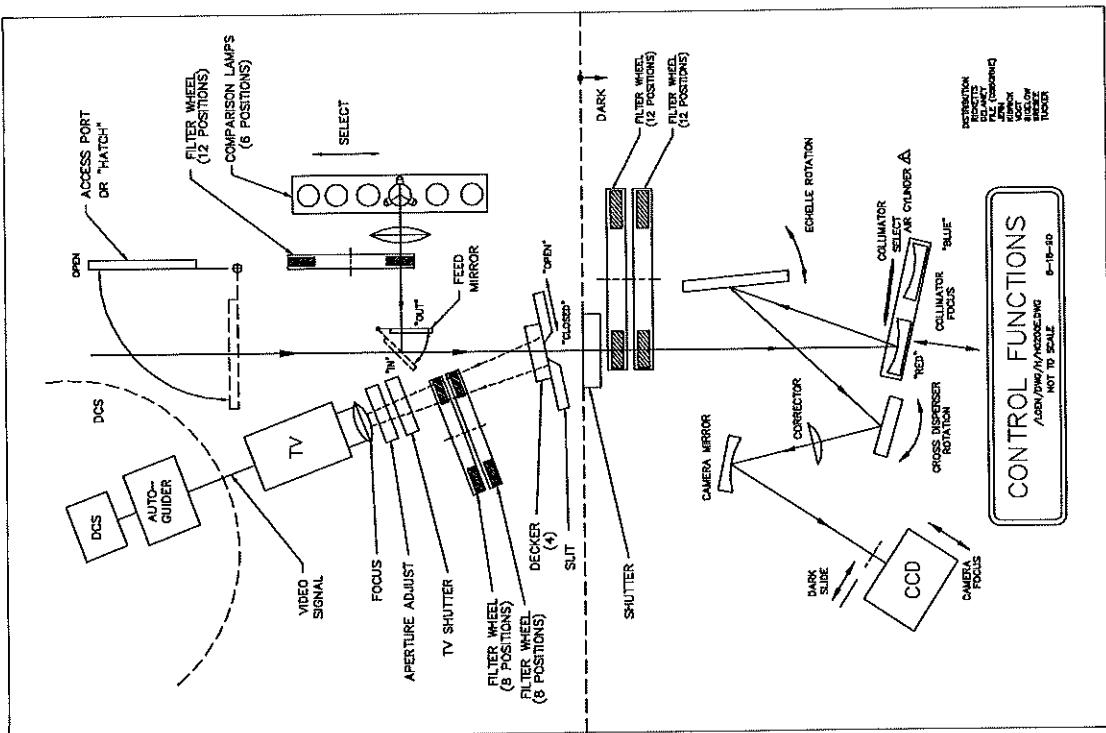
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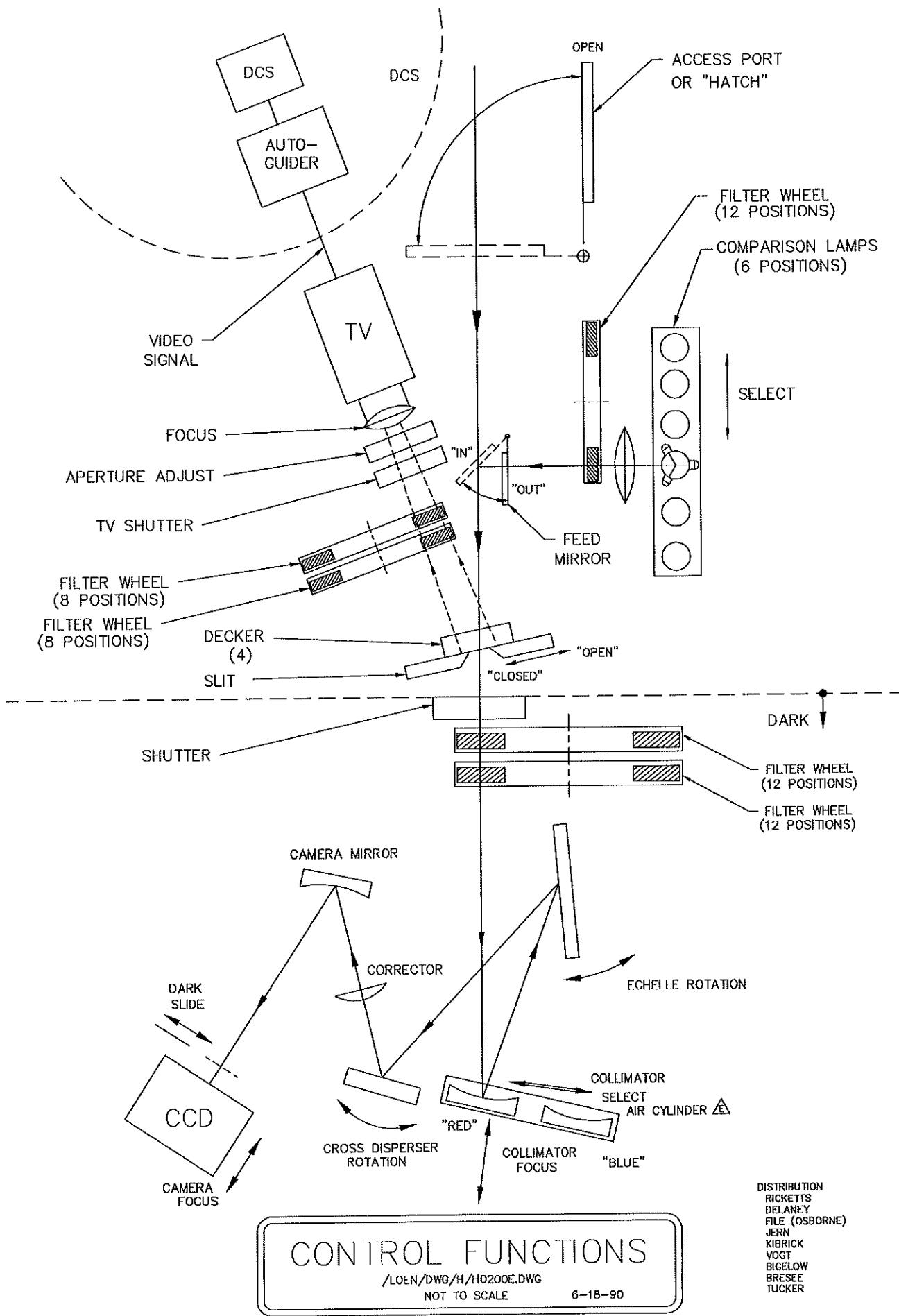




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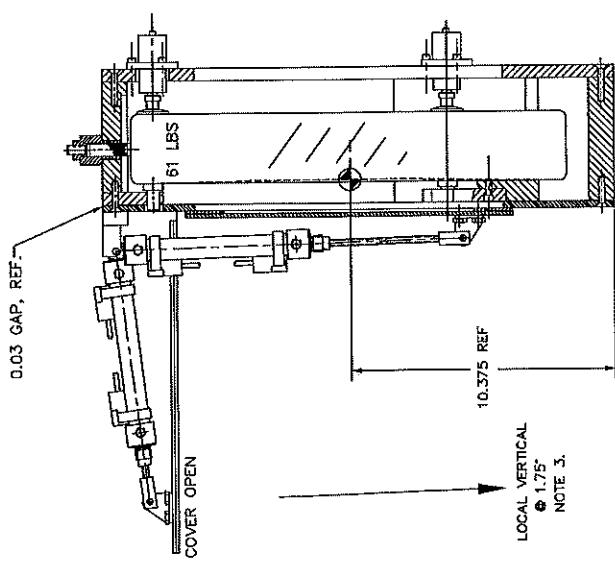
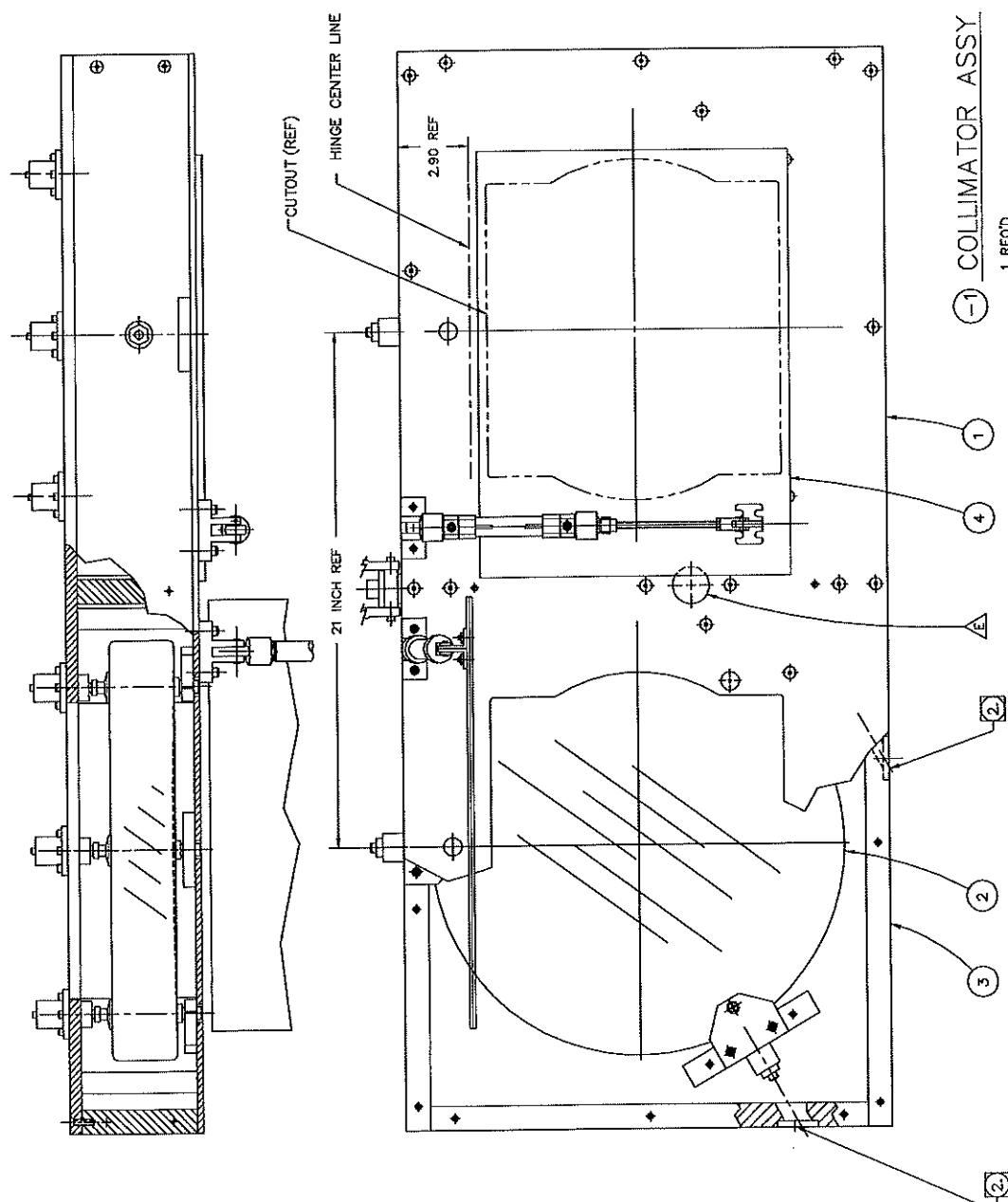
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Appendix C List of Drawings — Collimators

1. H2005 Locating Tree
2. H2000 Collimator Assembly
3. H2206 Spring Pad Assembly
4. H2209 Support Detail (and Temperature Compensator)
5. H2401 Transfer Mechanism (Shuttle)
6. H2415 Shuttle Pneumatic Diagram
7. H2460 Stage Locks
8. H2600 Rear View of Cell
9. H2610 Focus Drive Schematic
10. H2620 Drive Detail
11. H2623 Focus Drive Motor
12. H2660 Motor/Gearbox Assembly

H2000	COLLIMATOR ASSEMBLY		H2600	COLLIMATOR FOCUS		H2634	COLLIMATOR FOCUS	SPRING ASSY
H2001	COL. TOP PLATE DET	H2401	COL. MIRROR TRANSFER MECH.	H2601	" " DETAIL	H2635	COLLIMATOR FOCUS	SPRING ASSY
H2100	RED MIRROR			H2602	" " DETAIL	H2636	" " SPRING DETAIL	
H2110	BLUE MIRROR	H2404	COL. SUPPORT STAND DETAIL	H2610	" " DRIVE SCHEMATIC	H2637	" " SPRING DETAIL	
H2200	COLLIMATOR CELL ASSY	H2405	COL. CHANGER CORNER ASSY	H2620	" " DRIVE BLOCK ASSY	H2638	" " SPRING DETAIL	
H2201	" " DETAIL	H2406	" " " DETAIL	H2621	" " " DETAIL	H2639	" " DETAIL	
H2202	" " DETAIL	H2407	COL. SUPPORT VEE PADS	H2622	" " " DETAIL	H2640	DIAL INDICATORS ASSY	
H2203	" " DETAIL	H2408	" " BRACE DETAIL	H2623	" " " MOTOR	H2641	" " DETAIL	
H2204	" " DETAIL	H2410	COL. CHANGER FLAT MOUNT	H2624	" " BELT TENSIONER ASSY	H2645	MOTOR GUARD DETAIL	
H2205	" " DETAIL	H2411	" " " DETAIL	H2625	" " " DETAIL	H2650	COLLIMATOR FOCUS	SHUTTLE DETAIL
H2206	SPRING PAD ASSY	H2412	" " " DETAIL	H2627	" " BRAKE MOD.	H2651	" " " DETAIL	
H2207	" " DETAIL	H2413	" " " DETAIL	H2628	" " BRAKE ASSY'S	H2652	" " " DETAIL	
H2208	TOOL	H2415	" " PNEUMATIC DIAGRAM	H2629	" " BRAKE DETAILS	H2653	" " " DETAIL	
H2209	RADIAL SUPPORT ASSY	H2416	" " AIR CONTROL PANEL	H2630	" " LIMITS	H2654	" " " DETAIL	
H2210	" " DETAIL	H2417	" " " DETAIL	H2631	" " " DETAIL	H2655	" " " DETAIL	
H2300	COVER ASSEMBLY AND AIR SCHEMATIC DIAGRAM	H2420	" " LINK DETAIL	H2632	" " " DETAIL	H2656	" " " DETAIL	
H2421	" " "	H2422	" " AIR CYL. CLEVIS	H2633	" " " DETAIL	H2657	" " " DETAIL	
H2301	COVER/CYLINDER LAYOUT	H2423	" " AIR CYLINDER	H2603	ALIGNMENT FIXTURE	H2658	" " " DETAIL	
H2302	AIR CYLINDER & DOOR	H2460	COL. STAGE LOCKS	H2604	" " DETAIL	H2660	" " GEAR REDUCER	OBSCOLE
H2303	SPEED CONTROL MOD.	H2461	" " DETAIL	H2605	" " DETAIL	H2661	" " " ADAPTER	
H2304	" " DETAIL	H2462	" " DETAIL					
		H2463	" " DETAIL					
H2450	COLLIMATOR JUNCTION BOX ("AC STAGE")							
H2451	" " DETAIL							
H2452	COL. FOCUS BOX							
H2453	" " DETAIL							
A [see fig. 1] [see fig. 2]						Keck/Hires Collimator Locating Tree	Jno. 828 H205.A	



H2000-E	1	2	3	4
COVER ASSY	2	1	1	2
MIRROR CELL ASSY	1	2	2	2
COLIMATOR MIRROR	1	1	1	1
BASE PLATE	1	1	1	1

1/2

KECK/HIRES
COLIMATOR
ASSEMBLY

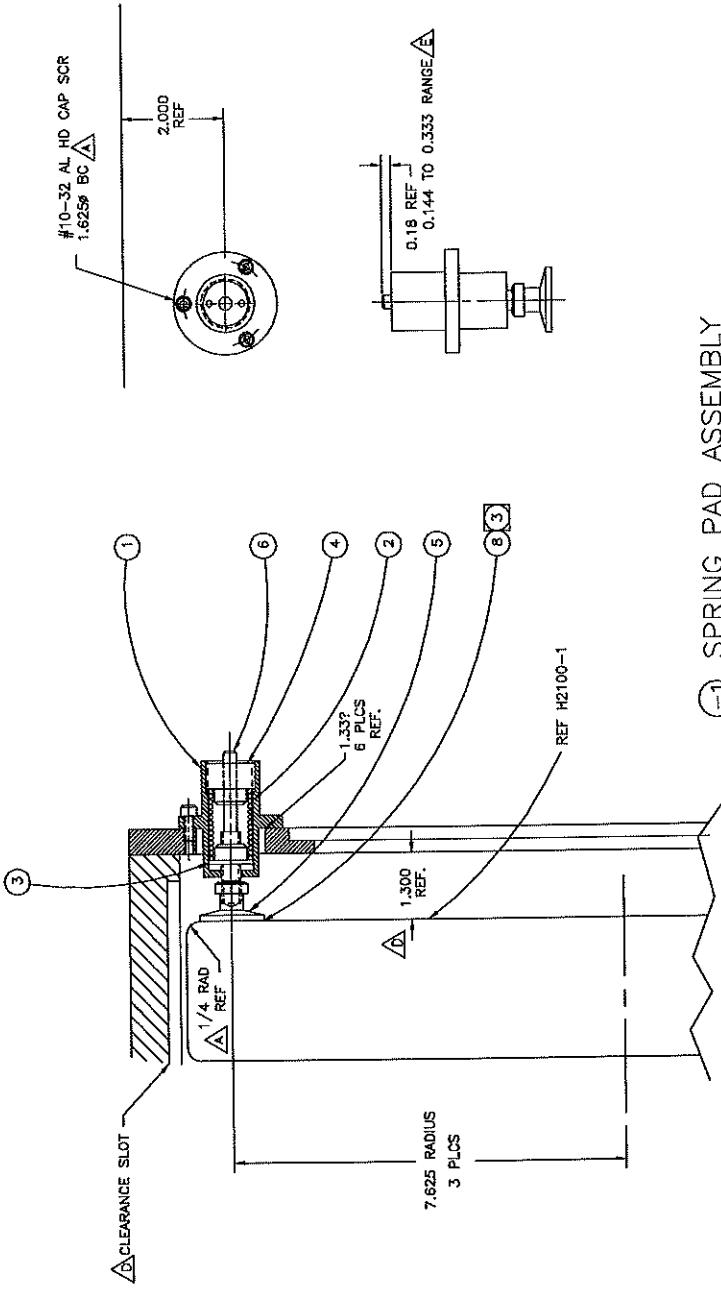
H2000.E

1. REF H2301, COVER/CYLINDER DIMENSIONS.
 2. COLIMATOR ACCESS HOLE (5 PLACES).
 3. MIRROR IS MOUNTED SQUARE TO THE CELL.
 THE CELL IS TILTED 1.75° FROM VERTICAL.

1 REQD

NOTES:

- A [] B [] C [] D [] E []
 1. [] HORN REFL. CHAMBERS.
 2. [] UPDATE TO AS-BUILT.
 3. [] MIRROR CLEARED OF THE MOUNTS.
 4. [] FLOOR.
 5. [] AND 1.5MM ALUMINUM.
 6. [] 10.375 IN.
 7. [] 11.22 IN.



(-1) SPRING PAD ASSEMBLY

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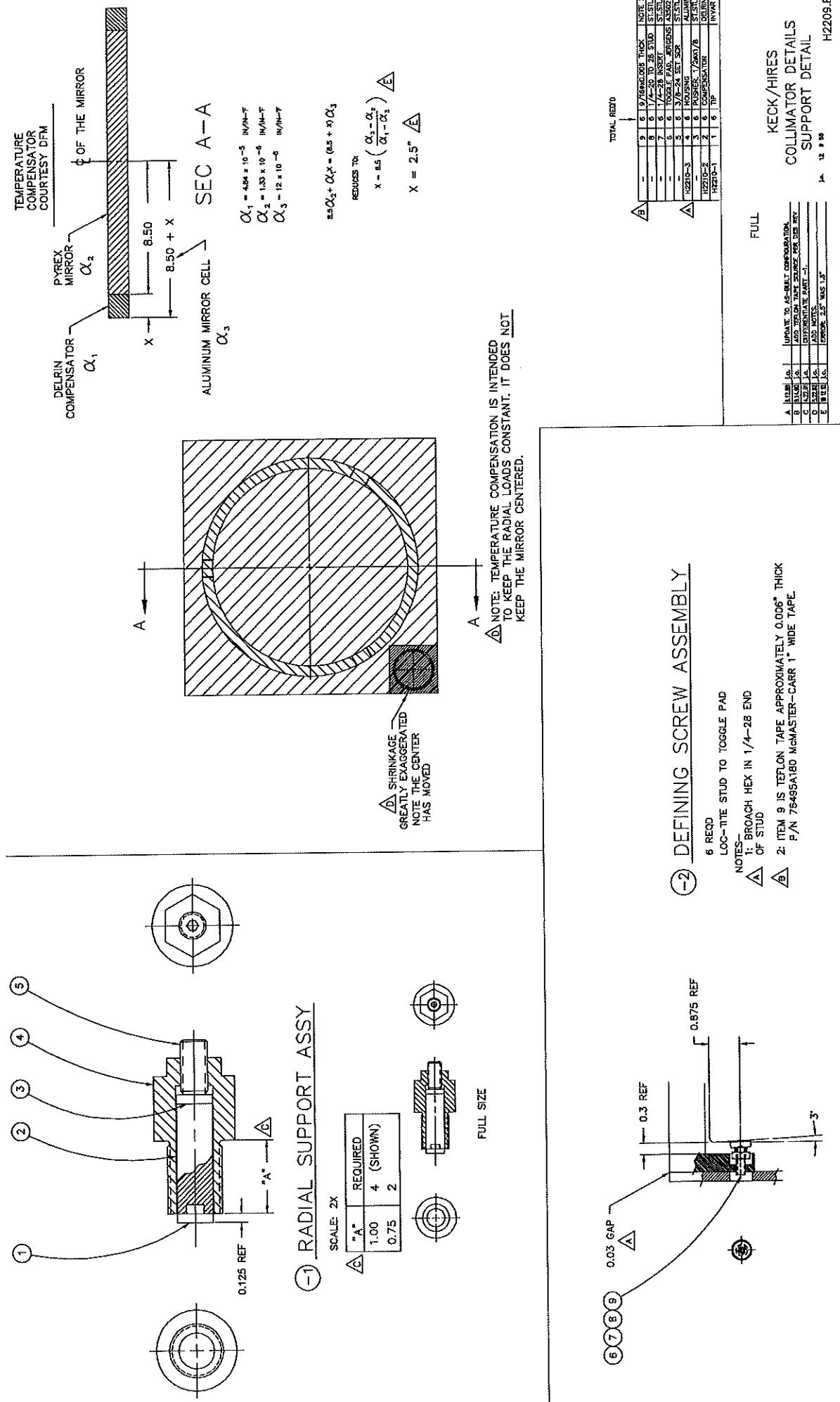
1. SPRING DATA
 3.5' FREE LENGTH
 0.085" STL WIRE
 1.19 STL HEIGHT
 (14 TURNS)
 OD IS 0.845,
 DESIGNED FOR 7/8" HOLE
 $K = 15.0 \text{ lbs/in}$
 DESIGN 2.115" COMPRESSION (32lbs)
 2. USE LOC-TITE ADHESIVE
 DESIGN 1.385"

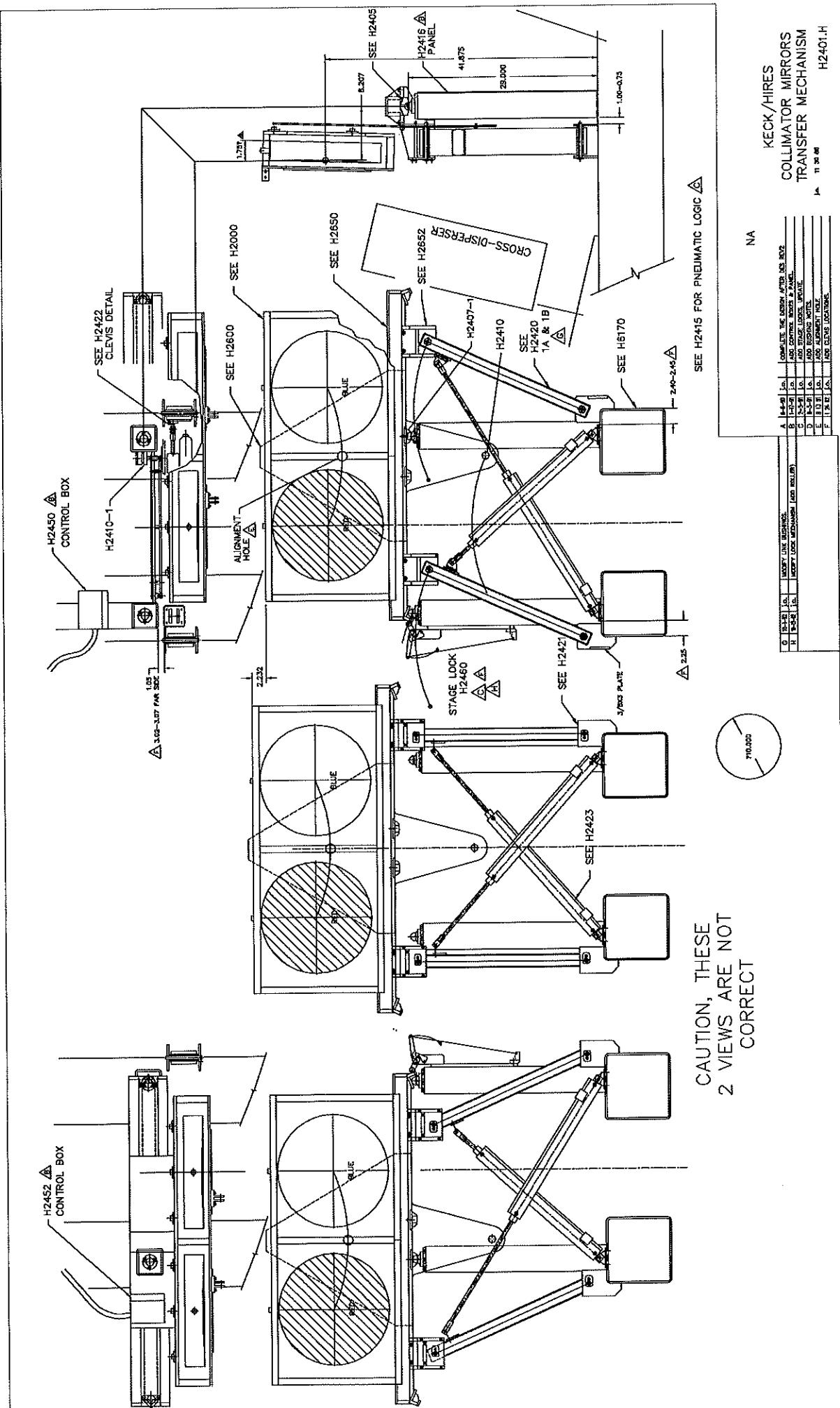
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P/N 764951AB0 MCMASTER-CARR 1" WIDE TAPE.**

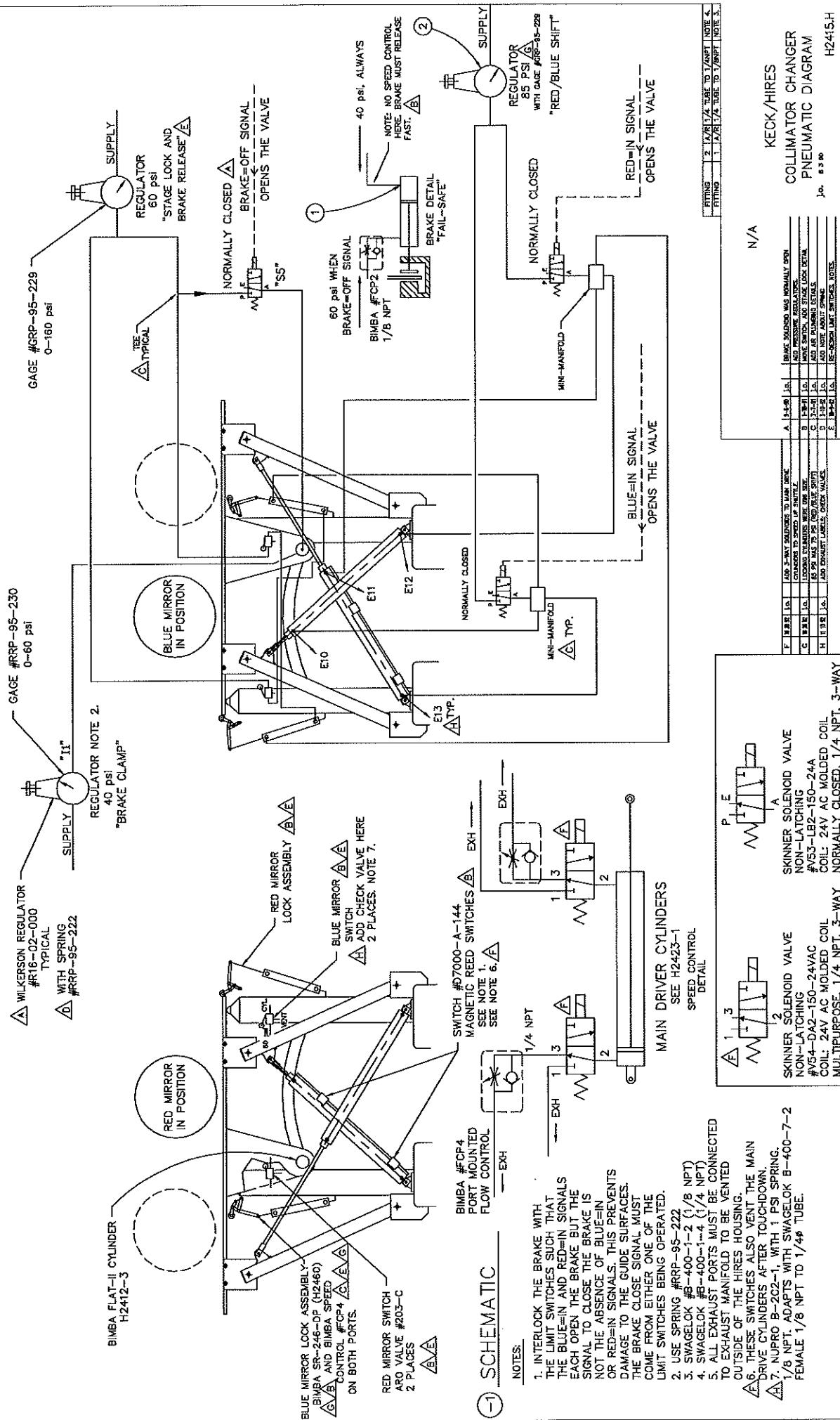
E. 101 S.E. 121. ADY NOTE CHURCH CULLOMATION		A 100% loc.	100% mes- C-205. 1/4 mes- C-402.
		NOTE	
C	5.5	10.	100% mes- C-205.
C	5.5	10.	100% mes- C-205.
C	5.5	10.	100% mes- C-205.
D	5.5	10.	100% mes- C-205.

KECK/HIRES
COLLIMATOR DETA
SPRING PAD AS

H2205







(-1) STAGE LOCK ASSEMBLY

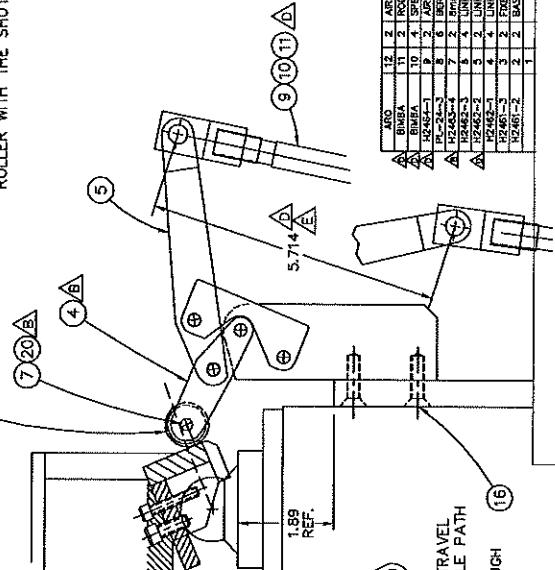
2 RECD. MAKE MIRROR IMAGES

NOTES:
△ 1. BIMBA AIR CYLINDER #SP-246-DP

- △ 2. BIMBA SPEED CONTROL #CP-4, 1/4 NPT
- △ 3. BIMBA ROD CLEVIS #D-231-3
- △ 4. ARO VALVE #D03-C 3-WAY VALVE
- △ 5. SEE H2445 FOR PNEUMATIC LOGIC DIAGRAM.
- △ 6. GRIND 45° CHAM TO 0.070 WIDE ALSO, CLEARANCE BEVEL ON LOWER PIECE
- △ 7. ENGAGEMENT FORCE IS ABOUT 200 LBS.
- △ 8. PLATE SHOWN IS SLOTTED TO ALIGN THE ROLLER WITH THE SHUTTLE

△ (9) (2) NOTE 7.

△ (7) (2)△



TIRE DETAIL

△

△ (5)

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△ (6)

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△ (3)

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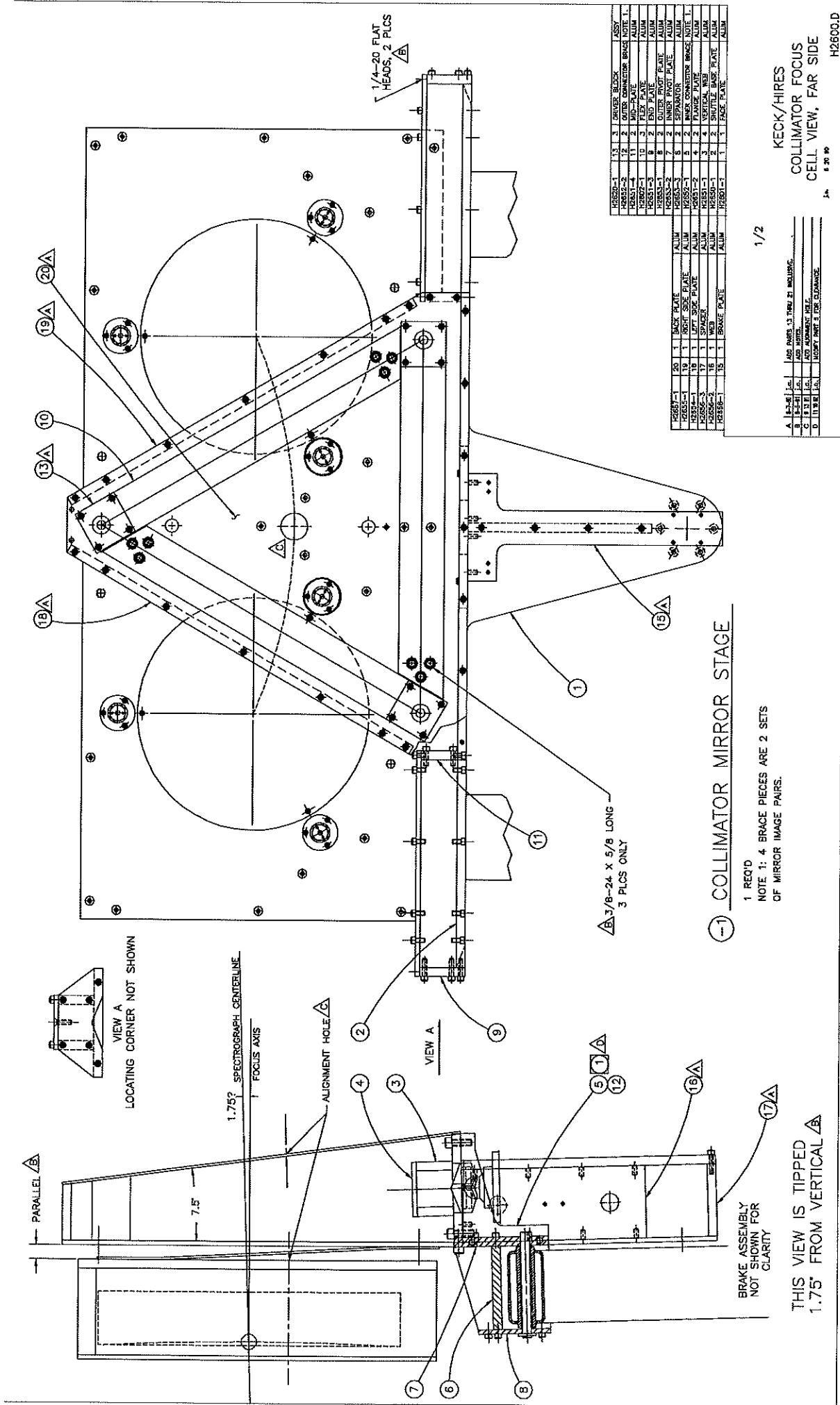
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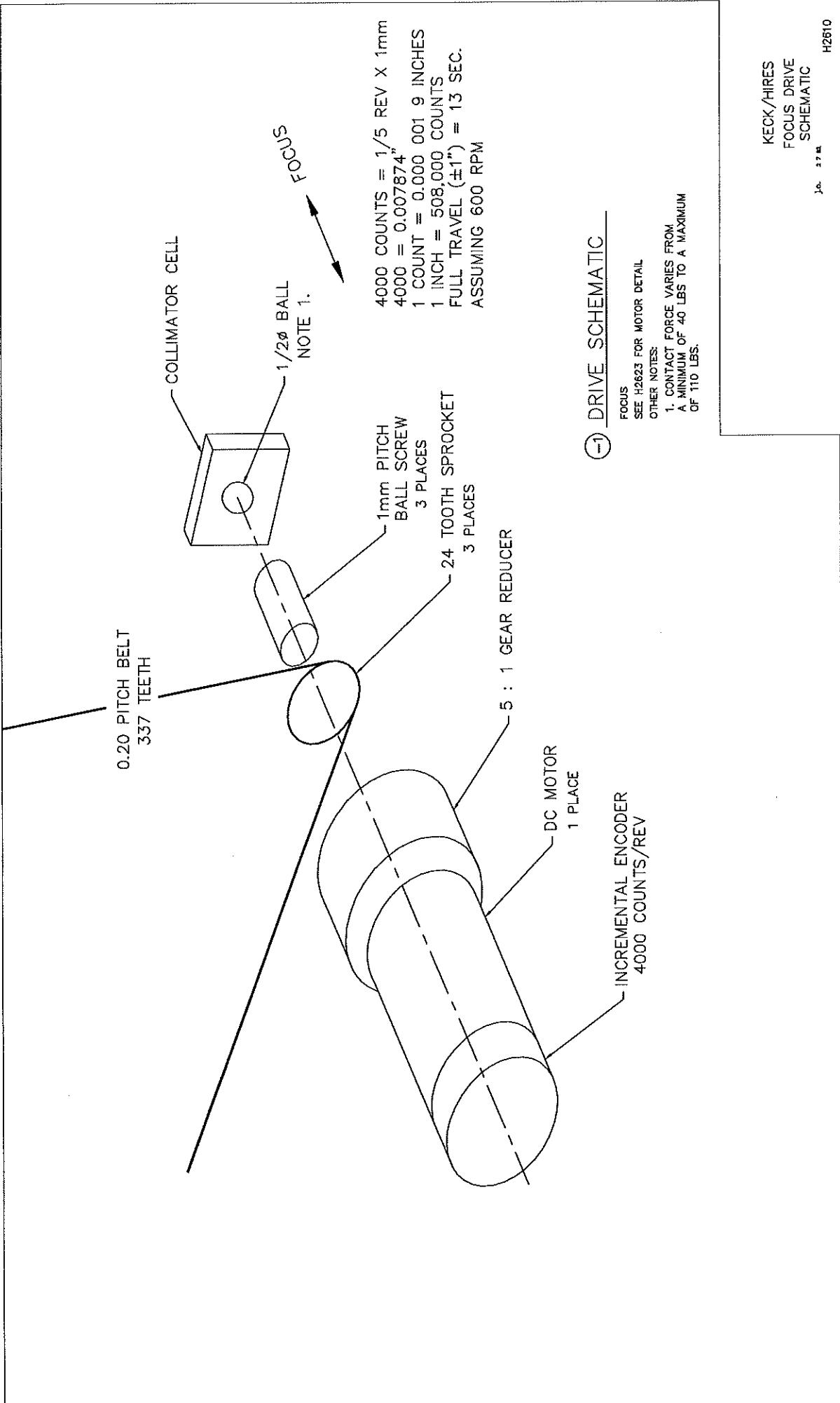
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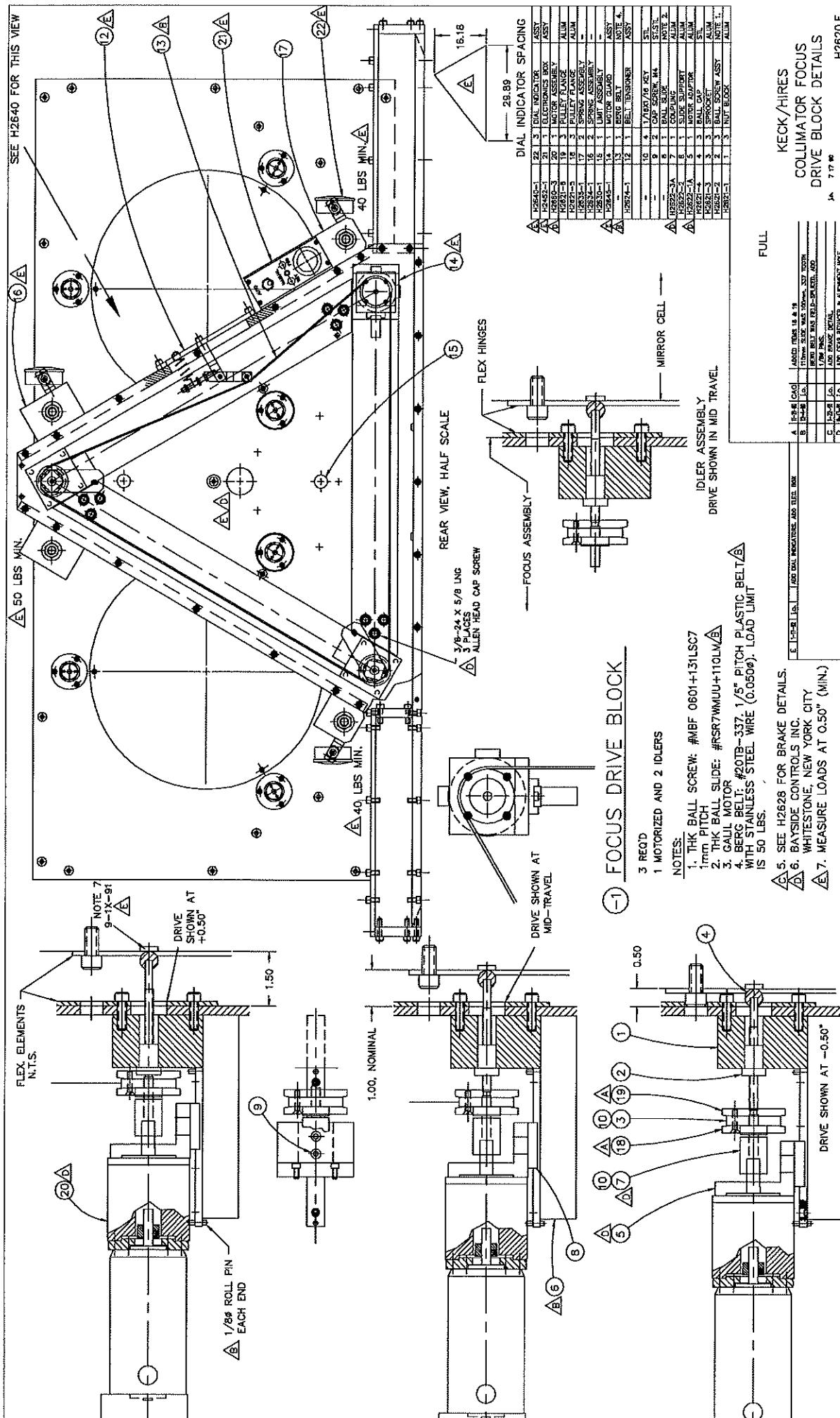
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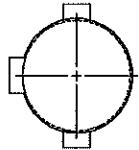
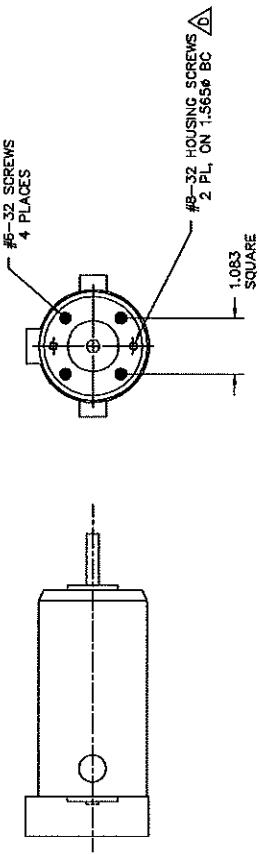


PITTMAN HIGH ALTITUDE
BRUSHES, TREATED
#60-47-4
PITTMAN
HARLEYSVILLE, PA

KEYSEAT FOR 1/16 KEY
MUST BE ADDED



1.008 x 0.10 SHOELDR \triangle



(-1) MOTOR

MOTOR:
GAUL #MOTOR-50-1000 ①
GAUL MOTION CONTROL, INC
PALO ALTO, CALIFORNIA
(415) 964-6494

MOTOR SPECIFICATIONS:
PITTMAN #4204-B32
AND NEWELL-PICKARD ENCODER
SERIES #EDS-601C
 \triangle ① THIS MOTOR IS NOW CUSTOM-MOUNTED
TO HP HEDS ENCODER; 10 WEEK DELIVERY

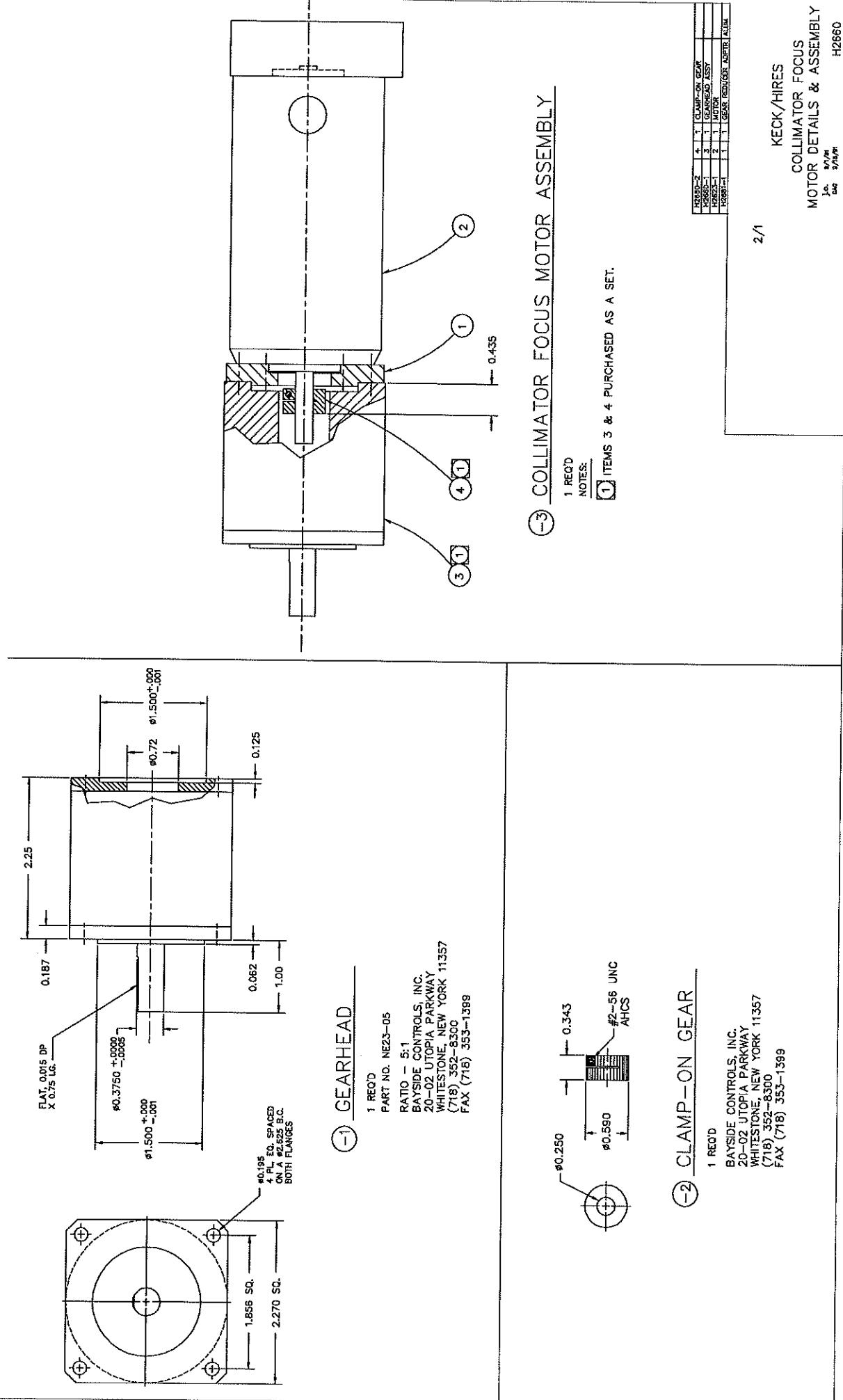
MOTOR ALSO COMES WITH BEI
ENCODER ATTACHED.
 \triangle DO NOT USE THIS MOTOR. TEMPERATURE
RANGE IS ONLY +80°C TO -10°C.

BEI ENCODER
MODEL MX 212-25-1000
SAN MARCOS, CA
 \triangle TEMPERATURE RANGE: +80°C TO -10°C
 \triangle LED LIGHT SOURCE

NOTE: THIS MOTOR IS USED
IN THE FOLLOWING ASSEMBLY
DRAWINGS:
H2620, COLLIMATOR FOCUS
H1110, FILTER WHEEL, PRIMARY
H1524, SLIT
H1538, SLIT ACCESSORY SERVER
H6705, COMPARISON SOURCES
H7300, CCD FOCUS / TEST ECHELLE DRIVE
 \triangle H4264, CROSS-DISPENSER DRIVE
 \triangle H7312, CCD FOCUS

FULL

KECK/HIRES
COLLIMATOR FOCUS
DRIVE MOTOR
724.00
H2623.D

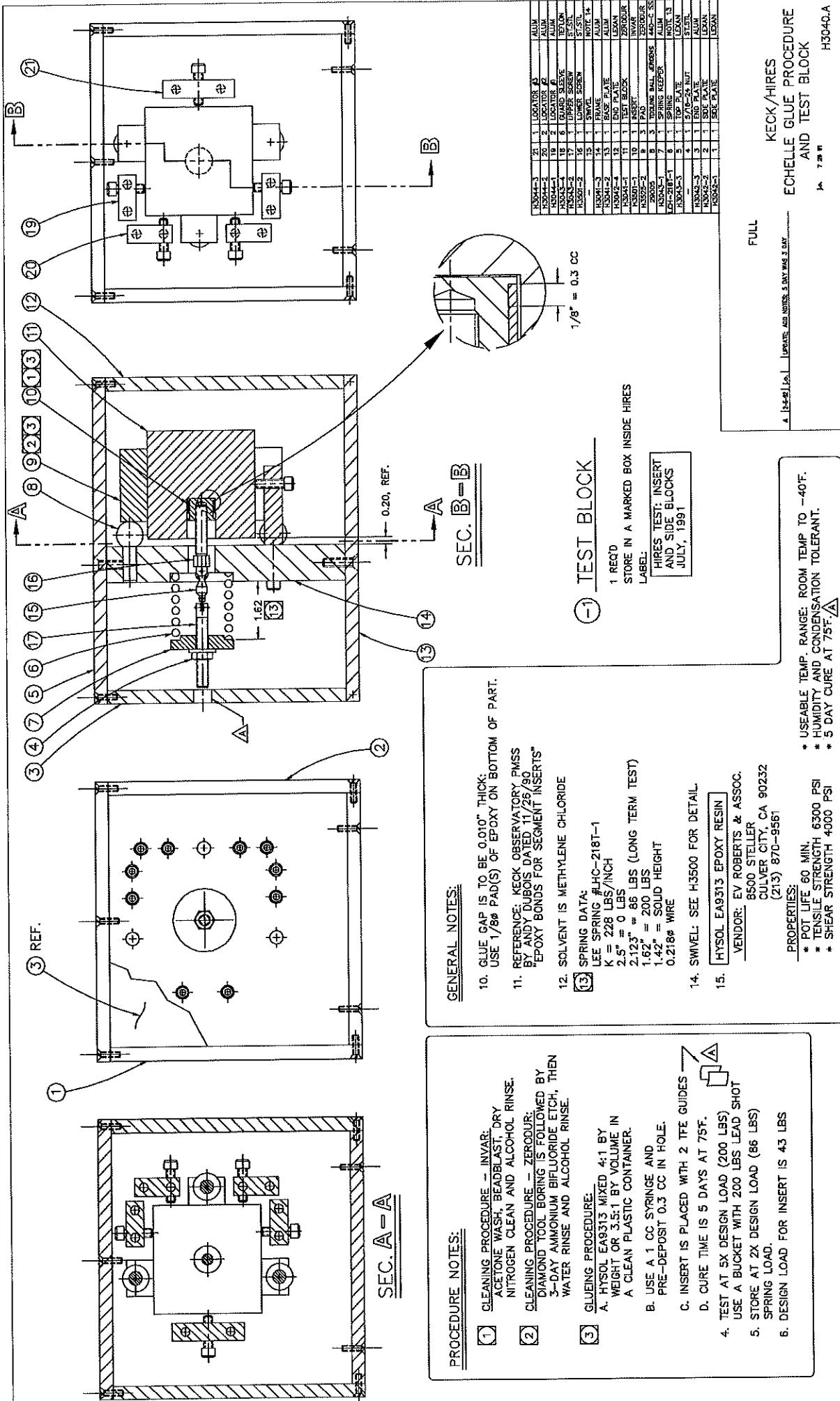


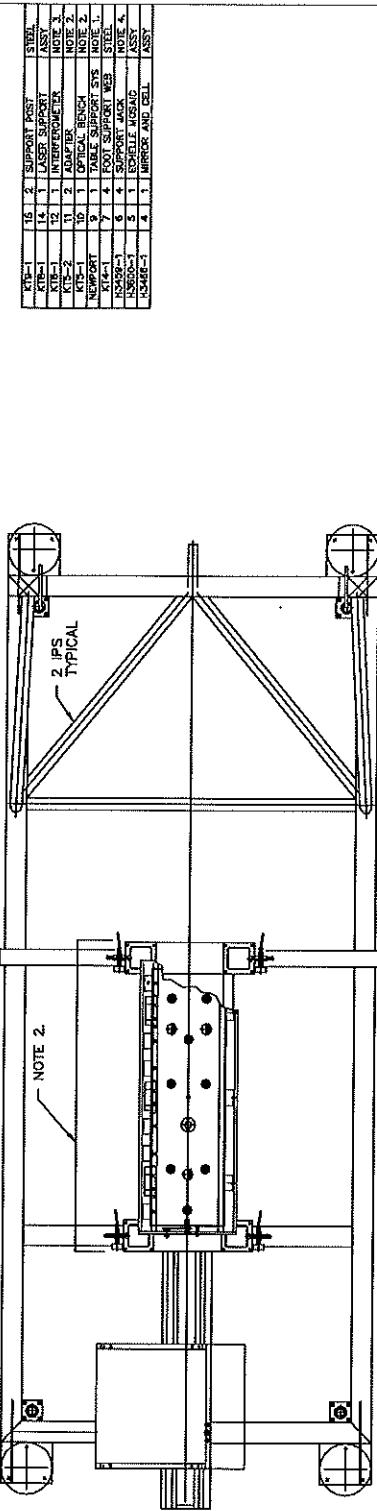
Appendix D List of Drawings

— Echelle Mosaic

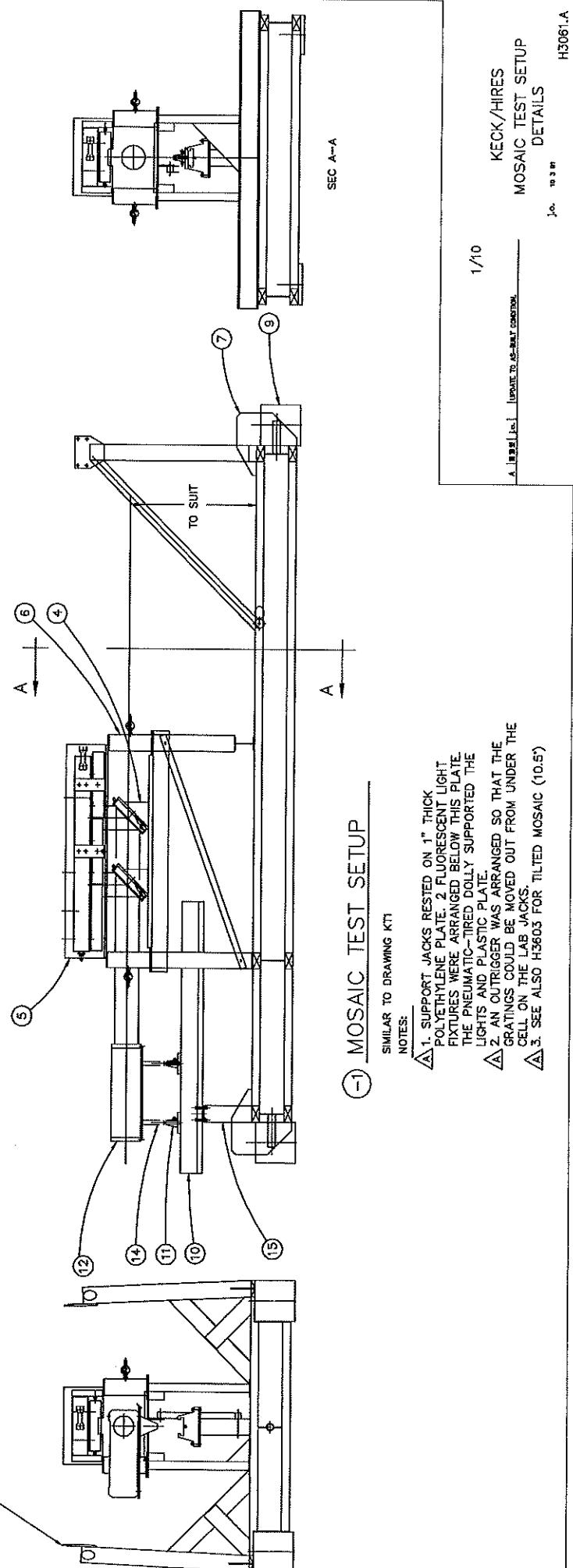
1. H3005 Locating Tree
2. H3040 Glue Procedure and Test Assembly
3. H3061 Mosaic Test Setup
4. H3100 Mosaic Assembly on Subplate
5. H3200 Echelle Drive Assembly and Schematic
6. H3240 Echelle Pivot Assembly
7. H3244 Alignment Mirror — Echelle Pivot
8. H3300 Subplate Support, Column
9. H3340 Subplate Support, Lateral Support
10. H3345 Lateral Support Detail
11. H3425 Echelle Grating Blaze Detail
12. H3454 Echelle Grating and Side Pad Assembly
13. H3470 Subplate
14. H3500 Grating Back Support Assembly
15. H3510 Grating Side Support, Type 1
16. H3511 Side Support Detail, Flex Rod
17. H3520 Side Support, Type 2
18. H3600 Echelle Cell Assembly
19. H3603 Testing, Interferometer Layout
20. H3700 Cover Assembly
21. H3703 Air Control Panel
22. H3801 Subplate Finite Element Analysis Results
23. H7300 Encoder Assembly and Drive
24. H7330 Limits, Drive

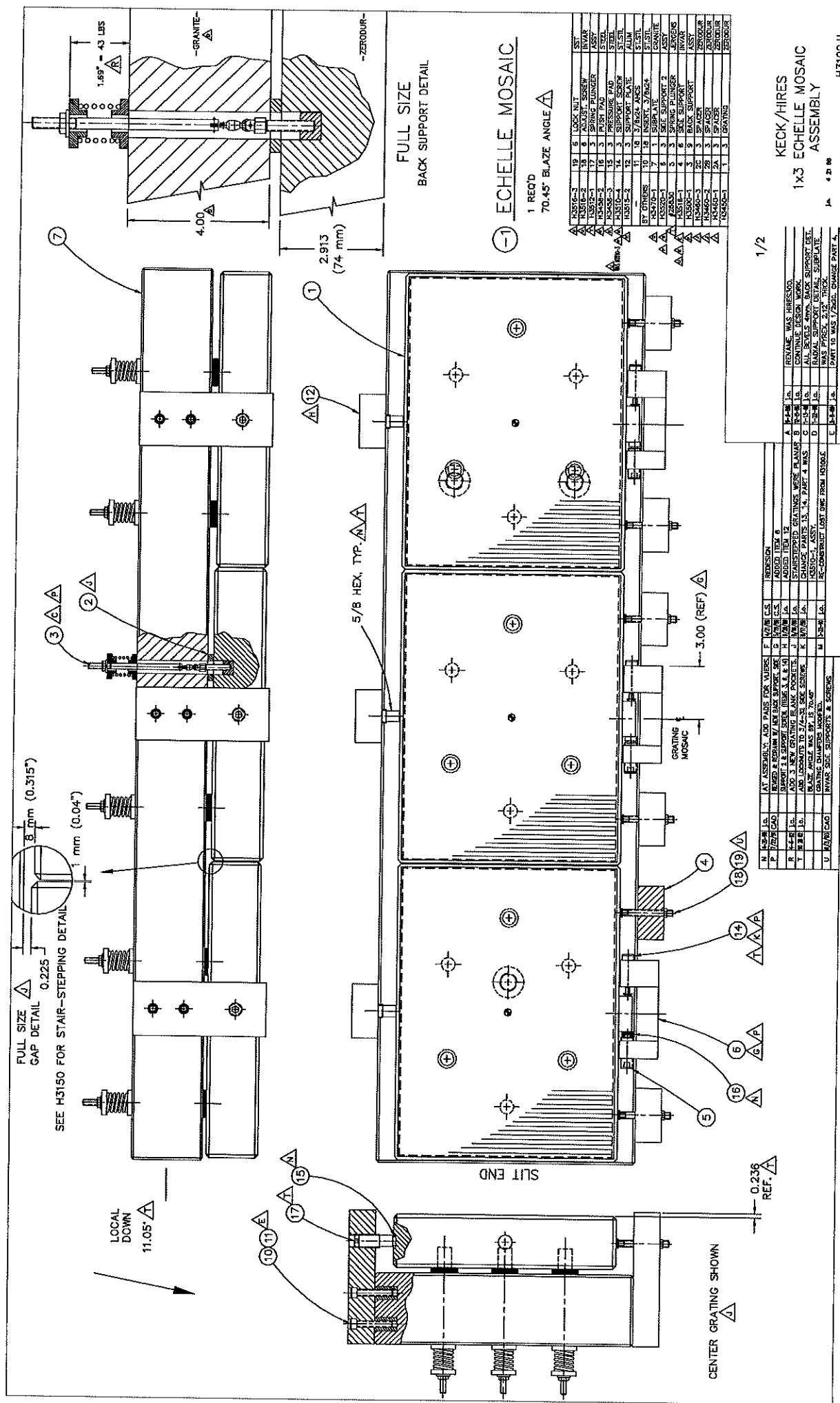
LOCATING TREE (THIS DRAWING)			
H3005		H3100 MOSAIC ASSEMBLY	
		H3150 MOSAIC: SHOWING STAIR-STEP	
		H3151 70.5° BLAZE ANGLE	
		H3200 DRIVE ASSEMBLY	
H3040	GLUE PROCEDURE	H3240 PIVOT ASSEMBLY	H3201 DRIVE LAYOUT
H3041	GLUE PROCEDURE	H3241 PIVOT DETAIL	H3202 " MOUNT BRACKETS
H3042	GLUE PROCEDURE	H3242 PIVOT DETAIL	H3203 " "
H3043	GLUE PROCEDURE	H3243 PIVOT DETAIL	H3205 " LIMITS
H3044	GLUE PROCEDURE	H3244 ALIGNMENT MIRROR ASSY	H3206 " "
H3045	GLUE PROCEDURE	H3300 COLUMN SUPPORT ASSEMBLY	H3207 " FLEX PLATE
H3046	GLUE PROCEDURE	H3310 " DETAIL	
H3050	JUNCTION BOX	H3315 WRENCH DETAILS	
H3051	JUNCTION BOX	H3320 COLUMN DETAILS, ASSEMBLY STEPS	
H3052	JUNCTION BOX	H3340 LATERAL SUPPORT ASSY	
H3061	MOSAIC TEST DETAIL	H3345 " DETAIL	
H3075	CART ASSEMBLY	H3350 LONGITUDINAL SUPPORT ASSY	H3500 BACK SUPPORT ASSEMBLY
H3076	CART DETAIL	H3400 GRATING BLANK	H3501 " DETAIL
H3077	CART DETAIL	H3425 BLANK DETAIL	H3502 " " & CAP
H3600	CELL ASSEMBLY	H3425 BLAZE ANGLE DETAIL	SIDE SUPPORT
H3610	HOUSING ASSEMBLY	H3450 E3 GRATING	" FLEX ROD DETAIL
H3615	HOUSING DETAILS	H3451 E2 GRATING	" SPRING PLUNGER
H3620	HOUSING DETAILS	H3452 E1 GRATING	" DETAIL
H3625	HOUSING DETAILS	H3453 MASTERS	INVAR
H3630	HOUSING DETAILS	H3454 GRATING AND SIDE PAD	SIDE SUPPORT 2
H3635	HOUSING DETAILS	H3455 ORIGINAL GLUE FIXTURE	" DETAIL
H3640	SAFETY PLATES	H3456 " DETAIL	(ZERODUR BLOCKS)
H3641	SAFETY PLATES	H3457 "	
H3700	COVER ASSEMBLY	H3458 ALIGNMENT JACKS & MISC PARTS	
H3702	" DETAIL: KINEMATICS	H3459 JACK STAND ASSY	
H3703	AIR CONTROL PANEL		
H3704	" DETAIL		
H3705	"		
H3755	COVER DETAILS	H3460 SPACERS	
H3801	F.E.A. RESULTS	H3461 ALIGNMENT NOTES	
		H3462 ALIGNMENT NOTES	
		H3463 ALIGNMENT NOTES	
		H3464 ALIGNMENT NOTES	
		H3465 TEST MIRROR & CELL	
		H3466 TEST MIRROR & CELL	
		H3467 GRANITE SUBPLATE	
MISC:	ALIGNMENT TOOLING		
H3602	INTERFEROMETER LAYOUT		
H3603			

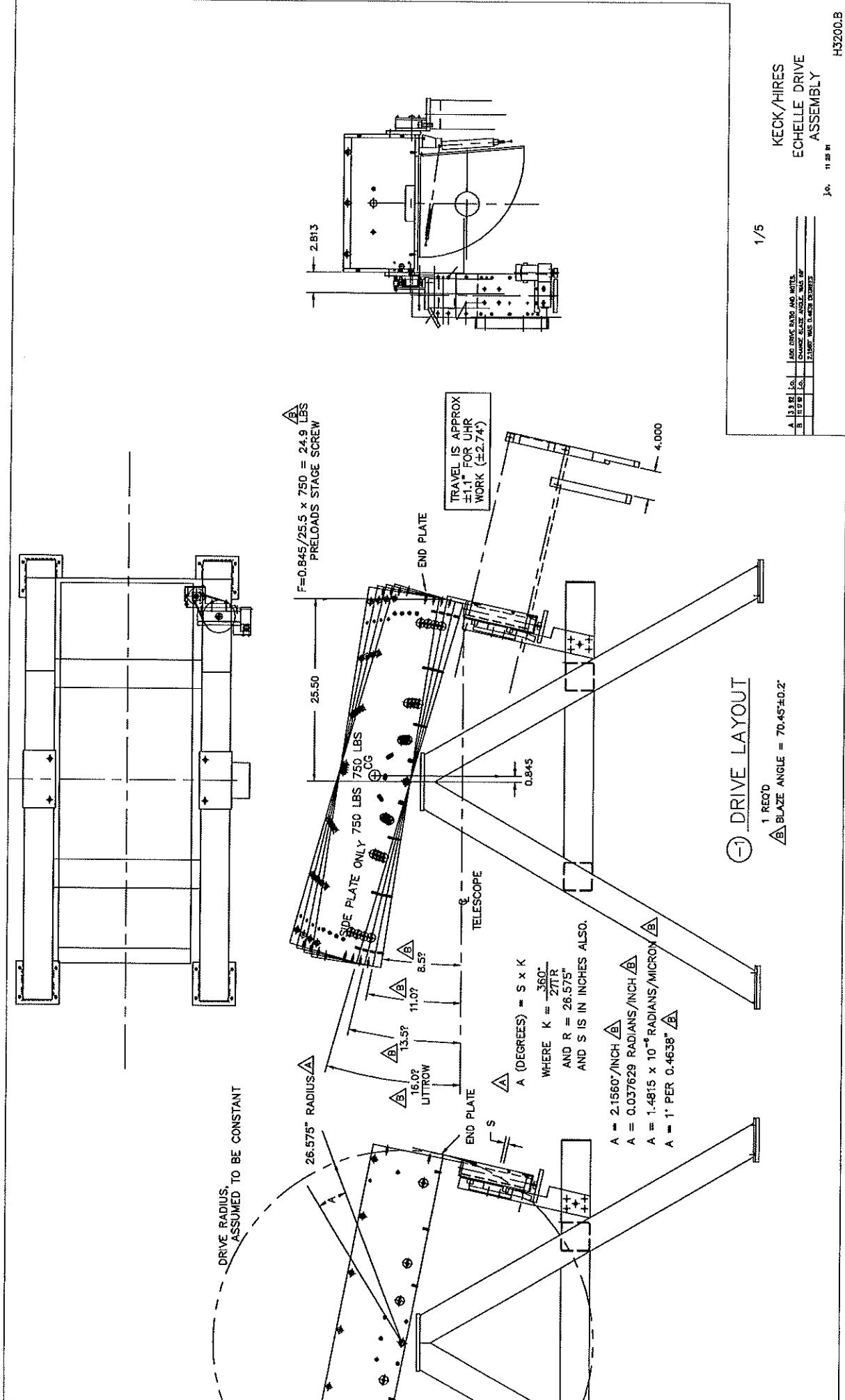


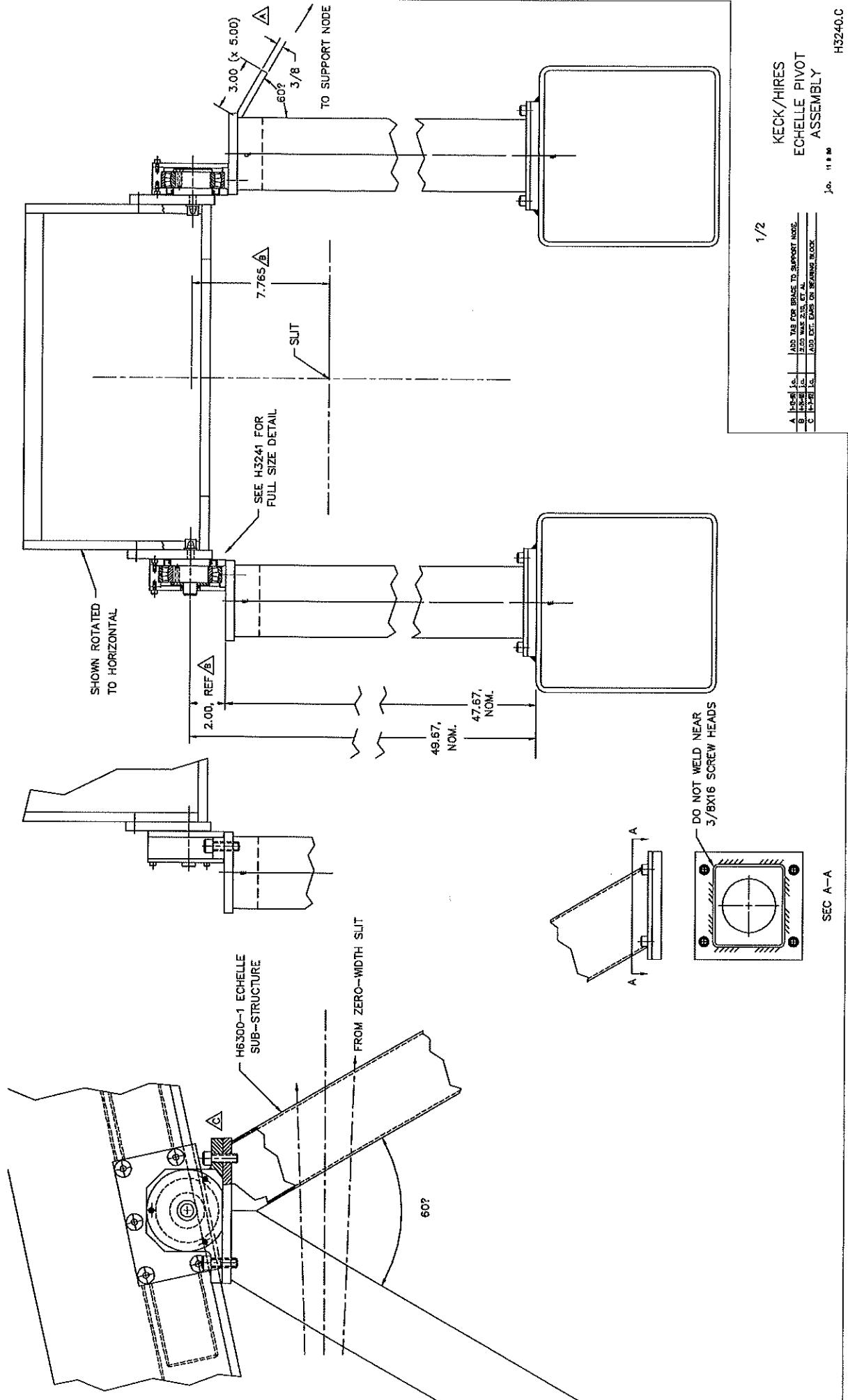


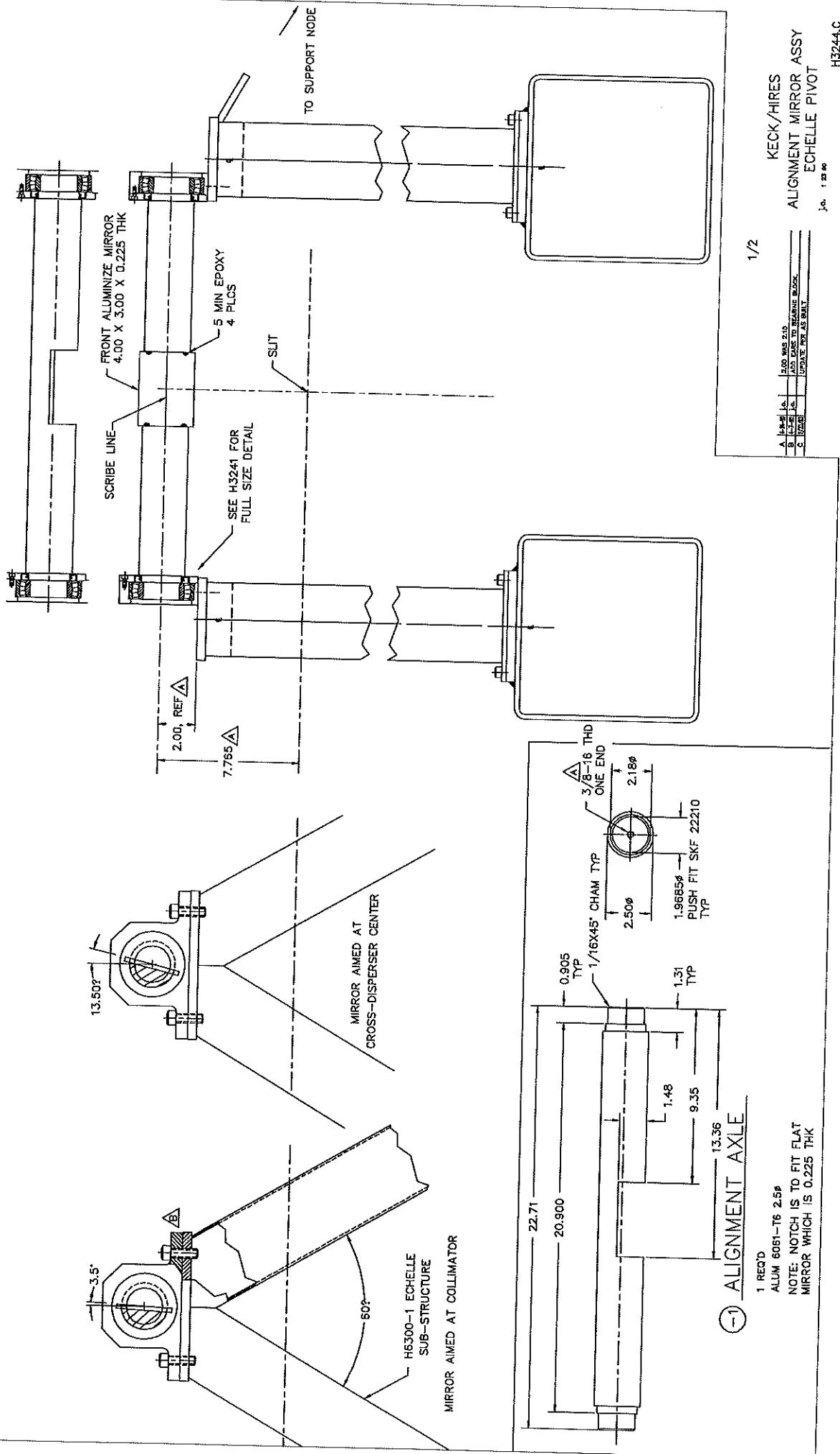
THIS STRUCTURE WAS USED
FOR TESTING THE KECK SECONDARY MIRROR.

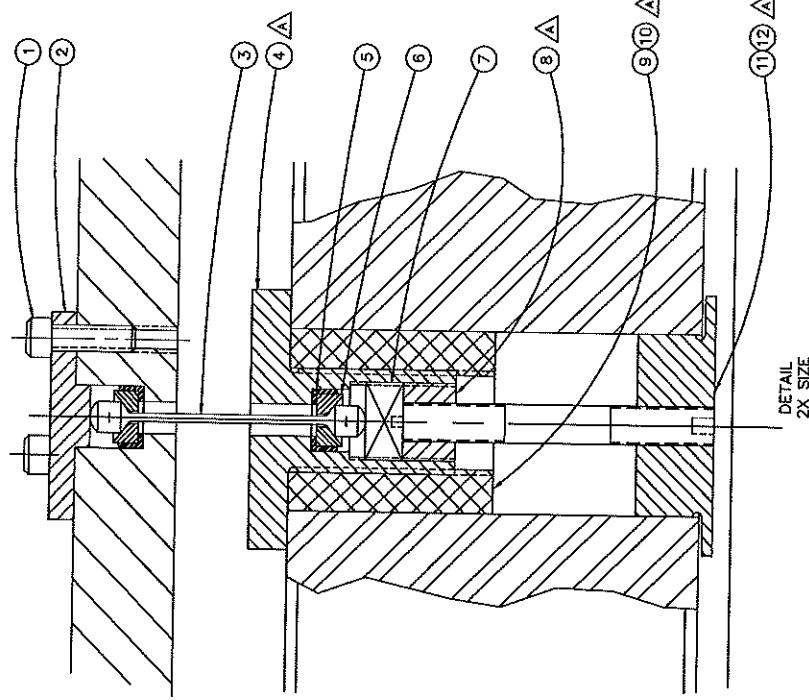
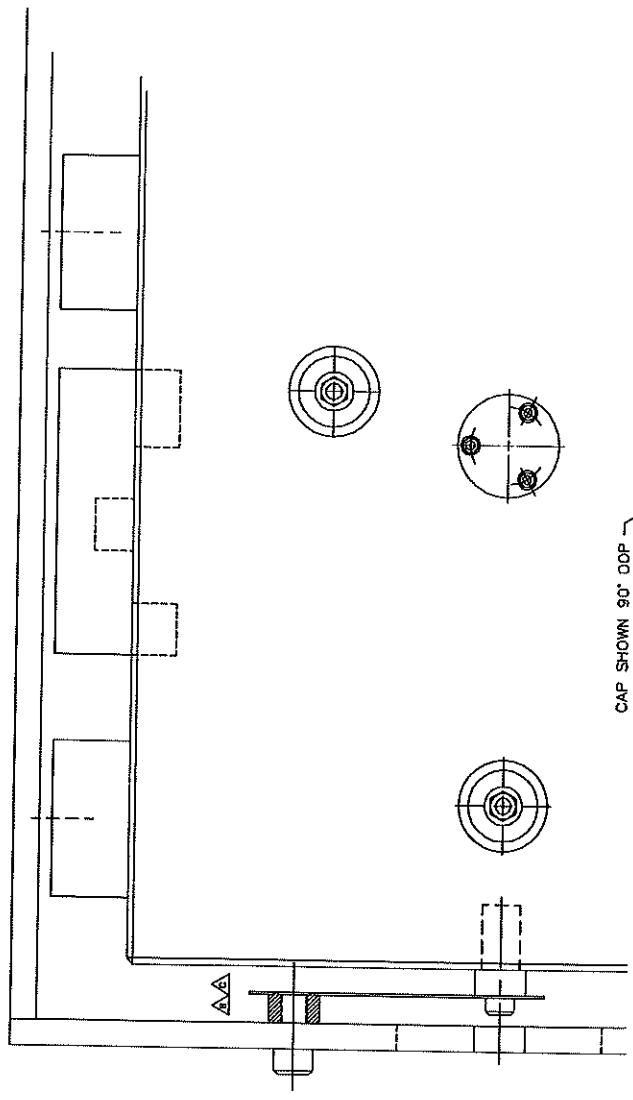








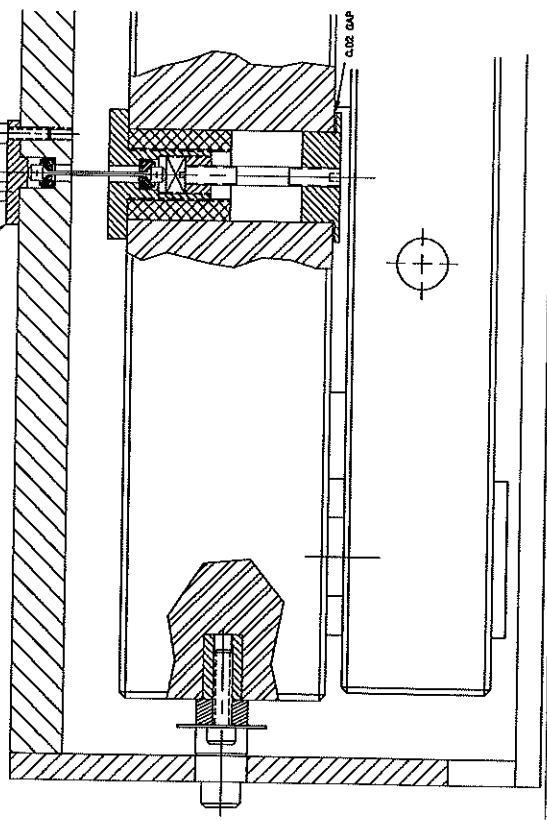




-1 COLUMN SUPPORT

3 REOD Δ

NOTE: SEE H3320 FOR ASSEMBLY
SEQUENCE.



Δ	H331-C-0	12	1	SPLIT COLLAR	ST. STE.
Δ	H331-C-5	11	1	SPLIT NUT	ST. STE.
Δ	BY OTHERS	10	1	ADAPTER	ST. STE.
Δ	BY OTHERS	9	1	LOCK	ST. STE.
Δ	H331-D-4	8	1	LOCK	ST. STE.
Δ	H331-D-3	7	1	PLUG	17-44-17
Δ	H331-D-3	6	2Pcs	SPLIT COLLAR	17-44-17
Δ	H332-C-2	5	2	RETAINER	17-44-17
Δ	CLOSURE	4	1	HOLDER	ST. STE.
Δ	CLOSURE	3	1	COLLAR	17-44-17
Δ	CLOSURE	2	1	COLLAR	17-44-17
Δ	CLOSURE	1	1	YARNED NUTS	ST. STE.

FULL

KECK/HIRES
ECHELLE SUBPLATE
COLUMN SUPPORT

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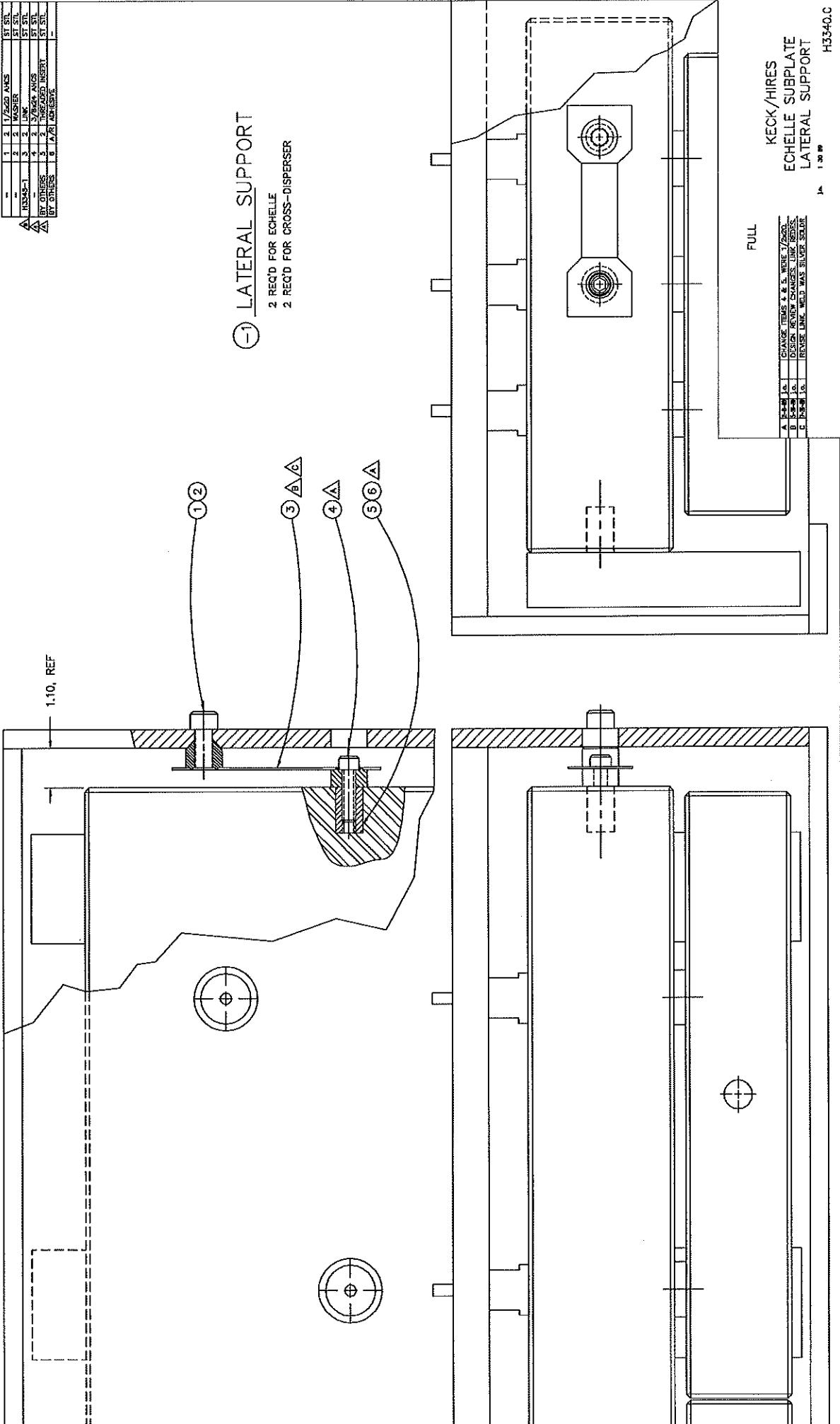
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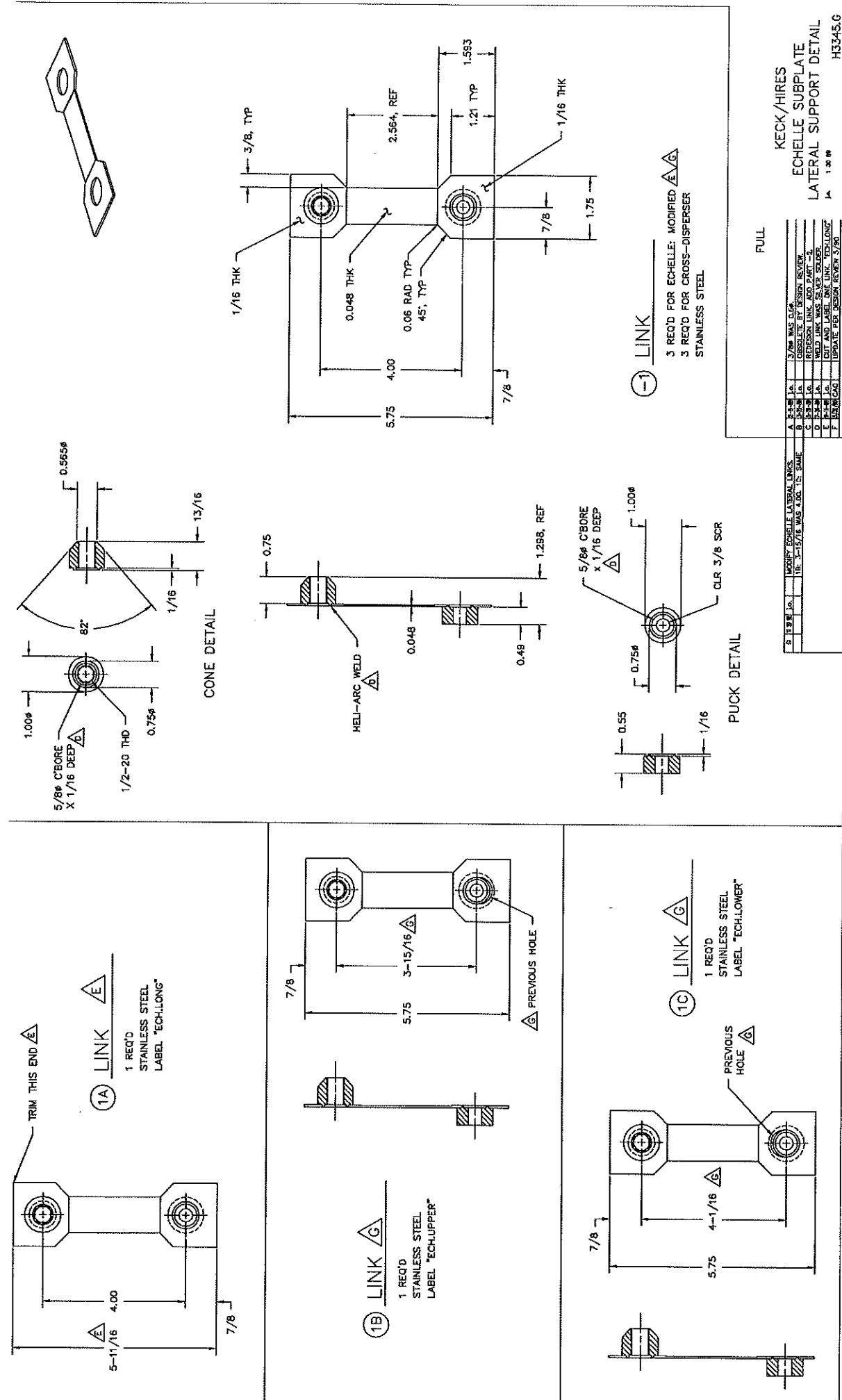
GG

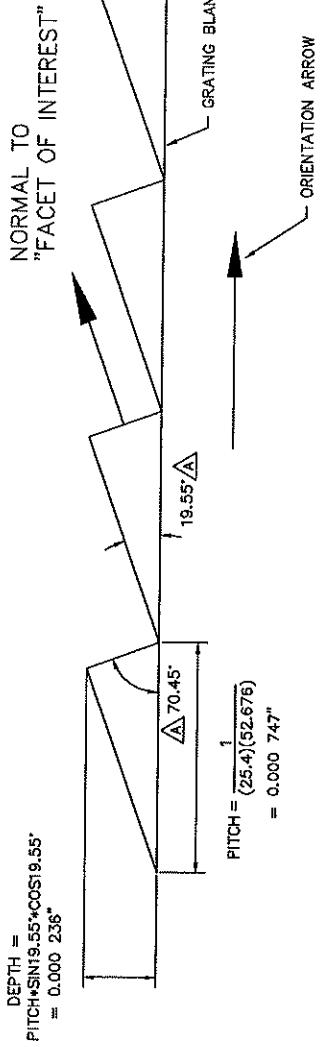
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① BLAZE DETAIL

CORE VERSION OF HIRES
 NOTES.

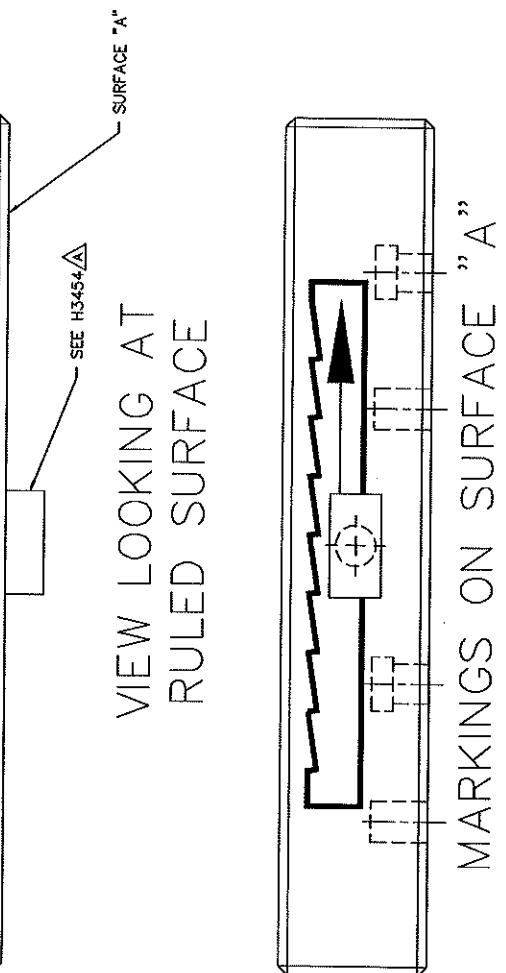
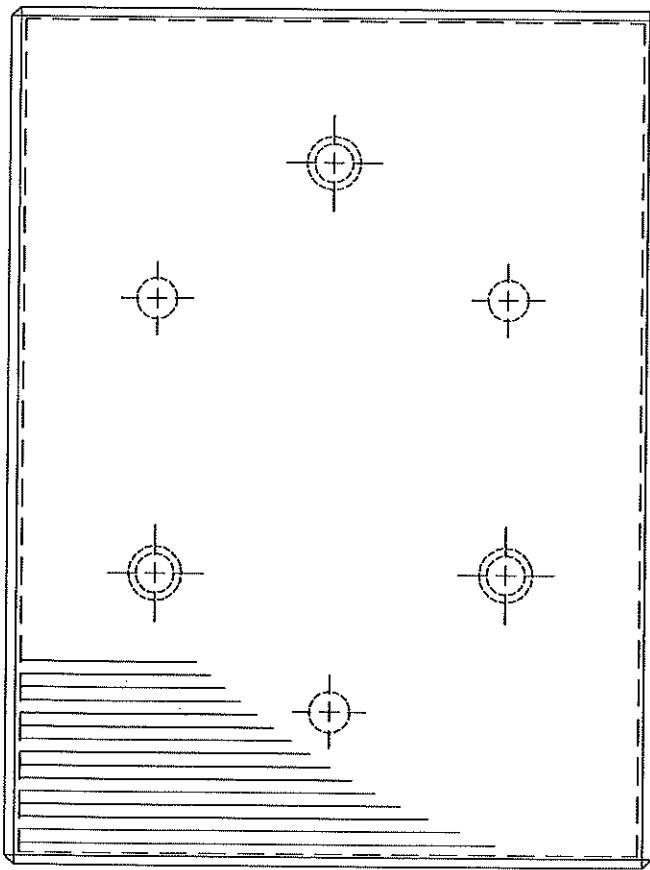
1. 52.676 LINES/mm
2. BLAZE ANGLE IS 70.45° (6238 Å)
3. R=2.6 ECHELLE GRATING

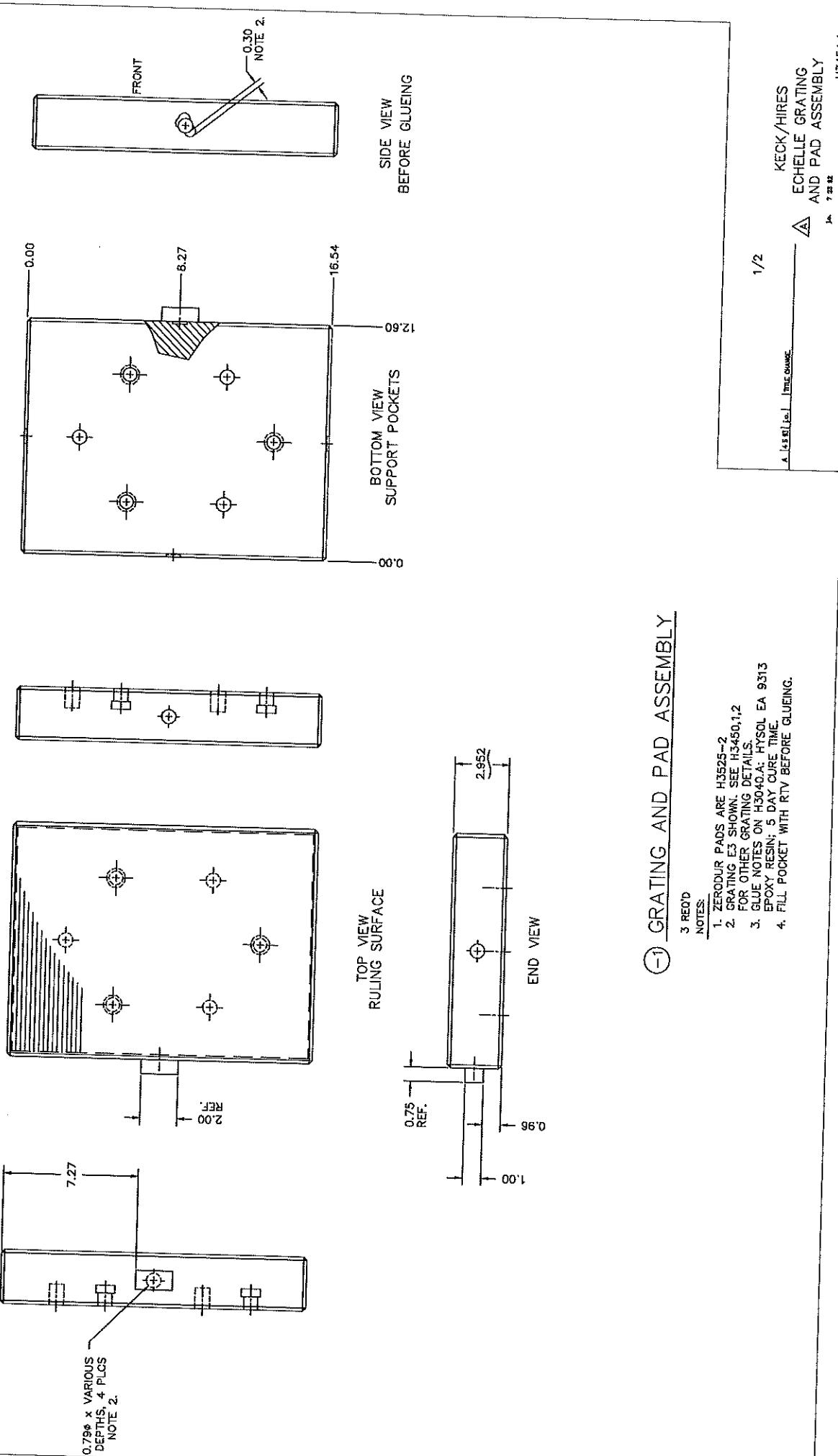


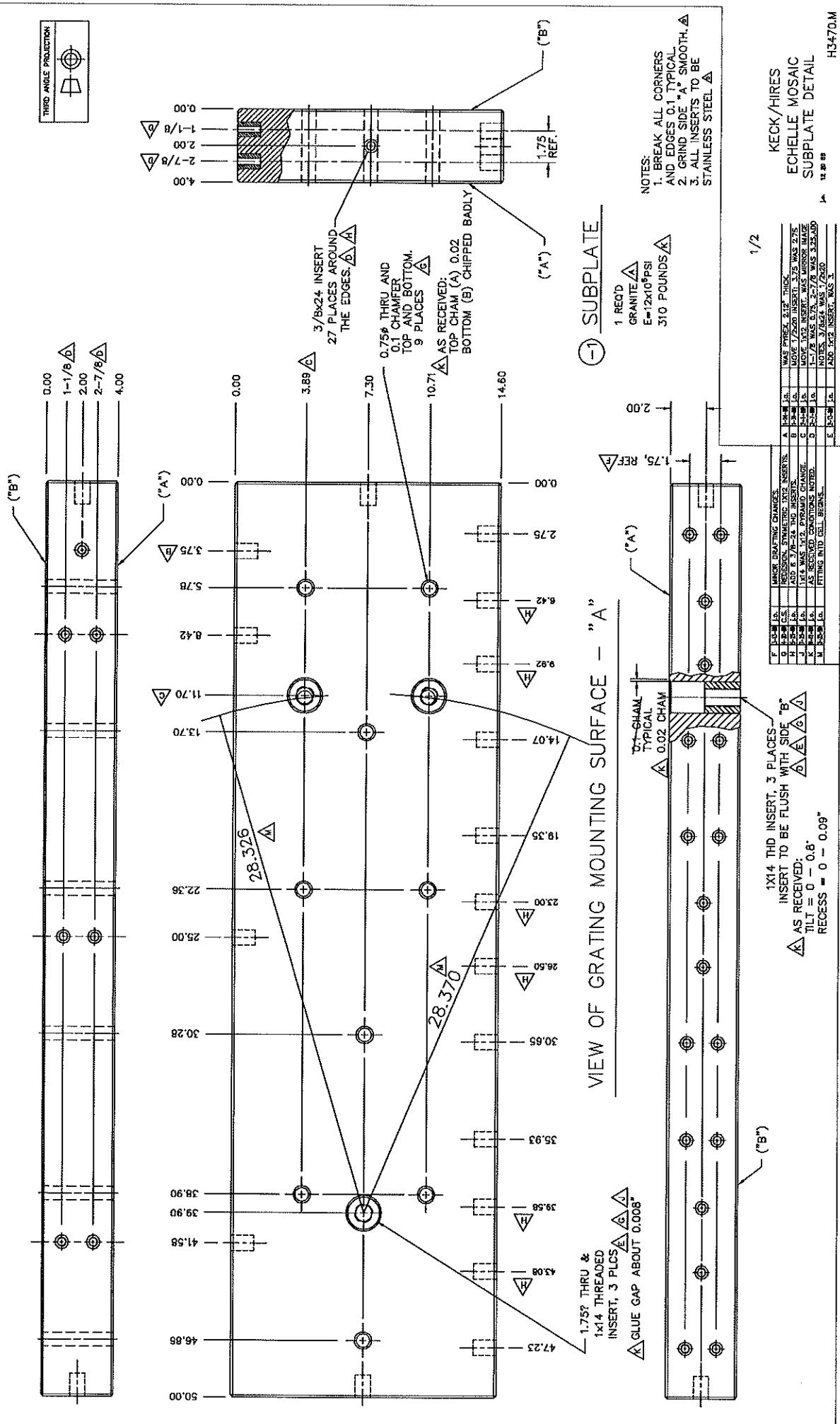
FULL

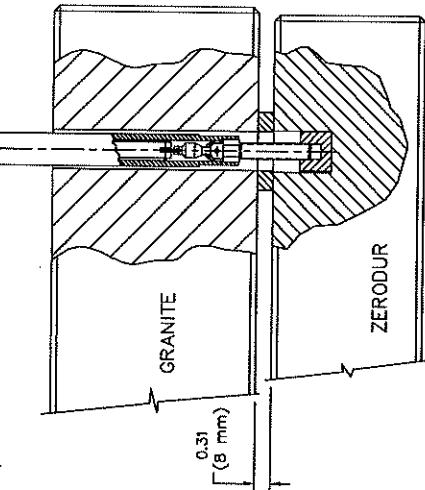
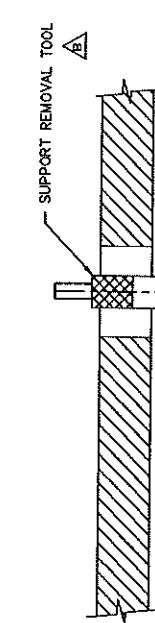
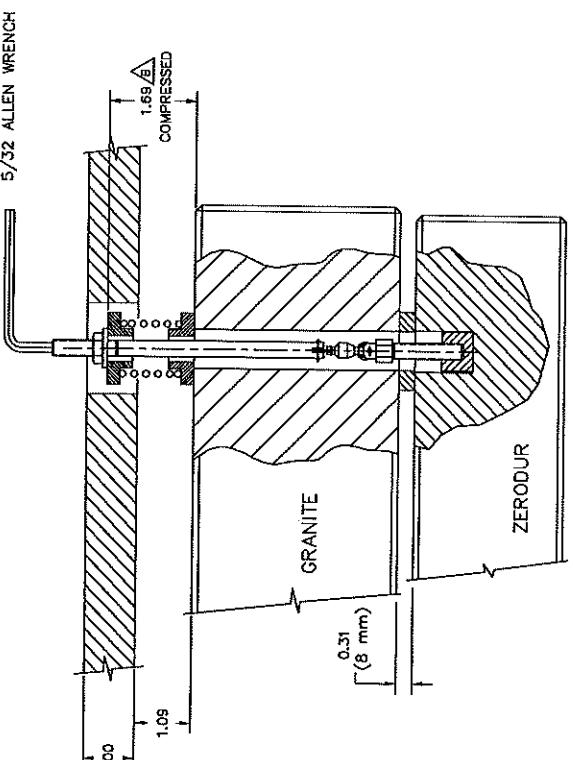
KECK/HIRES
 GRATING BLAZE
 ECHELLE

H3425.A









(1) BACK SUPPORT ASSEMBLY

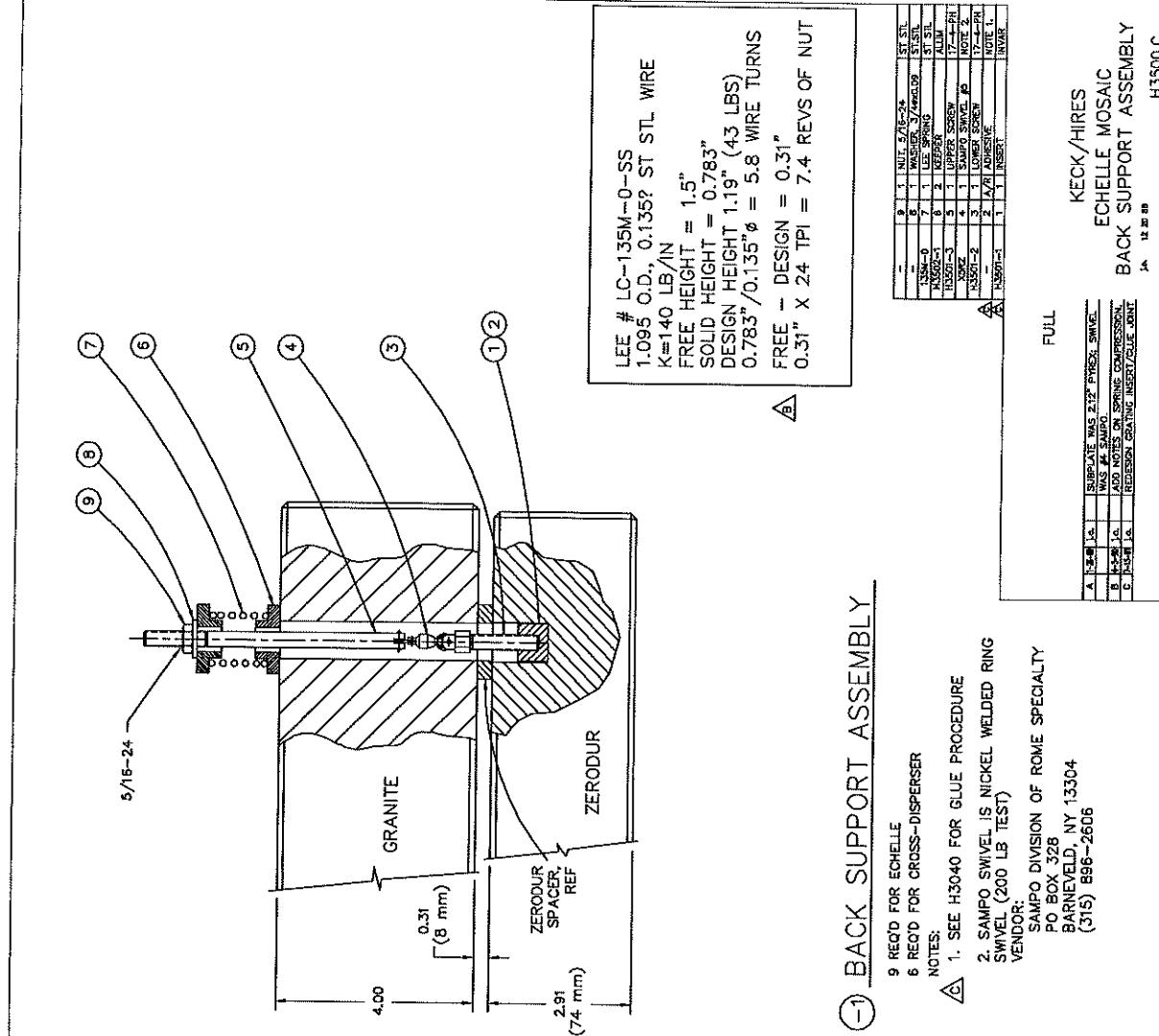
9 REQ'D FOR ECHELLE
6 REQ'D FOR CROSS-DISPERSER
NOTES:
△ 1. SEE H3040 FOR GLUE PROCEDURE
2. SAMPO SWIVEL IS NICKEL WELDED RING
SWIVEL (200 LB TEST)
VENDOR:
SAMPO DIVISION OF ROME SPECIALTY
PO BOX 328
BARNETT, NY 13304
(315) 896-2606

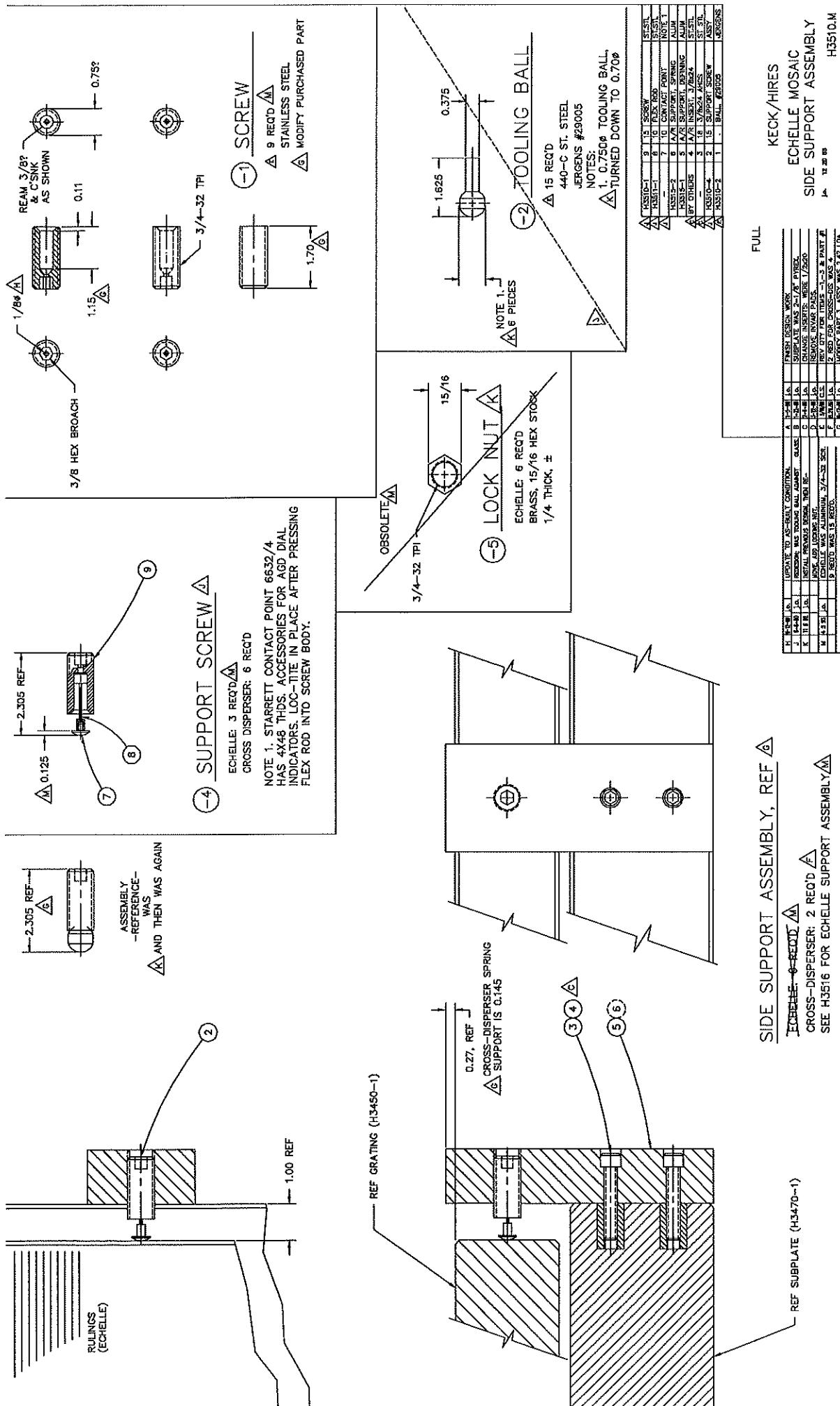
A. 1.5" I.D.
SUBPLATE WAS 2.12" DIAMETER
WAS 4.5" DIAM.
B. 1.25" I.D.
ADD NOTES ON SPRING COMPRESSION
REDUCE CRATING INSERT/CASE JACKET
C. 1.5" I.D.
1.25" I.D.
1.0" I.D.
1.0" I.D.

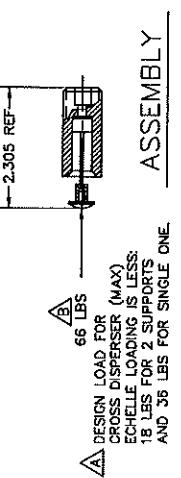
FULL

KECK/HIRES
ECHELLE MOSAIC
BACK SUPPORT ASSEMBLY

H3500.C







ASSEMBLY

△ BUCKLING NOTES:

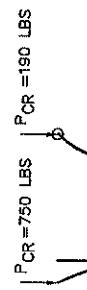
1. EULER BUCKLING LIMIT IS 750 LBS BASED ON N=2 (PINNED END AND FIXED END), DIAMETER = 0.069, LENGTH = 1" AND E = 30×10^6 PSI

A MORE CONSERVATIVE ESTIMATE WOULD BE FOR N=1 (BOTH ENDS PINNED AND FREE TO ROTATE) IN WHICH CASE, PCR IS 190 LBS.

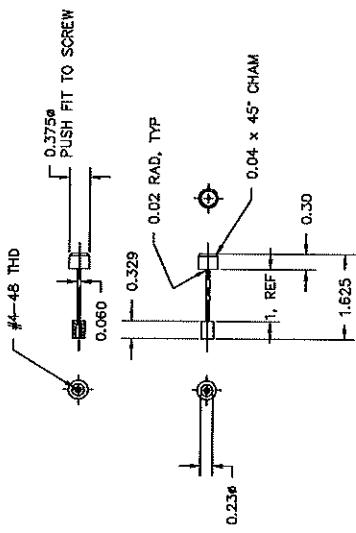
2. EULER'S FORMULA:

$$PCR = \frac{N^2 \pi^2 EI}{L^2}$$

$$\text{WHERE } I = \frac{\pi r^4}{4}$$



N = 2
N = 1



(-1) FLEX ROD

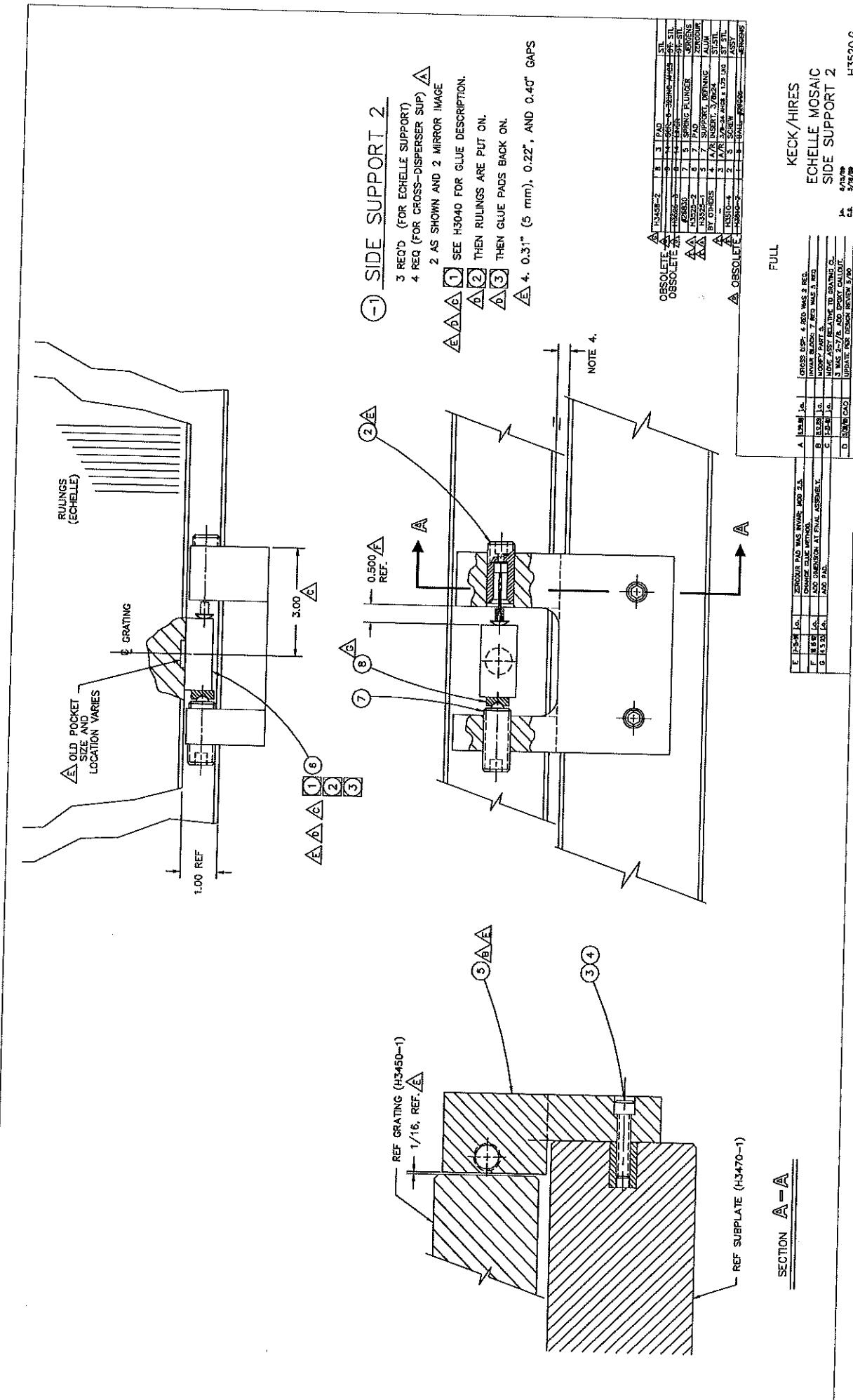
ECHELLE: 3 RECD

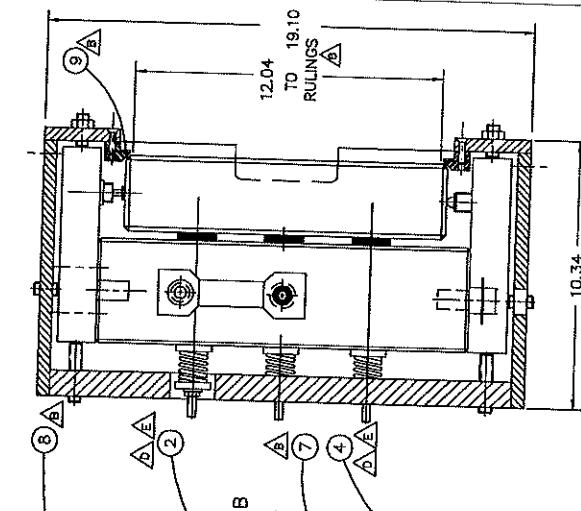
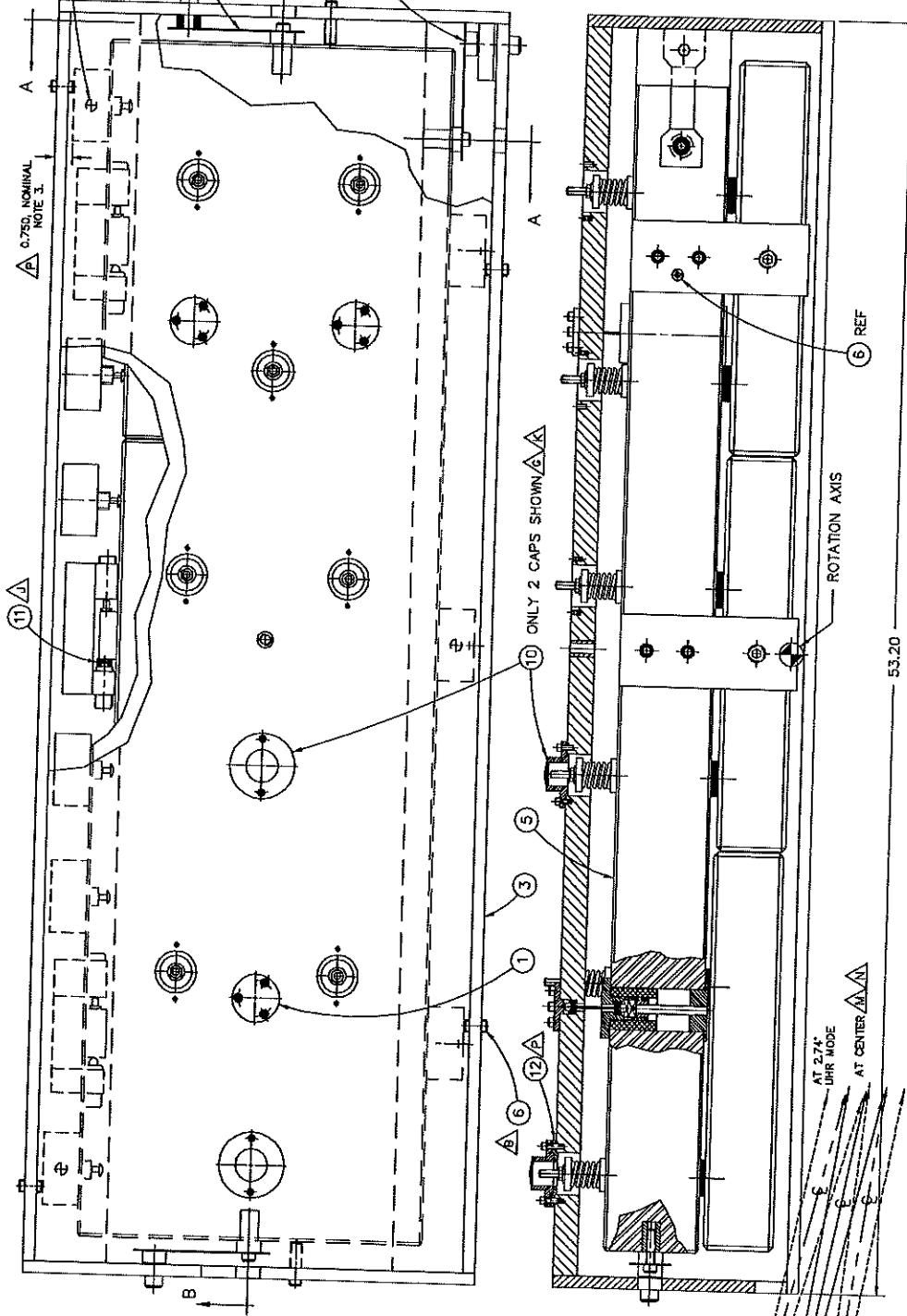
CROSS DISPERSER: 6 RECD
17-4-PH STAINLESS STEEL

NOTE: ALUMINUM SIDE SUPPORT BRACKET WILL CHANGE 0.004" IN TEMPERATURE RANGE EXPECTED. THIS ROD WILL OFFSET 0.004" WITH A FORCE OF 0.22 LBS. THIS IS A TOLERABLE FORCE AT THE GRATING EDGE SURFACE.

FULL

A	1.25 in.	1.00 BUCKLING CALC.	MOSAIC SIDE SUPPORT
B	1.25 in.	1.00 BUCKLING CALC.	FLEX ROD DETAIL
C	1.00 in.	1.00 BUCKLING CALC.	H3511.B





(-1) GRATING AND CELL ASSEMBLY

- 1 RECD
- 2 BLAZE ANGLE = 70.45°
- 3 SAFETY PLATES ARE 0.07" CLEAR OF GRATINGS
- 4 STOPS ARE 0.04" CLEAR

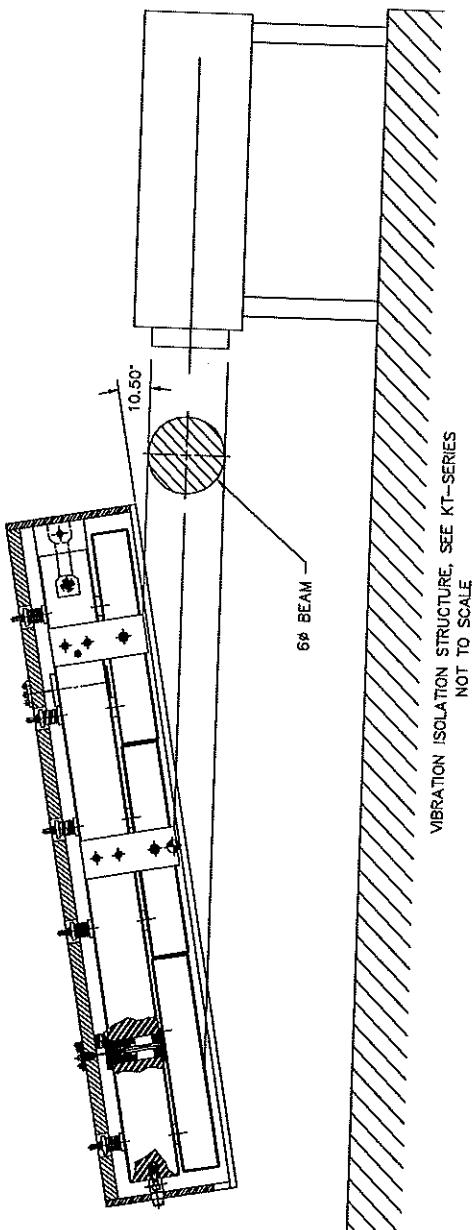
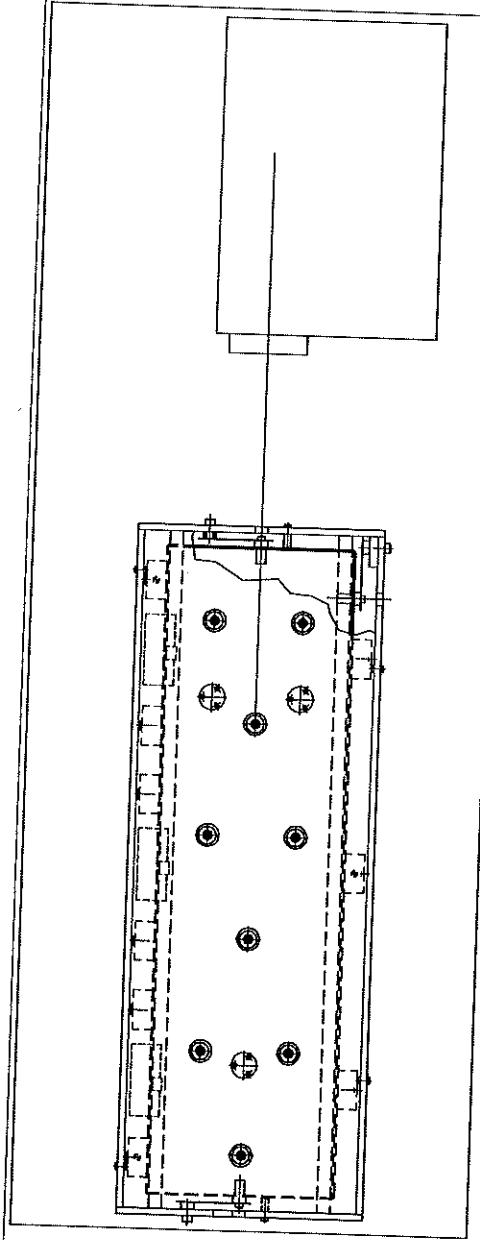
1/2

KECK/HIRES
ECHELLE MOSAIC
CELL ASSEMBLY

H360.P

1	SPACER	3	SPACER
2	PLATE PAD	4	PLATE PAD
3	CAP	5	CAP
4	SAFETY PLATE	6	SAFETY PLATE
5	UP/DOWN LOCK	7	END LOCK
6	END LOCK	8	END LOCK
7	END LOCK	9	END LOCK
8	ECHELLE MOSAIC ASSY	10	ECHELLE MOSAIC ASSY
9	OPTIONAL SUPPORT ASSY	11	OPTIONAL SUPPORT ASSY
10	ROSSING SUPPORT ASSY	12	ROSSING SUPPORT ASSY
11	COLLIMATOR SUPPORT ASSY	13	COLLIMATOR SUPPORT ASSY
12	STABILIZED MASTERSITE	14	STABILIZED MASTERSITE

1	REF	2	3	4	5	6	7	8	9	10	11	12	13	14
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2	REF	3	4	5	6	7	8	9	10	11	12	13	14	15
3	REF	4	5	6	7	8	9	10	11	12	13	14	15	16
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88	REF	89	90	91	92	93								



① ECHELLE TEST SETUP

NOTES:

1. SEE H3475 FOR FINAL INTERFERGRAM
2. SEE H3061 FOR LAYOUT WITH FOLDING FLAT.

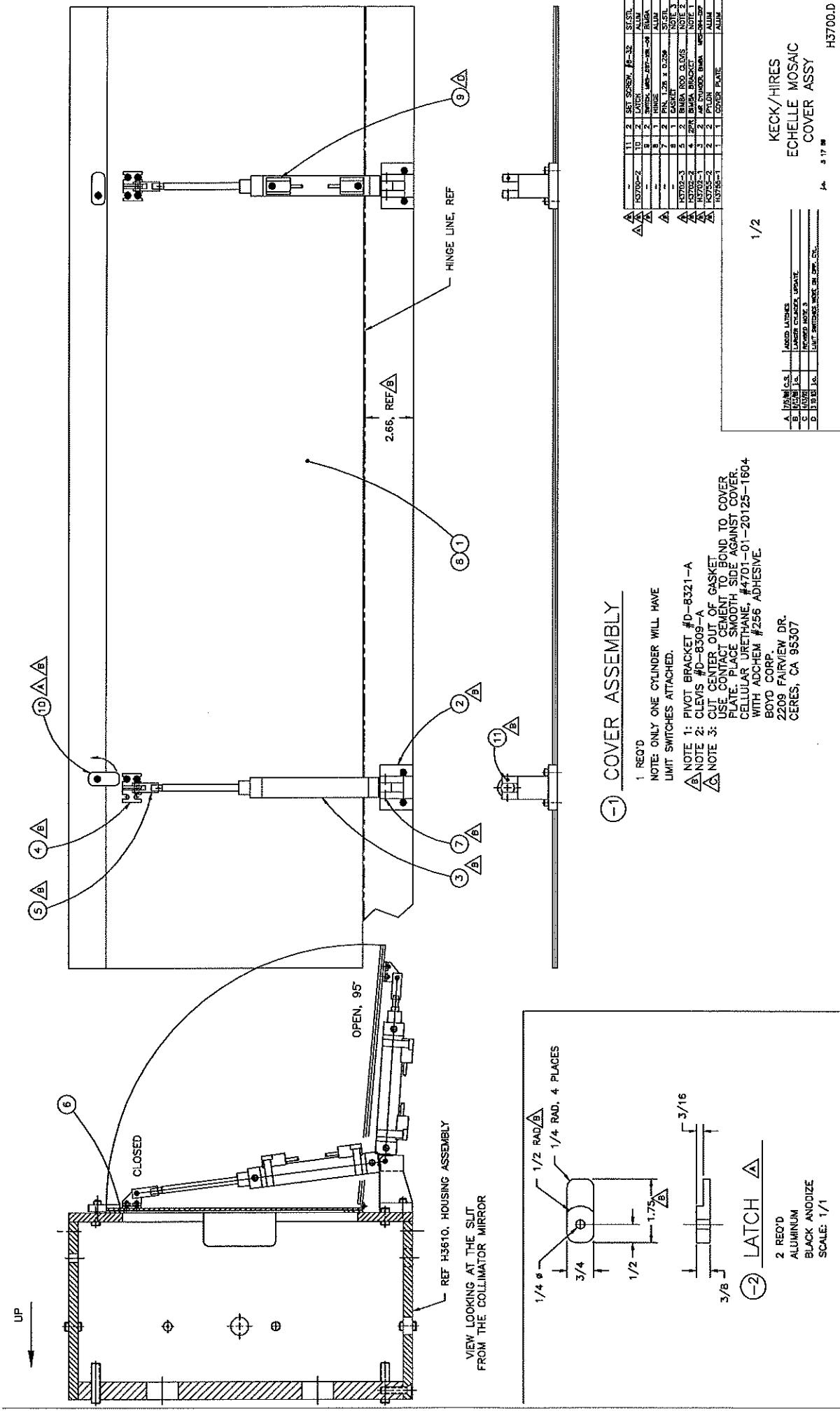
1/4

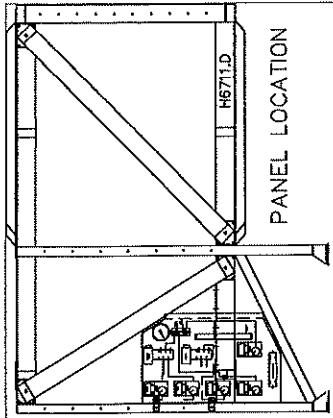
A. ~~TEST~~ loc. [] Update to ~~current~~ condition.

KECK/HIRES
ECHELLE MOSAIC TESTING
INTERFEROMETER LAYOUT
H3603.A

10.50

VIBRATION ISOLATION STRUCTURE, SEE KT-SERIES
NOT TO SCALE

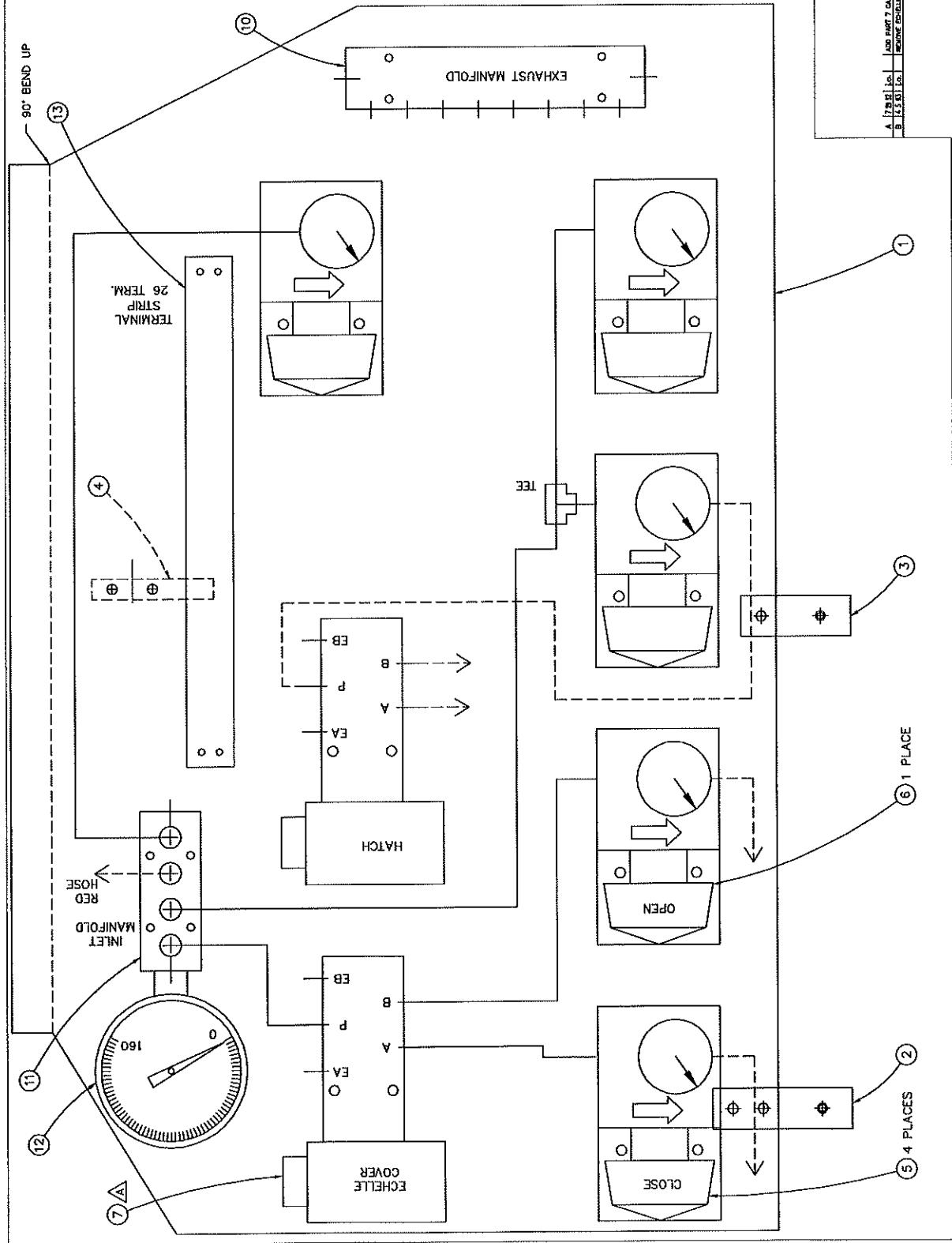




(-1) PANEL ASSEMBLY

1 REQ'D

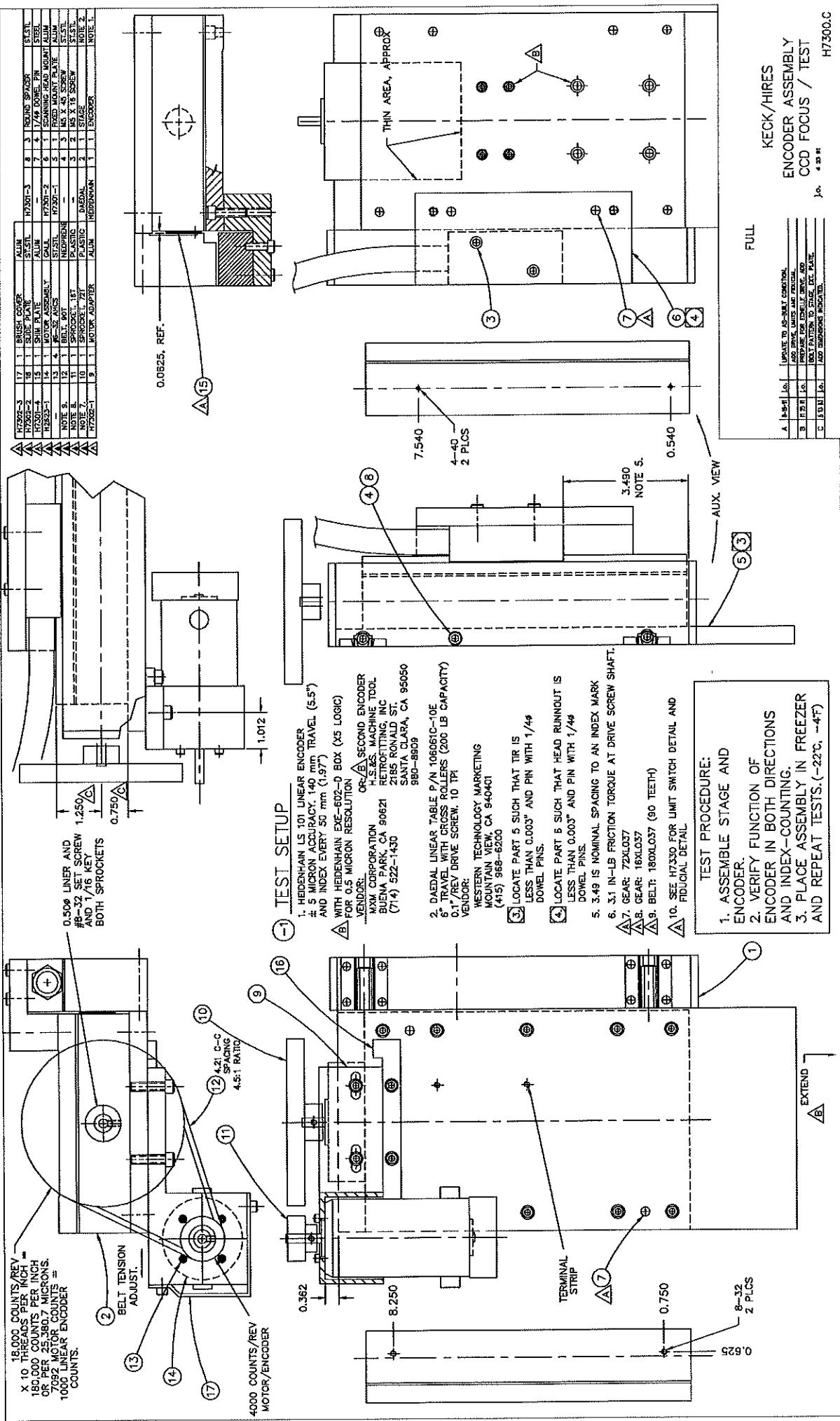
NOTES:
 1. WILKERSON REGULATOR #R16-02-000
 COMES WITH 0-160 PSI SPRING
 USE GAGE #RRP-95-229.
 2. WILKERSON REGULATOR #R16-02-000
 WITH SPRING #RRP-95-222 FOR
 0-60 PSI RANGE. USE GAGE #RRP-95-2
 3. SKINNER MAGNETLATCH 4-WAY VALVE
 #3535RBNA2150-244A. 24VAC COIL.
 4. 1/4INPT PORTS, ALUMINUM HOUSING W/SS
 SLEEVE, BUNA-N SEALS, 1/2INPT CONDUIT
 OUTLET.
 5. JERGENS #11B02, ALUMINUM, 8 1/4IN
 PORTS AND 2 3/8INPT PORTS (ENDS).
 6. JERGENS #11B01, ALUMINUM, 4 1/4IN
 PORTS AND 3 3/8INPT PORTS (ENDS).
 8. GAGE: McMaster-Carr #4023K55.
 -160PSI



18,000 COUNTS/REV
X 10 THREADS/INCH
180,000 COUNTS/PER INCH
OR PER .055387 MIGRONS.
1000 LINEAR ENCODER COUNTS.

MOTOR/ENCODER COUNTS.

1000 LINEAR ENCODER COUNTS.



(-1) LIMIT SWITCH ASSEMBLY

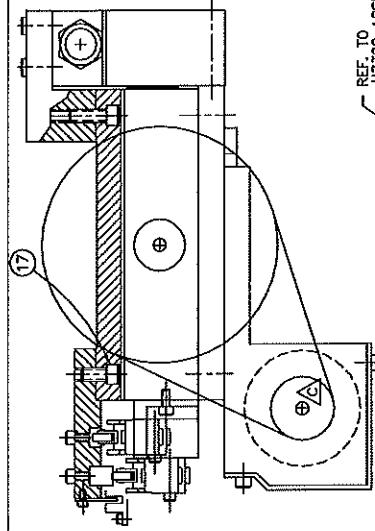
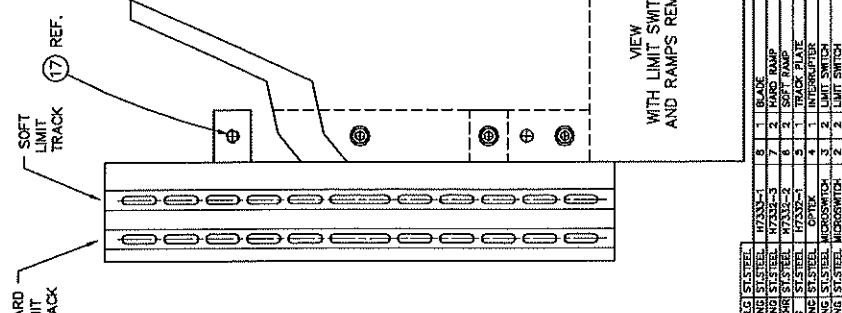
1 REQD FOR ENCODER TESTING

NOTES:
 1. TO REVERSE THE SENSE OF THE OPTICAL
 INTERRUPTER, RE-LOCATE HERE.
 2. "HARD" OR FINAL LIMITS: MICROSWITCH
 #D-2RV22-A7
 3. "SOFT" OR LOGIC LIMITS: MICROSWITCH
 #BZ-2RW822-A2
 4. OPTEK #OPB-970-T55.

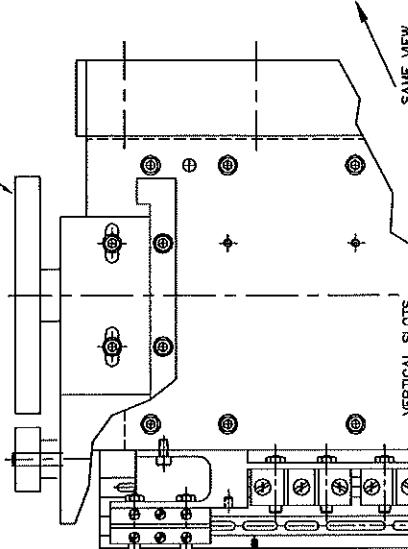
VENDOR:

BELL INDUSTRIES
 1161 N. FAIRDAKS AVE
 SUNNYVALE, CA 94086

△ THIS ASSEMBLY WAS MODIFIED AND MOVED
 TO H3201 AS PART OF THE ECHELLE MOSAIC
 DRIVE ASSEMBLY.



REF. TO
H7300 ASSEMBLY



SAME VIEW

SAME VIEW

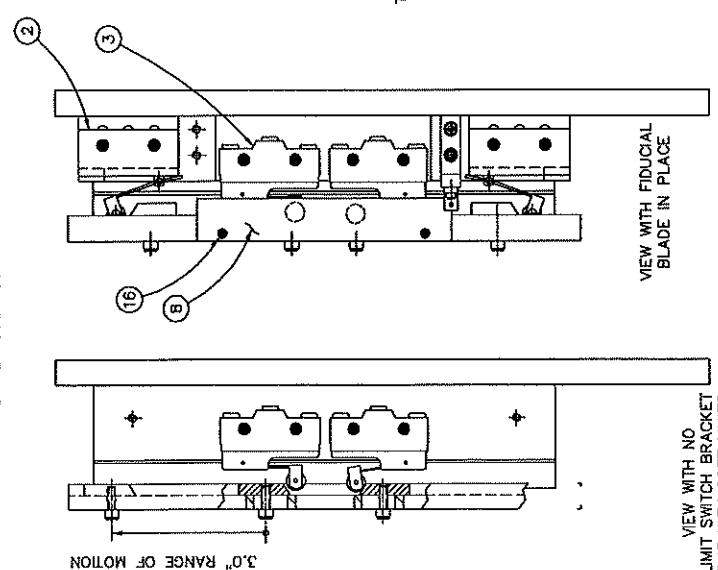
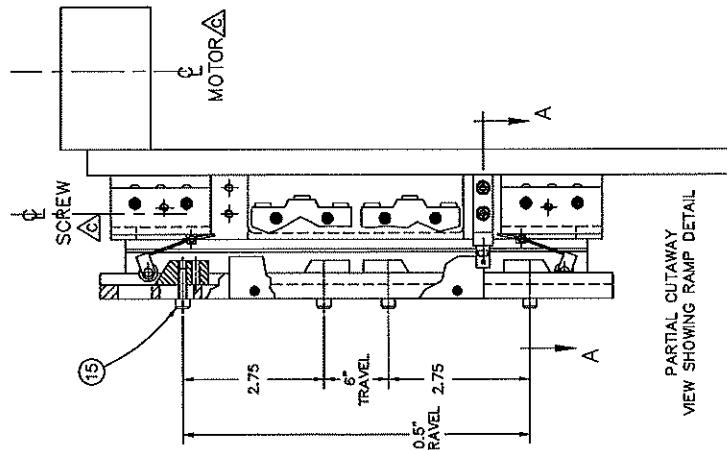
17	2	#1/4-20 ANCHOR/RC STUTTER	8	1 BLADE	8	1 BLADE
16	2	#4-40 ANCHOR/6 LNB STUTTER	7	2 HARD RAMP	7	2 HARD RAMP
15	4	#6-32 ANCHOR/6 LNB STUTTER	7	2 SOFT RAMP	6	2 SOFT RAMP
14	3	#6-32 HEX NUT/6 SHR STUTTER	7	1 TRACK PLATE	5	1 TRACK PLATE
13	5	#6-32 ANCHOR/6 LNB STUTTER	7	1 INTERRUPTER	4	1 INTERRUPTER
12	2	#6-32 ANCHOR/2 LNB STUTTER	3	2 LIMIT SWITCH	3	2 LIMIT SWITCH
11	2	#6-32 ANCHOR/4 LNB STUTTER	2	2 SWING BRACKET	2	2 SWING BRACKET
10	2	#6-32 ANCHOR/2 LNB STUTTER	1	1 SWING BRACKET	1	1 SWING BRACKET

FULL

KECK/HRES
 ENCODER ASSEMBLY
 DETAILS

H7330.C

J.O. 5-18 M



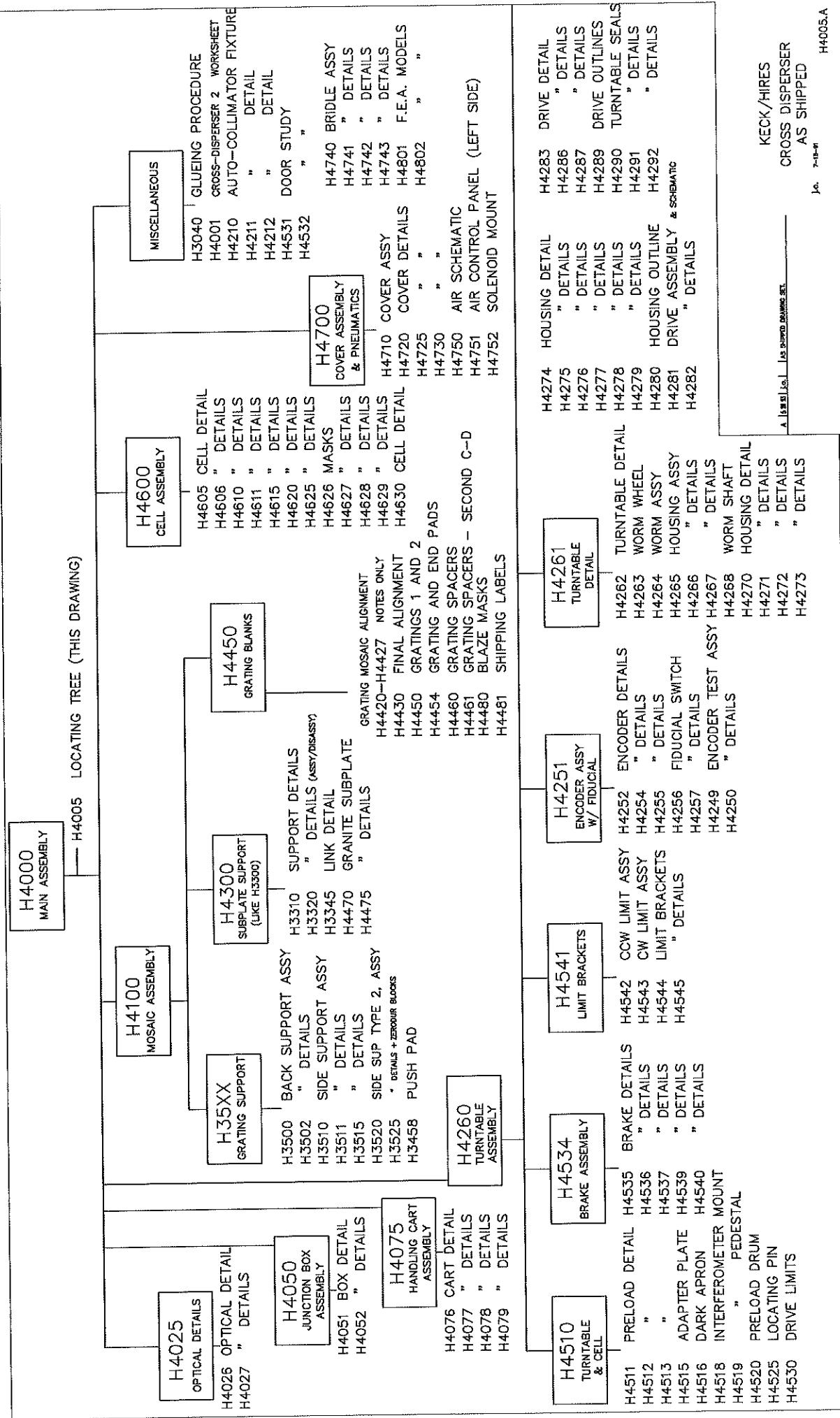
PARTIAL CUTAWAY
 VIEW WITH FIDUCIAL
 BLADE IN PLACE

VIEW WITH NO
 LIMIT SWITCH BRACKET
 BUT IN PLACE

PARTIAL CUTAWAY
 VIEW SHOWING RAMP DETAIL

Appendix E List of Drawings — Cross-Disperser Mosaic

1. H4005 Locating Tree
2. H4000 Cross Disperser Assembly and Cell
3. H4026 Optical Detail and Blaze
4. H4210 Alignment Fixture for Turntable Mounting
5. H4251 Encoder Assembly
6. H4264 Worm Drive Assembly
7. H4281 Drive Assembly and Schematic
8. H4300 Subplate Column Support Assembly
9. H4421 Alignment Notes
10. H4424 Alignment Notes
11. H4450 Grating Detail
12. H4454 Grating and Side Pads
13. H4470 Subplate
14. H4511 Turntable Preload
15. H4530 Turntable Drive Limits
16. H4534 Turntable Brake Assembly
17. H4740 Cell Carrier Assembly
18. H4750 Cell Doors Pneumatic Diagram
19. H4751 Pneumatic Details
20. H4802 Subplate Finite Analysis Results



(-1) CROSS DISPERSER

1 REFD

NOTES:

- 1. CORE IS "RED/BLUE" MOSAC
- 2. CAP IS NOMINALLY 0.04 IN.
- 3. MEASUREMENTS ENCL.
- 4. OLD POCKETS IN THE REAR OF THIS GRATING. ALSO,
- 5. UPPER OVERHANGS LOWER BY 0.014"
- 6. SIDE POCKETS.
- 7. UNKNOWN REASON.

BLAZE DRAWING
H4480

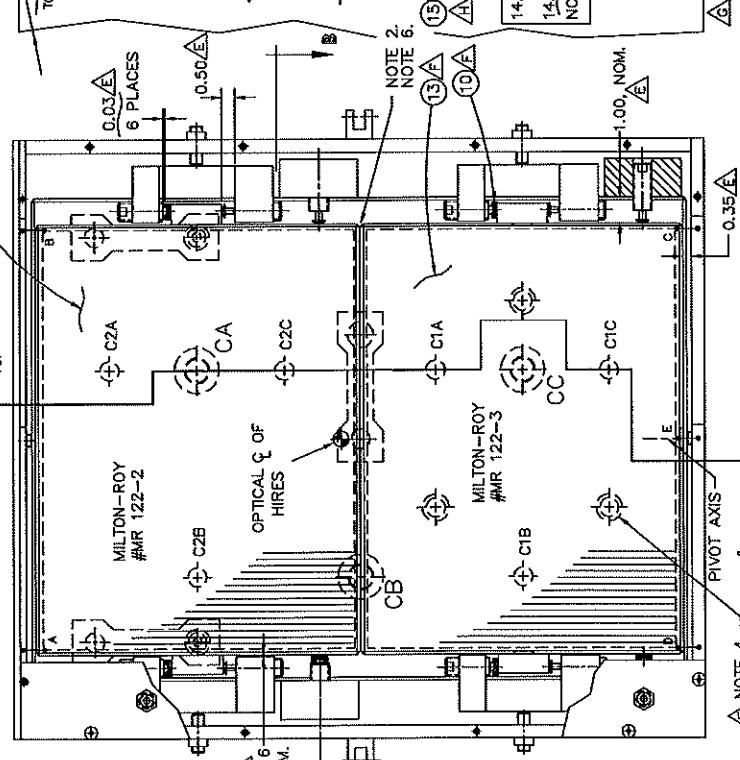
OPEN-SENSING
SWITCH

H 4480 (6) HAS MASS. NOTE 7.

ADD 0.014" FROM PLANE OF
ALO. NET ERROR IS 0.004" IN
D. (ABOUT 2%)
E. (NOT USED.)
THESE DIMENSIONS ARE ADJUSTABLE
BY SUPPORT ASSEMBLY, ITEM 3.

THEORETICAL:

1.018 NEAR △ NOTE 3.
0.768 FAR △ NOTE 3.
STRAIGHT LINE FROM SLIT TO COLIMATOR EDGE



1/2

KECK/HIRES
2x1 CROSS DISPERSER
ASSEMBLY

H4400.H

H4400.H

SET SEC. CHAMFER	
5	MAK. LOGOS 1/2-30
1	ASSURE
4	UPGRADE TO AS-BUILT LINE
5	1 PER COVERS
3	LINK
1	COLLUM SUPPORT
3	ASSURE
2	ASSURE
1	ASSURE
1	ASSURE

LAST 3 WERE SAME AS EDITION.

REGRIND FEATURES AT GAP.

UPDATE TO AS-BUILT LINE

REGRIND W/ LATEST MOSAC ASSY

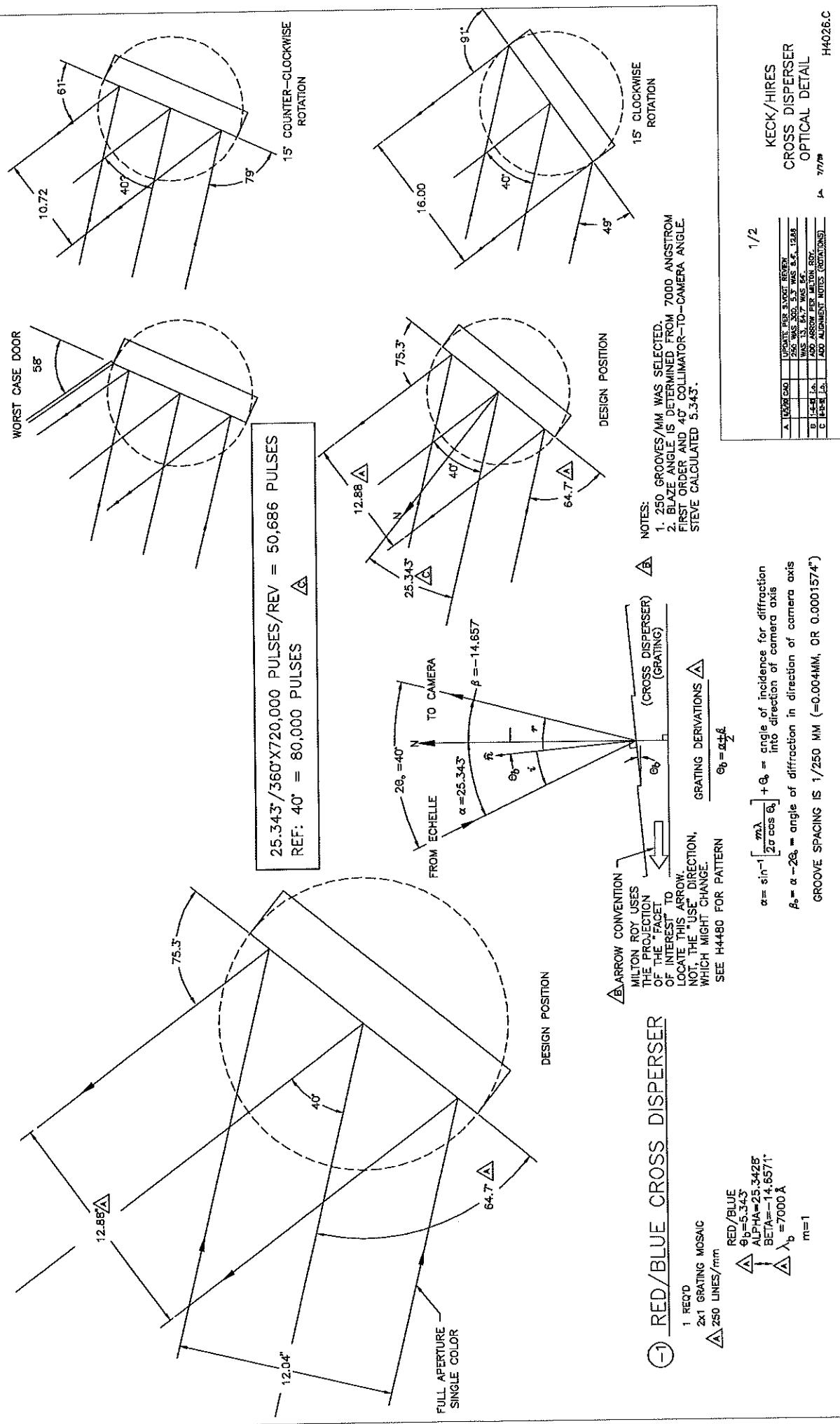
2x1 ASSEMBLY AT ASSEMBLY POINT

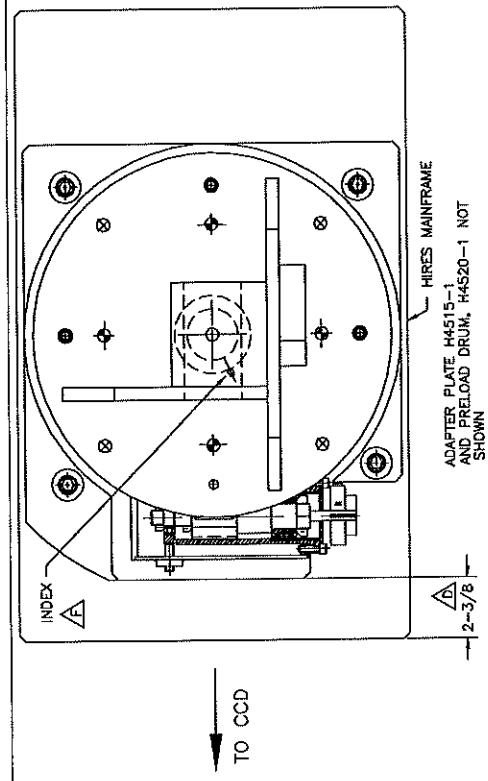
REGULUS SUPPLY 400 SITES

FINAL ASSEMBLY DATE

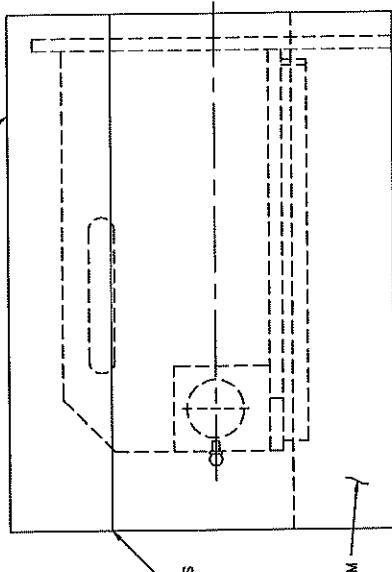
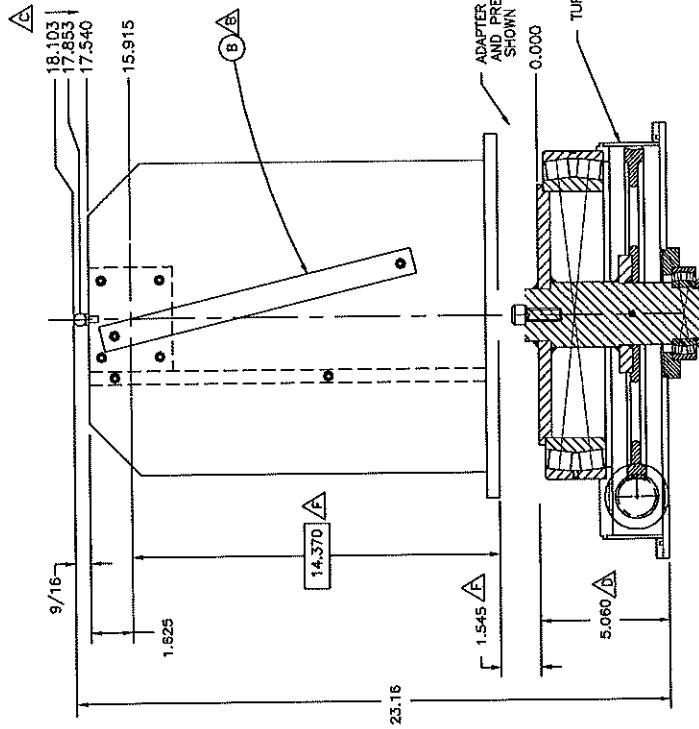
Q 3281 Le.

Q 3281 Le.





ADAPTER PLATE H4515-1
AND PRELOAD DRUM, H4520-1 NOT
SHOWN

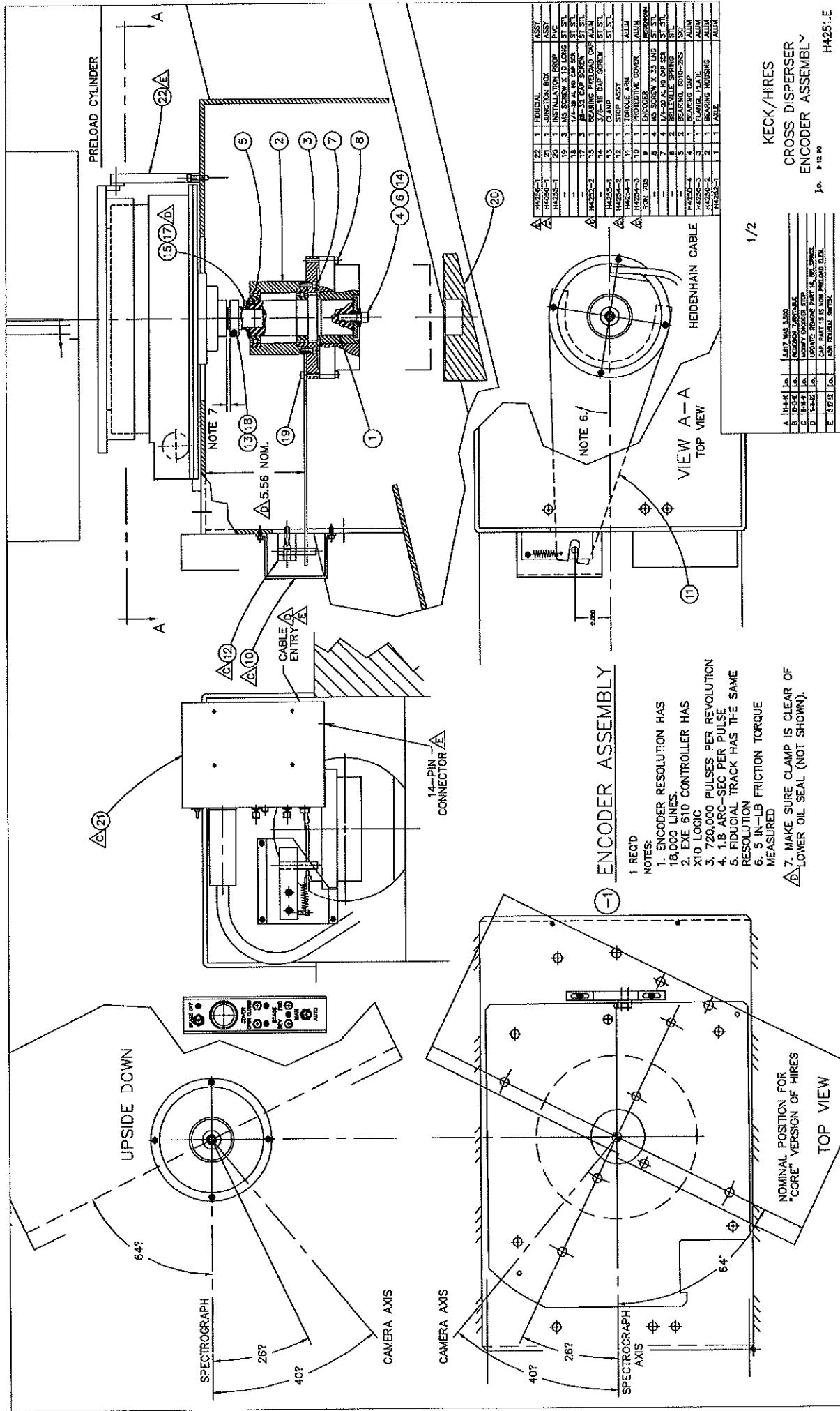


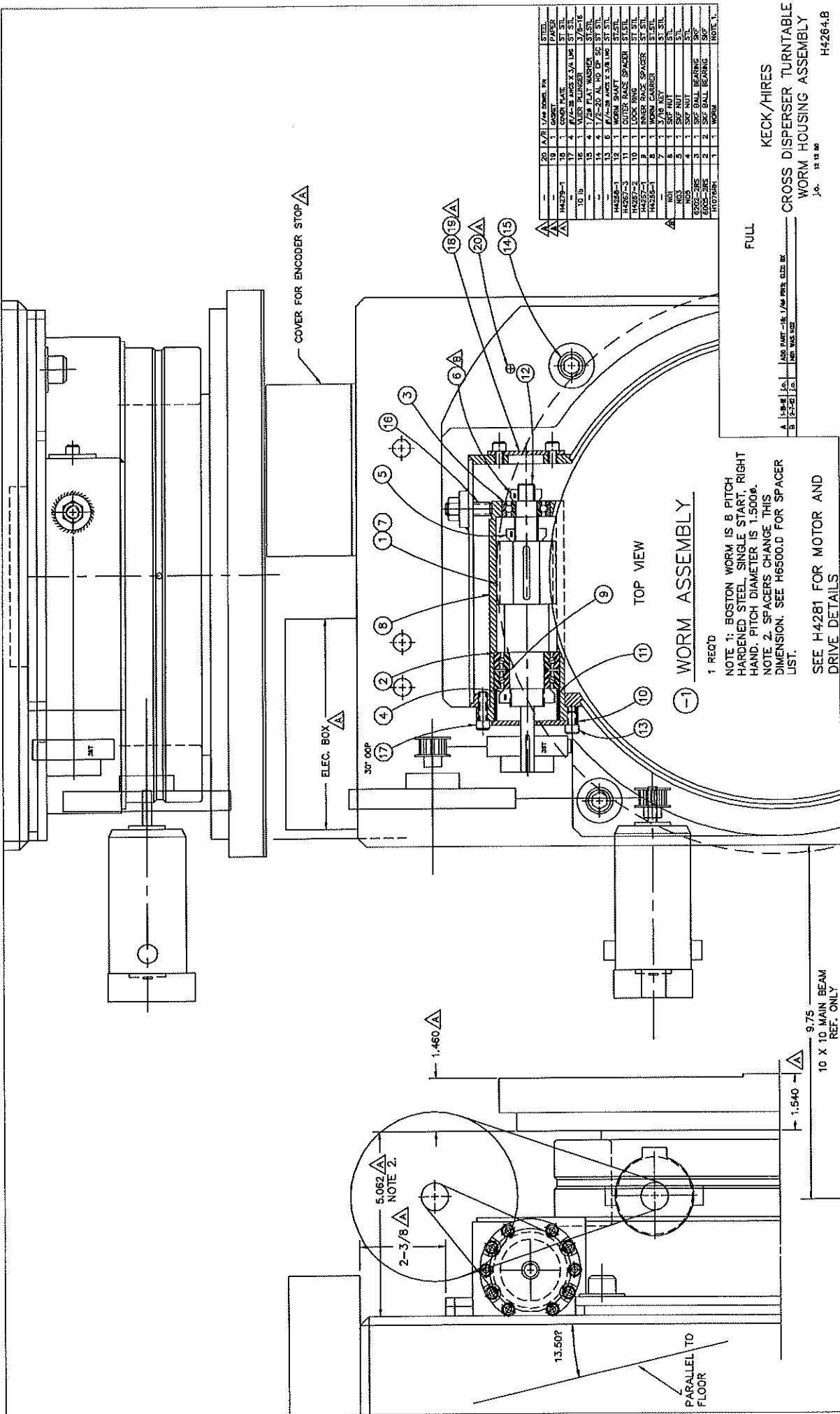
(-1) AUTO-COLLIMATOR HOLDER

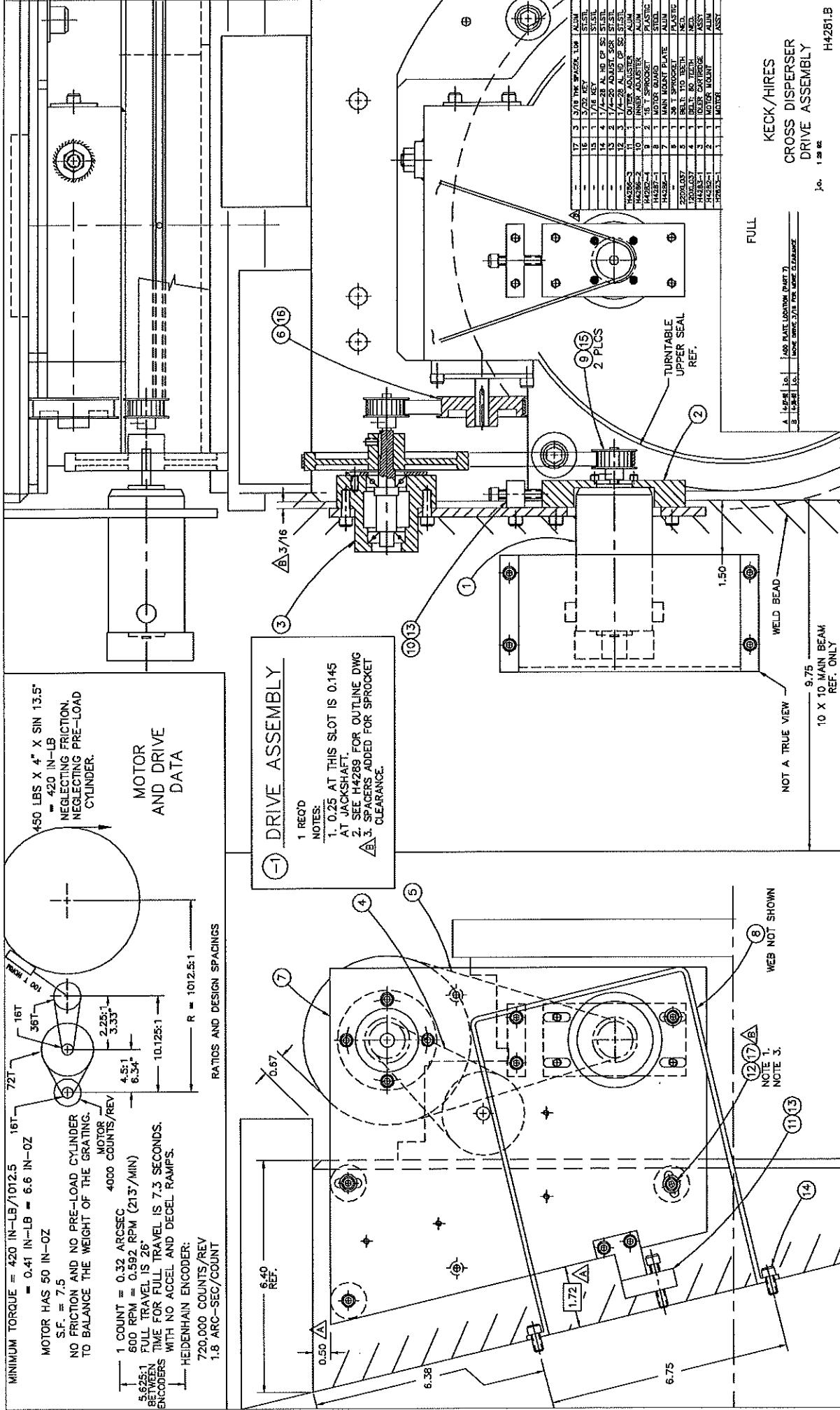
- 1 REQD
TURNTABLE: REF H4260
- NOTES:
- 1. DAVIDSON OPTRONICS D275 ALIGNMENT TELESCOPE, AUTOCOLLIMATING.
 - 2. RED AND GREEN PARTS.
 - 3. SCHRY-WAY CASE: MD30-20X15X11-4

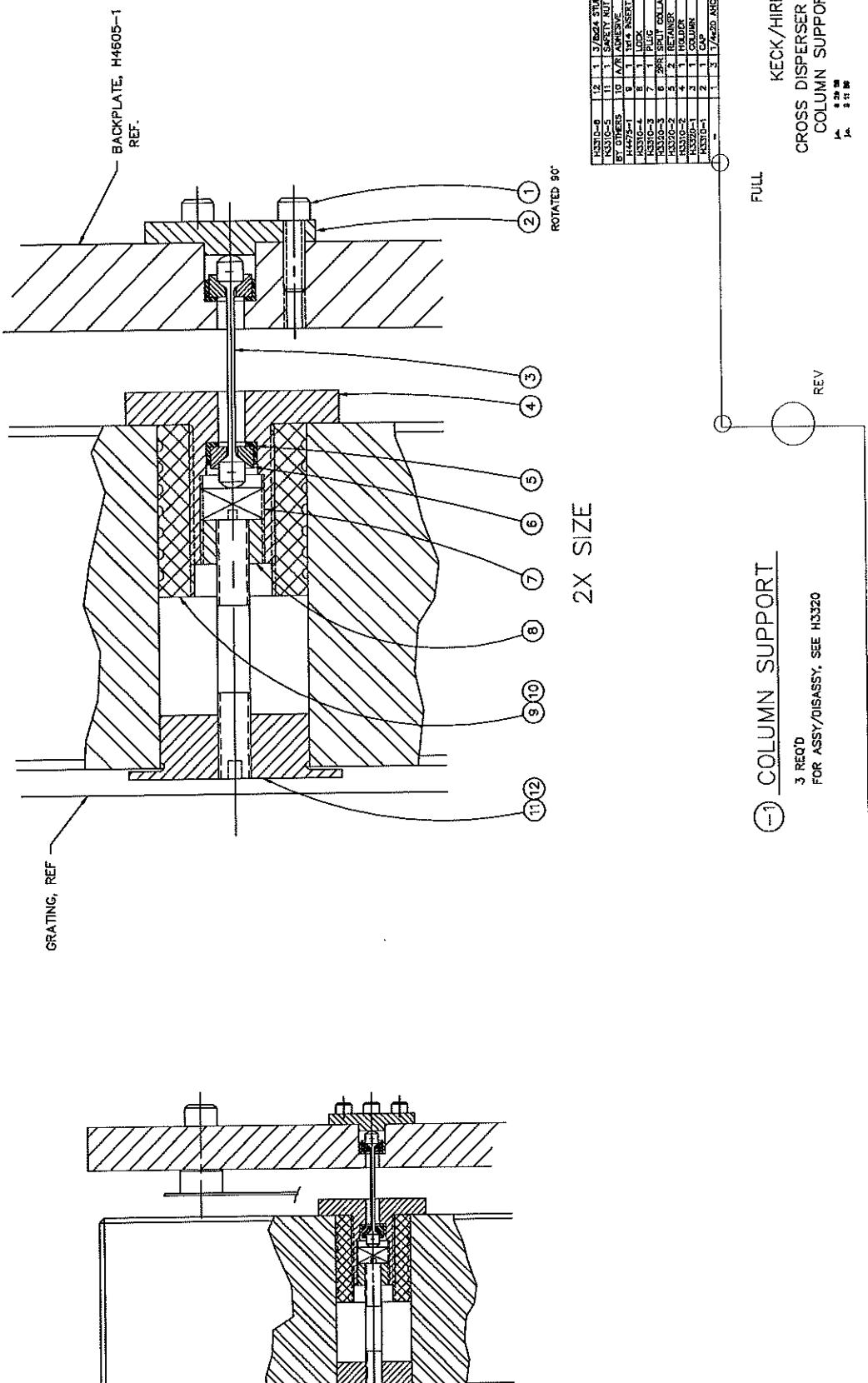
1/2	KECK/HIRES ALIGNMENT FIXTURE CROSS-DISPERSER	H4260.F

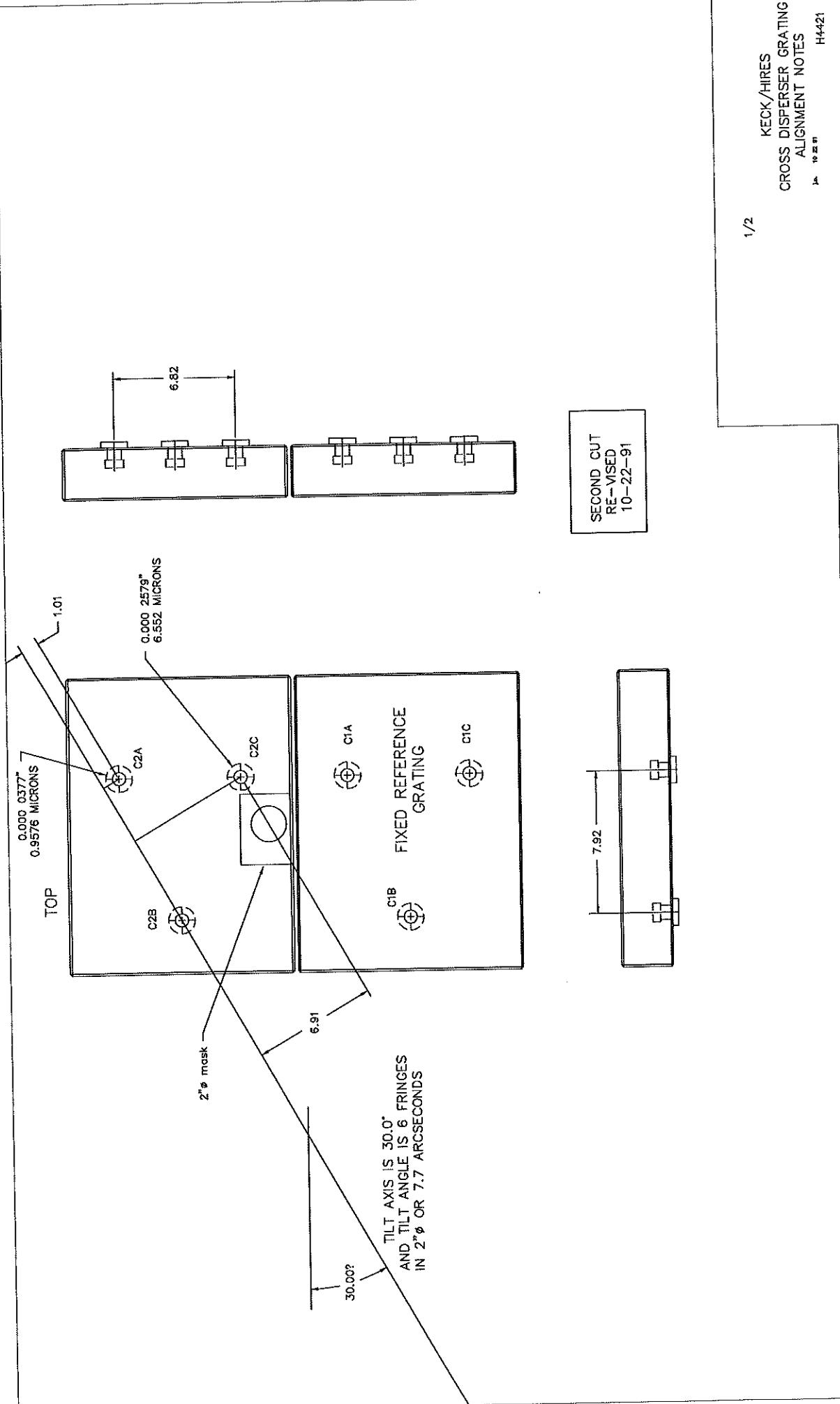
△ H4520-1	10	1	PILOT	ALUM
△ H4515-1	2	1	BOX	ALUM
△ H4211-5	3	1	LEVEL, REST	ALUM
-	7	1	TOOLING BALL	ALUM
△ H4211-4	6	1	BRACE	ALUM
△ H4211-3	5	1	INTERNAL SUPPORT RADS	ALUM
△ H4211-2	4	1	INTERNAL SUPPORT RADS	ALUM
△ H4211-1	3	1	INTERNAL SUPPORT RADS	ALUM
△ H4211-0	2	3	3/4 IN. AL. HS. 3/8 IN. SH.	ST. STEEL



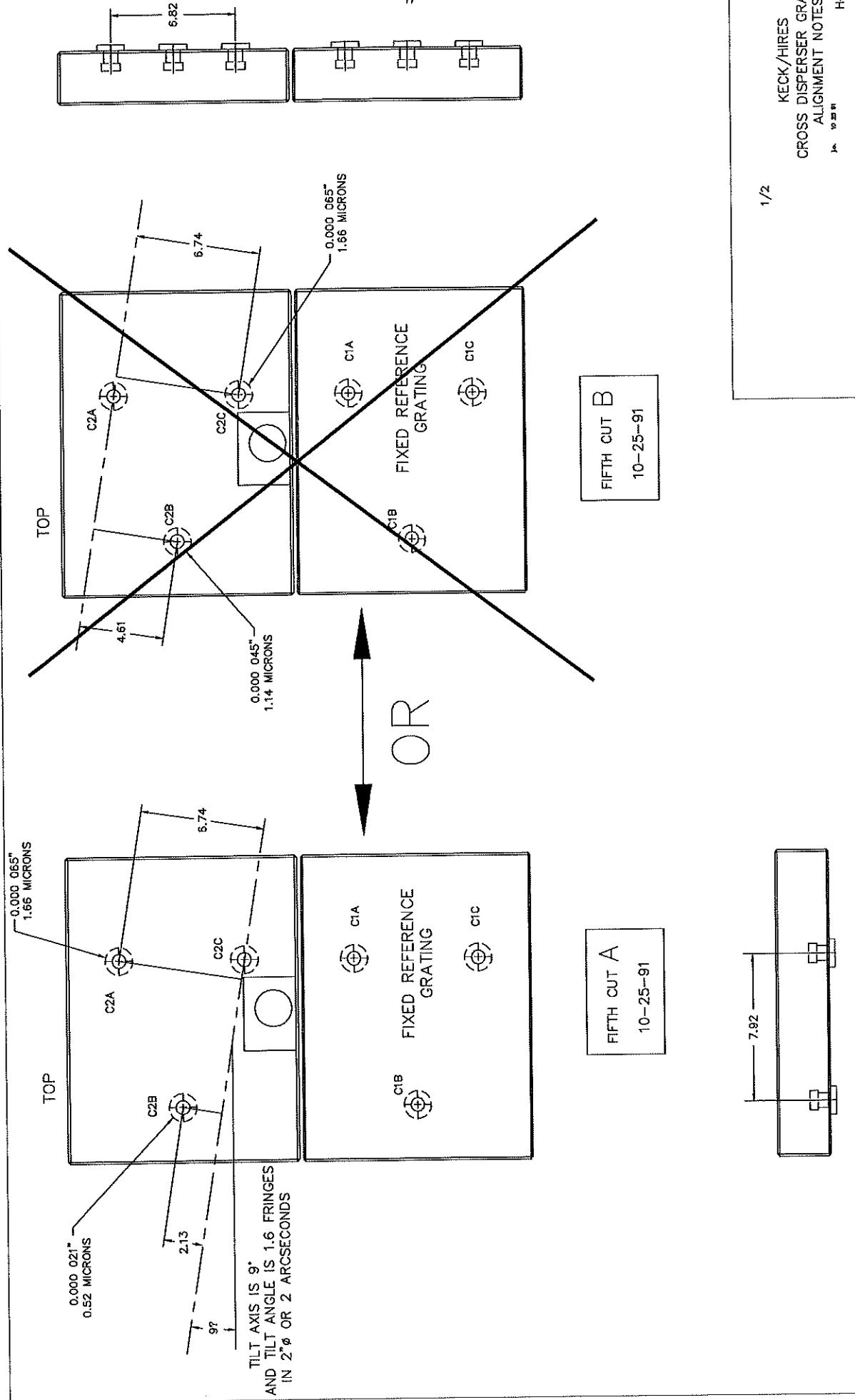


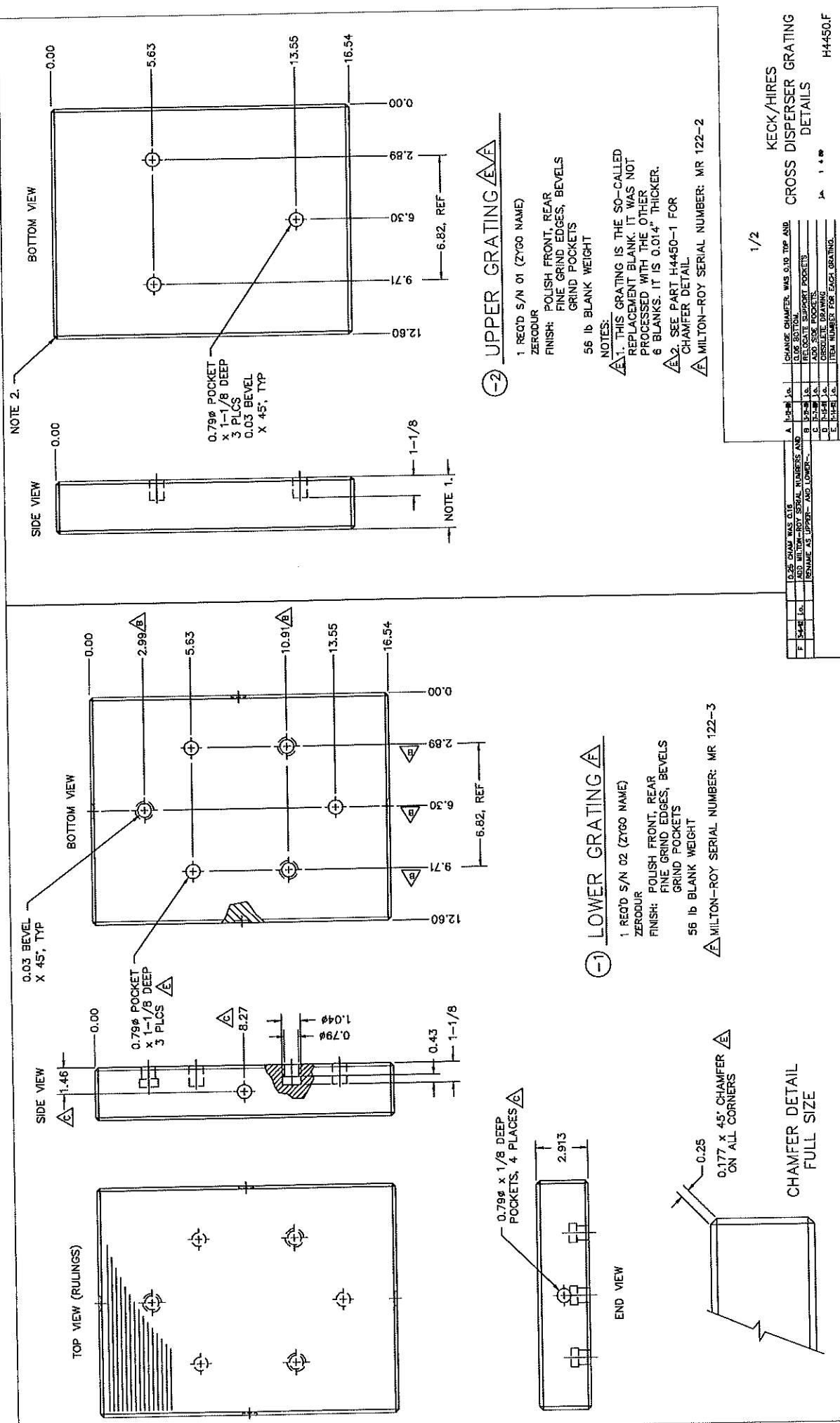


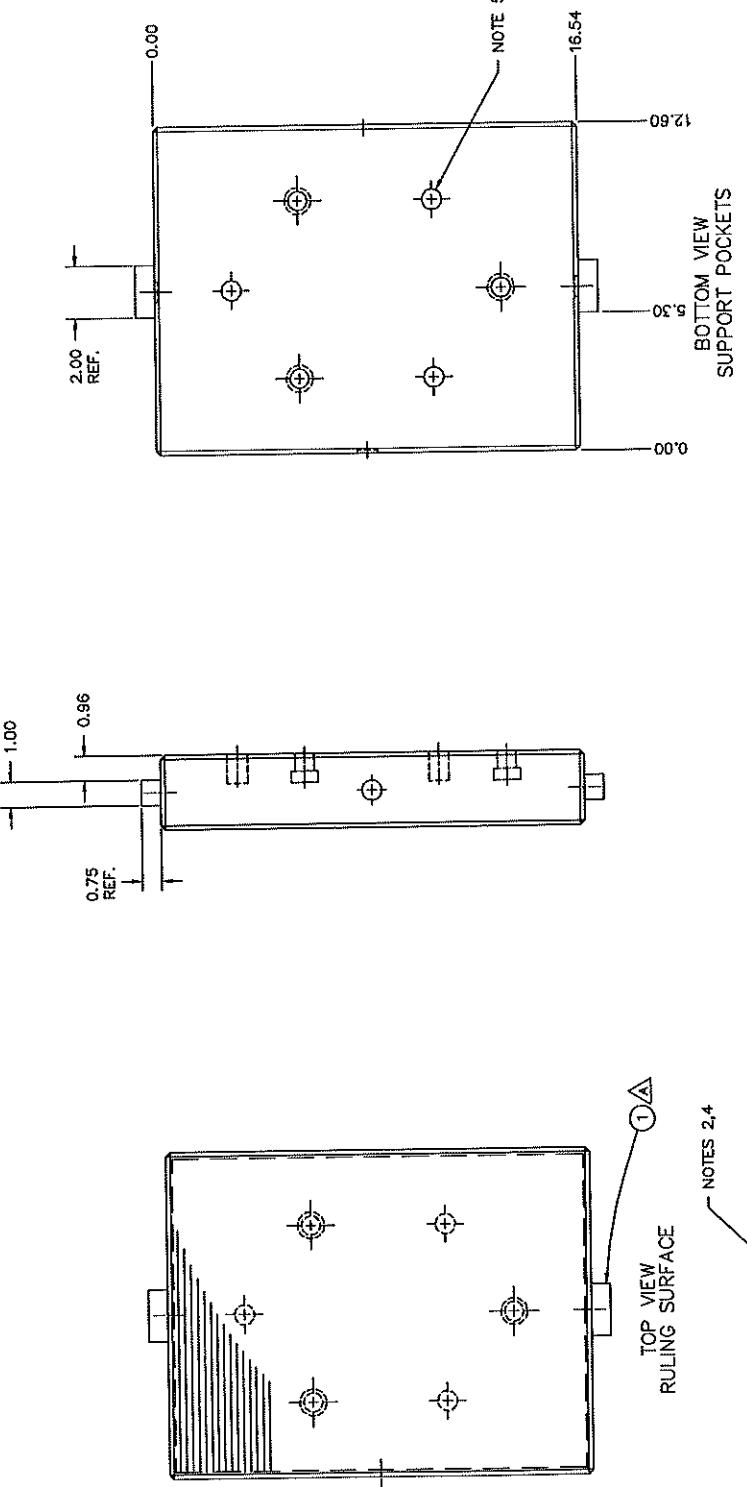




CUT #5







① CROSS-DISPERSER GRATING A

REPORT

1. GRADE

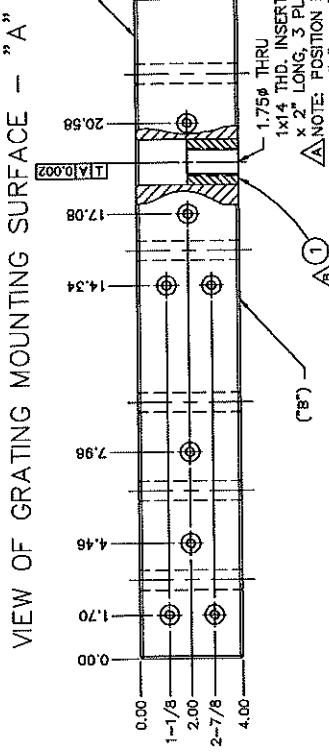
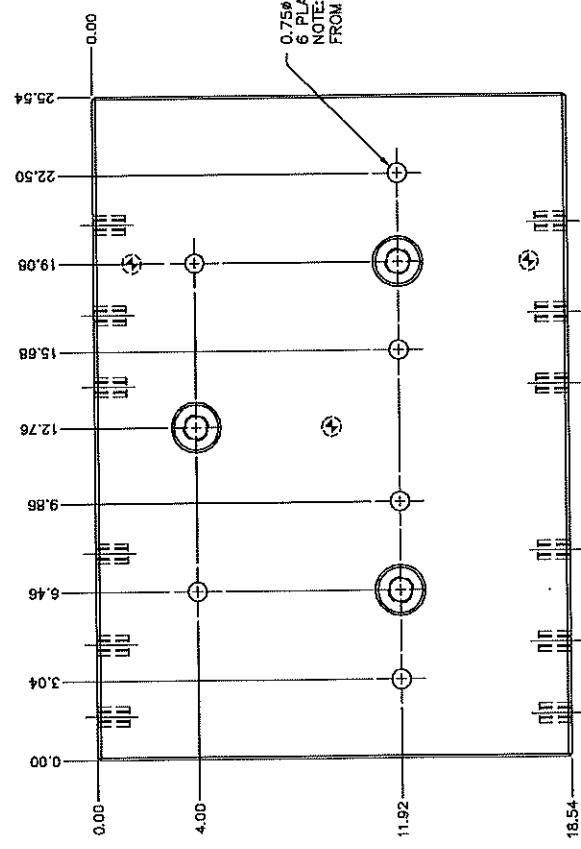
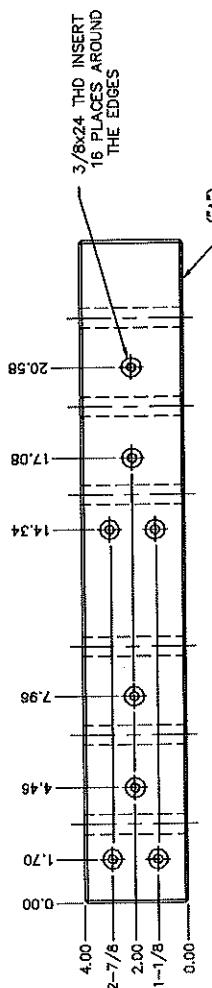
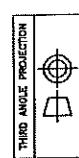
2. GRATING H4450-2 HAS NO SIDE POCKETS.
 3. GLUE NOTES ON H3040-A: HYDOL EA 9313
 4. FILL POCKET WITH RTV BEFORE GLUENG.
EPOXY RESIN; 5 DAY CURE TIME.
 5. GRATING H4450-2 HAS ONLY 3 REAR POCKETS.

PER ASSY

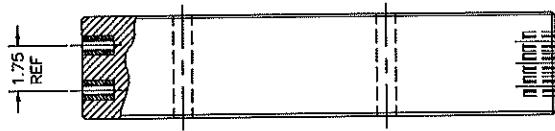
1

1/2

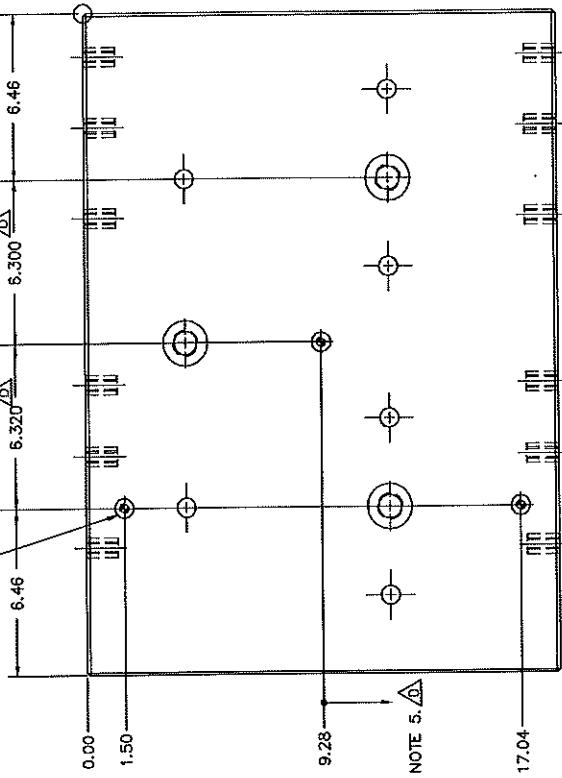
RECK/HIRES
GRATING AND END PADS
GLUE DETAIL



VIEW OF REAR SURFACE - "B"



3/8-24 THD INSERTS
3 PLACES
BLIND



(1) SUBPLATE

2 REC'D. \triangle VENDOR:
PYRAMID GRANITE
GRANITE "BLACK GABRO"
E=12x10⁶ PSI

NOTES:

1. BREAK ALL CORNERS
2. GRIND SIDE "A" SMOOTH.
3. ALL INSERTS TO BE

STAINLESS STEEL

4. THESE FEATURES ARE OFFSET FROM CENTER BY 0.01. THIS WAS BY MISTAKE.
5. THIS FEATURE IS OFFSET FROM CENTER BY 0.01. THIS WAS BY MISTAKE.
6. WEIGHT = 206 LBS

2x1 CROSS DISPERSER
SUBPLATE

H4475-2
1/2

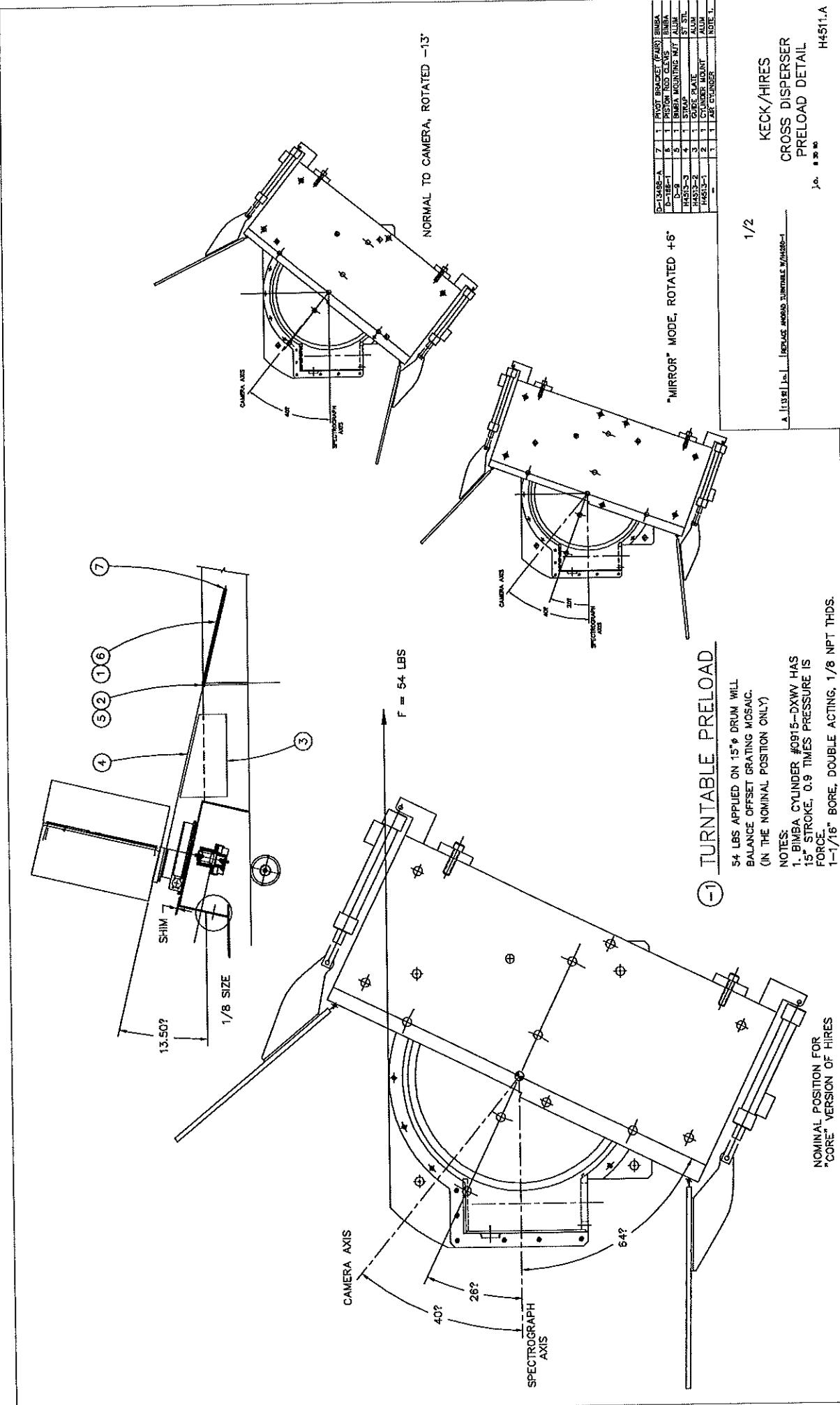
KECK/HIRES

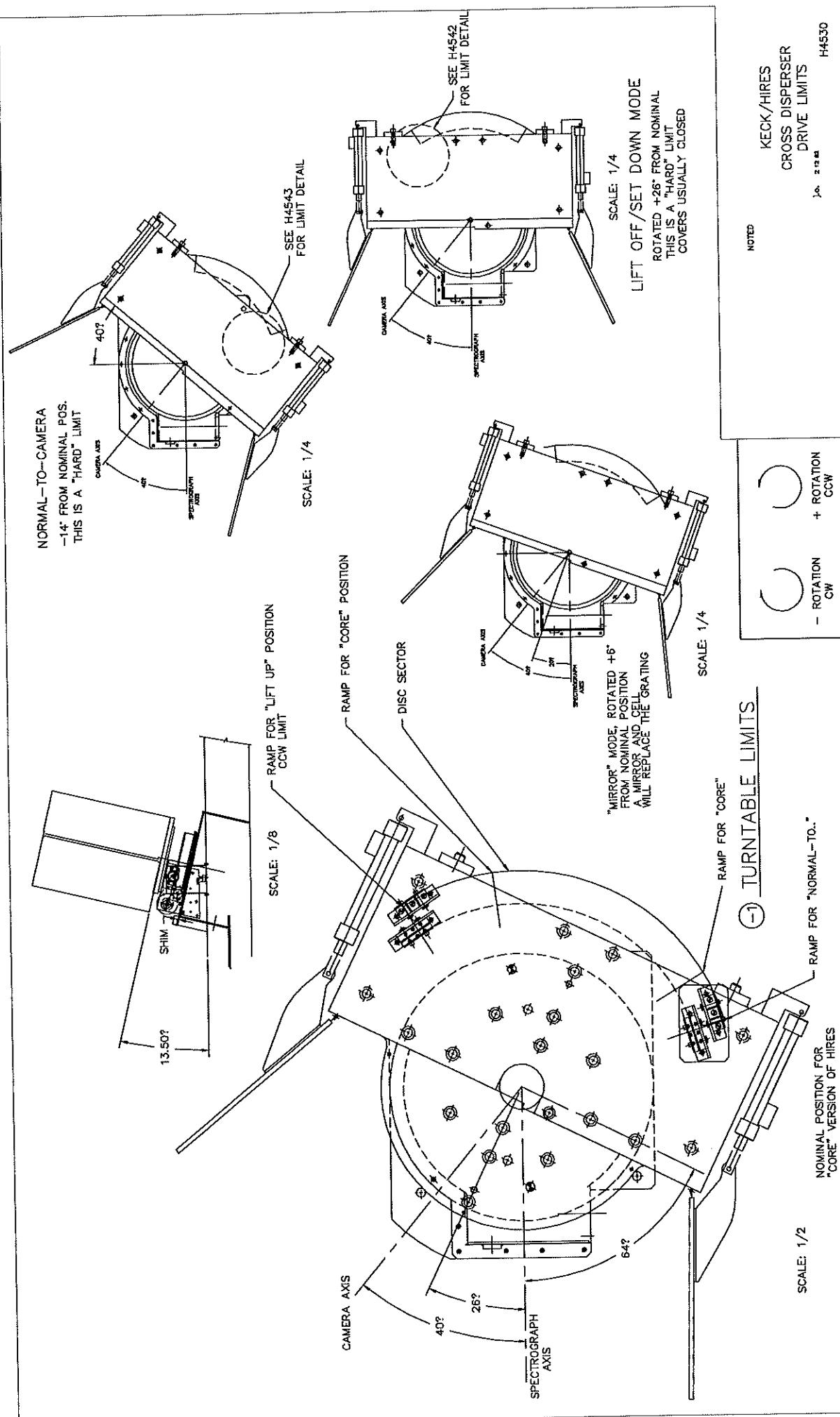
2x1 CROSS DISPERSER
SUBPLATE

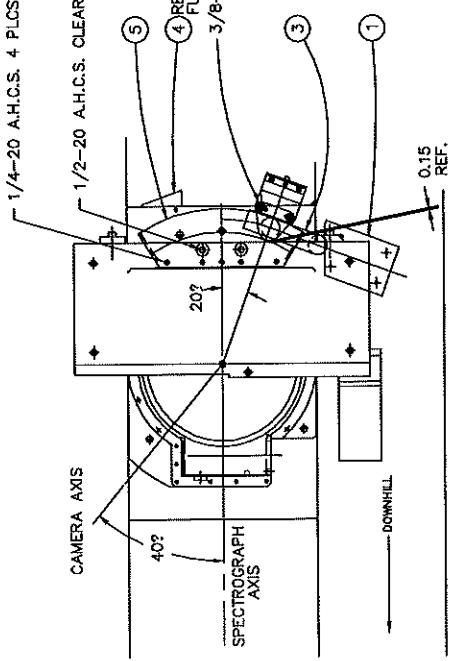
H4475-2

A	REF	1.758	1	3	154. INSERT	STNL
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1/2

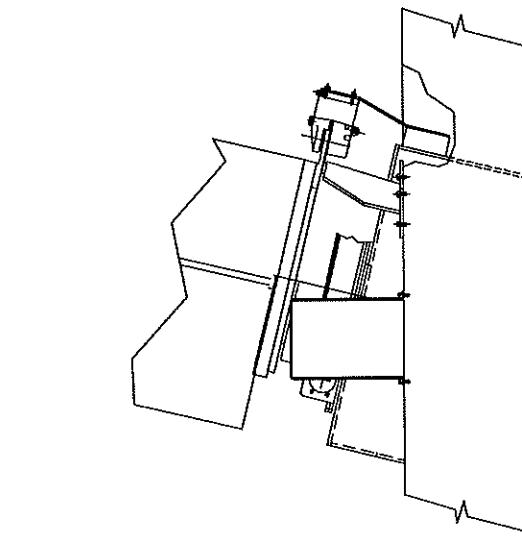
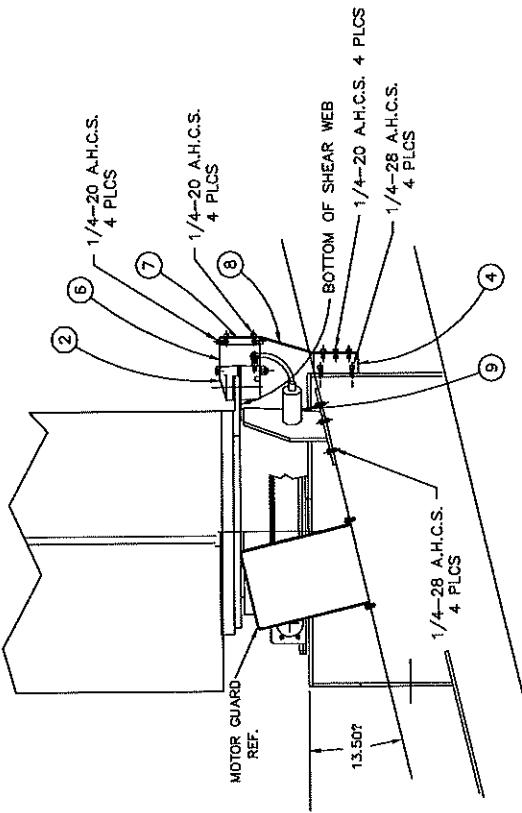






NORMAL TO CAMERA, ROTATED -14°
FROM NOMINAL POSITION
THIS IS A "HARD" LIMIT

ROTATED +26° FROM NOMINAL
THIS IS A "HARD" LIMIT
COVERS ARE CLOSED



(-1) BRAKE

NOTES:

1. BRAKE FUNCTION IS "NORMALLY ON" THAT IS, WHENEVER THERE IS AIR PRESSURE IN THE HIRIS ENCLOSURE, THE BRAKE WILL BE PREVENTING THE STAGE FROM ROTATING. A SIGNAL TO THE SOLENOID VALVE TO STOP THE AIR WILL BE PRESENT BEFORE DRIVING THE STAGE.
2. SEE ALSO H4533 FOR LIMITS

△ 3. SKINNER #V55LB2150-25A, 3-WAY SOLENOID VALVE, 11 WATTS, 1/2NPT PORTS, SS BODY, 1/2NPT CONDUIT OUTLET.

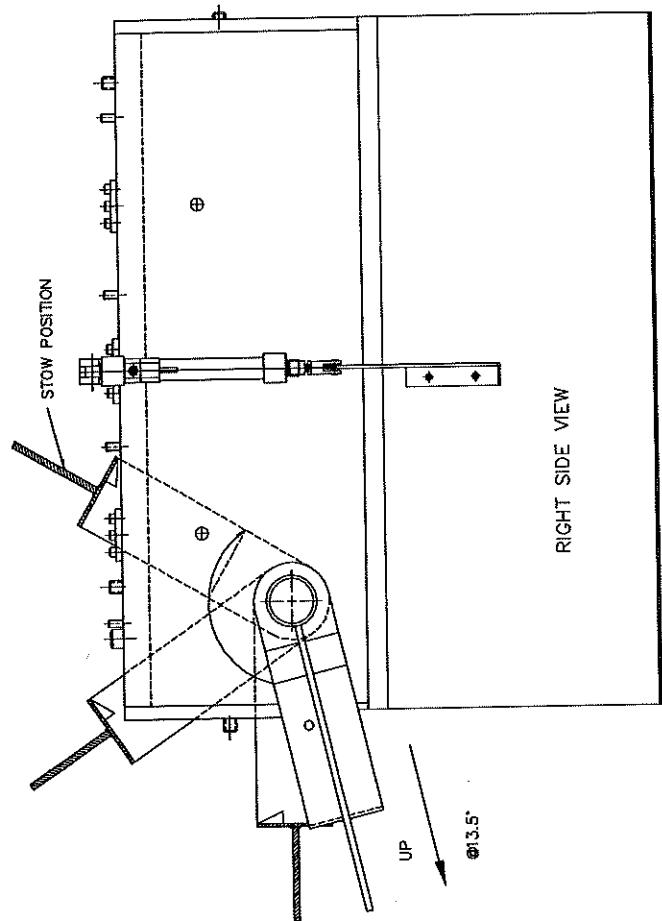
	REF.	DESCRIPTION	NOTE 1...
H4533-1	6	SOLENOID VALVE	AIR IN
H4533-2	7	PLATE	AIR IN
H4533-3	5	PLATE	AIR IN
H4533-4	5	FLX PLATE	AIR IN
H4533-5	5	PSO SENS	AIR IN
H4533-6	4	MOUNT BLOCK	AIR IN
H4533-7	3	SPEAR WEB	AIR IN
H4533-8	2	TO-2-MATIC BRAKE	AIR IN
H4533-9	1	SHEAR WEB MOUNT	AIR IN

1/4

KECK/HIRIS
CROSS-DISPERSER
BRAKE SUPPORT ASSY

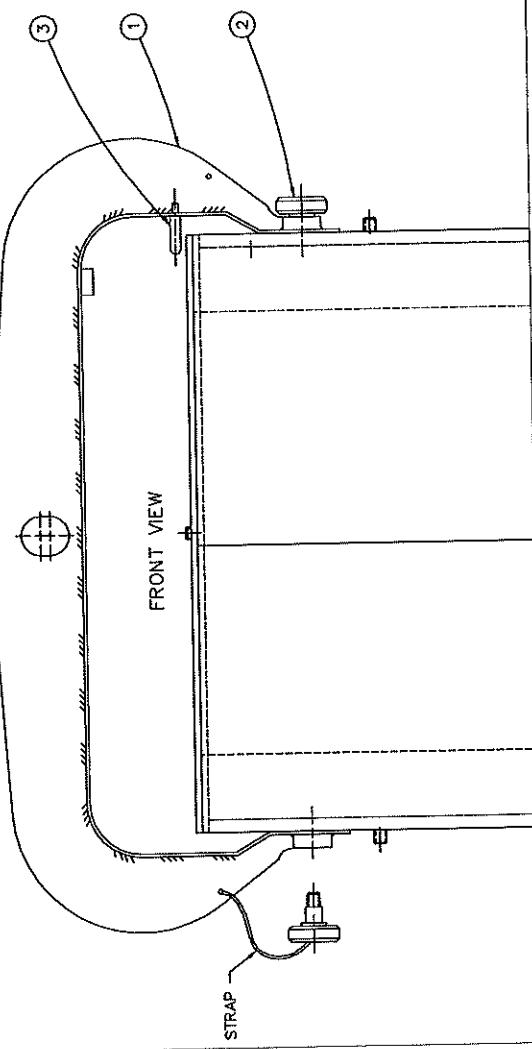
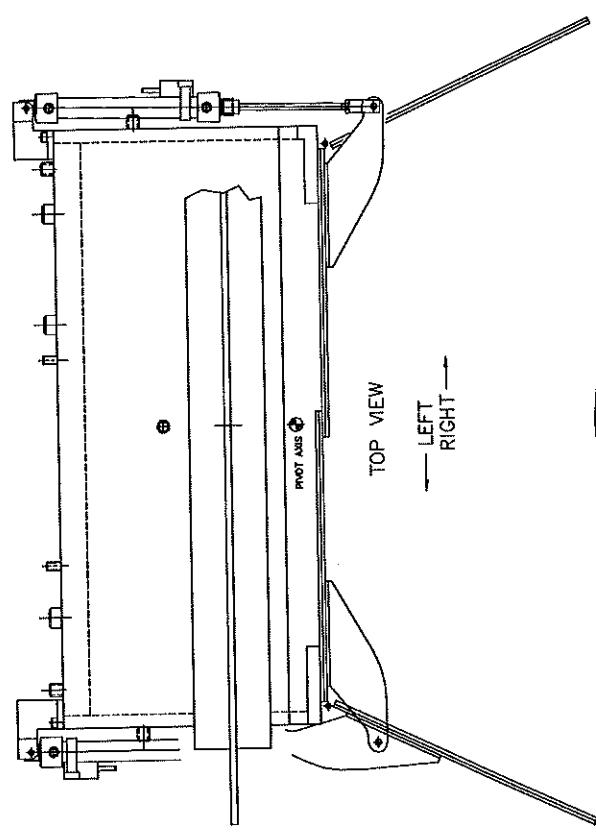
H4534.B

10. 3 20 22



(-1) CARRIER ASSEMBLY

1 REQ'D

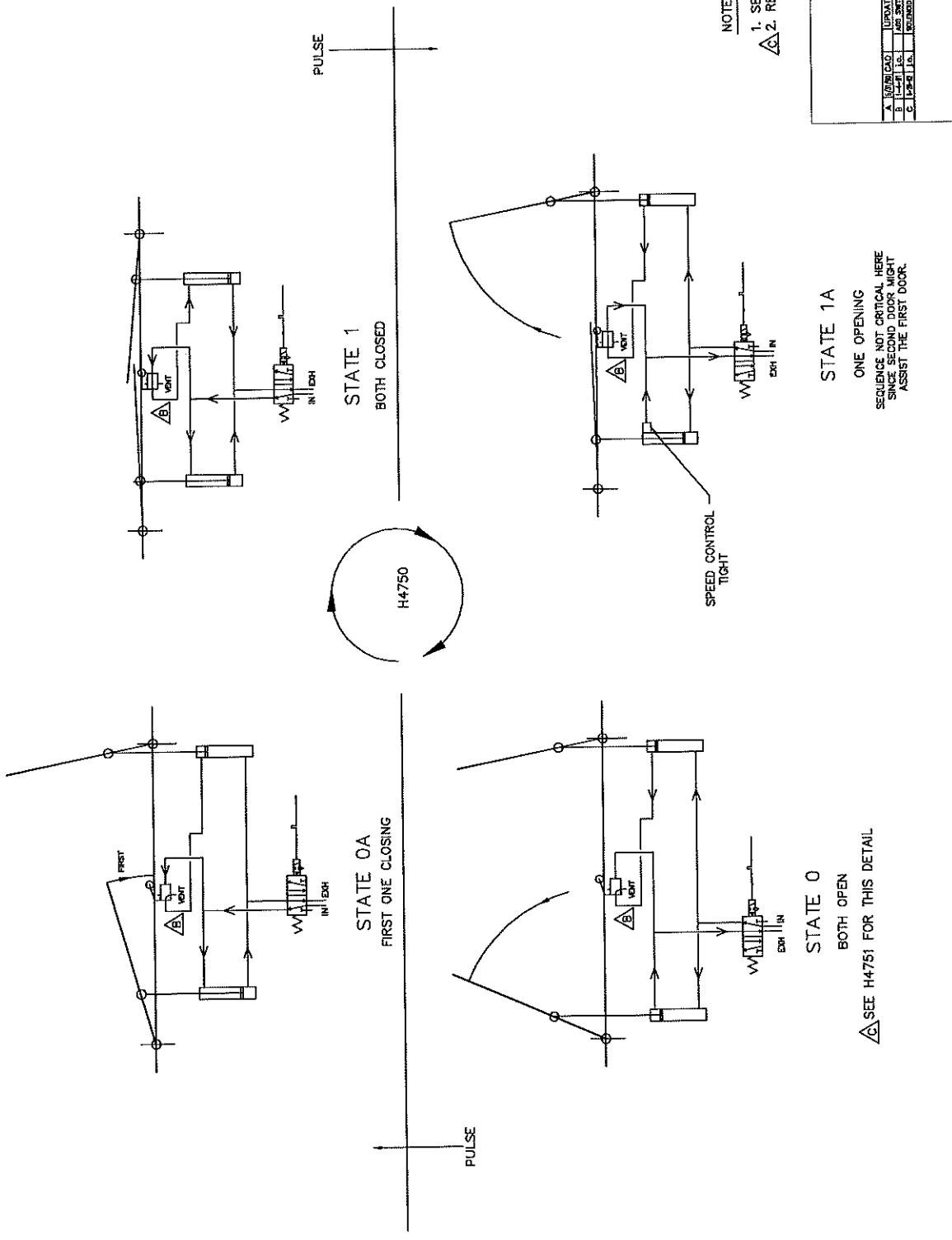
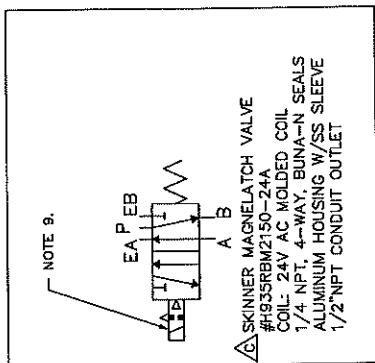
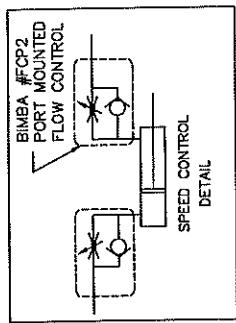


H4742-2	5	1	STOP	RUBBER
H4742-1	2	2	SCREW	SIL.
H4741-1	1	1	BRAKE, MEDIGENT	STE.

1/2

KECK/HIRES
CROSS DISPERSER
CARRIER ASSEMBLY

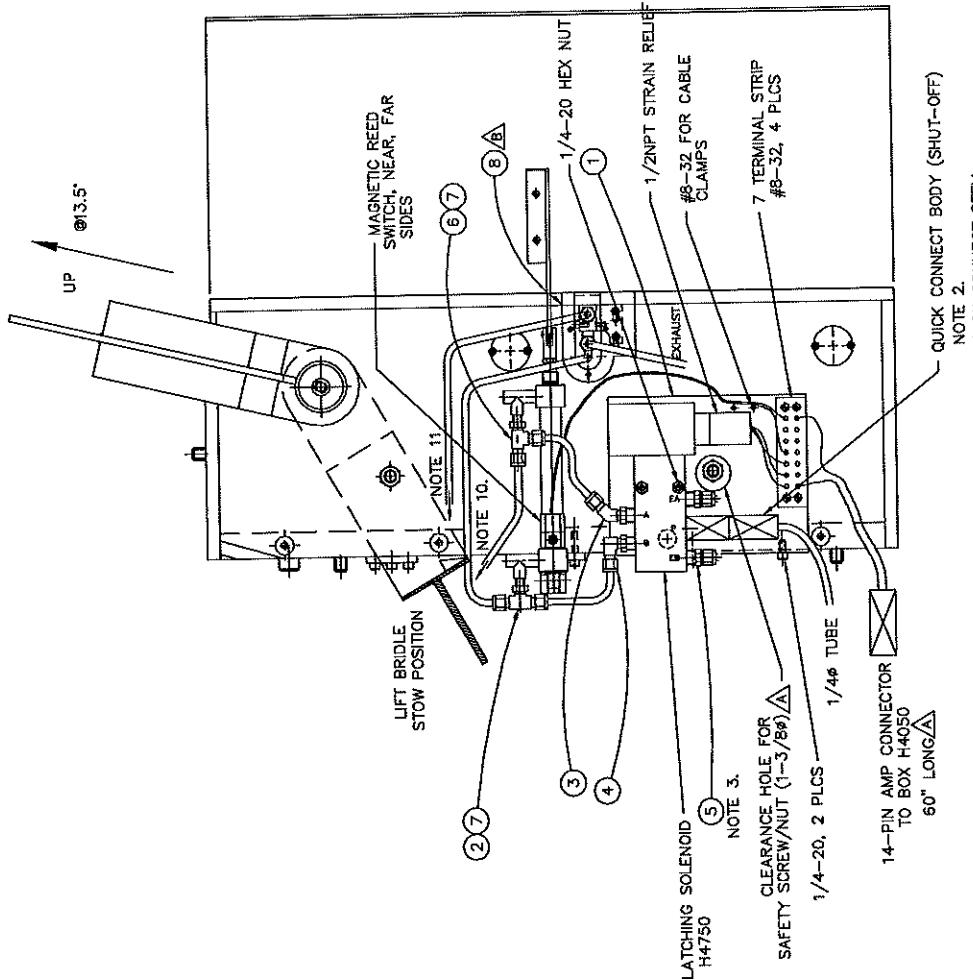
J.C. 3-2-80 H4740.A



(-1) AIR SOLENOID DETAIL

NOTES:

1. SWAGELOK B-QC4-S-4PM (1/4 NPT, MALE)
2. SWAGELOK B-QC4-B-400 (1/4" TUBE)
3. EXHAUST PORTS USE SWAGELOK 1/4" TUBE
4. SWAGELOK B-400-3TM (MALE RUN TEE, 1/8 NPT, 1/4" TUBE)
5. SWAGELOK B-400-5TM (MALE BRANCH TEE, 1/8 NPT, 1/4" TUBE)
6. SWAGELOK B-400-5-4 (45° 1/4" NPT TO 1/4" TUBE)
7. SWAGELOK B-400-2-4 (90° 1/4" NPT TO 1/4" TUBE)
8. SWAGELOK B-400-1-4 (MALE CONNECTOR 1/4" NPT TO 1/4" TUBE)
9. BIMBA FCF2, SPEED CONTROL (1/8 NPT)
10. GOES TO FRONT OF FAR SIDE CYLINDER.
11. GOES TO REAR (FIXED PIVOT END) OF FAR SIDE CYLINDER.



1/2

LEFT SIDE VIEW

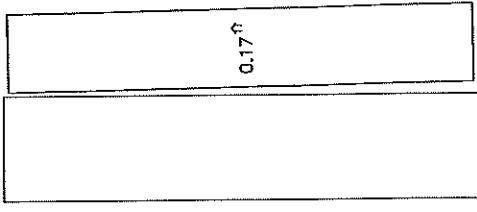
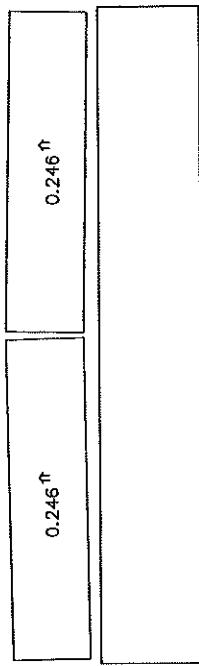
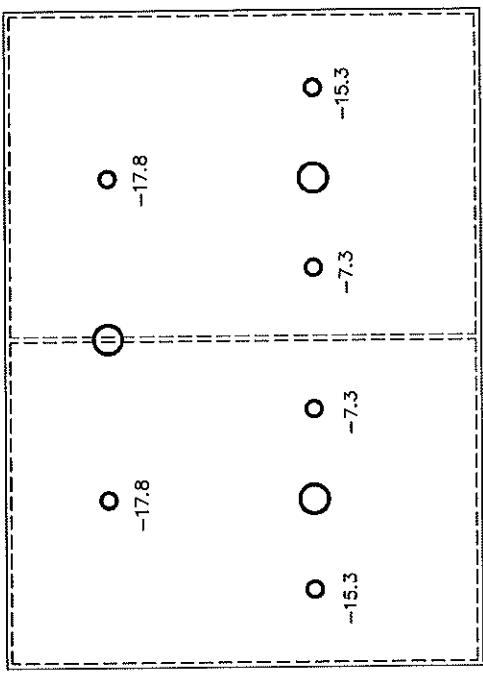
KECK / HRES

A	1 3/8" L.C.	INPUT TO ASSEMBLY CONNECTION	PC
B	1 3/8" L.C.	EXHAUST TUBE & DUST COVER	NOTE 9.
			NOTE 4.
			NOTE 5.
			NOTE 6.
			NOTE 7.
			NOTE 8.
			NOTE 9.
			NOTE 10.
			NOTE 11.

CROSS DISPENSER
AIR SOLENOID MOUNTING
10. 1 3/8" L.C.

H4751.B

DEFLECTION IN MICRO-INCHES



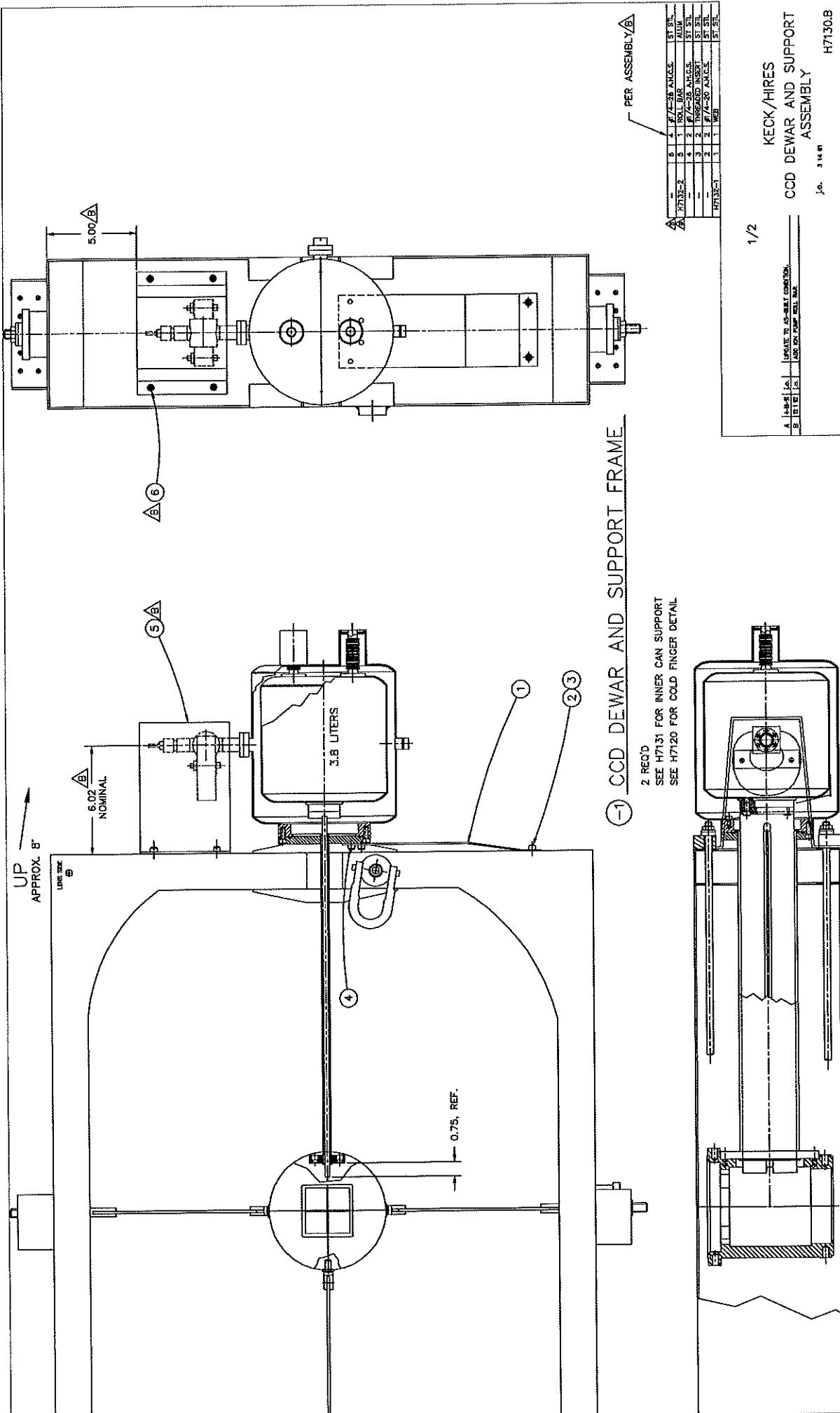
TIITS IN ARC-SECONDS

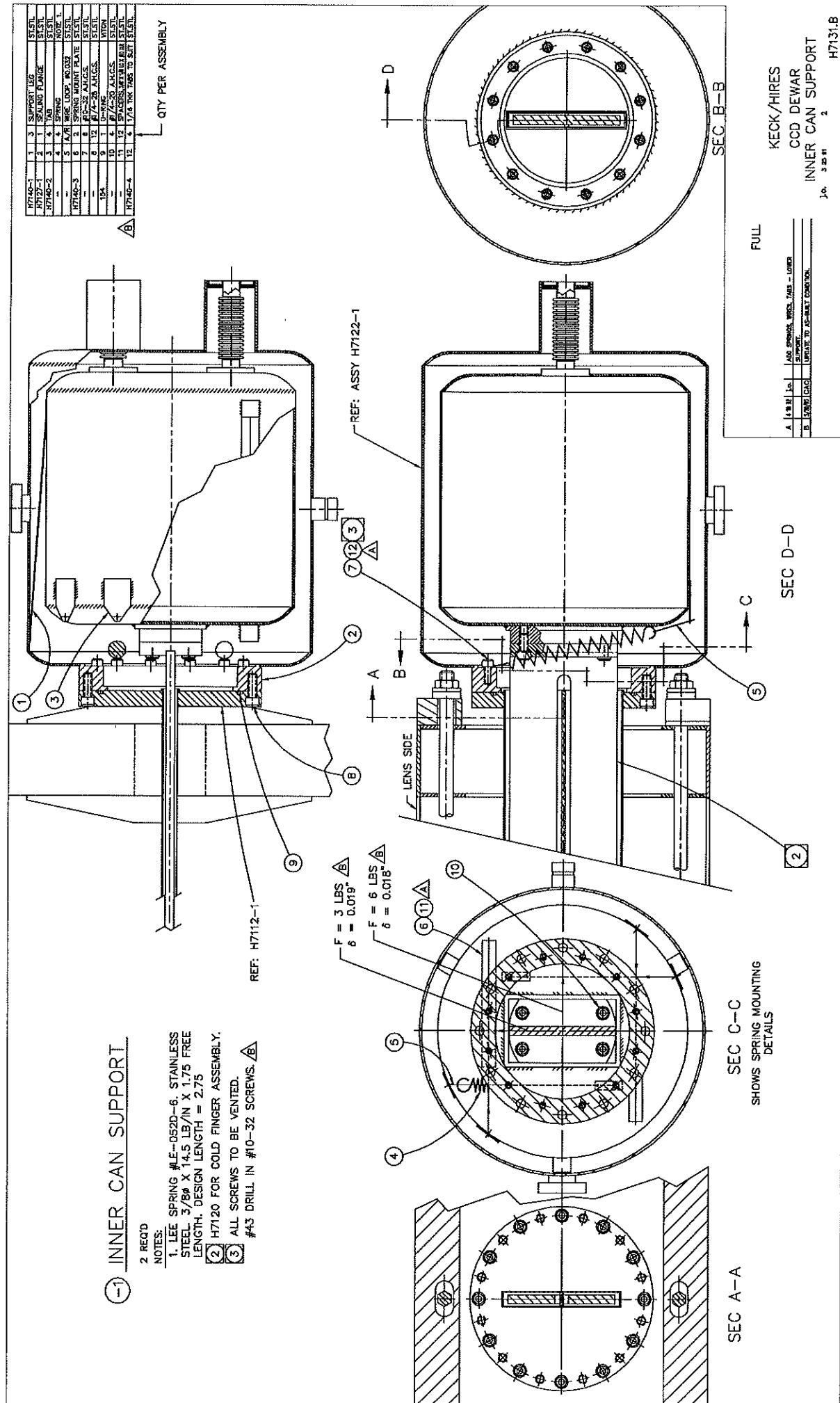
KECK/HIRES
CROSS DISPERSER
FEA RESULTS
H4802

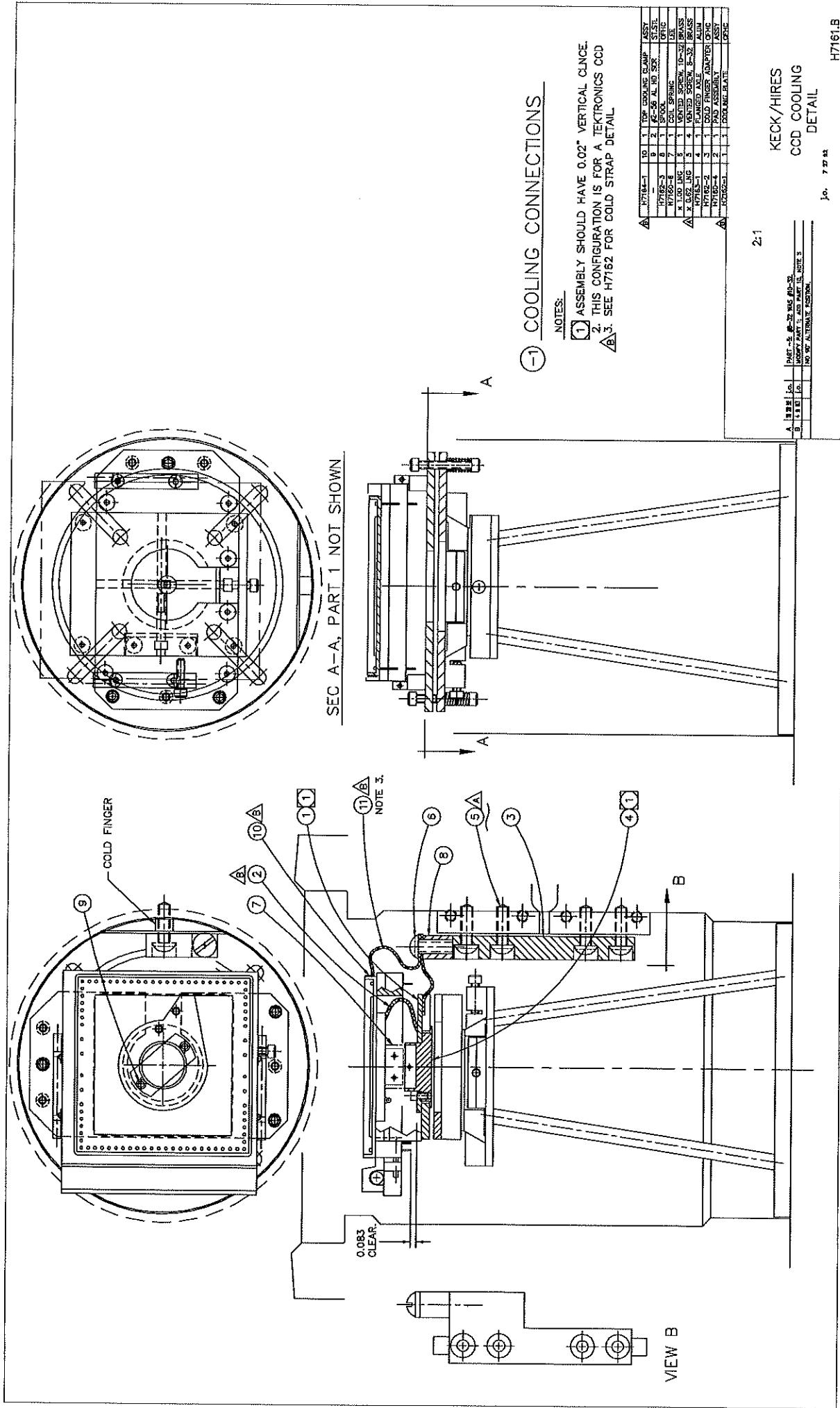
Appendix F List of Drawings — CCD Dewar

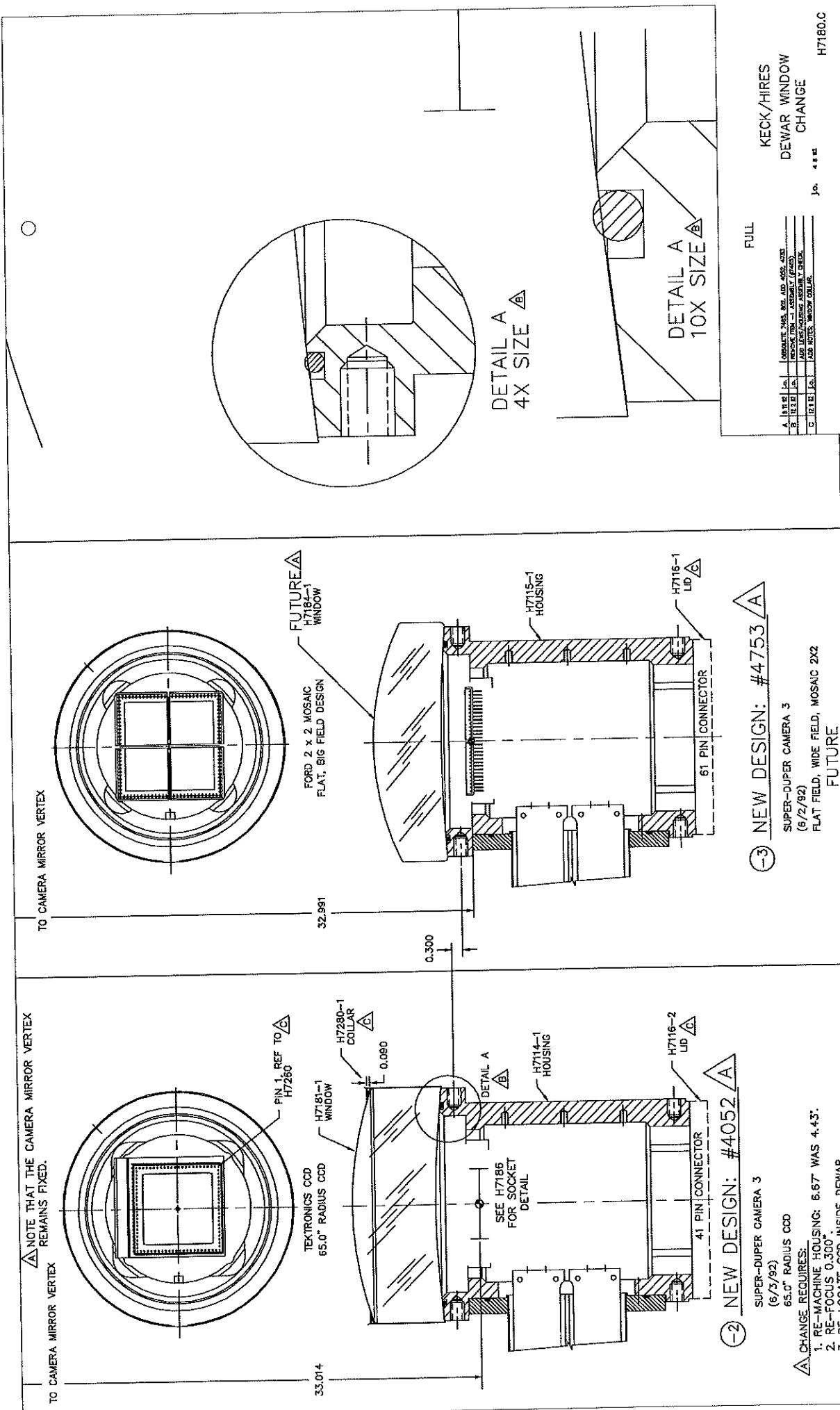
1. H7005 Locating Tree
2. H7130 CCD Dewar and Support Frame
3. H7131 CCD Dewar Inner Can and Support
4. H7161 CCD Cooling Detail
5. H7180 CCD Dewar Window (Field Flattener)
6. H7187 CCD Socket Alignment Tool Assembly
7. H7192 CCD and X-Y Stage Assembly
8. H7209 Dewar Support Frame Assembly
9. H7250 Dewar and Stage Assembly
10. H7260 Dewar With CCD and Echelle Format Layout
11. H7312 CCD Focus Drive Sub-Assembly
12. H7350 CCD Drive Schematic
13. H7500 CCD Electronics (Pre-Amp) Housing
14. H7600 Liquid Nitrogen Fill System
15. H7601 Liquid Nitrogen Level Sensor
16. H7800 Dewar Lifter and Cart
17. H7815 Dewar Cart Assembly
18. EL-1027-2S Ion Pump Modification

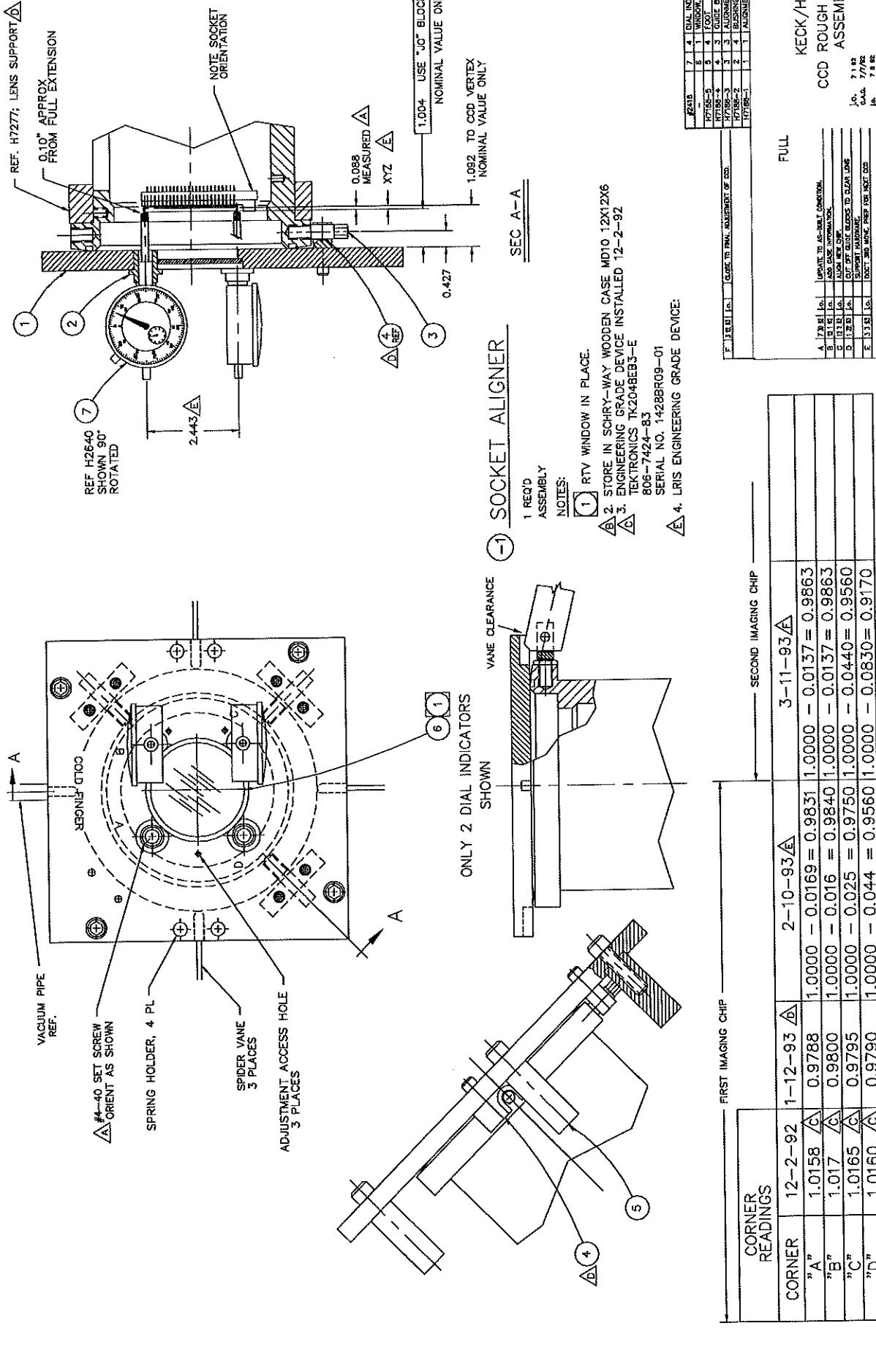
H7005	DEWAR LOCATING TREE	(65° RAD)	H7180	WINDOW CHANGE (TO #B02) ASSEMBLY	H7225	DEWAR SUPPORT BASE WELDING LOCATION	H7322	CCD FOCUS DETAIL
H7006	PRELIMINARY DEWAR ASSEMBLY		H7181	DEWAR WINDOW (LENS)	H7230	DEWAR SUPPORT FRAME DETAIL	H7323	" "
	TEK CCD WITH 27 MICRON PIXELS		H7182	WINDOW HOLDER (TEST)	H7231	DEWAR SUPPORT FRAME SPIDER DETAIL	H7324	CCD FOCUS, SLIT SIDE DRIVE ASSY
H7010	RAYTRACE THRU FIELD FLATTENER		H7183	LENS HOLDER (TEST, DARKSLIDE)	H7233	DEWAR SUPPORT FRAME DETAIL	H7326	" "
H7011	DETAIL OF FIELD FLATTENER		H7184	DEWAR WINDOW (FUTURE)	H7234	DEWAR SUPPORT FRAME DETAIL	H7327	" "
H7050	CCD CAMERA JUNCTION BOX		H7186	DEWAR SOCKET, TEK	H7235	DEWAR SUPPORT FRAME DETAIL	H7328	" "
H7051	" DETAILS		H7187	ROUGH ALIGNER	H7236	DEWAR, FRAME, BASE ASSEMBLY DETAIL	H7329	" "
H7052	" DETAILS		H7188	" DETAILS	H7250	DEWAR, FRAME, BASE ASSEMBLY	H7340	DRIVE STRUCTURE 3-D
H7060	ALIGNMENT TOOL (MIRROR)		H7190	SOCKET DETAILS	H7251	DEWAR, FRAME, FLEX ARM BRACKETS	H7350	KINEMATICS OF FOCUS DRIVE
H7110	VACUUM HOUSING ASSEMBLY		H7191	" DETAILS	H7252	"	H7351	BELT & PULLEY DETAIL
H7112	TRANSFER PIPE ASSEMBLY		H7192	" DETAILS, X-Y STAGE	H7253	"	H7352	DRIVE DETAIL (AND LIMITS)
H7113	" DETAILS		H7193	" DETAILS, SLIDES	H7260	CCD PIN 1 DEFINED	H7355	DRIVE DETAIL (LIMITS)
H7114	VACUUM HOUSING BODY, ^{FORD} TEK		H7194	" DETAILS	H7261	CCD CONTROLLER LOCATION	H7356	DRIVE DETAIL (LIMITS)
H7115	VACUUM HOUSING BODY, ^{FORD} TEK		H7195	" DETAILS, SPIDER	H7270	DARKSLIDE ASSEMBLY	H7357	DRIVE DETAIL (LIMITS)
H7116	COLD DEWAR LID DETAILS		H7196	" DETAILS	H7271	DARKSLIDE DETAILS	H7358	DRIVE DETAIL (LIMITS)
H7117	FLANGE		H7197	" DETAILS	H7273	"	H7359	DRIVE DETAIL
H7118	LID DETAIL		H7198	MASK SEE H7410	H7274	"	H7360	DRIVE DETAIL - TENSIONER
H7119	"		H7199	SOCKET ADJUSTMENTS	H7276	DEWAR LENS SUPPORT	H7365	CCD HEADER DETAIL (FORWARD)
H7120	COLD FINGER ASSEMBLY				H7277	"	H7401	TEKTRONICS CCD
H7121	ION PUMP				H7278	"	H7402	FOCUS TESTING
H7122	LIQ. NITROGEN CAN ASSY		H7100	PRELIMINARY DEWAR ASSEMBLY	H7279	"	H7410	MASK
H7123	INNER CAN DETAIL		H7200	DEWAR TEST ASSEMBLY	H7280	DEWAR LENS COLLAR	H7411	MEASURE CCD
H7124	VACUUM DETAIL			/TEK CCD WITH 27 MICRON PIXELS	H7281	"	H7500	CCD ELEC. BOX - PRE-AMP
H7125	VACUUM DETAIL		H7201	DEWAR TEST PLATE, ROUND	H7282	DARKSLIDE DETAILS	H7501	PRE-AMP BOX DETAIL
H7126	OUTER CYLINDER		H7202	DEWAR TEST PLATE, OBLONG	H7283	"	H7502	PRE-AMP BOX DETAIL
H7127	SEALING FLANGE		H7205	" DETAILS	H7293	"	H7503	PRE-AMP BOX DETAIL
H7128	41-PIN SOCKET		H7206	" DETAILS	H7294	"	H1452	CCD CONTROLLER BOX
H7129	VACUUM DETAIL		H7207	" DETAILS	H7295	"	H7600	LIQUID NITROGEN TRANSFER
H7130	DEWAR AND SUPPORT FRAME		H7208	DEWAR TEST - BIG WINDOW	H7296	"	H7601	SENSOR ASSEMBLY
H7131	INNER CAN SUPPORT		H7209	DEWAR SUPPORT FRAME ASSEMBLY	H7297	"	H7602	SUPPLY DEWAR MOD.
H7132	SUPPORT DETAIL		H7210	" DETAILS	H7298	"	H7603	DETAIL
H7133	COLD FINGER		H7211	" DETAILS	H7303	CCD FOCUS: PUSHRODS ASSEMBLY	H7810	SUPPLY DEWAR ASSY
H7136	COLD FINGER SUPPORT		H7212	" DETAILS		"	H7815	DEWAR
H7140	INNER SUPPORT DETAIL		H7213	3-D OF DEWAR FRAME	H7304	"	H7816	CART ASSY
H7160	COOLING DETAIL		H7214	DEWAR SUPPORT FRAME MACHINING DETAIL	H7306	LOWER DRIVE ASSEMBLY	H7817	WHEEL
H7161	"		H7215	DEWAR SUPPORT BASE ASSEMBLY	H7307	CCD FOCUS: PUSHROD DETAIL	H7818	CART DETAIL
H7162	"		H7216	" DETAILS	H7310	"	H7823	LIFTER DETAIL
H7163	"		H7217	" DETAILS	H7311	"	H7824	LIFTER DETAIL
H7170	FORD SPIDER		H7218	" DETAILS	H7312	CCD FOCUS SCREW ASSEMBLY	H7825	LIFTER DETAIL
H7171	DARKSLIDE & SHUTTER FOR CCD TESTING		H7219	" DETAILS	H7313	"	H7826	LIFTER DETAIL
H7172	"		H7220	" DETAILS	H7314	"	H7827	LIFTER DETAIL
H7173	"		H7221	" DETAILS	H7315	CCD FOCUS SCREW DETAIL	H7830	LIFTER BEAM
					H7316	"	H7831	BEAM DETAIL
					H7317	"	H7832	VENT ASSEMBLY
						"		
H7300	ENCODER ASSEMBLY - CCD FOCUS TEST		H7350	ENCODER TEST DETAILS	H7317	"		KECK/HIRES
H7301	" DETAIL		H7331	" DETAIL	H7318	"		DEWAR LOCATING TREE
H7302	" DETAIL		H7332	" DETAIL	H7319	"		FINAL VERSION <u>A</u>
			H7333	" DETAIL	H7320	"		
					H7321	"		
								JO. 83511 SHIP TO BIRMINGHAM H7005.A

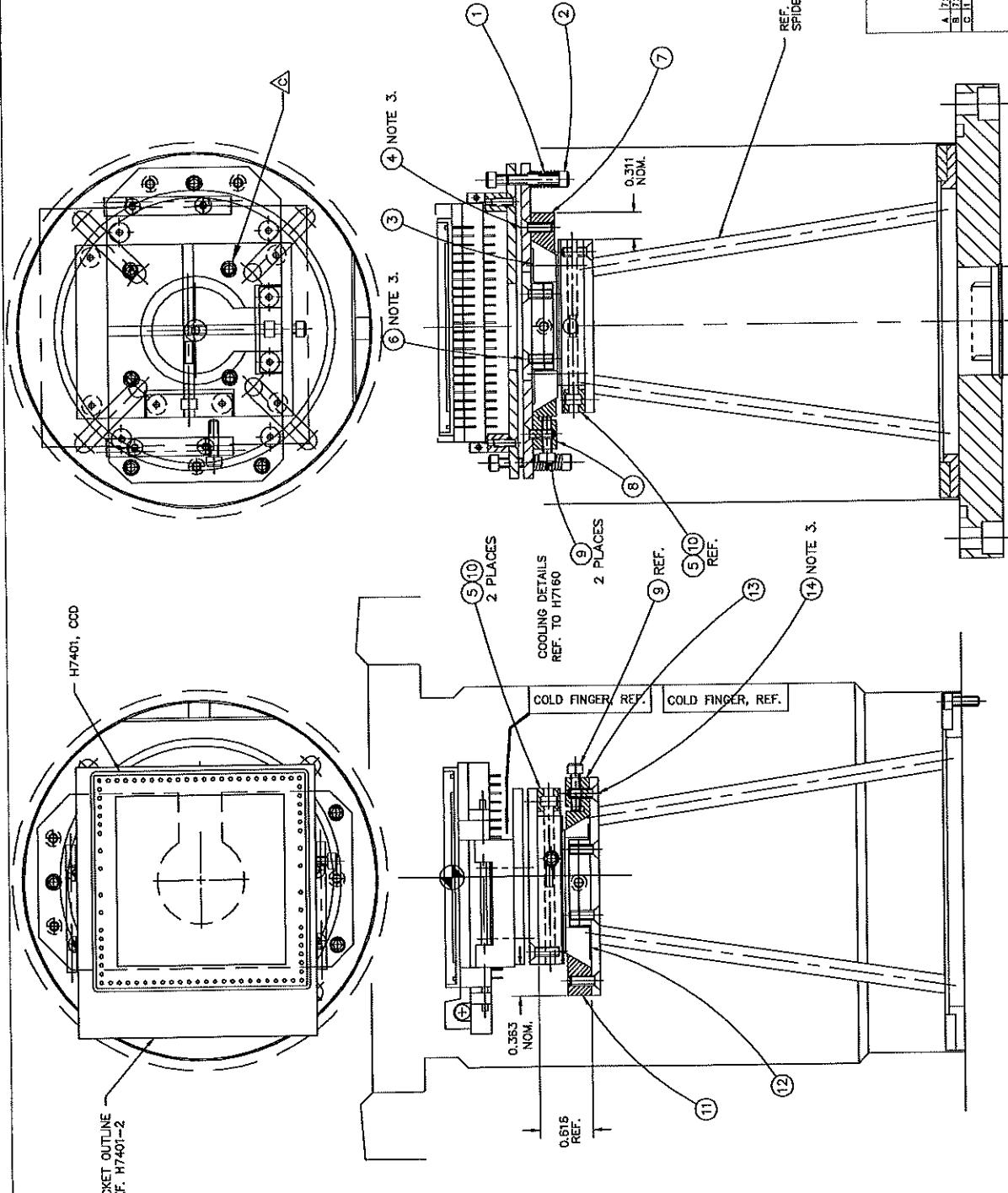












(-1) X-Y STAGE ASSEMBLY

1

NOTES:

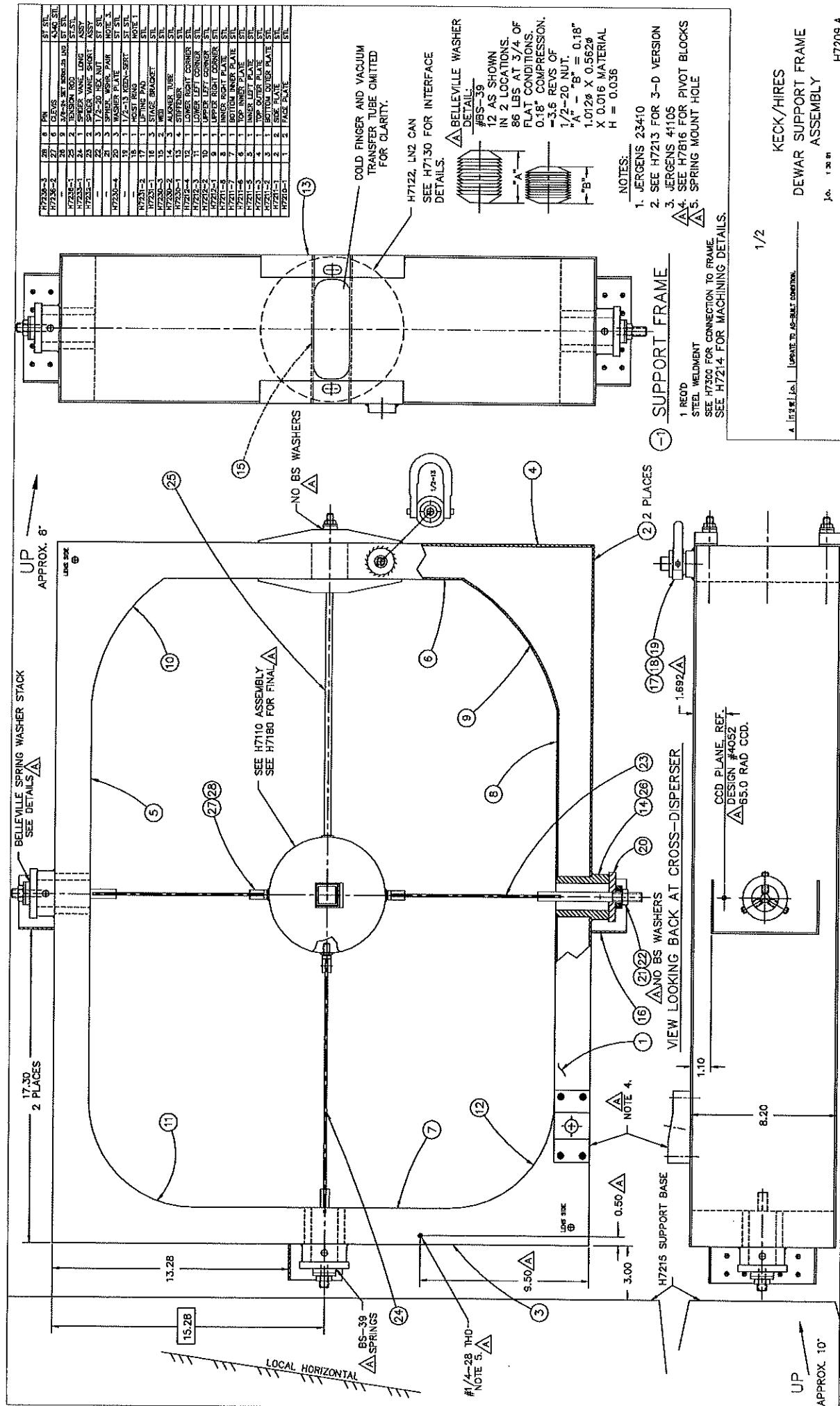
1. LEE, 23 LB-IN, 9/16 FREE LENGTH,
0.028" WIRE, 10 TURNS, D-264 SOLID
HEIGHT - 6-3/4 LB-S.
 2. SEE H7199 FOR X-Y STAGE
ADJUSTING PROCEDURE.
 3. USE SOCKET DRIVE FLAT HEAD SCREWS
(1/16 HEX).
 4. SEE H7793 FOR UPPER/LOWER SLIDE ASSY
THIS ASSEMBLY IS FOR THE TRONICS
CCD WITH 65MM CONVEX RADIUS.
 5.  A

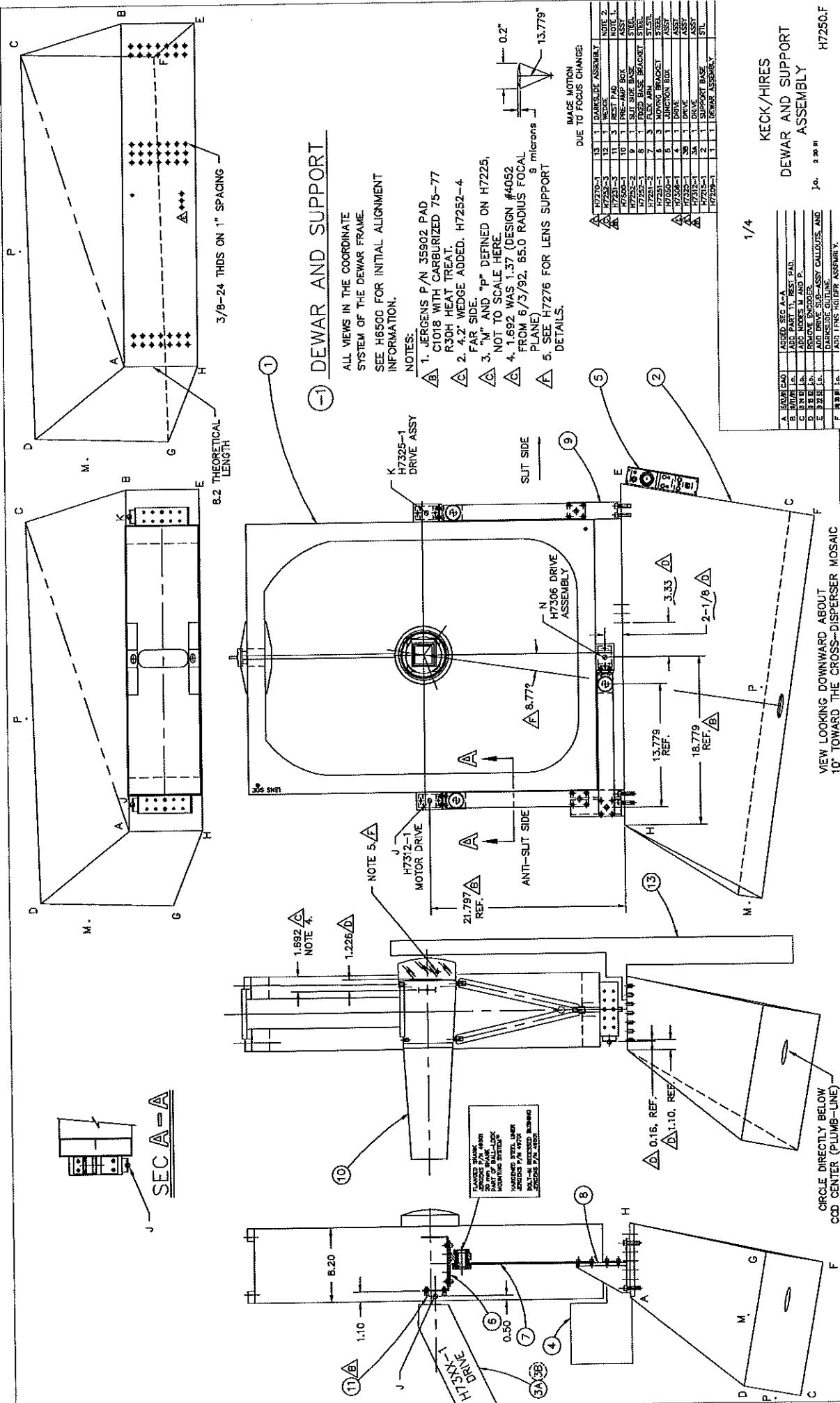
	4	4D	RT	HD SER	STSL
H7574-3	13	1	UPPER	LOCKING	SBS
H7570-2	12	1	LOWER	SIDE	AUIN
H7574-2	11	1	UPPER	SIDE	AUIN
X 10 LONG	10	2	4D	AL	CP SER
X 10 LONG	9	2	4D	AL	CP SER
H7574-4	8	1	UPPER	LOCKING	SBS
H7574-4	7	1	UPPER	LOCKING	SBS
H7574-1	5	4	UPPER	LOCKING	SBS
H7574-1	5	2	SCUT	BUSHING	SBS
H7570-1	4	4	UPPER	LOCKING	SBS
X 101 LONG	3	1	UPPER	LOCKING	SBS
X 101 LONG	2	3	SCUT	CP SER	SBS

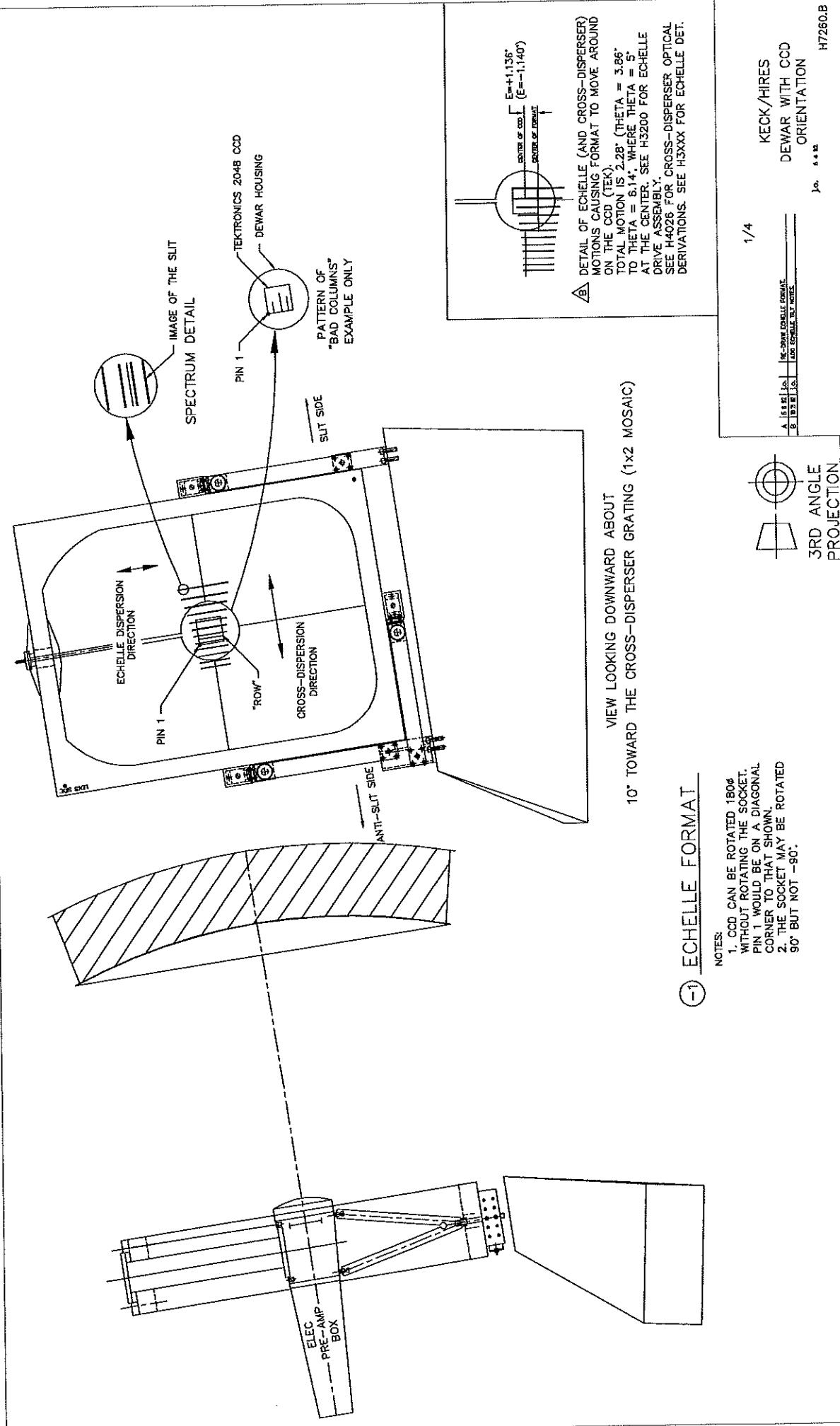
2-1

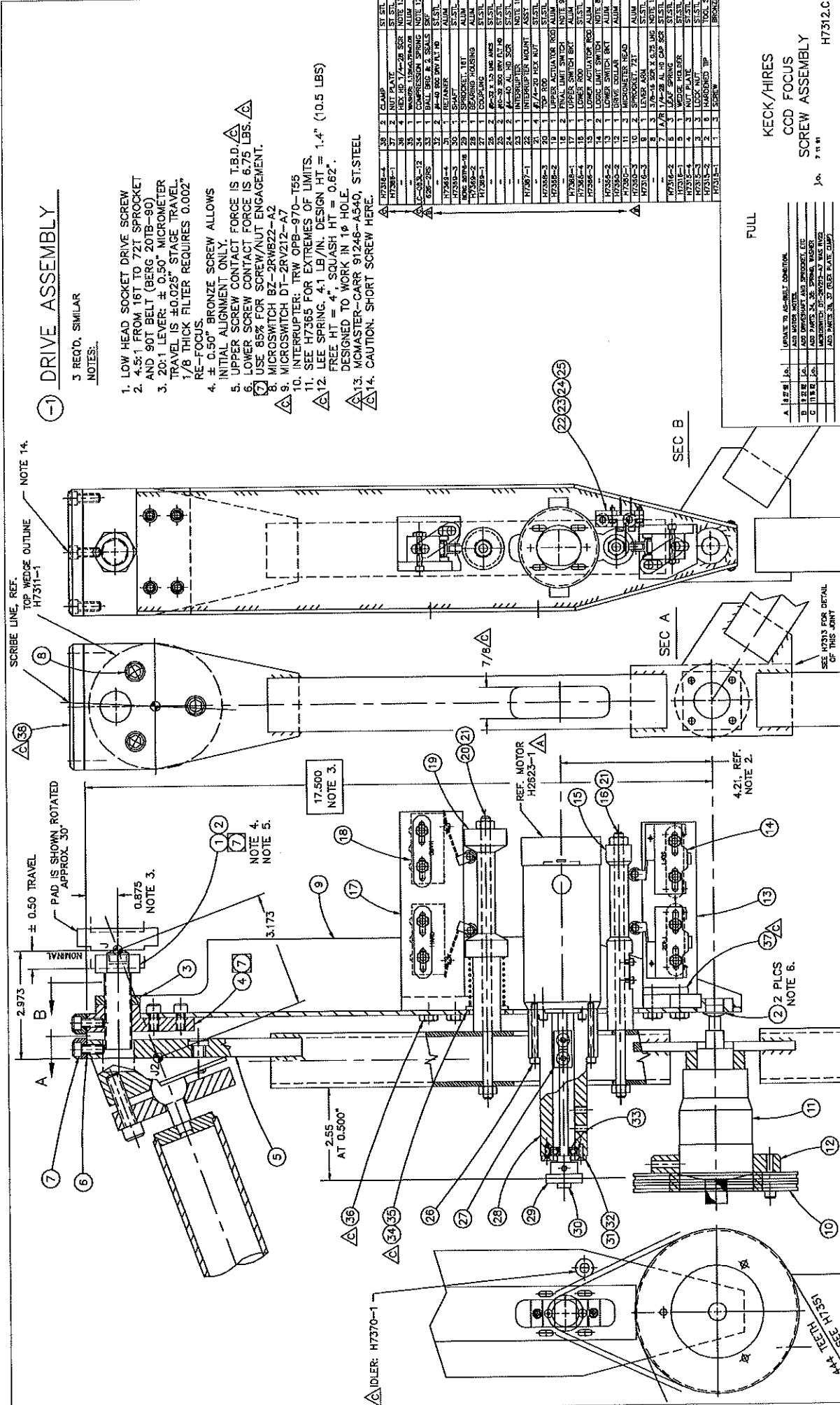
KECK/HIRES
CCD DEWAR
X-Y STAGE

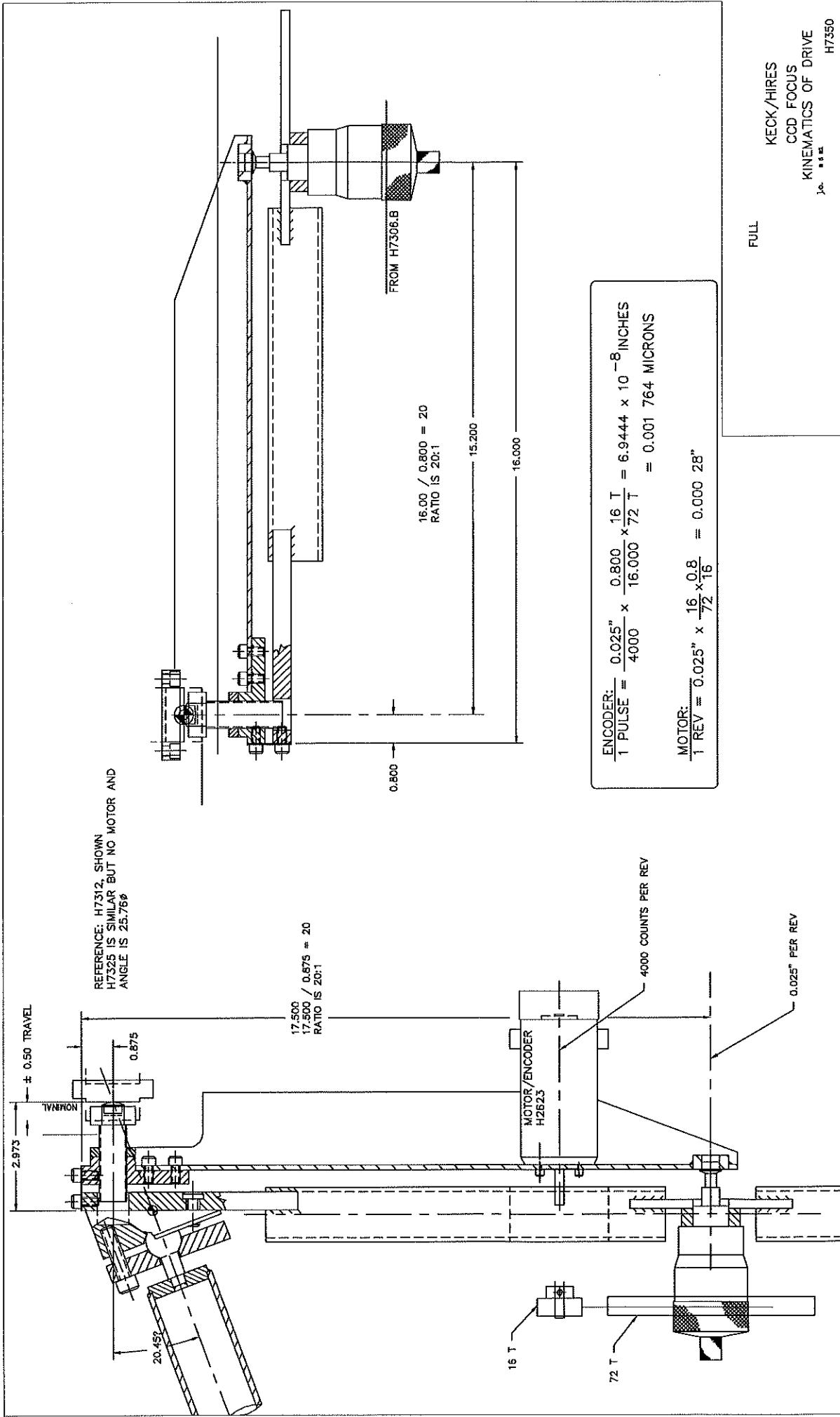
H7192.C

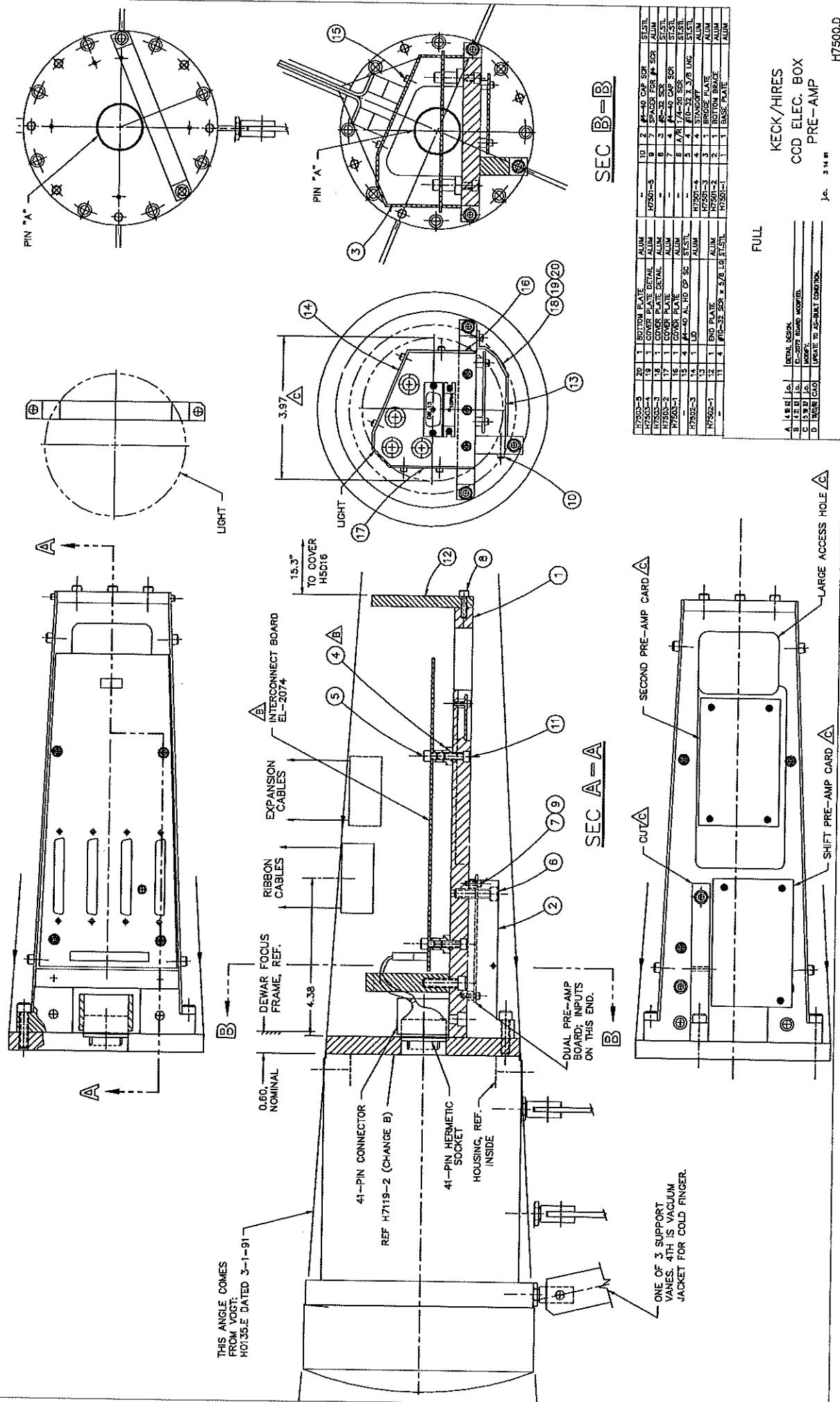


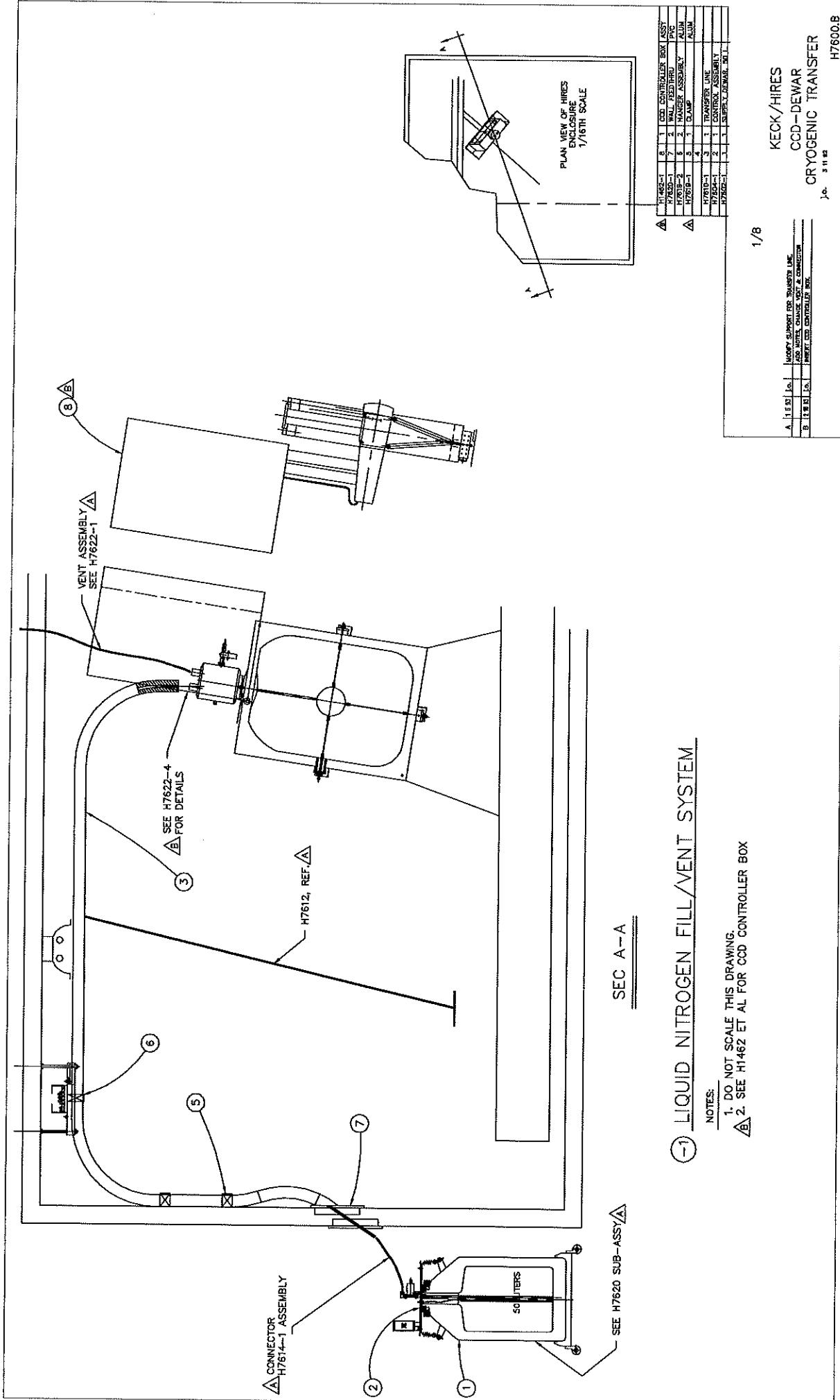


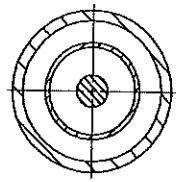




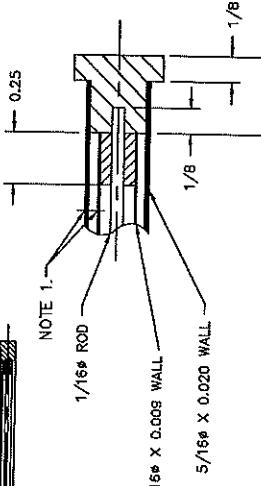
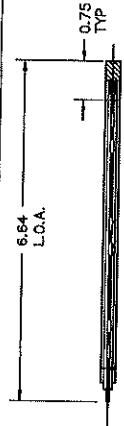








SECTION
10 TIMES SIZE

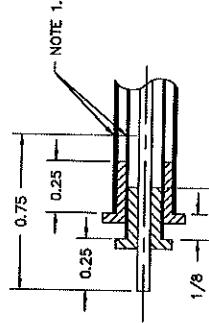


BOTTOM END DETAIL

1 REQD
NOTES:

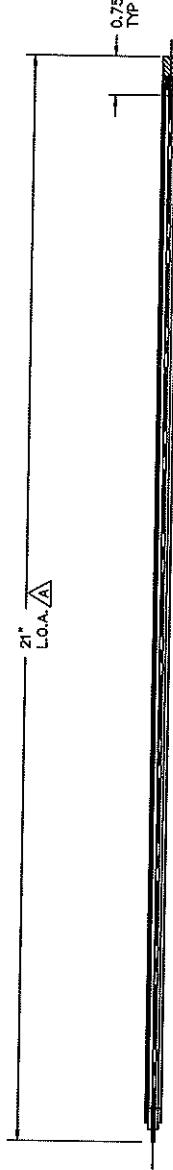
1. 3/32" BOTH TUBES, LINE UP
2. AS SHOWN, BOTH ENDS ARE THE SAME.
3. BRASS TUBING
4. LEXAN SPACERS, PRESS FIT
5. FLANGES ARE FOR REMOVAL
6. DESIGN BORROWED FROM LAWRENCE BERKELEY LAB REPORT LBL-19674 DATED MAY 1985, D.A. LANDIS

(-1) DEWAR SENSOR



TOP END DETAIL

BOTTOM END DETAIL



(-2) SUPPLY DEWAR SENSOR

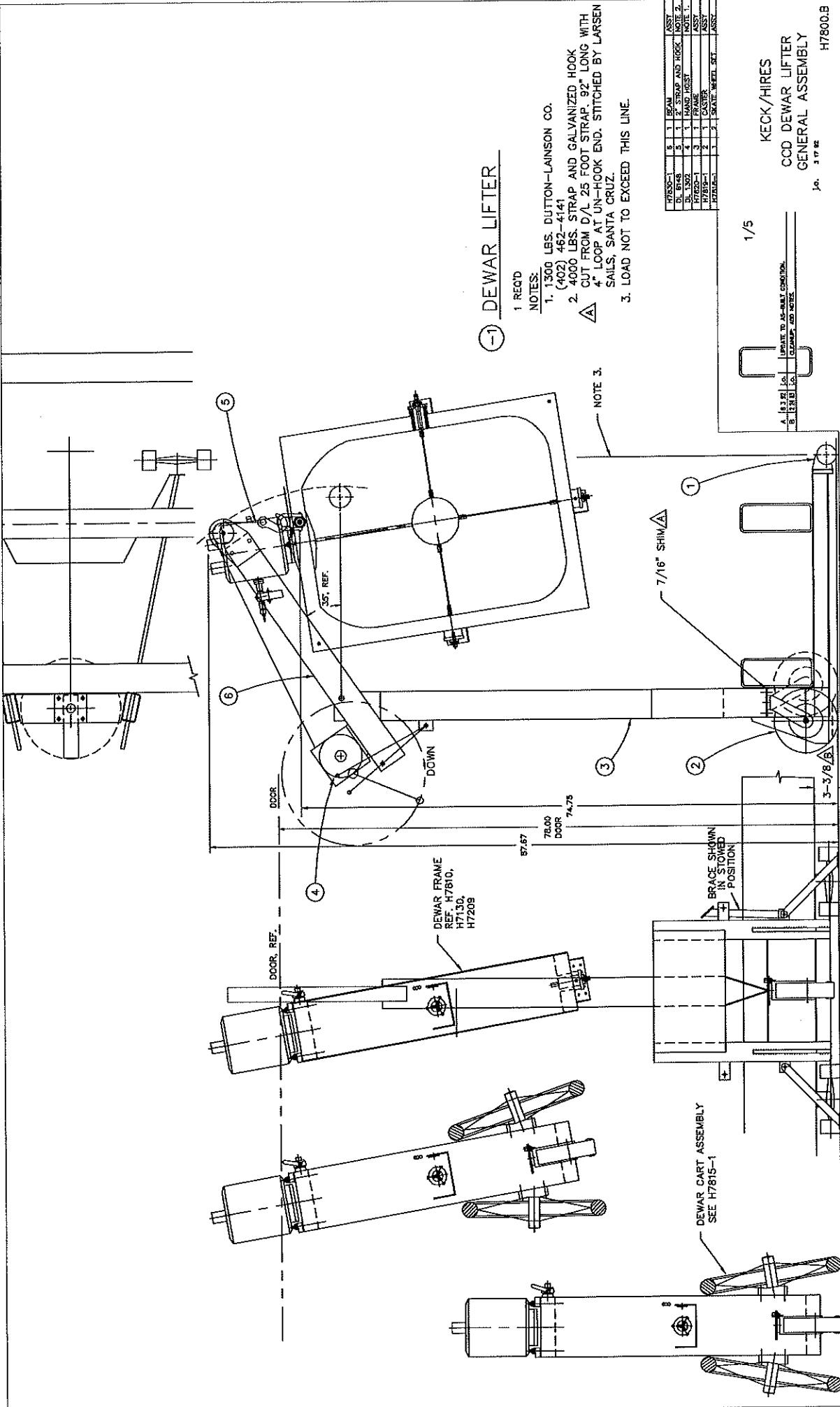
1 REQD
NOTES:

1. 3/32" BOTH TUBES, LINE UP
2. AS SHOWN, BOTH ENDS ARE THE SAME.
3. BRASS TUBING
4. LEXAN SPACERS, PRESS FIT
5. FLANGES ARE FOR REMOVAL
6. USE WITH TAYLOR WHARTON 50 LITER DEWAR SEE H7602 FOR DETAIL

FULL

1	3/32"	IN.	DEWAR LENGTH ESTIMATED
2	3/16"	IN.	ADC SECTION NEW AT THE SIDE

KECK/HIRES
LIQUID NITROGEN FILL
SENSOR ASSEMBLY
Jo. 3 18 82
H7601.B



(1) DEWAR CART ASSEMBLY

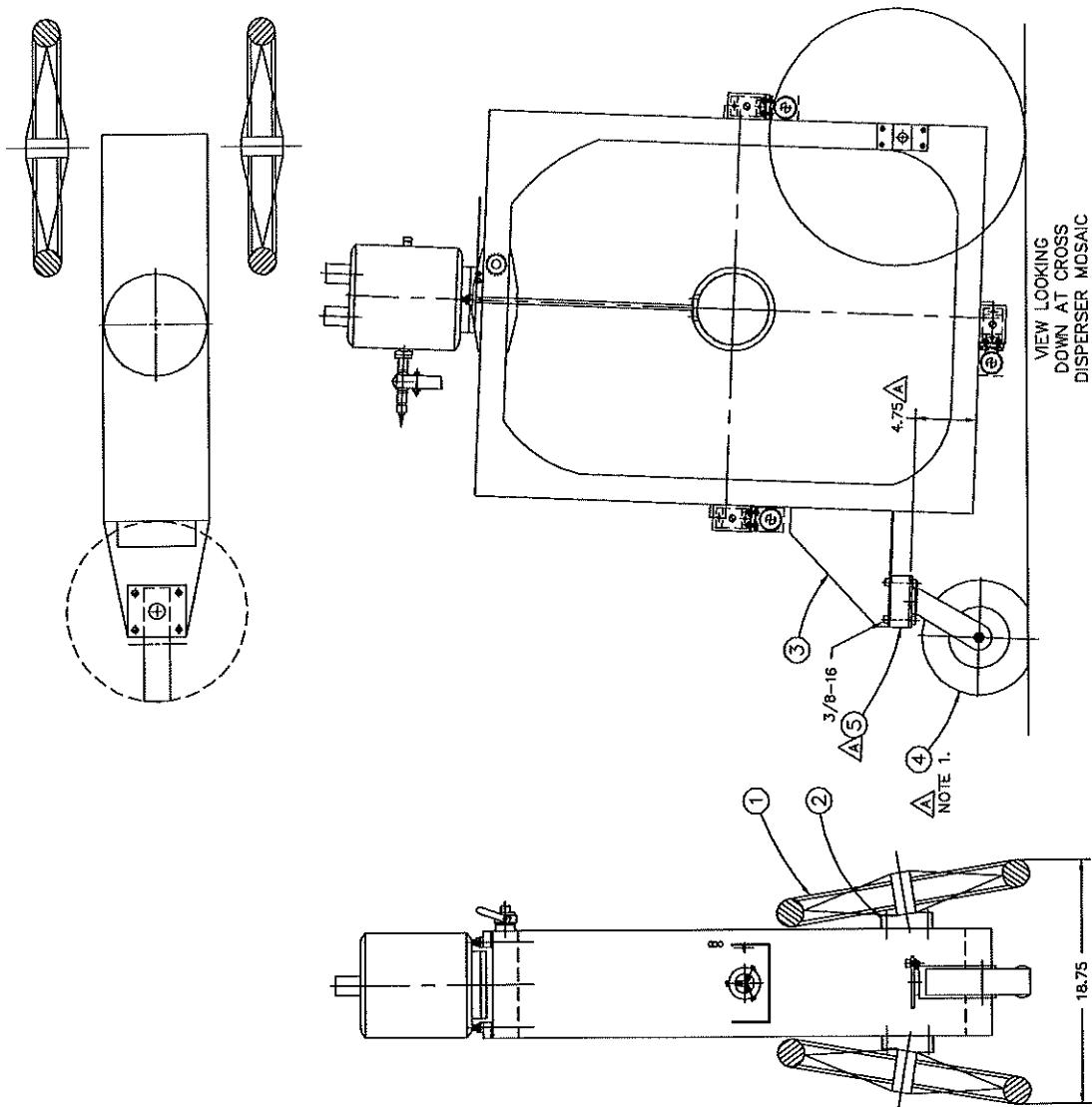
2 REQD **A**

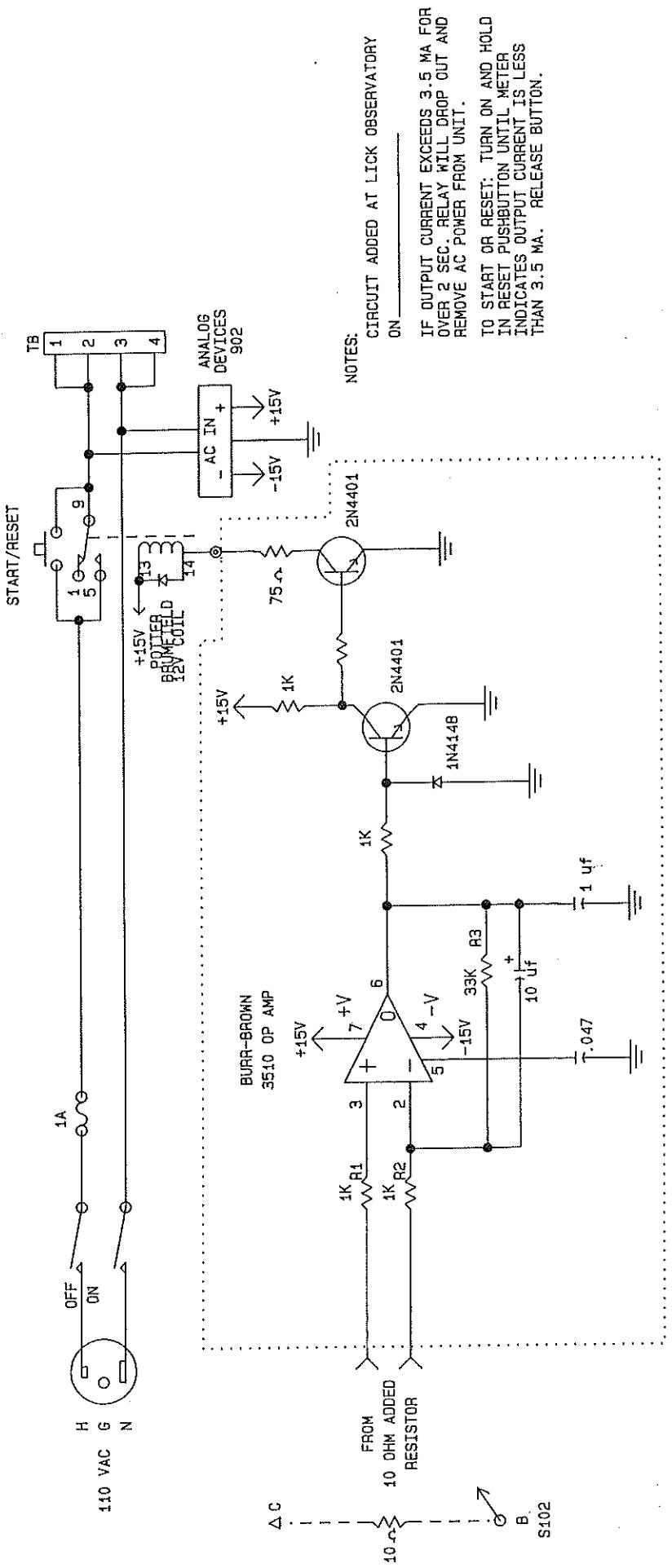
NOTES:
1. UNDER-INFLATE TIRE AS SHOWN
2 PER ASSEMBLY

ITEM	DESCRIPTION	QUANTITY
17037-3	SPACER	1
17039-1	CASTER MOUNT	1
17037-2	CASTER	1
3707-1	PIVOT BLOCK	2
3707-1	PNEUMATIC WHEEL	2

1/4

KECK/HIRES
DEWAR CART
ASSEMBLY
J.O. 3-25-82 H7815.A





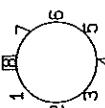
UNIVERSITY OF CALIFORNIA LICK OBSERVATORY

OVERLOAD PROTECTION FOR VARIAN ION PUMP

MARK	DATE	DRAWN BY	CHKD BY	REVISION
1-24-90	CD	ENTERED IN PCAD		

Dwg. No.	
EL-1027-2S	

TOP VIEW
3510 OP AMP

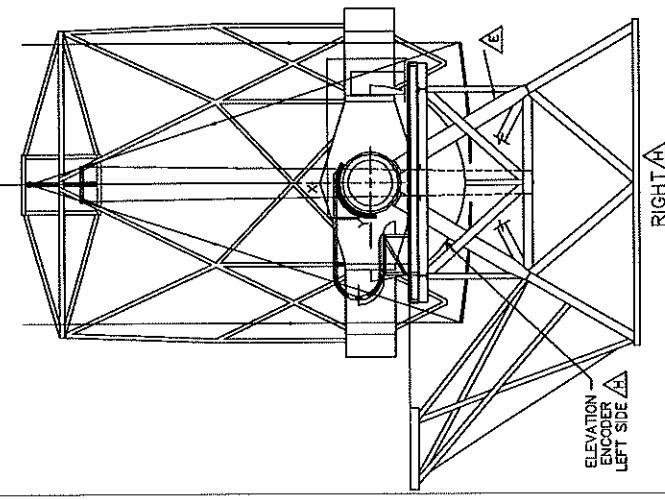
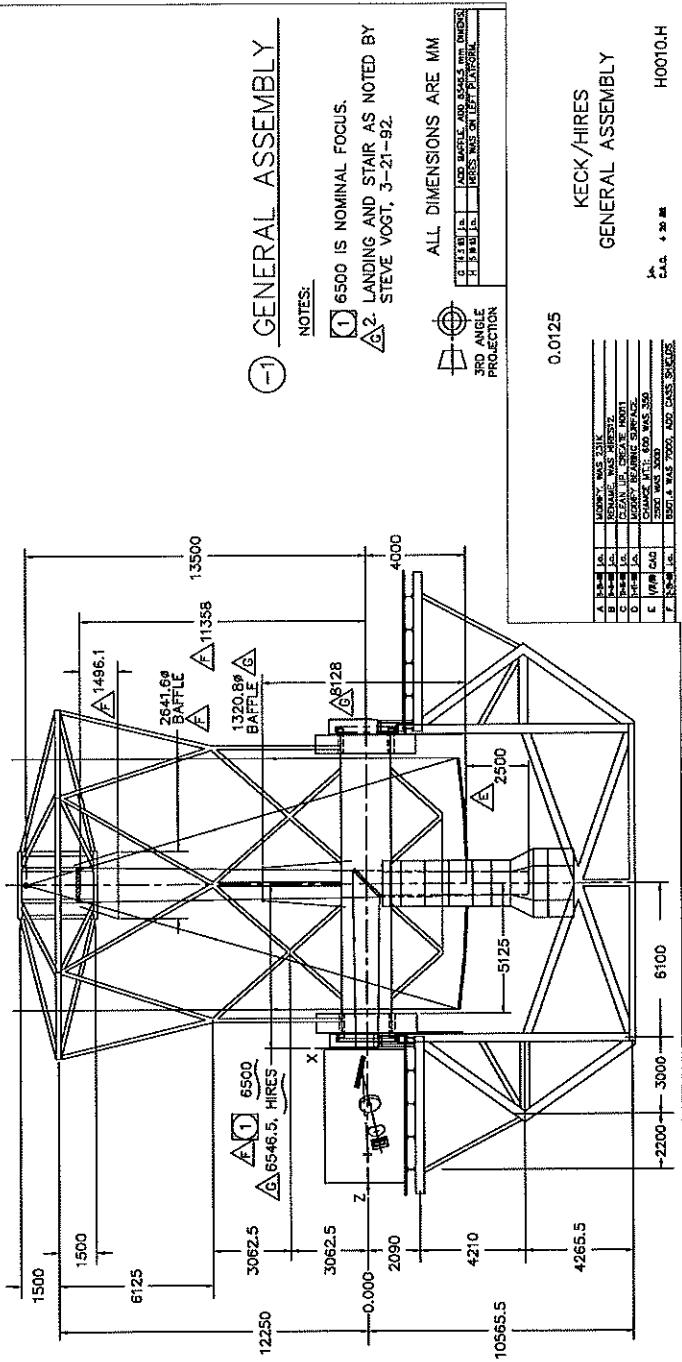
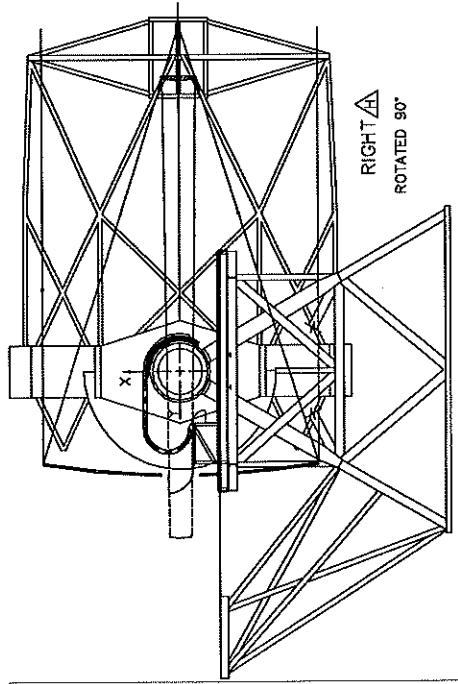
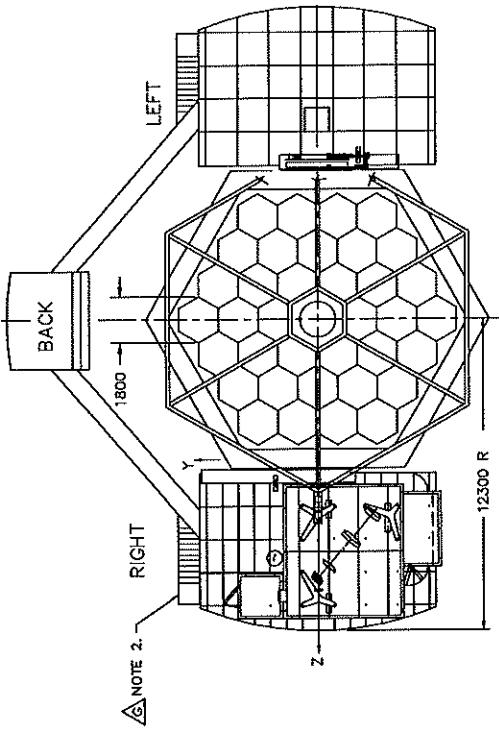


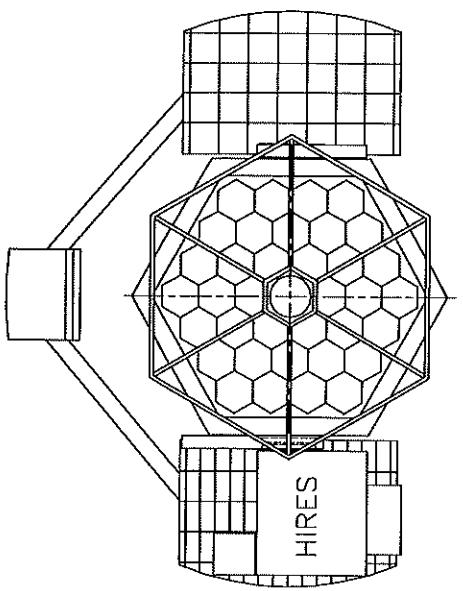
Appendix G List of Drawings

— Housing & Structure

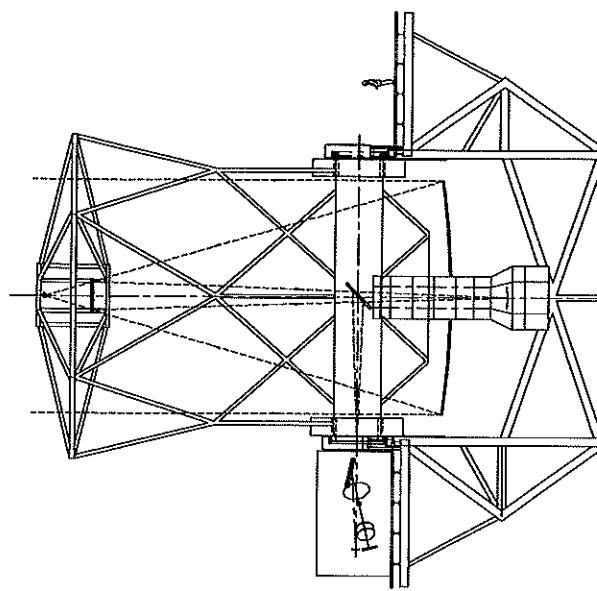
1. H6000 Locating Tree
2. H0010 General Telescope Assembly
3. P0010 General Layout
4. H0012 Elevation Axle Detail
5. H0102 Final Optical Layout
6. H0148 Super-Duper 3 Layout
7. H6100 Main Assembly Drawing
8. H6102 Nasmyth Deck and Leg Detail
9. H6120 Support Foot Detail (Ball-On-Flat)
10. H6170 Optical Bench Assembly (Main Structure)
11. H6174 Support Foot Detail (Ball-In-Groove)
12. H6177 Support Foot Detail (Ball-In-Cone)
13. H6200 Main Support Framework
14. H6220 Interferometer Mounting for Mosaic Testing
15. H6222 Details of Testing
16. H6400 Main Housing Assembly
17. H6410 3-D of Main Housing
18. H6425 Electronics Vault Assembly
19. H6429 Hatch Assembly
20. H6460 Light Trap Assembly
21. H6461 Light Trap Details
22. H6470 Housing and Freezer During Testing at UCSC

H6000	LOCATING TREE (THIS DRAWING)	H6400	HOUSING ASSEMBLY
H6050	JUNCTION BOX/HATCH DETAILS	H6401	LIGHTING PLAN IN BIG ROOM
H6051	" DETAILS	H6402	THERMAL MODEL (MATHEMATICAL)
H6052	" DETAILS	H6403	SEE HOXYZ STICK FILE
H6100	MAIN ASSEMBLY DRAWING	H6404	"
H6101	NASMYTH INTERFACE	H6405	"
H6102	NASMYTH DECK BOLT PATTERN	H6406	"
H6103	FLOOR DETAIL - CUTOUTS	H6407	"
H6104	PATTERNS	H6408	"
H6105	PATTERNS	H6409	3D HOUSING DRAWING
H6106	PATTERNS	H6410	TRANSITION CONE ASSEMBLY
H6107	FEET SPACERS	H6411	" DETAILS
H6110	SUPPORT FOOT - BALL/GROOVE	H6412	" DETAILS
H6111	PATTERNS	H6413	" NOSE: CYLINDER ASSEMBLY
H6112	PATTERNS	H6414	" HOUSING RACEWAY ASSEMBLY
H6113	WEB DETAIL	H6415	" DETAILS
H6114	SUPPORT FOOT DETAIL	H6416	" DETAILS
H6115	SUPPORT FOOT - BALL/CONE	H6417	" DETAILS
H6116	PATTERNS	H6420	OUTER CONE DETAIL
H6117	PATTERNS	H6421	" NOSE DETAILS
H6118	WEB DETAIL	H6422	" DETAILS
H6119	SUPPORT FOOT DETAIL	H6423	" DETAILS
H6120	SUPPORT FOOT - BALL/PLANE	H6424	" DETAILS
H6121	PATTERNS	H6425	ELECTRONICS VAULT ASSEMBLY
H6122	SUPPORT FOOT DETAIL	H6427	HATCH ASSEMBLY
H6123	SUPPORT FOOT DETAIL	H6428	HATCH: PNEUMATIC DOOR DETAIL
H6140	MAT PATTERNS	H6429	HATCH: PNEUM. DOOR ASSEMBLY
H6141	" DETAILS	H6430	BALLY BUILDING QUOTATION DRAWING
H6142	" DETAILS	H6431	NOSE: BRACES
H6143	" DETAILS	H6432	NOSE: INNER PANEL
H6144	" DETAILS	H6433	THERMAL BAFFLES
H6145	" DETAILS	H6434	THERMAL BAFFLES
H6146	" DETAILS	H6440	INSIDE FRONT WALL DETAIL
H6147	" DETAILS	H6450	THERMAL TESTING
H6148	" DETAILS	H6460	LIGHT TRAP ROOM
H6151	TURNTABLE MOUNT ASSEMBLY	H6461	" DETAILS
H6160	" DETAILS	H6200	OPTICAL BENCH DETAILS
H6170	OPTICAL BENCH ASSEMBLY	H6201	" DETAILS
H6172	BALL/FLAT ASSEMBLY	H6202	" DETAILS
H6173	" DETAILS	H6203	" DETAILS
H6174	CROSSED ROLLER ASSEMBLY	H6204	" DETAILS
H6175	OPTICAL BENCH DETAILS	H6205	" DETAILS
H6177	BALL/CONE ASSEMBLY	H6206	" DETAILS
H6179	CROSSED ROLLER DETAILS	H6207	" DETAILS
H6180	OPTICAL BENCH DETAILS	H6220	ZYGO INTERFEROMETER
H6181	" DETAILS	H6221	MOUNTING FOR
H6182	CROSSED ROLLER DETAIL	H6222	SEE ALSO H4518.9 MOSAIC TESTING
H6185	" DETAILS	H6223	{ TEST COLLIMATOR
			JUNCTION PANEL
			H6491
			H6492
			H6493
			H6494
			H6495
			H6496
			KECK/HIRES
			STRUCTURE & HOUSING
			DRAWING INDEX
			Job. 8301 H6000.A

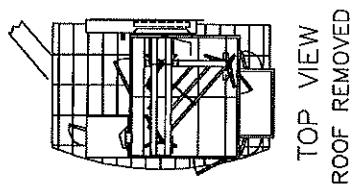




TOP VIEW

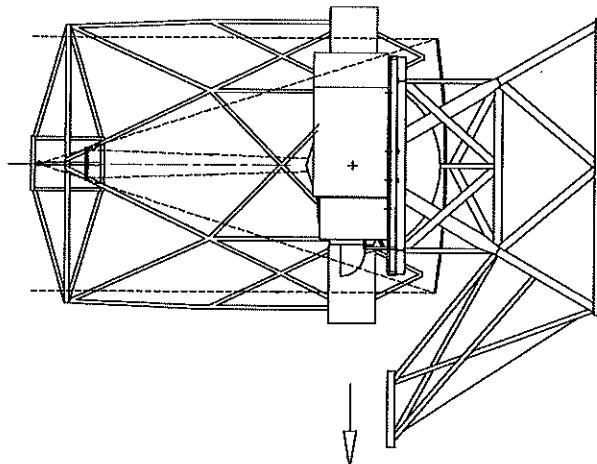


FRONT VIEW
P010.A
JG 3 30 93

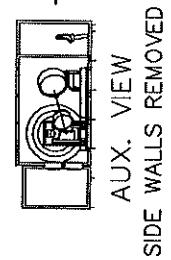


TOP VIEW
ROOF REMOVED

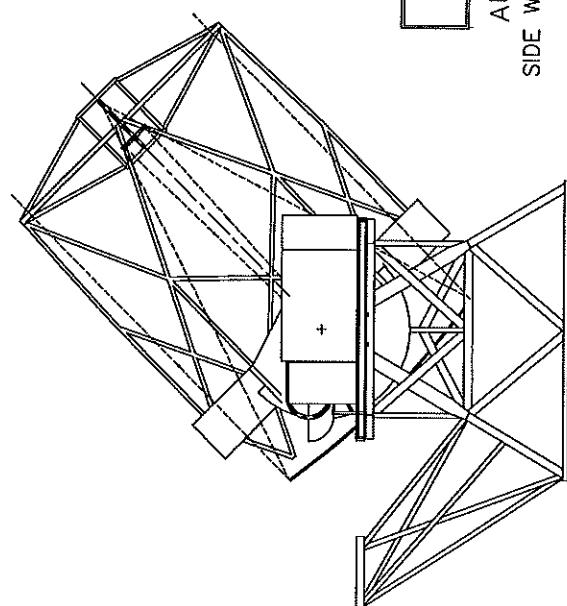
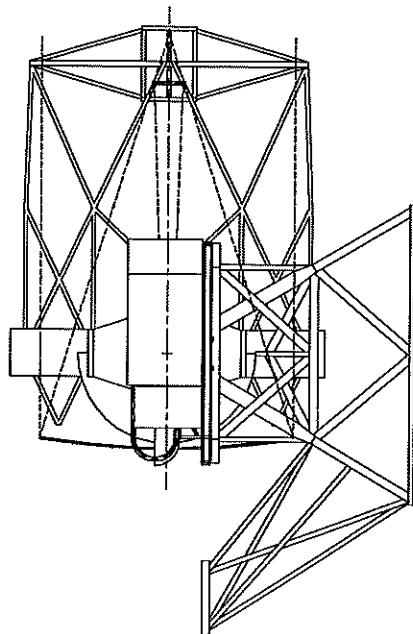
HIGH RESOLUTION SPECTROGRAPH
MOUNTED ON THE KECK TELESCOPE



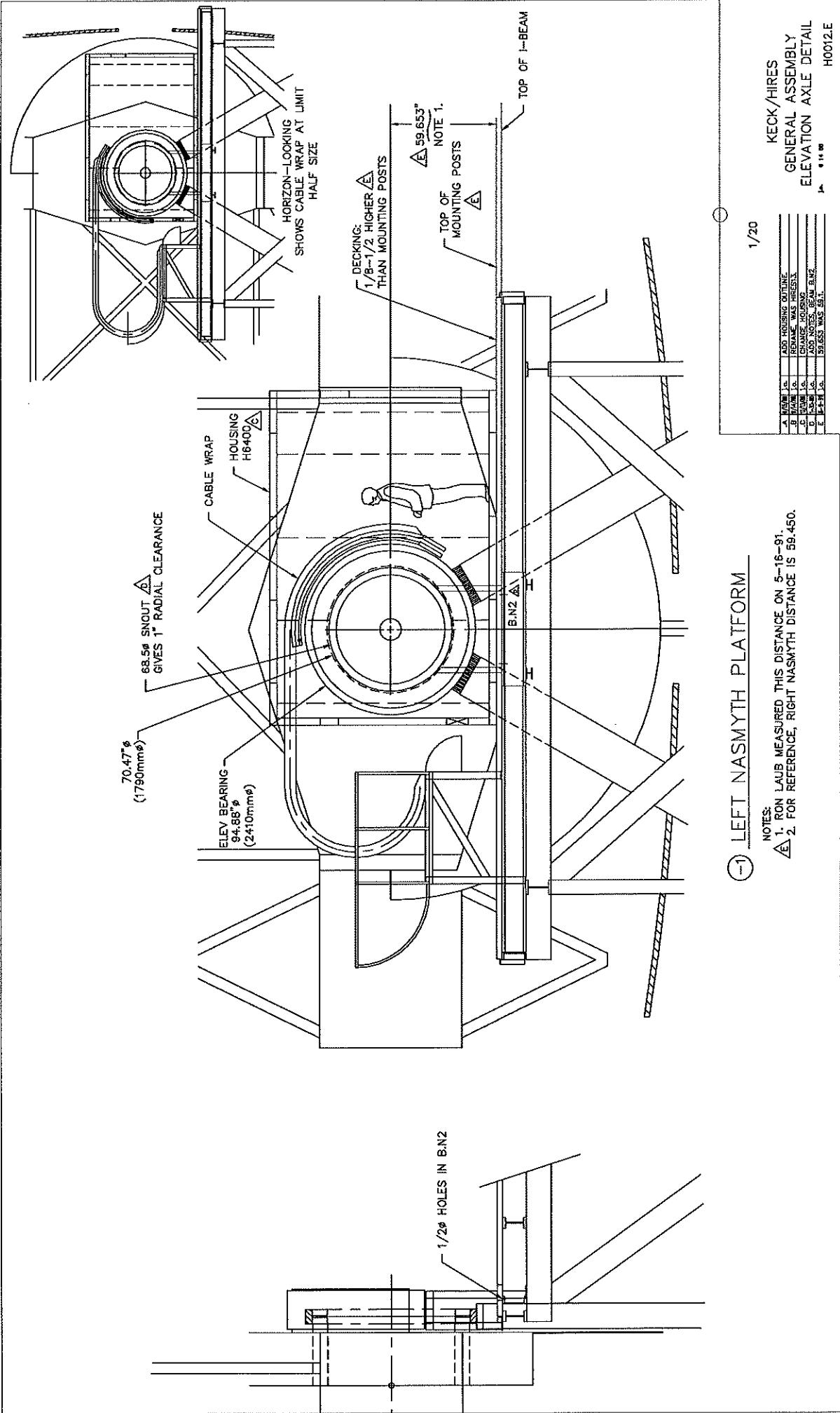
SIDE VIEW

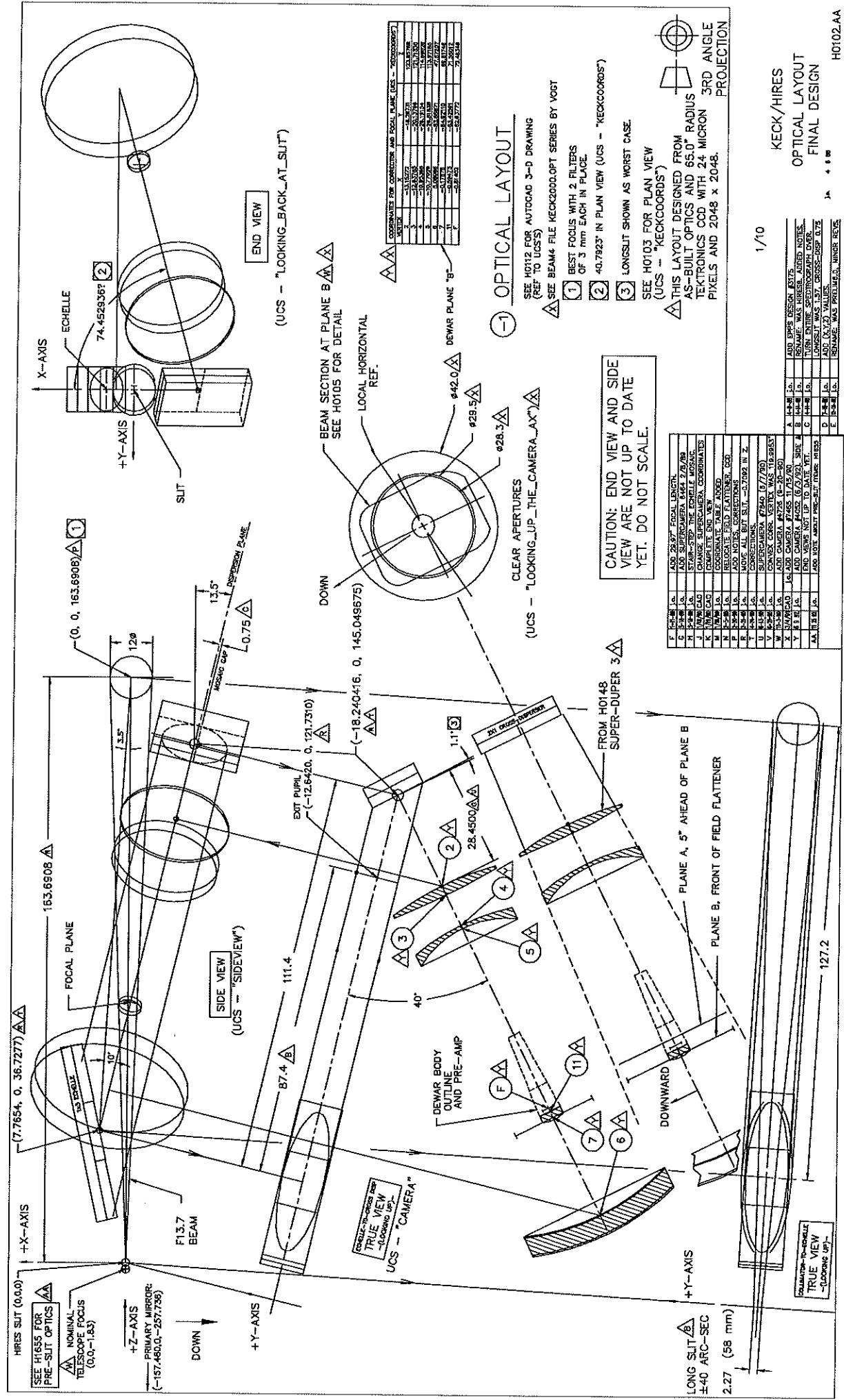


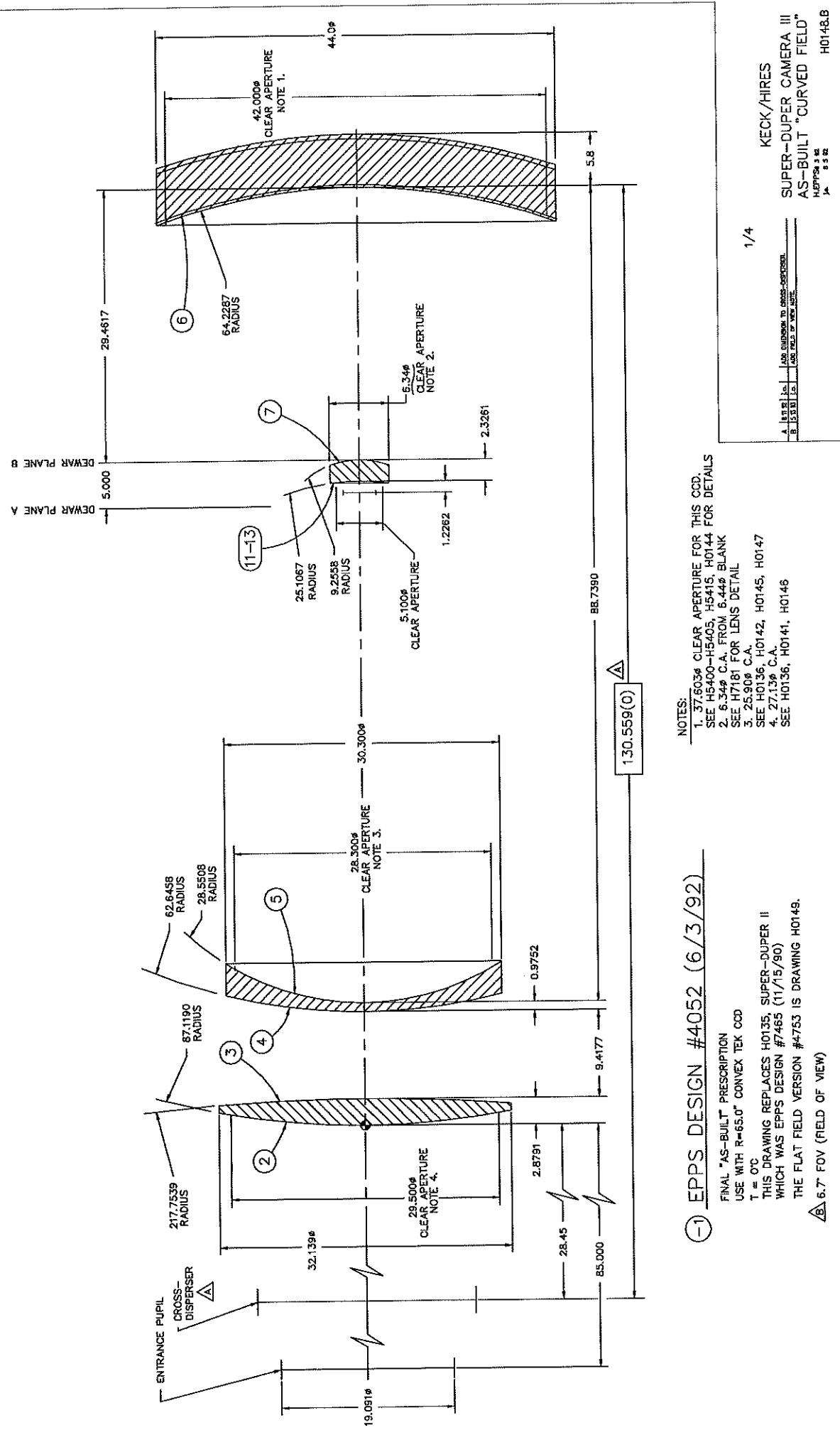
AUX. VIEW
SIDE WALLS REMOVED

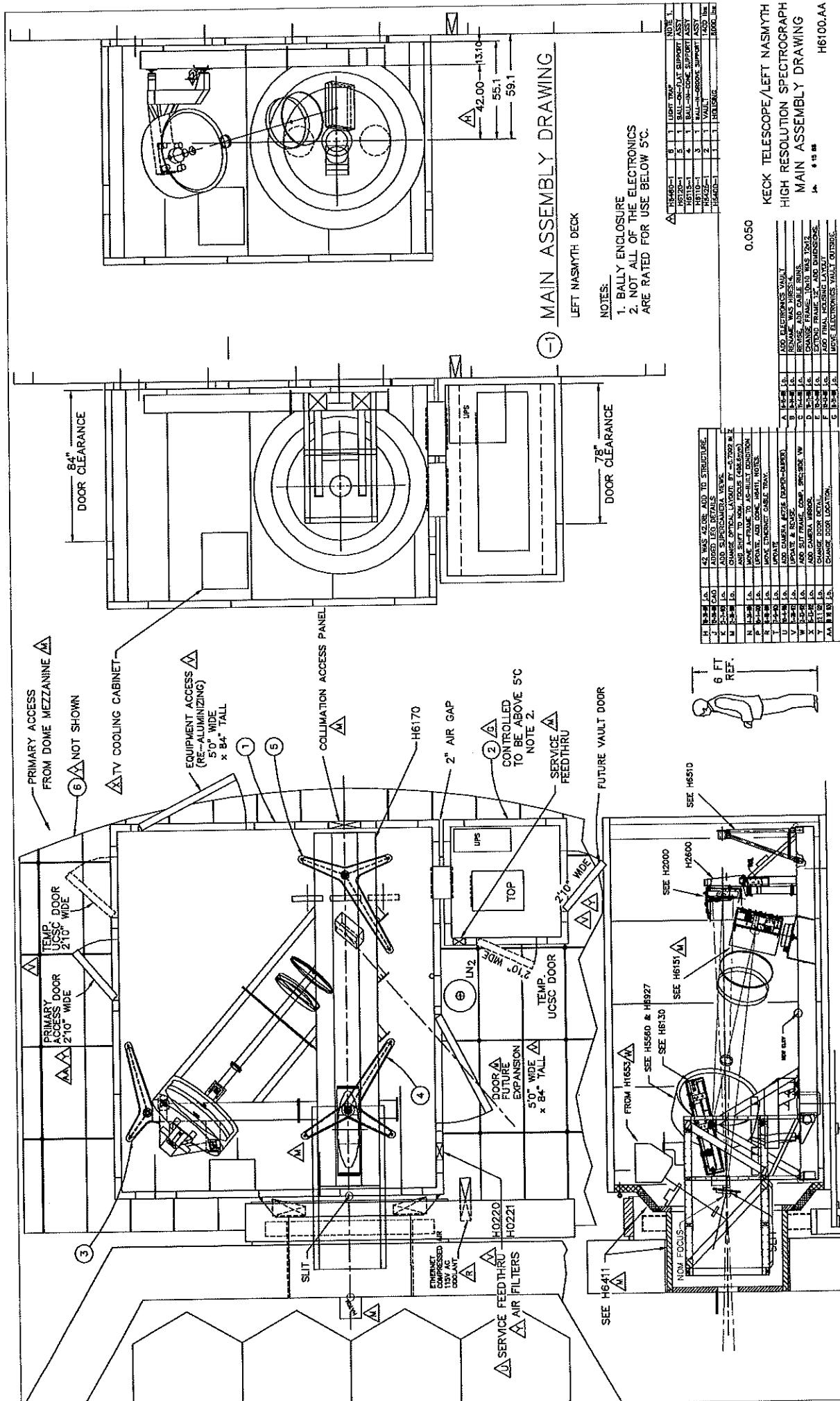


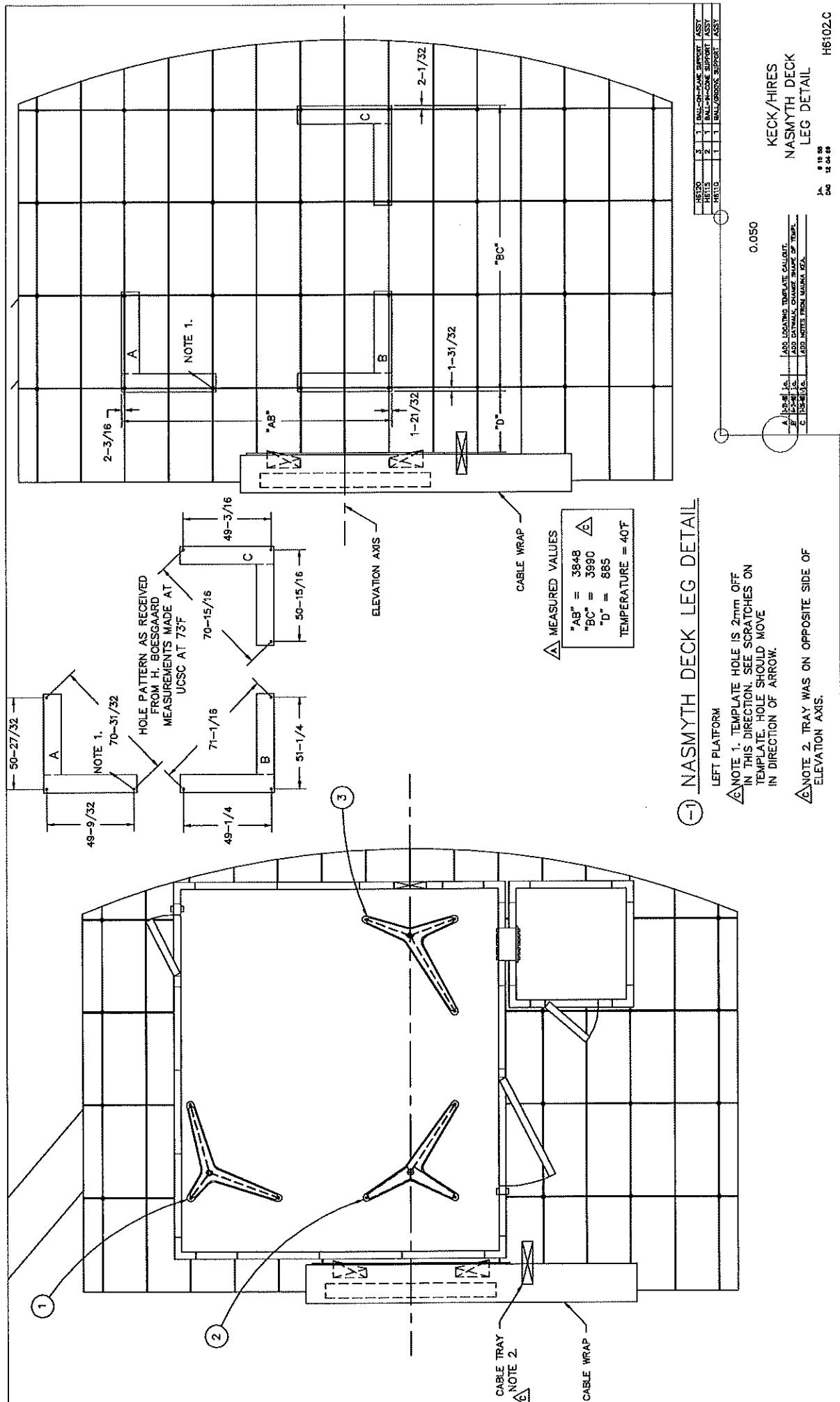
SIDE VIEWS WITH ROTATION

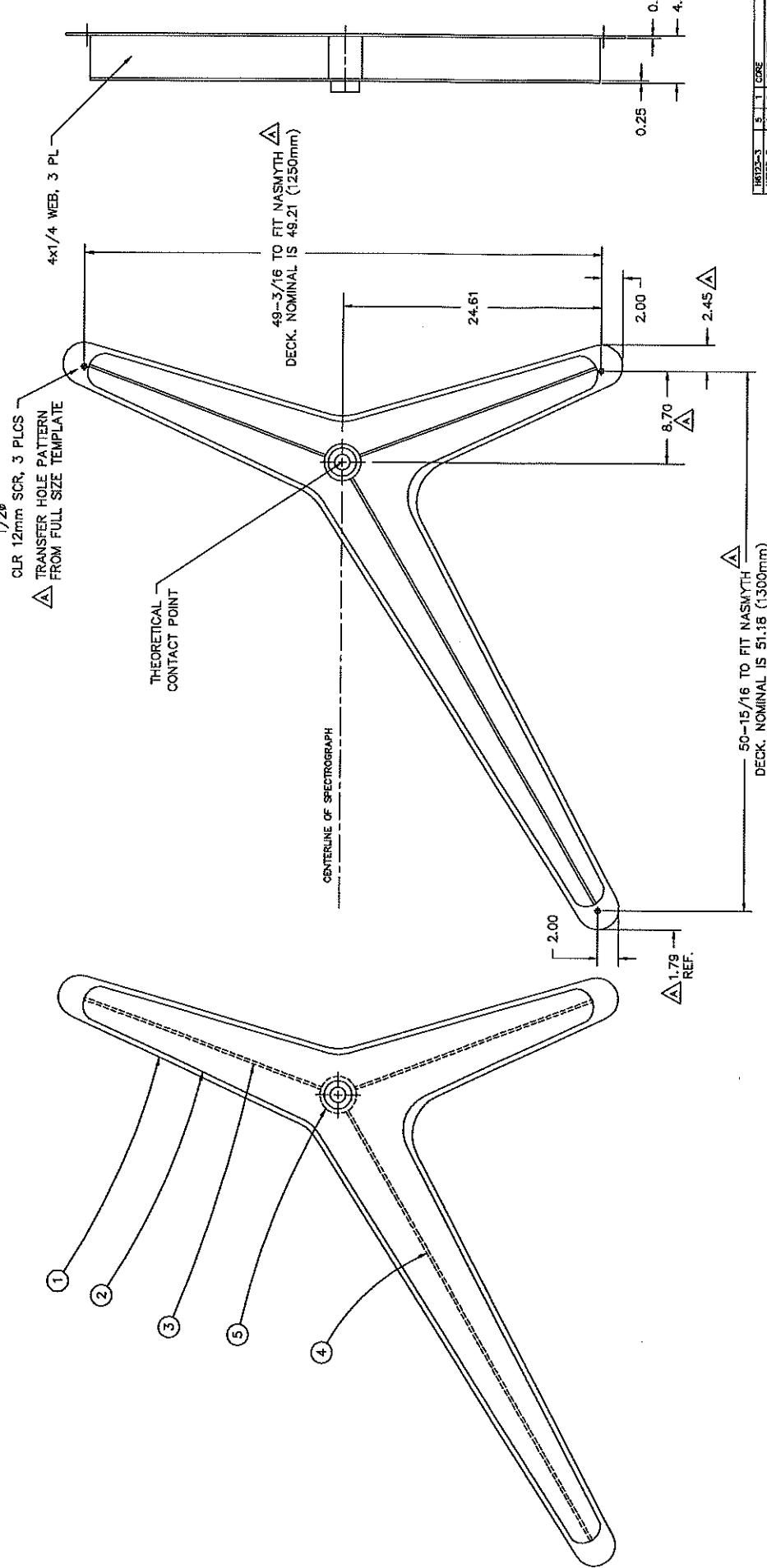










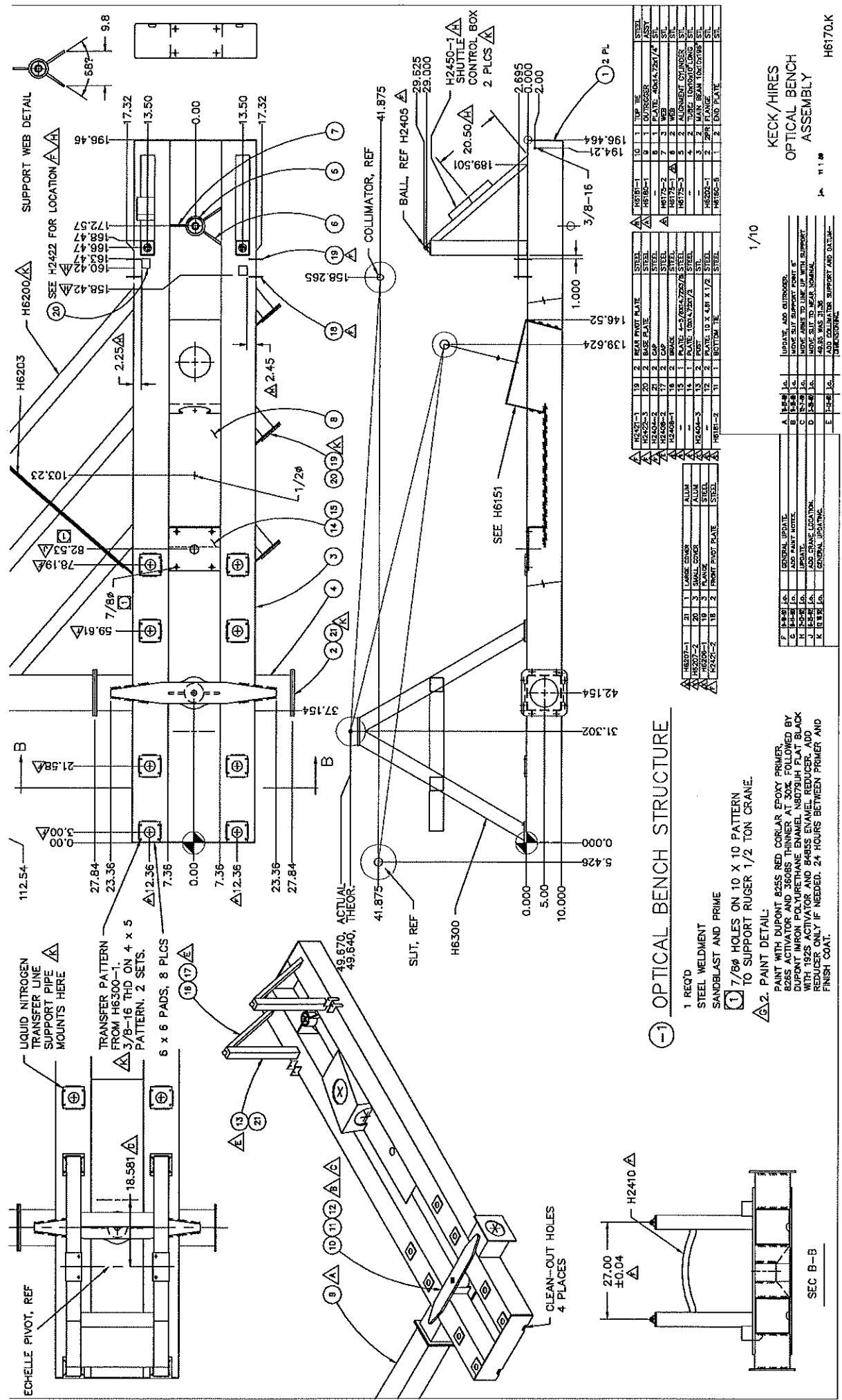


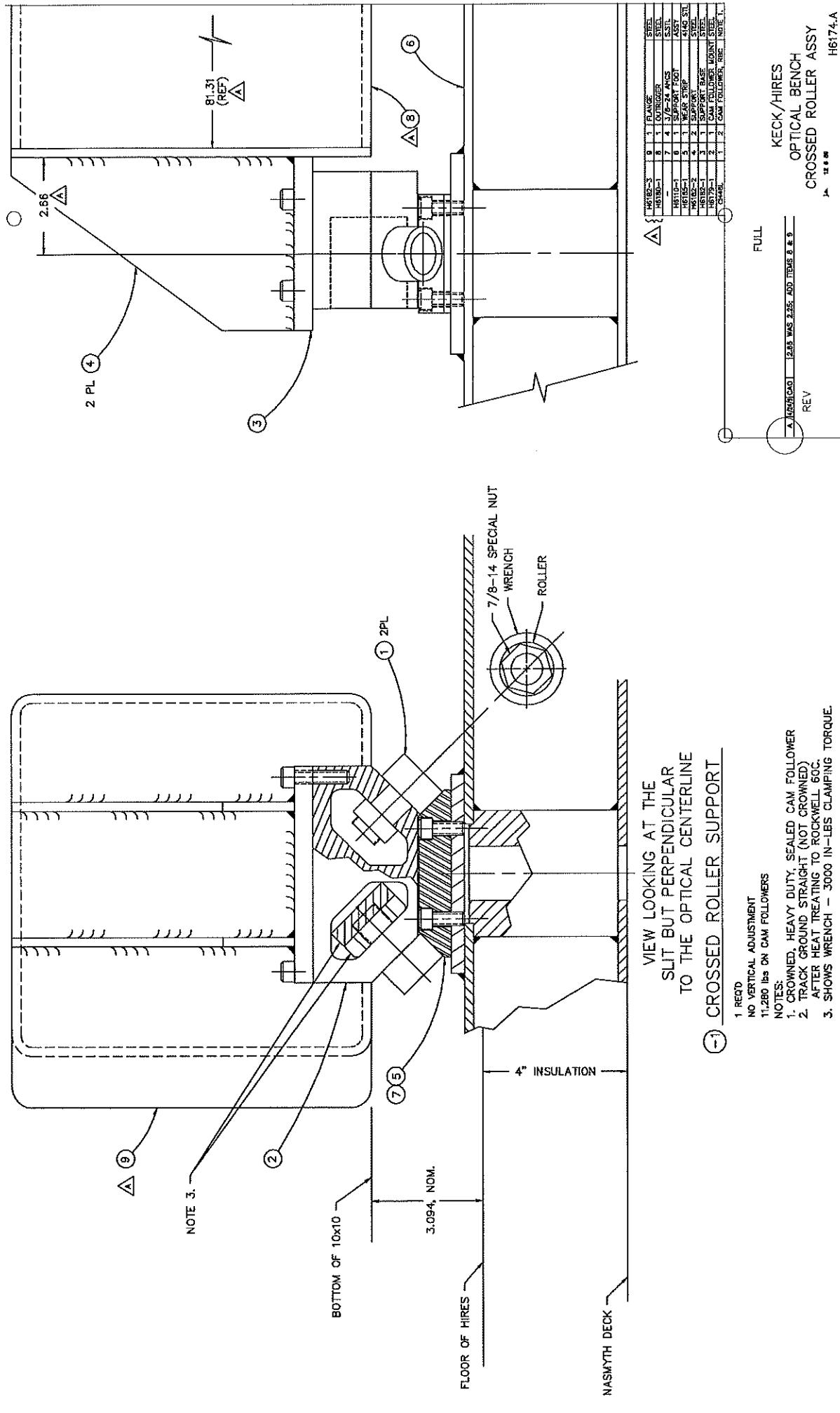
① BALL-ON-FLAT SUPPORT ASSEMBLY

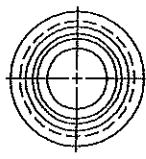
1 RECORD
STEEL WELMENT

KECK/HIRES
SUPPORT FOOT
BALL-ON-PLANE
Rev. n 36 40
H6120.A

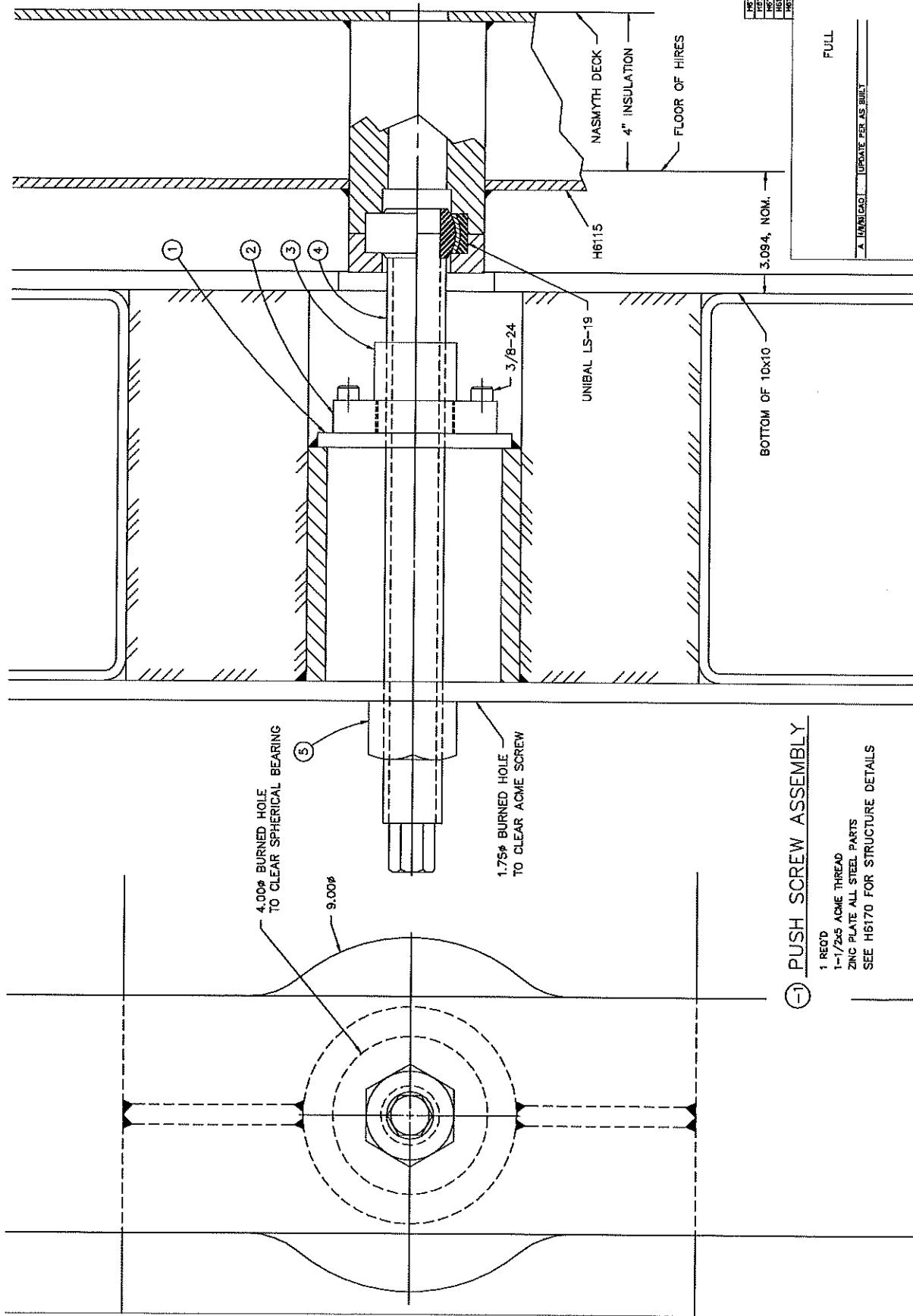
1/4

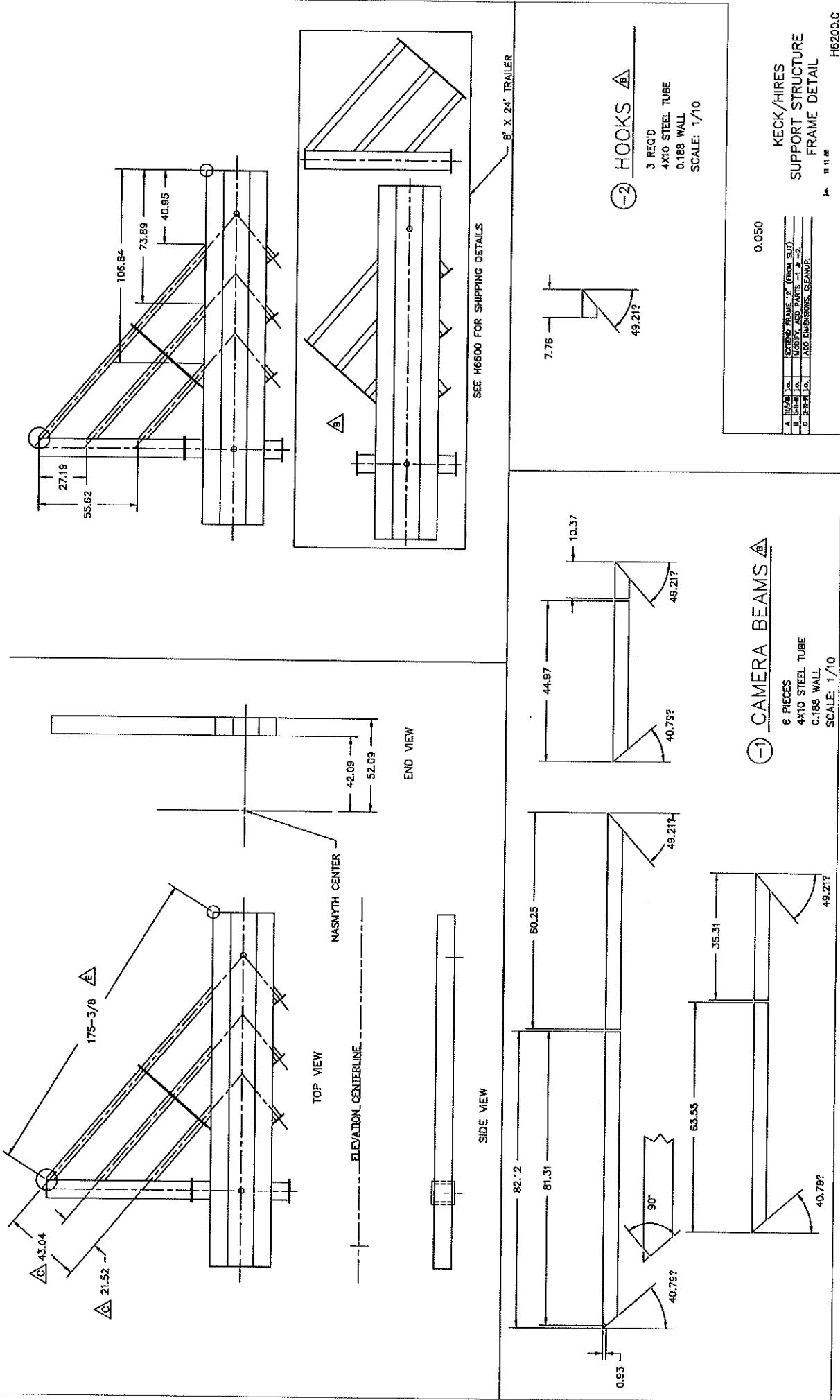


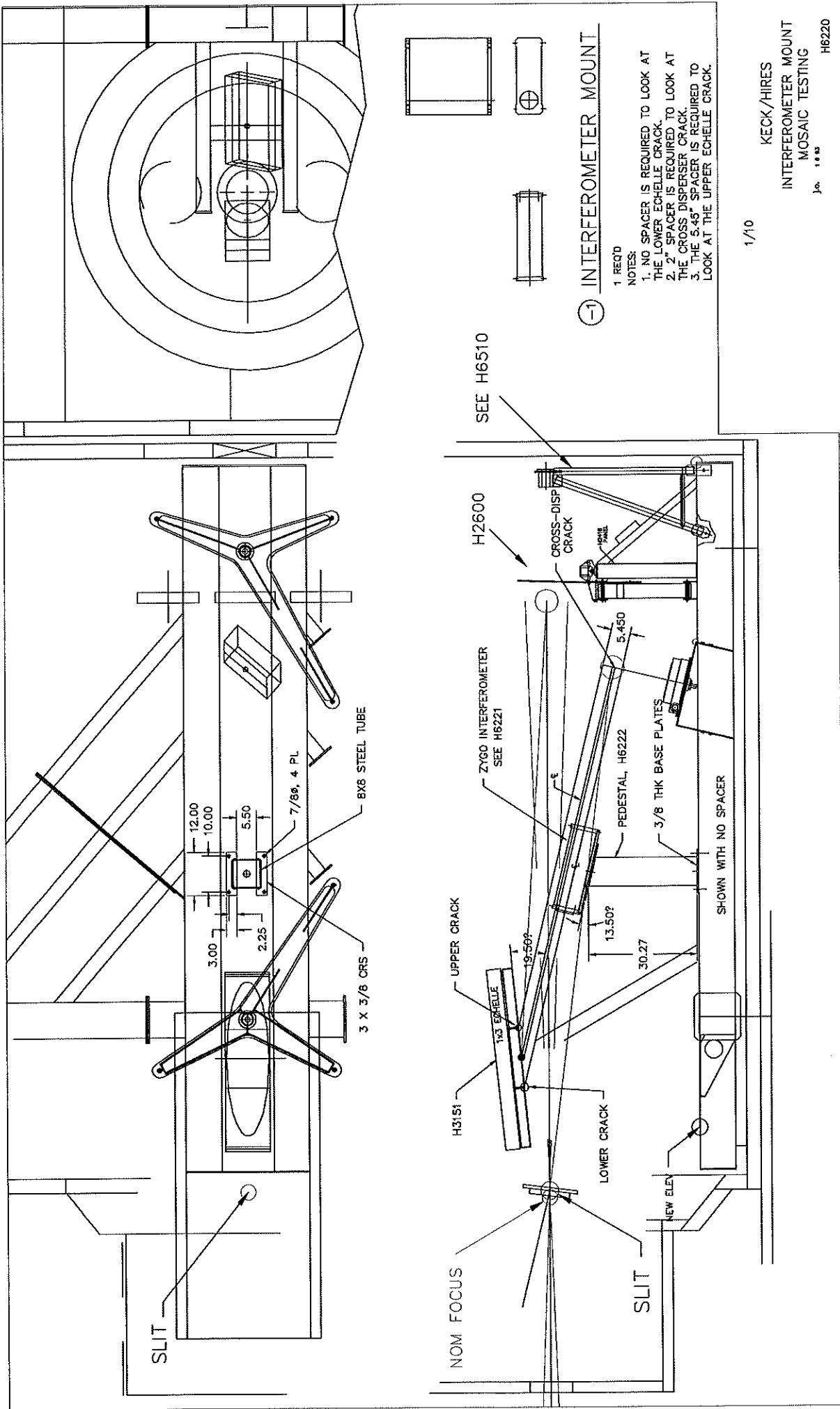


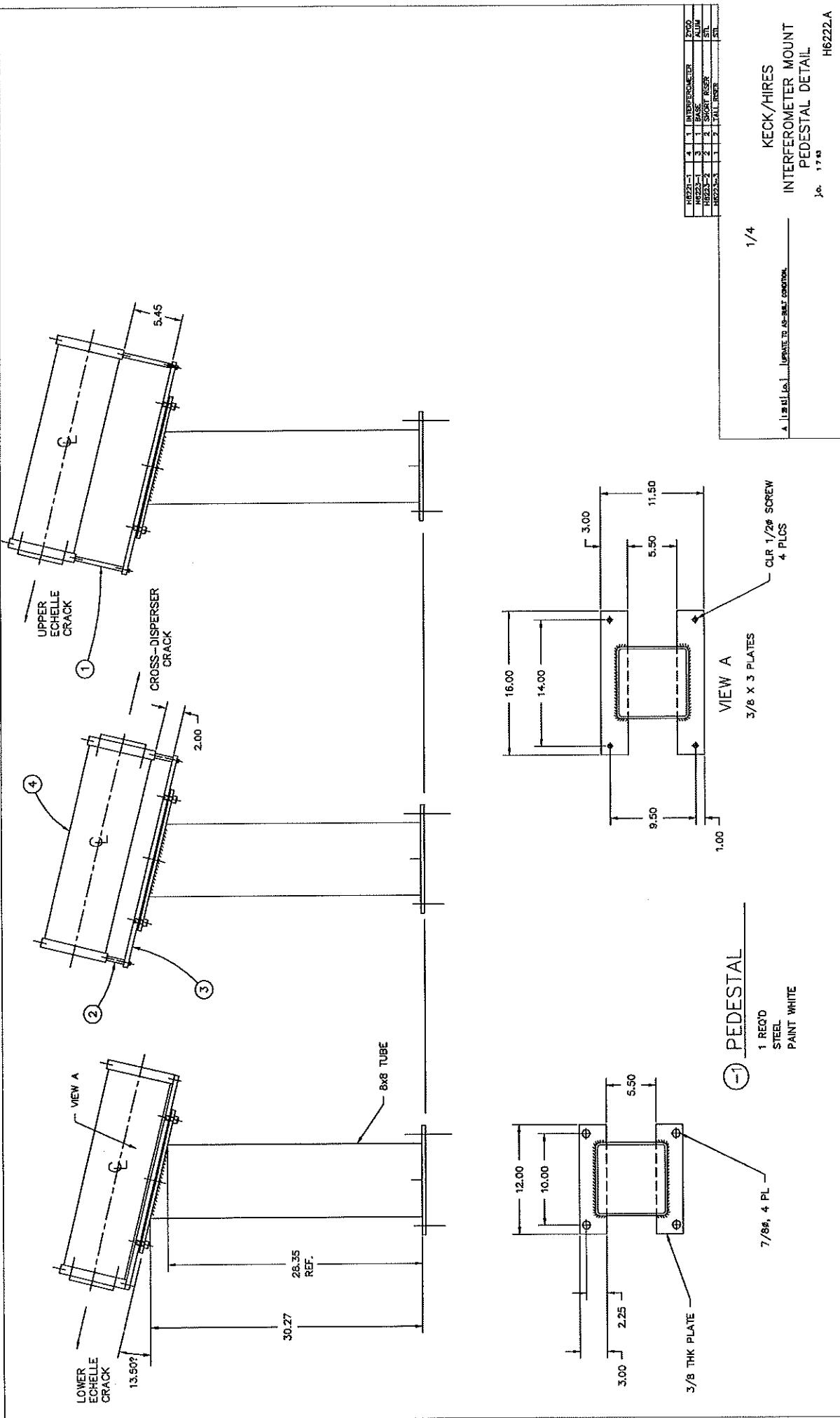


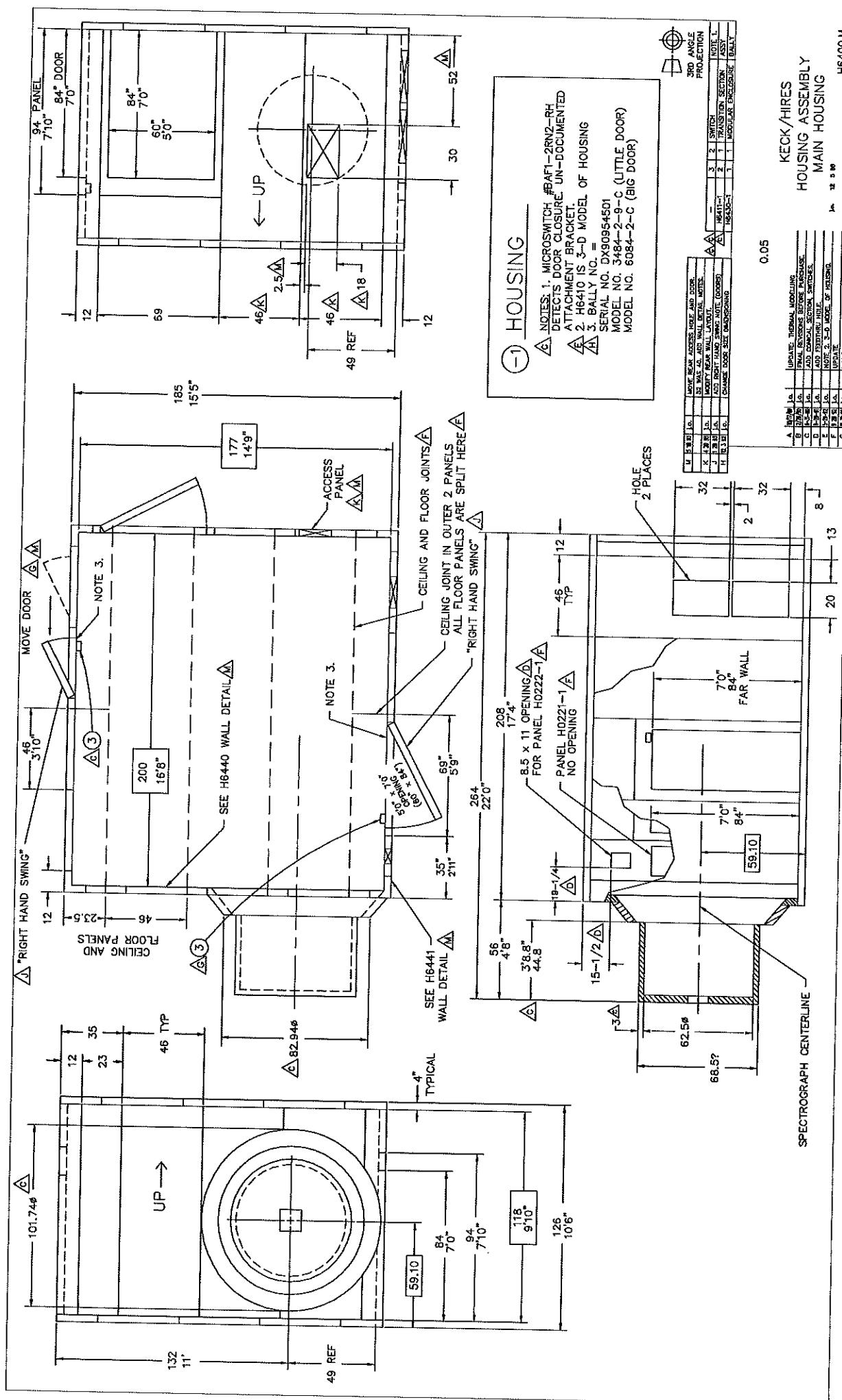
UNIBAL LS-19
HEIM CO., 63,000 LBS
1.25" WIDE, 2.625" OD
1.1875" ID
BRONZE INSERT
26 BALL

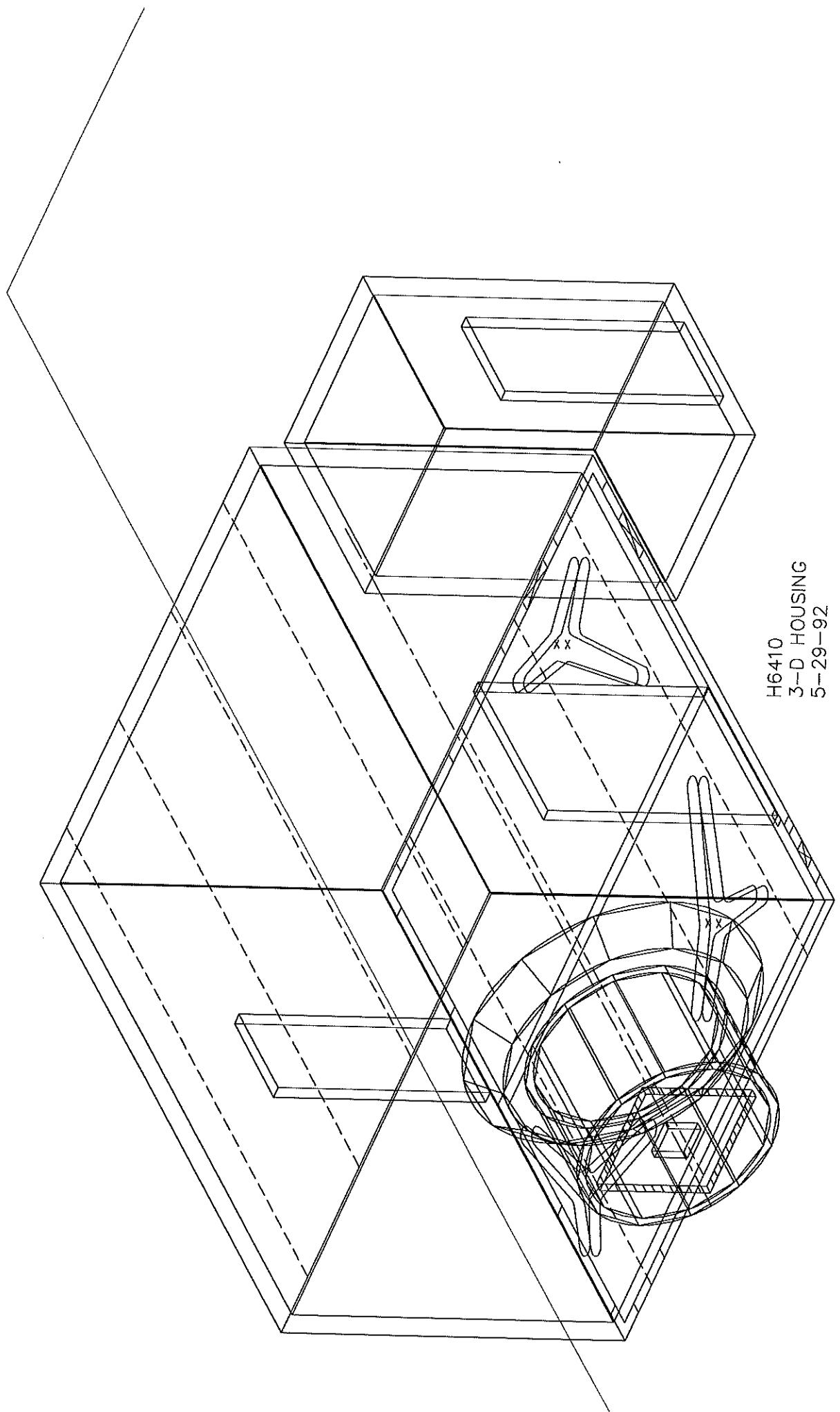




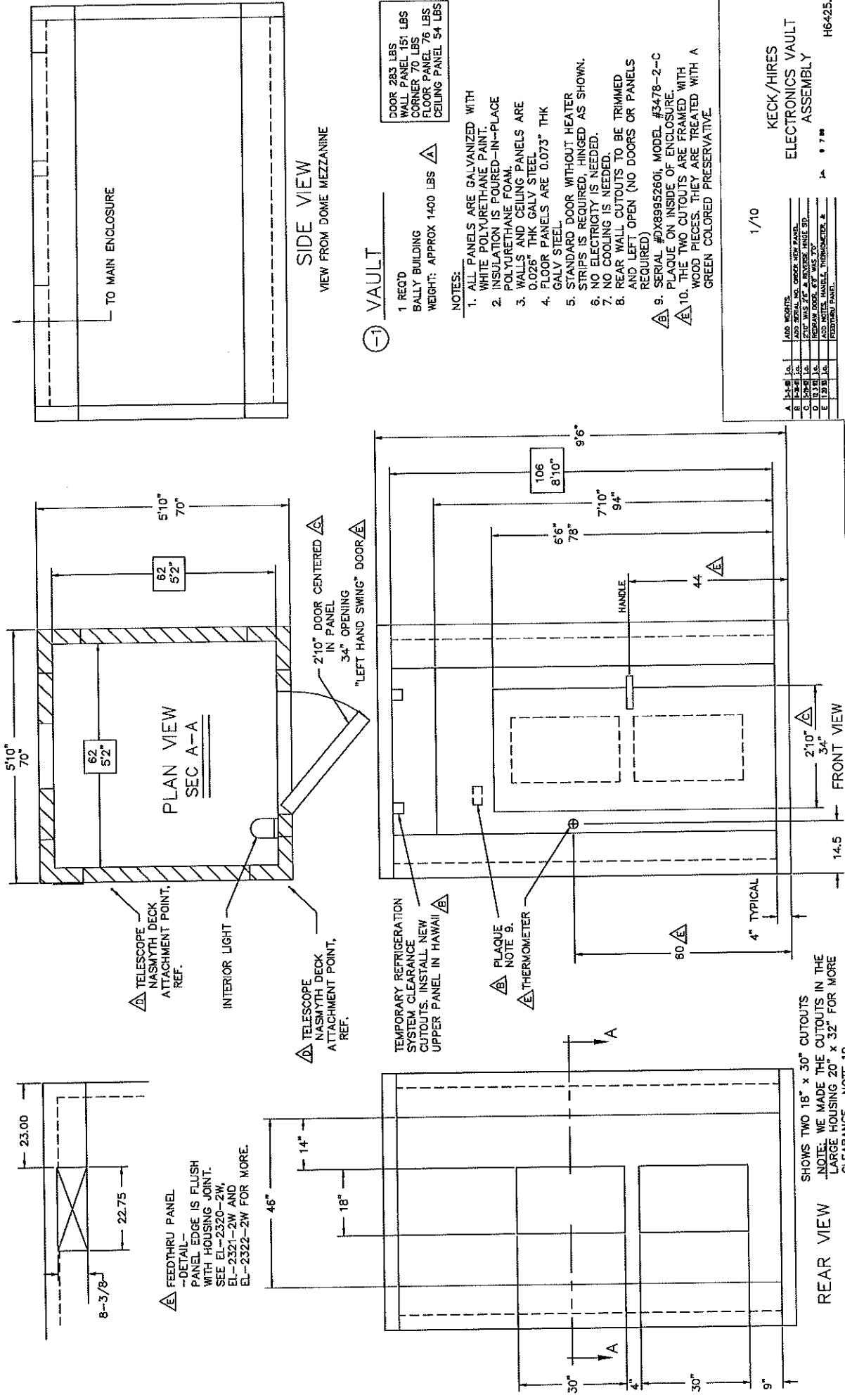


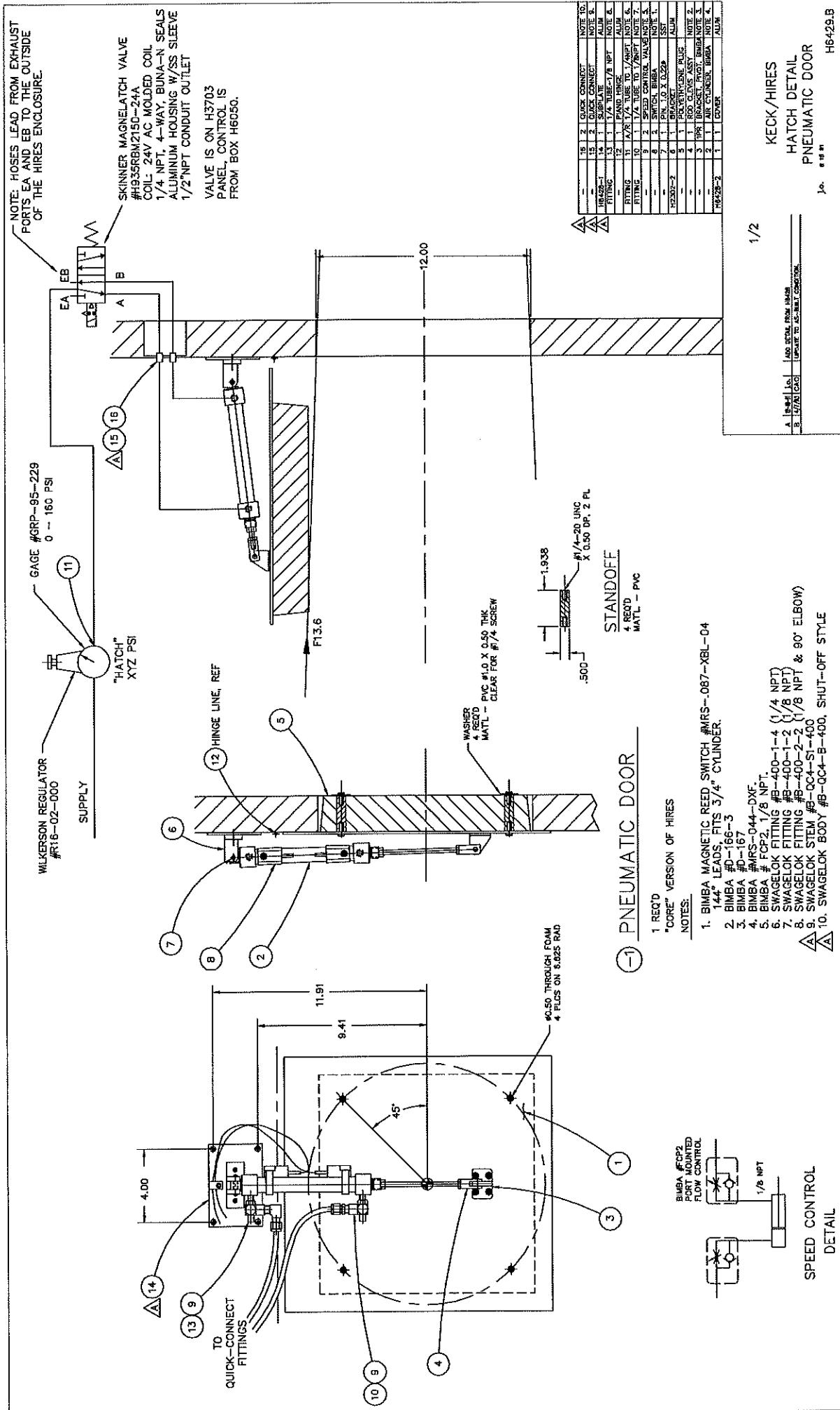




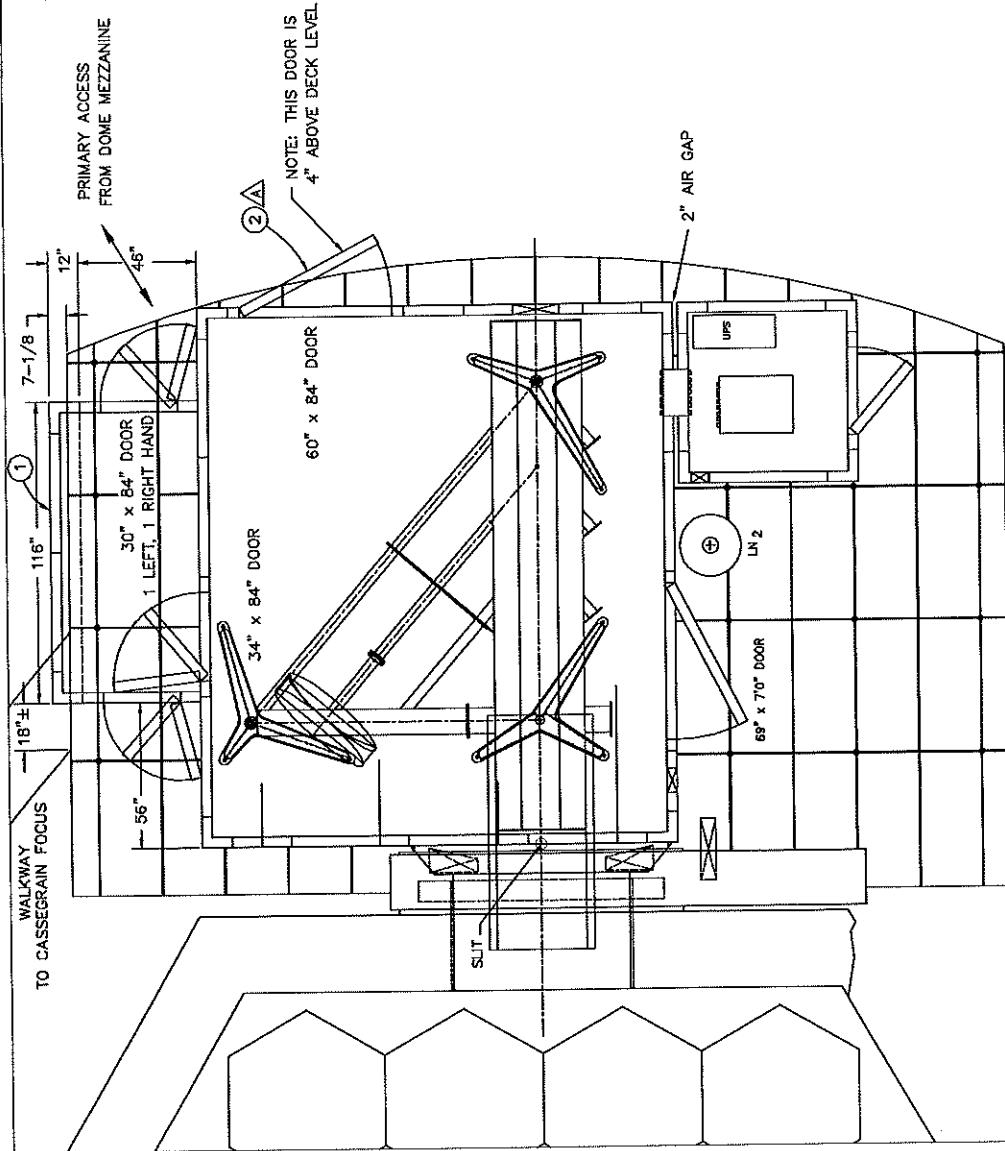


H6410
3-D HOUSING
5-29-92





PENDING
CHANGE.
RIGHT
NASMYTH
PLATFORM.



TOP VIEW 1/20

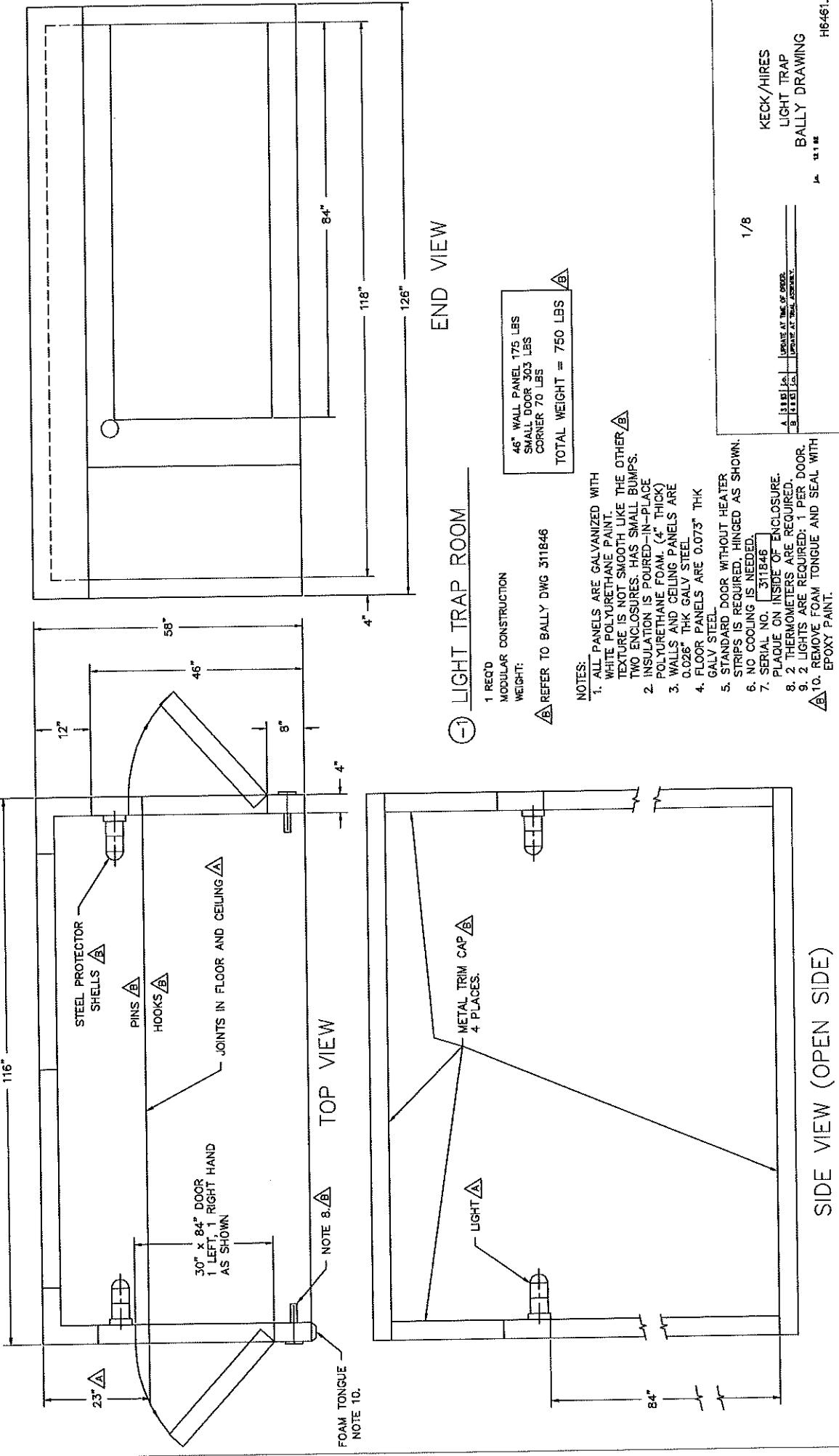
(-1) LIGHT TRAP ASSEMBLED

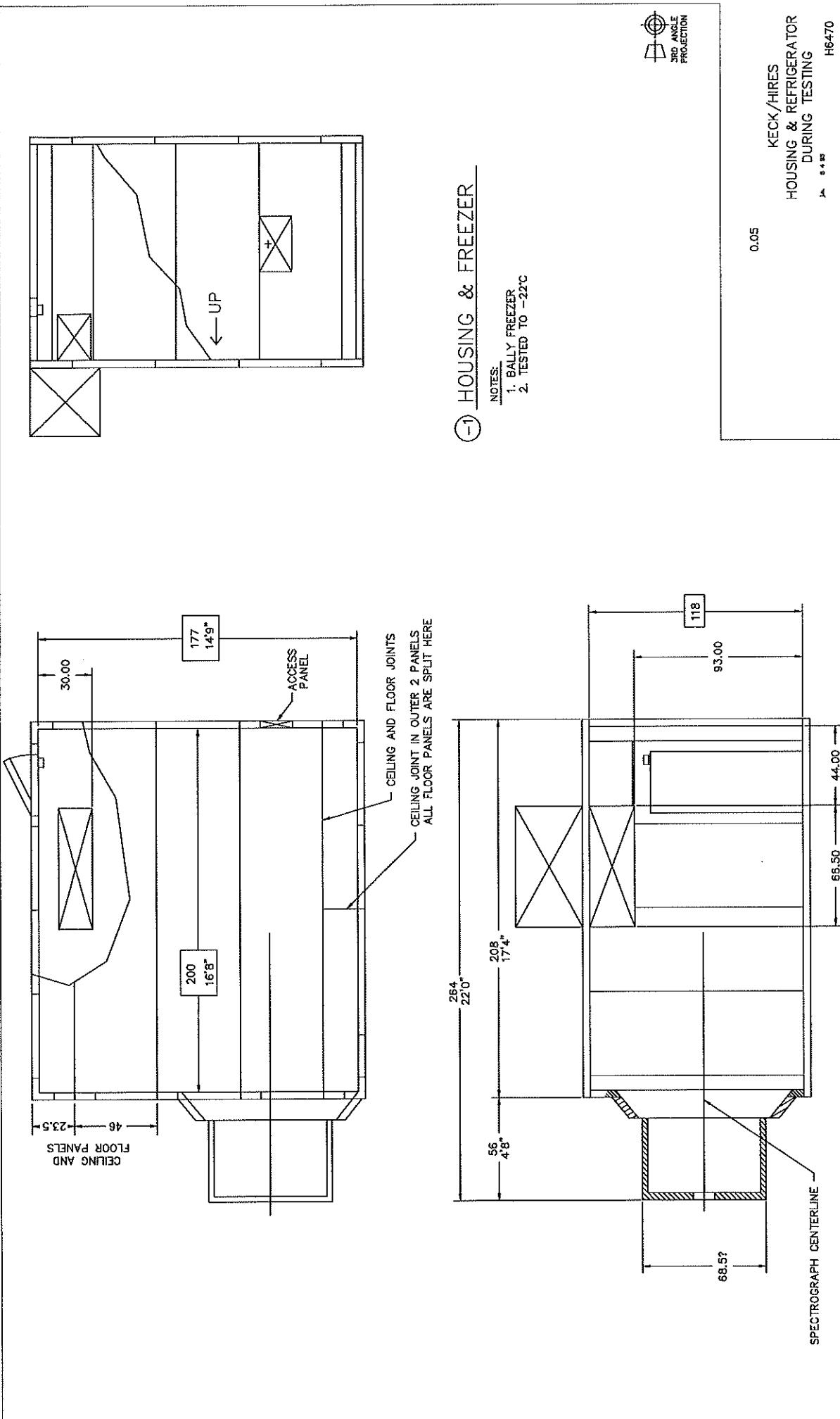
1 REQ'D
REFERS TO H6400 FOR HOUSING DETAIL
NOTES:
1. BALLY ENCLOSURE
2. SEE H6100 FOR ASSEMBLY ON
TELESCOPE NASMYTH PLATFORM

△ H6400-1 2 1 BIG DOOR
△ H6400-2 1 1 DOME TRAP

KECK/HIRES
HOUSING AND LIGHT TRAP
ASSEMBLY
A. ISSUE 1. 100 PART 2. RE-ENTL. MASS. START. _____
1a. 12 1/2

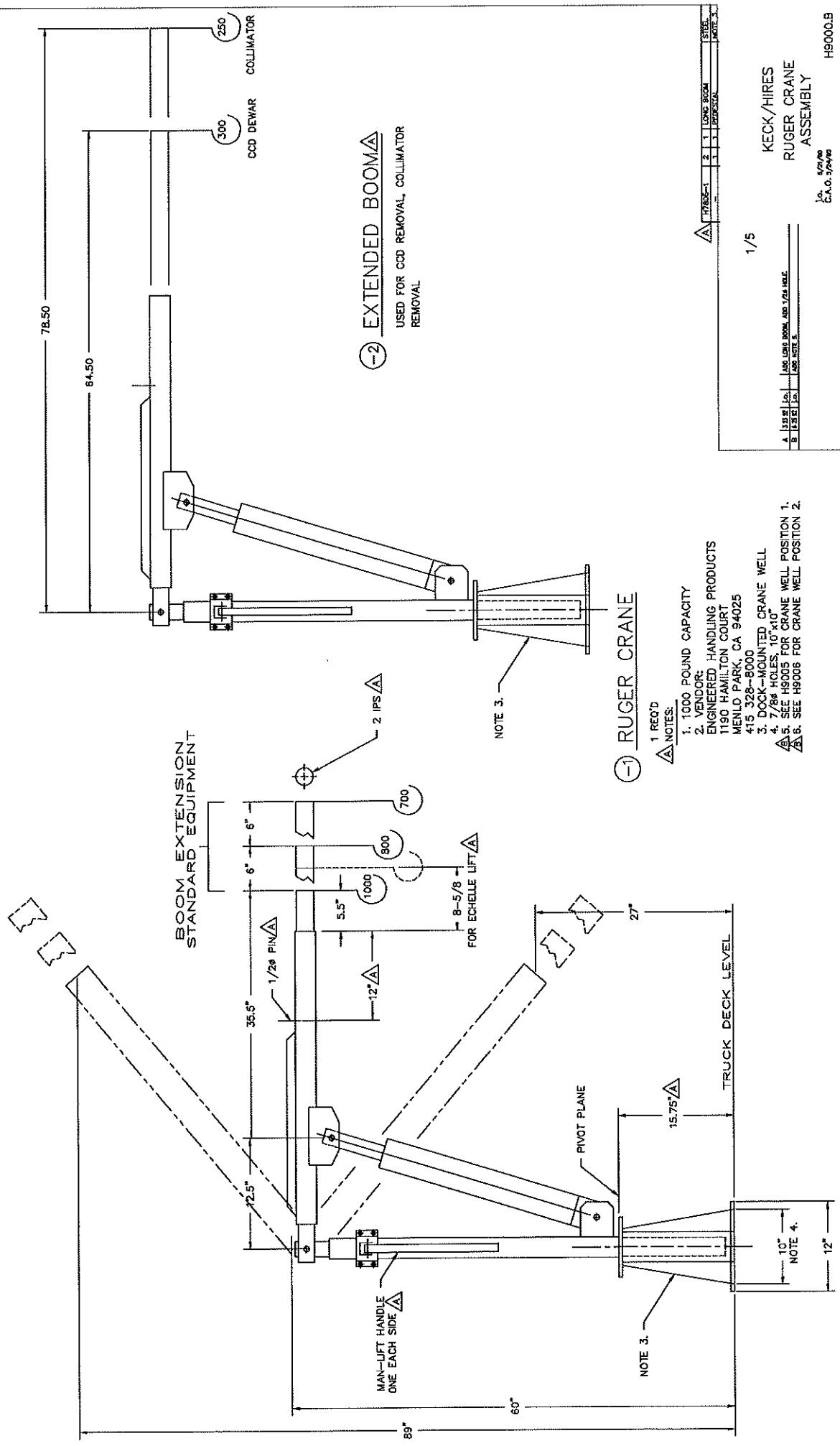
H6400.A





Appendix H List of Drawings — Shipping & Initial Alignment

1. H9000 Ruger Crane Assembly and Extended Boom
2. H9005 Ruger Crane, Position 1
3. H9006 Ruger Crane, Position 2
4. H3075 Echelle Mosaic Cart
5. H4075 Cross Disperser Cart
6. H6605 Main Frame Casters
7. H6936 Camera Mirror Moving Details



(-1) CRANE MOUNTED IN SOCKET 1

NOTES:

1. CROSS-DISPERSER ACCESS

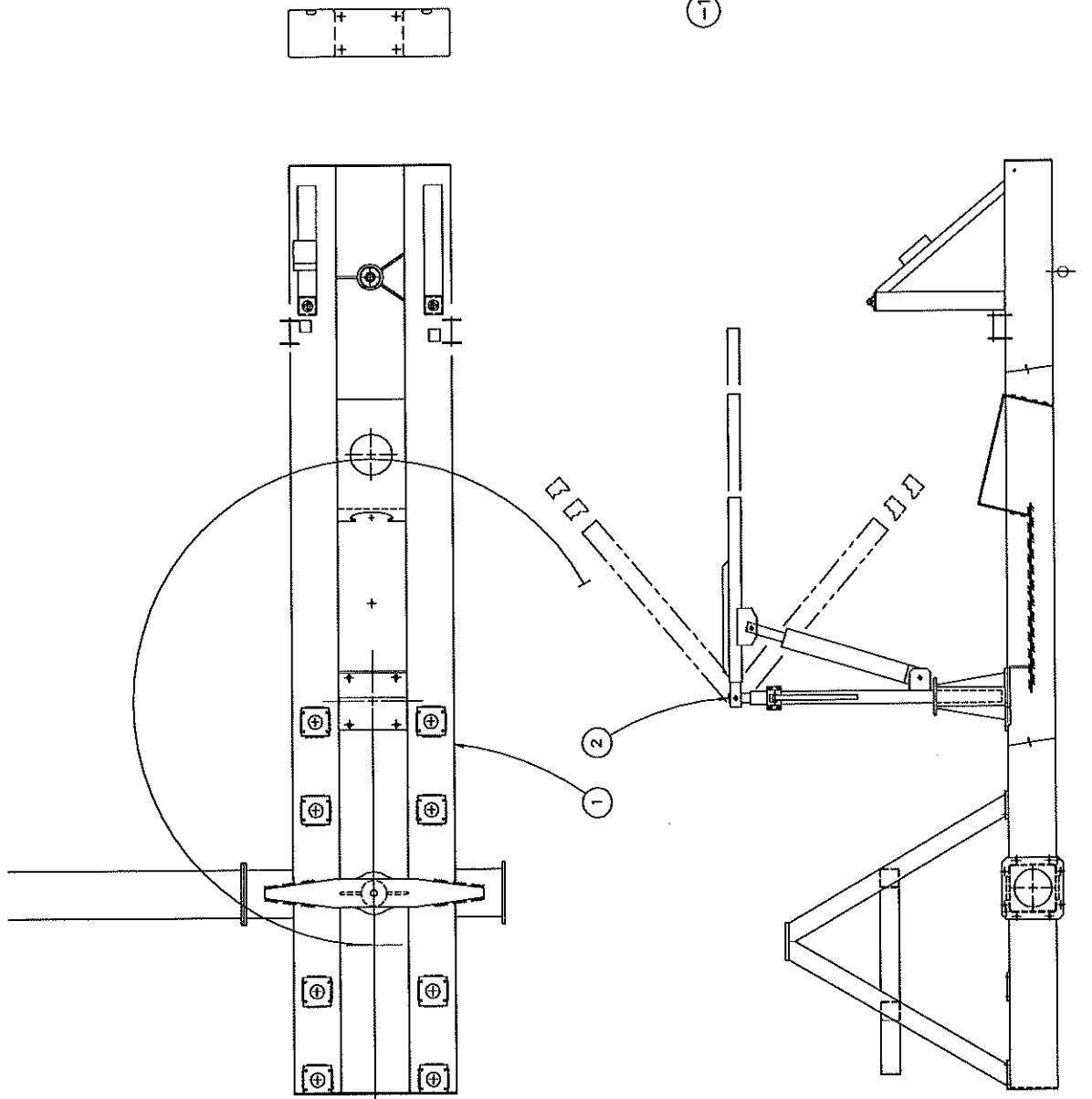
HE000-1 2 1 RIVER CRANE
HE010-1 1 1 OPTICAL BRANCH STATION

1/10

KECK/HIRES
CRANE
POSITION 1

HE005

2010/2
S/N 01/2010



H9006

KECK/HIRES
CRANE
POSITION 2

10
354x22

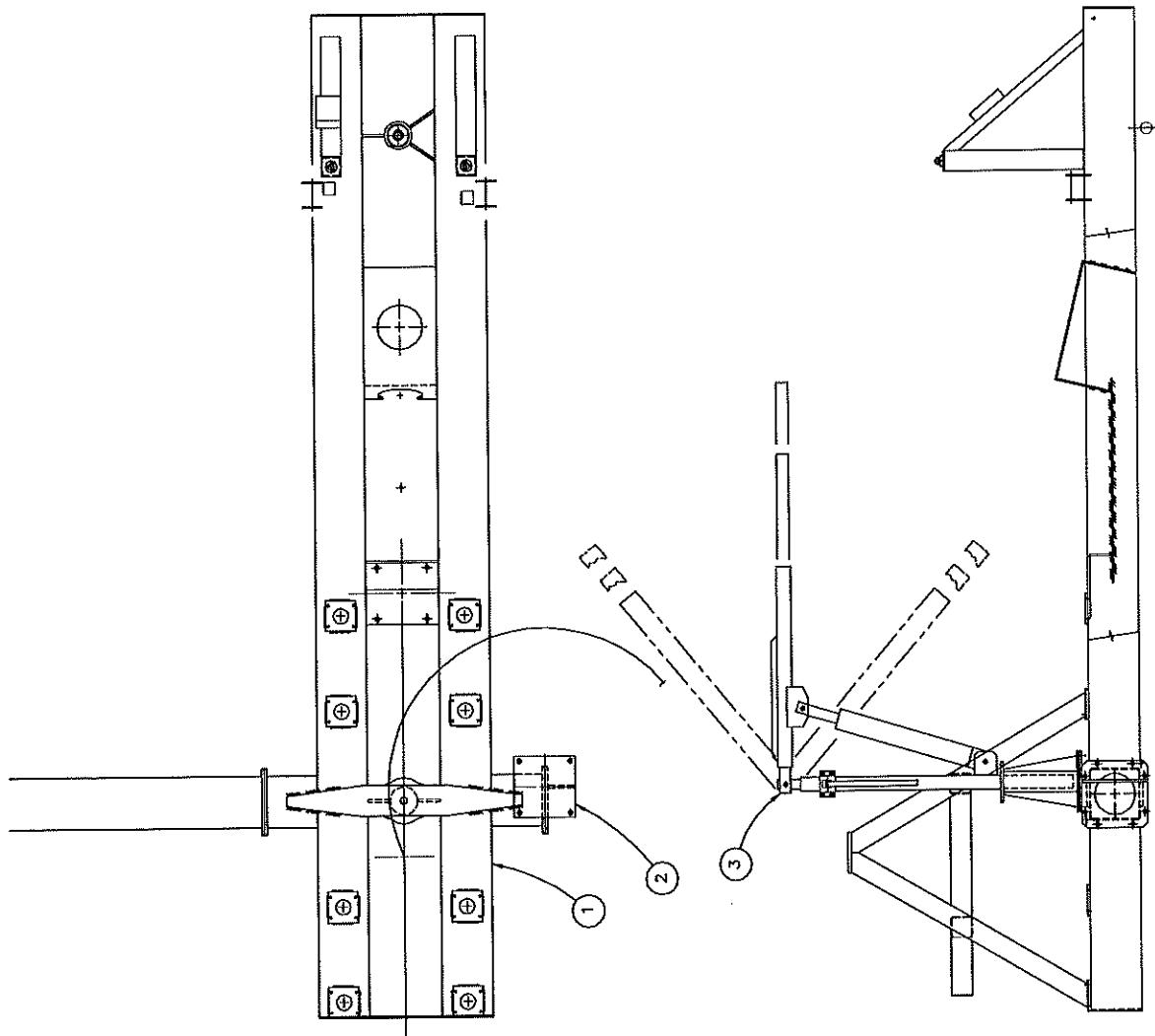
1/10

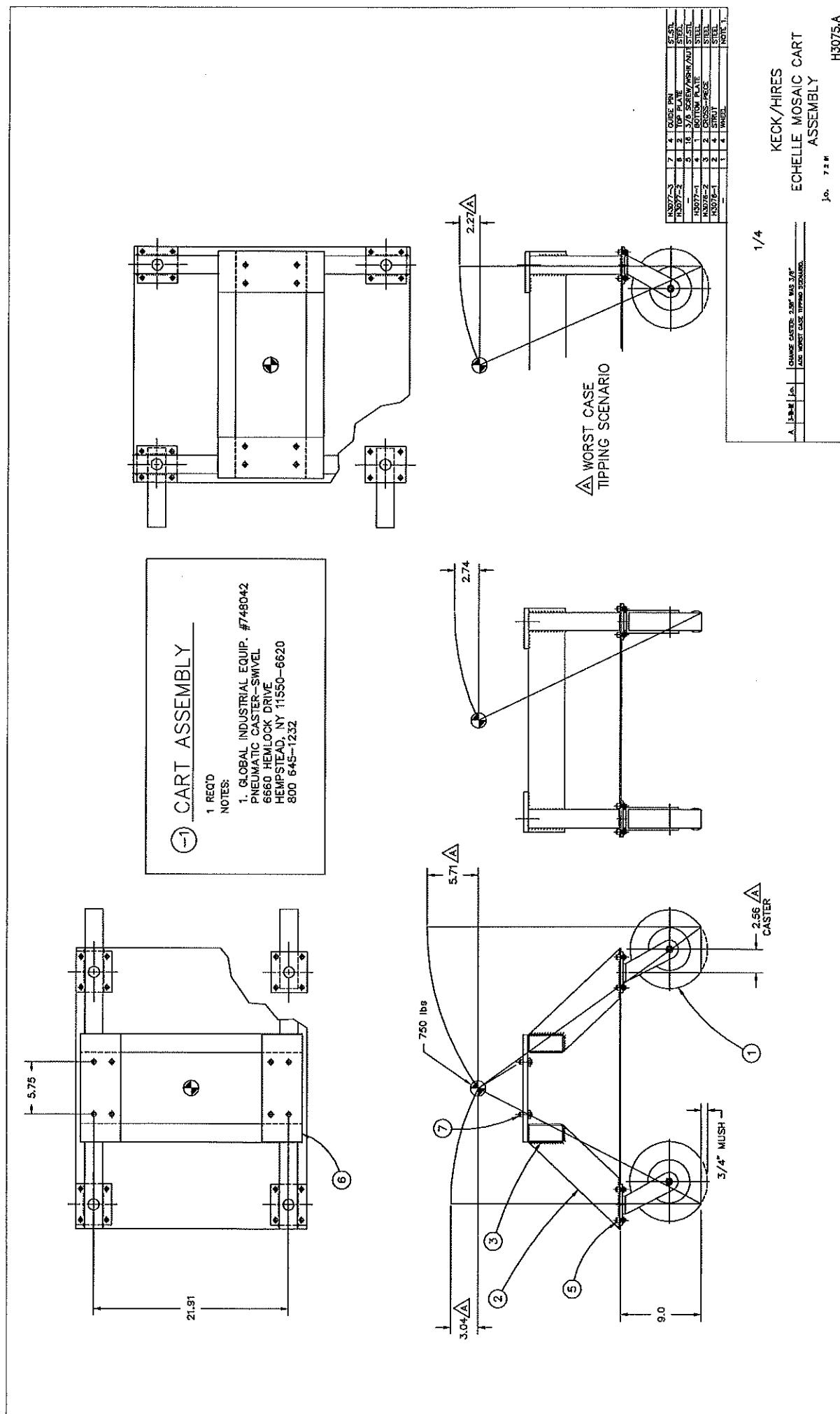
H6607-1	3	1	ROLLER CRANE BASE
H9006-1	2	1	ROLLER CRANE
H6750-1	1	1	OPTIONAL BRUSH STRUCTURE

① CRANE MOUNTED IN SOCKET 2

NOTES:

1. ECHELLE MOSAIC ACCESS
2. CRANE STORAGE POSITION

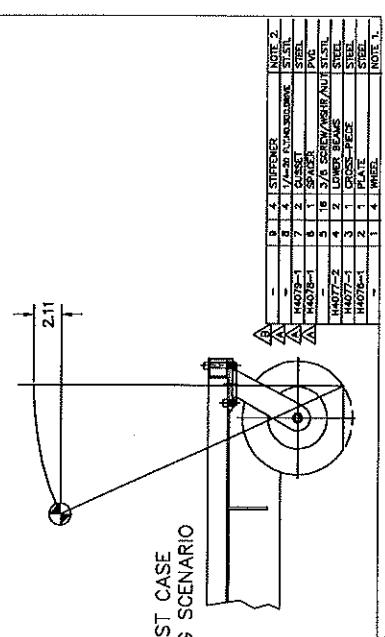
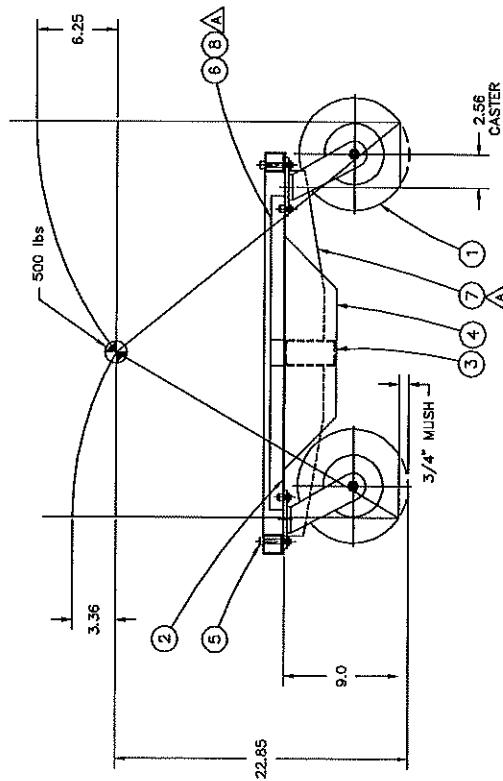
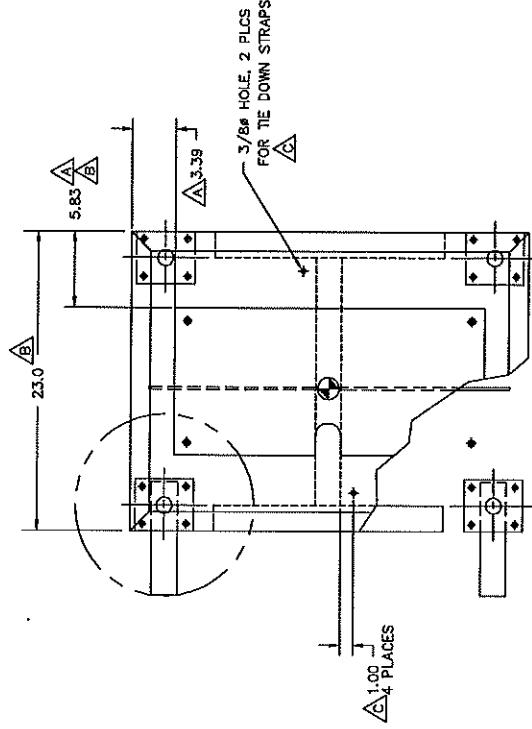




(-1) CART ASSEMBLY

1 REQD
NOTES:

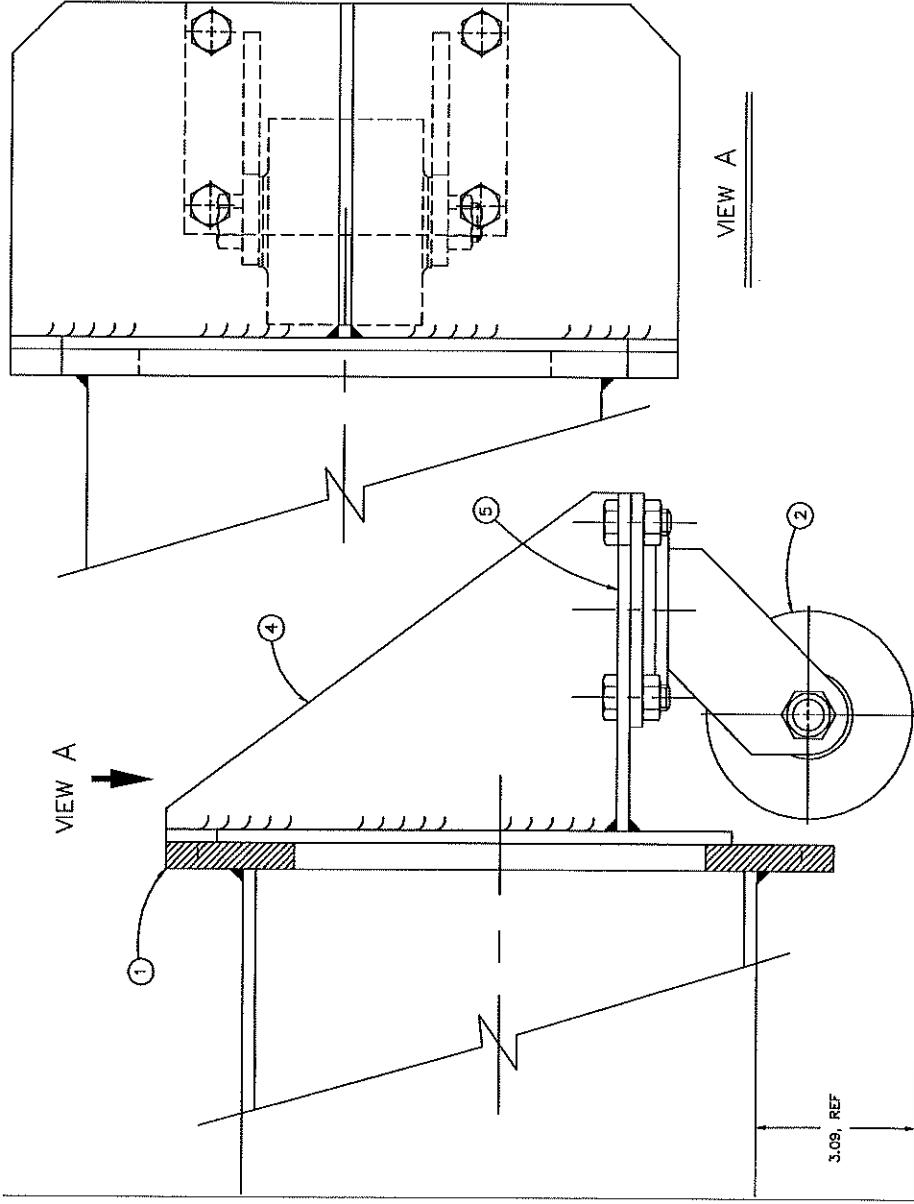
1. GLOBAL INDUSTRIAL EQUIP. #748042
PNEUMATIC CASTER-SWIVEL
6660 HEMLOCK DRIVE
HEMPSTEAD, NY 11550-6620
800 645-1232
△ B2. 1.5 x 1.5 STEEL TUBING.



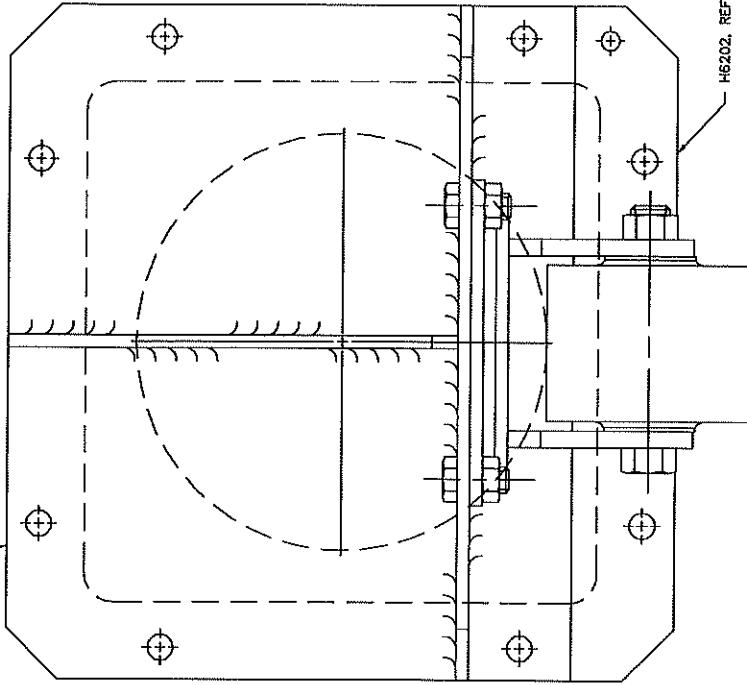
1/4

KECK/HIRES
CROSS DISPERSER CART
ASSEMBLY

JO. 3122 H4075.C



(3)



ITEM	QTY	DESCRIPTION	STEEL
H6605-3	5	CASTER MOUNT	STEEL
H6605-2	4	SUPPORT	STEEL
H6605-1	3	CASTER	STEEL
H6605	2	CASTERS	STEEL
H6605	1	CASTER	STEEL

FULL

KECK/HIRES
CASTERS
ASSEMBLY

H6605.A

A 1st ed. loc. [] add part note.

VIEW A

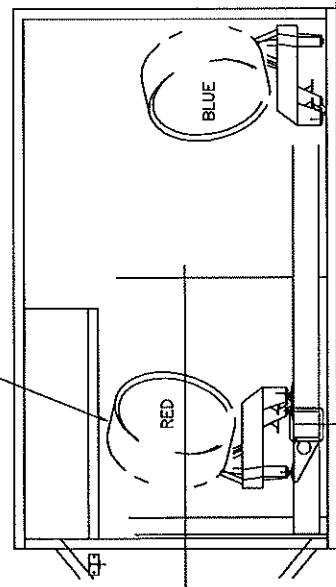
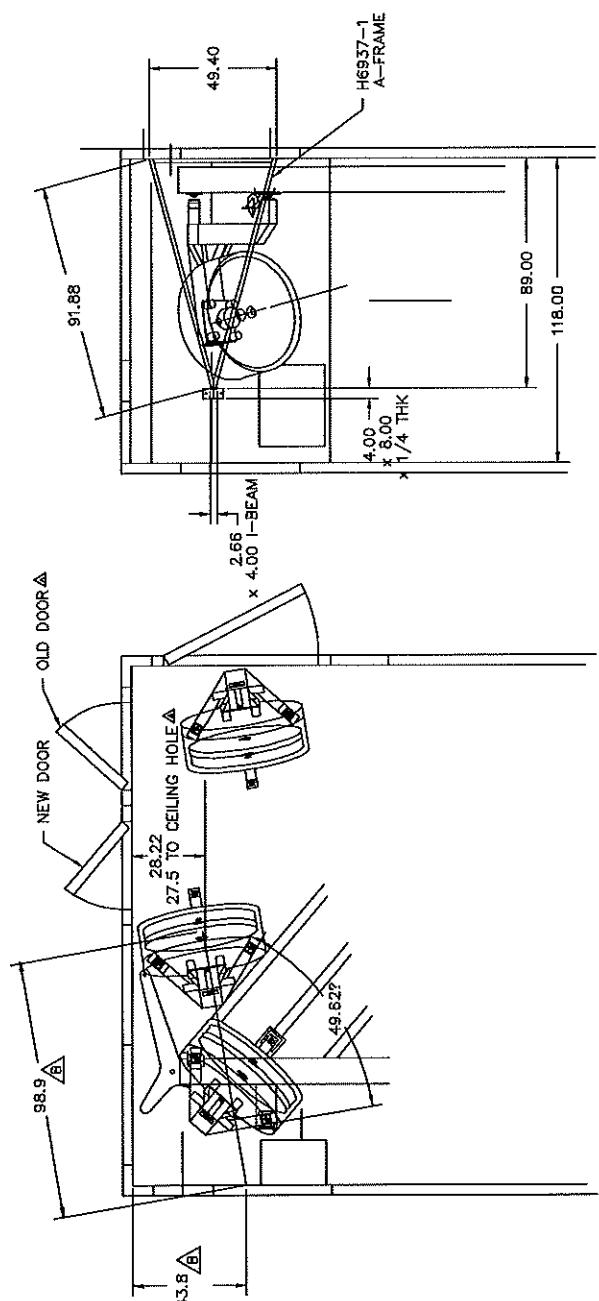
VIEW A

3.09, REF

(-1) CASTER ASSEMBLY

3 REQD
ALBION #81-C4-04501
40X3 SEMI-STEEL WHEELS
CAPACITY: 1650 LBS
SWIVEL STYLE
POWDER COAT WHITE





H6936.B
PLAN VIEW
11-7-92
SEE H1465 FOR ROOF SUPPORT BEAM
△ SEE H6440 FOR A-FRAME LOCATION

.05

KECK/HIRES
CAMERA MIRROR LIFT
LAYOUT

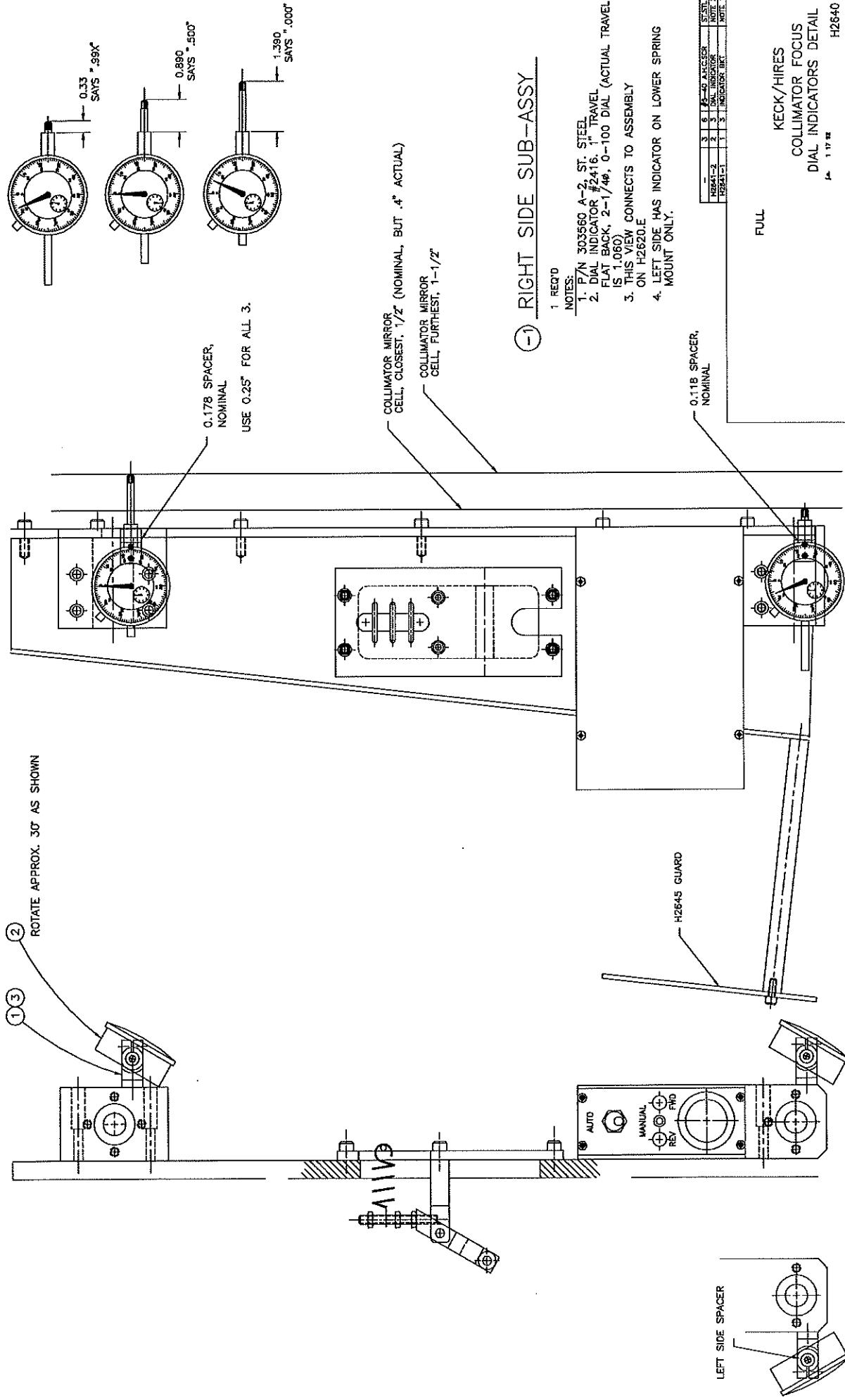
H6936.B

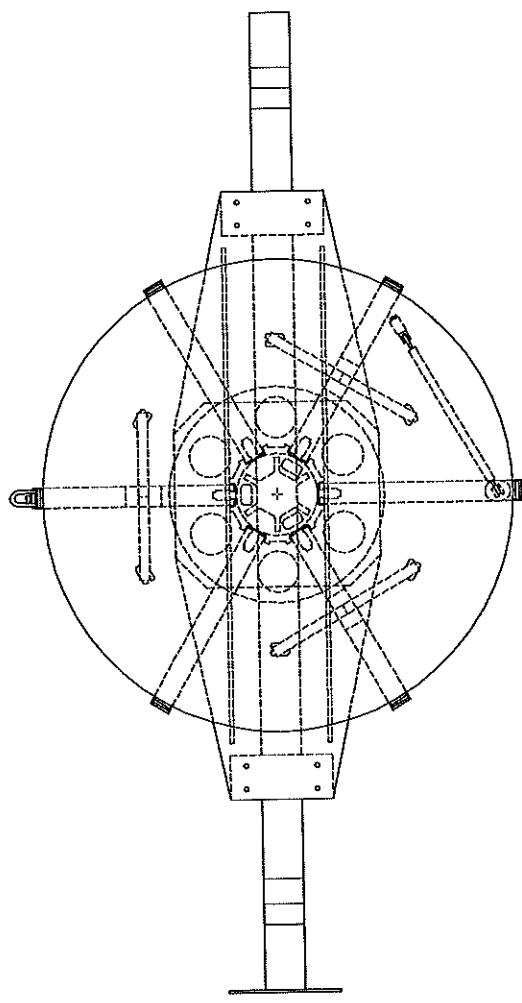
B SIGN IN. SIGN OUT

4

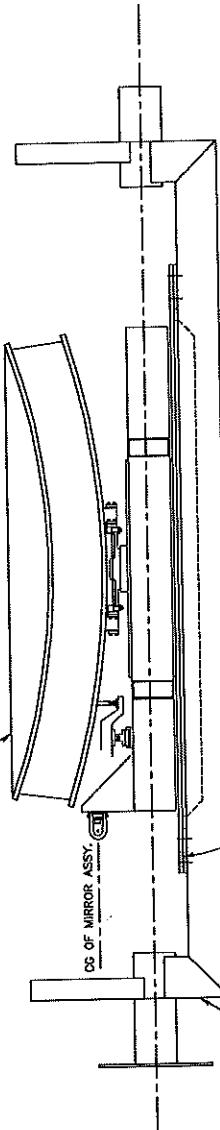
Appendix I List of Drawings — Re-Aluminizing and Re-Aligning

1. H2640 Collimator Focus, Dial Indicator Detail
2. H5420 Hextek (Camera Mirror) Aluminizing Fixture Assembly
3. H6500 Alignment Scenario
4. H6510 Alignment Telescope Fixture, Rear
5. H6530 Alignment Telescope Fixture, Front
6. H6560 Alignment Target, Echelle Pivot





CAMERA MIRROR ASSY
H5560

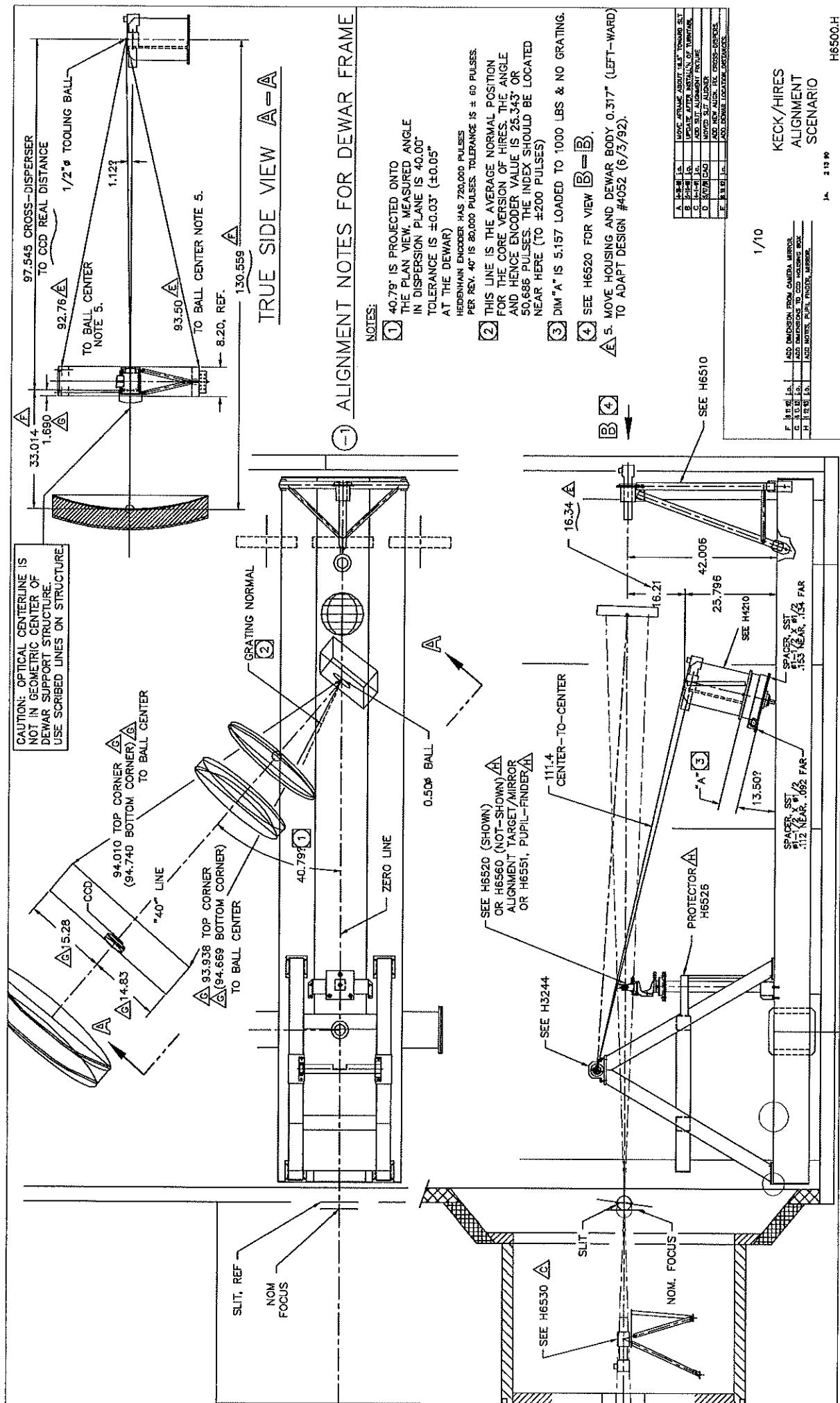


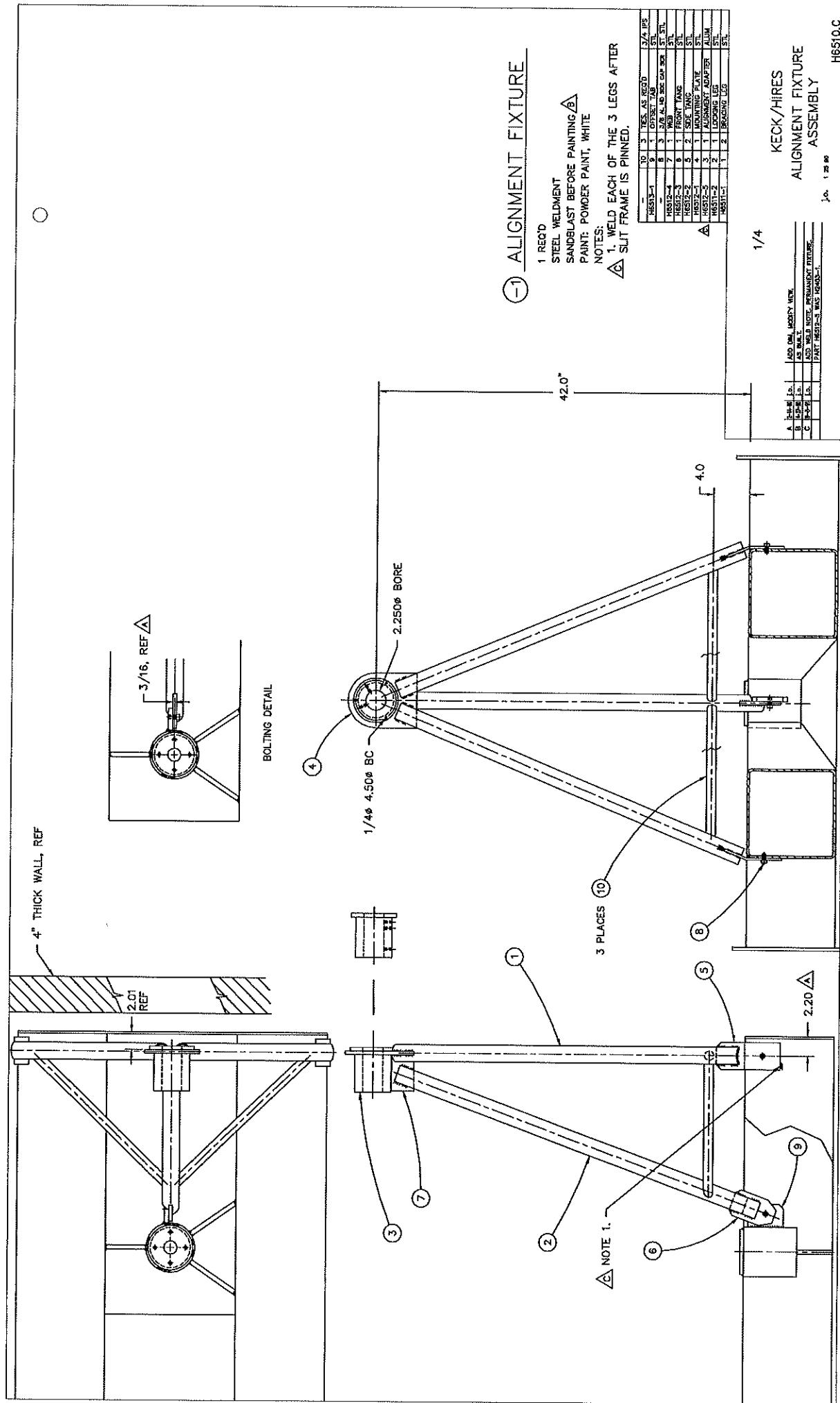
REF KECK P/N 643-C0023
KECK SECONDARY MIRROR CELL
HANDLING FIXTURE

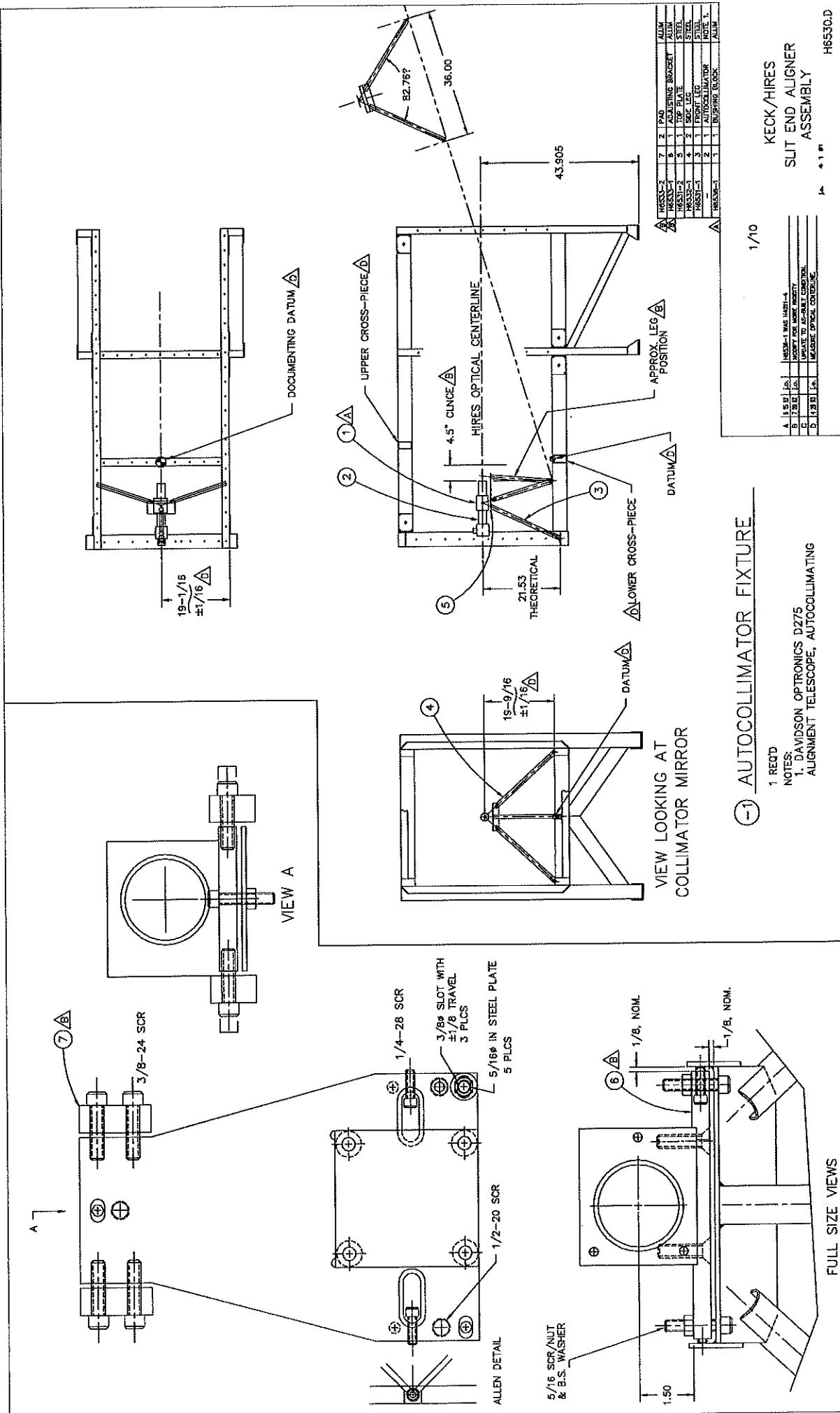
ALUMINIZING FIXTURE DETAIL
H5521 & H5522
△ ENGRAVE: HIRES, CAMERA MIRROR, ALUMINIZING FIXTURE
△ DO NOT ANODIZE
△ VENT ALL SCREW HOLES
△ DO NOT USE GREASE TO ASSEMBLE SCREWS

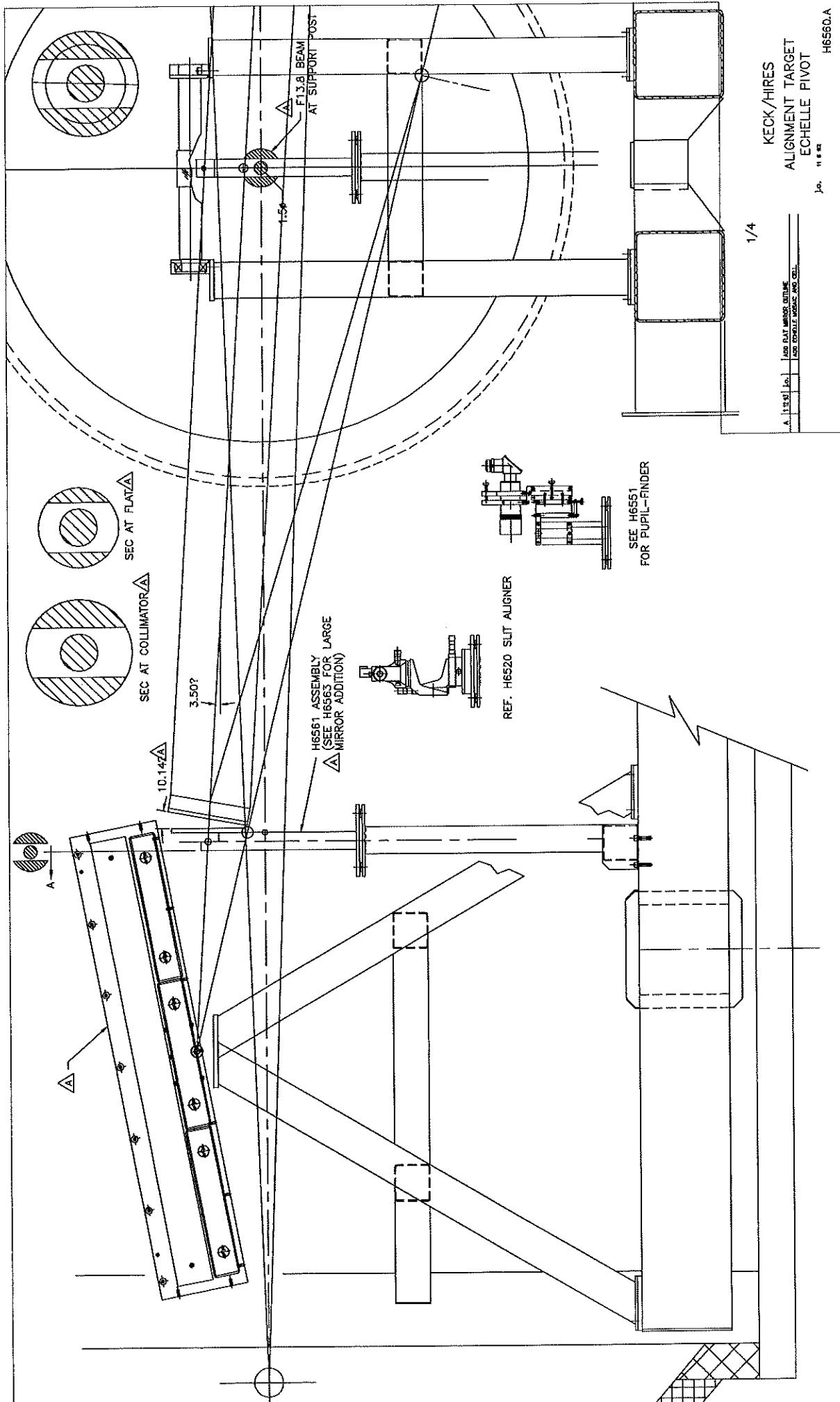
.2

KECK/HIRES
CAMERA MIRROR
ALUMINIZING FIXTURE
H5520.A
A [255] I.e. [large note: MIRROR SURFACE]
△ C.R. 10/20/98
△ C.R. 10/20/98
△ C.R. 10/20/98
△ C.R. 10/20/98









Appendix J List of Drawings

— Other Comments

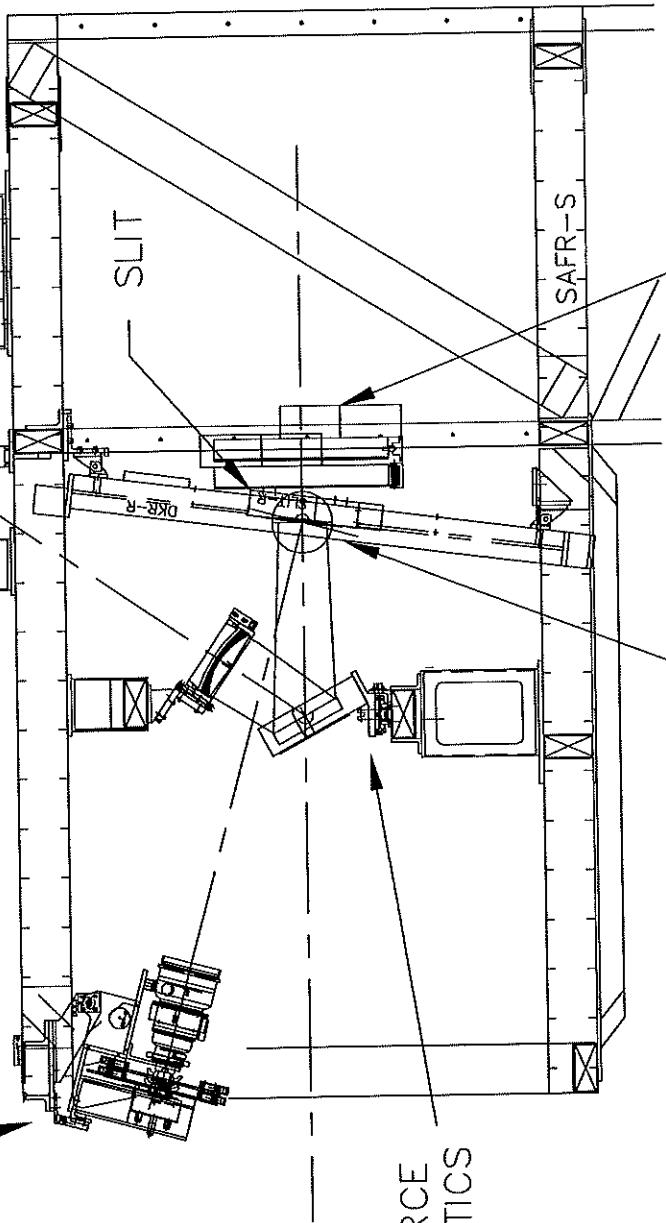
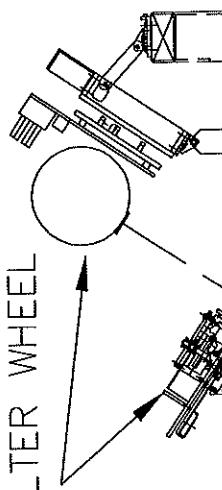
1.

Appendix K List of Drawings — Slit Area

1. P1001 Slit Area Overview and Layout
2. H6711 Slit Area Structure Sub-Assembly

GUIDER TV AND
FILTER WHEELS (2)

COMPARISON SOURCES
AND FILTER WHEEL

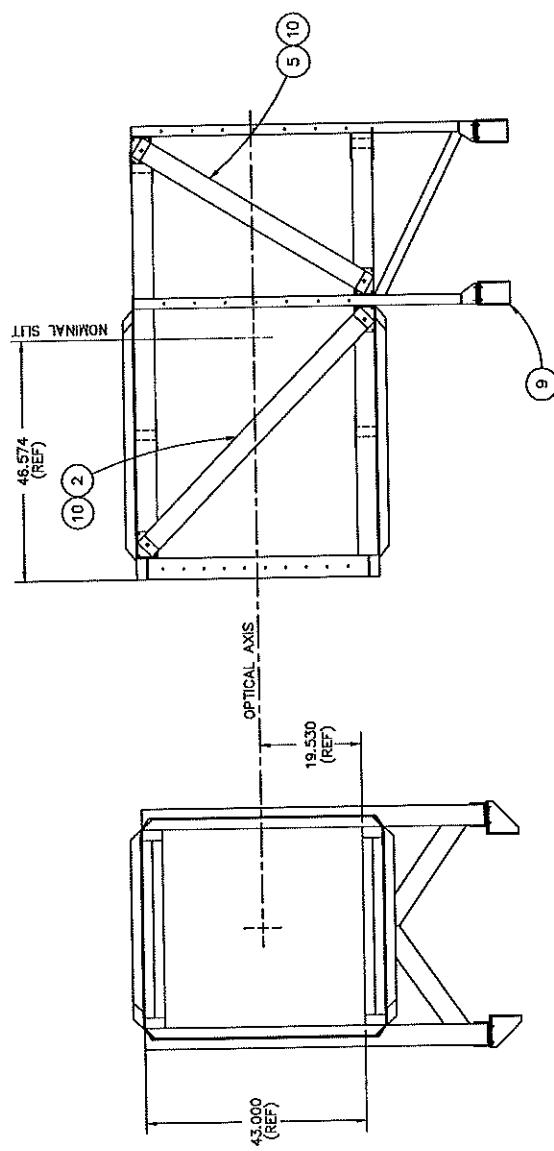
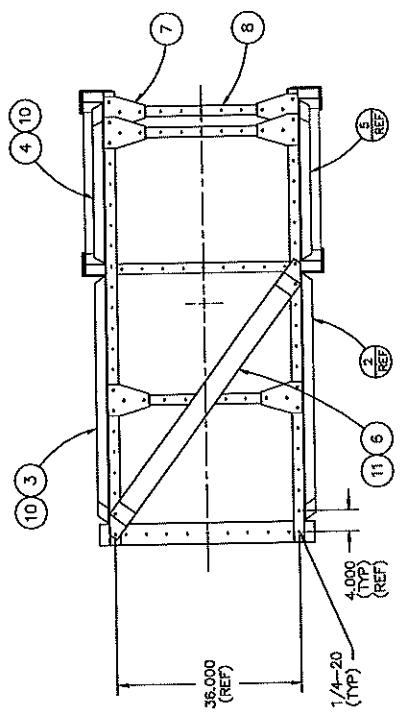


COMP. SOURCE
STEERING OPTICS

DECKER STAGE
4 POSITIONS

SPECTROGRAPH
FILTERWHEELS (2)

/H/P/P1001.DWG
1/24/92 BCB



ITEM	DESCRIPTION	SIZE
H6716-1	END PLATE	5/32
H6716-2	END PLATE	5/32
H6716-3	SOLID WELD	5/32
H6716-4	SOLID WELD	5/32
H6716-5	WELD	5/32
H6716-6	WELD	5/32
H6716-7	WELD	5/32
H6716-8	WELD	5/32
H6716-9	WELD	5/32
H6716-10	WELD	5/32
H6716-11	WELD	5/32

NOTES:
1. POWDER PAINT: FULLER O'BRIEN FLAT
BLACK EFB 534-SO.

KECK/HIRES
SLIT AREA STRUCTURE
ASSEMBLY

H6711.D

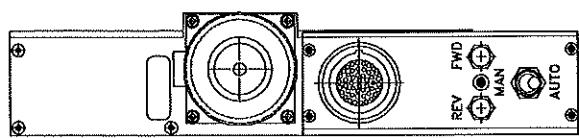
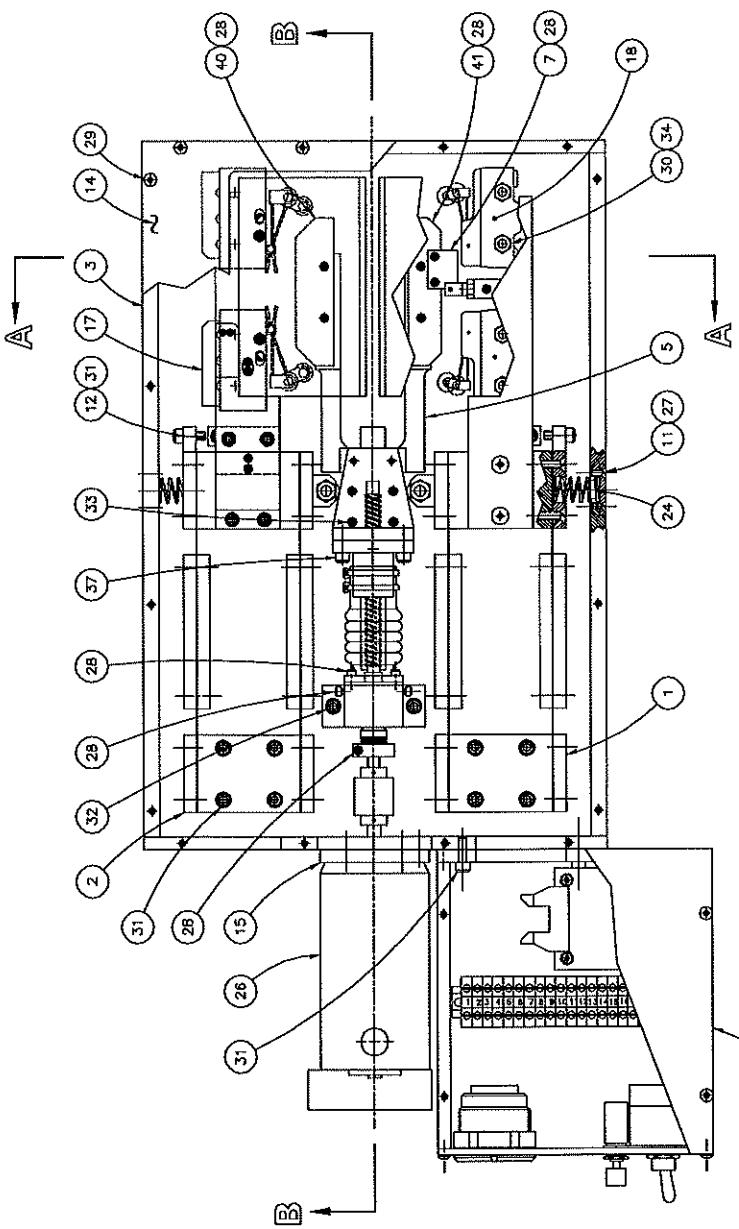
S.A.	6/17/80
D.A.	6/27/80

1/10

Appendix L List of Drawings

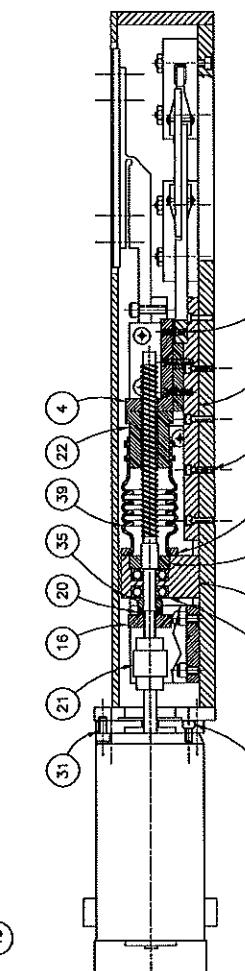
— Slit Assembly

1. H1324 Slit Assembly
2. H1326 Slit Drive Schematic Information
3. H1322 Slit Jaw / Spring Assembly



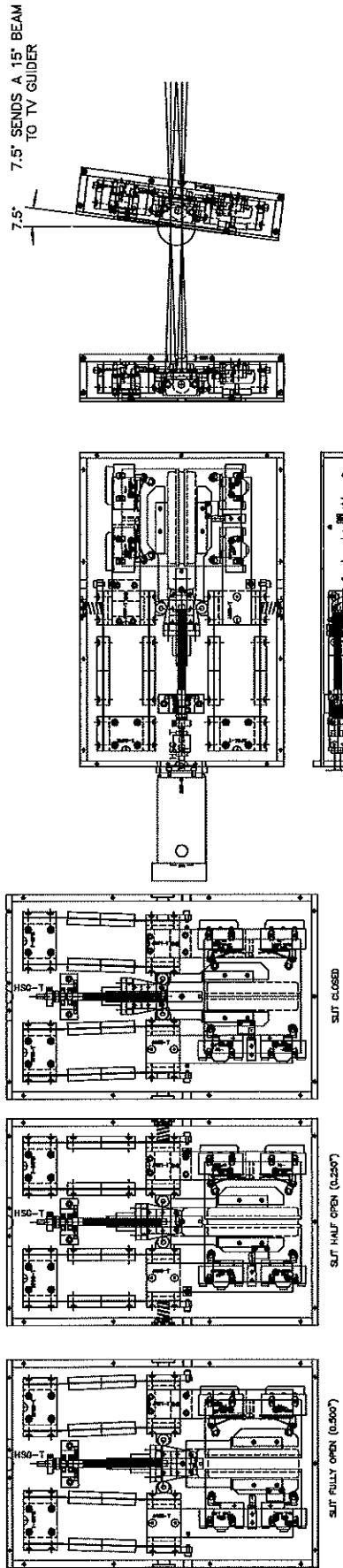
① SLIT ASSEMBLY

A.	B.	C.	D.	E.
ROUT L.S.	CHANGED ITEM 13			
2 ROUT L.S.	MATERIAL CHANGED			
C URGENT	ADDED PART NO. TO THE SHEET			
D URGENT	ROUTED LIFT SHIMS			
E URGENT	ROUTED NO. PINS			



A.	B.	C.	D.	E.
1 CULPING				
H1317-5	41 1 TRIGGER LINE SWITCH ALUM	NOTE 10		
H1317-2	42 1 TRIGLOC LINE SWITCH			
-	43 1 SPDT SW			
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-	34			

SLIT ASSEMBLY SHOWN 1/2 SIZE



THEORETICAL POSITIONING RESOLUTION:

MOTOR ENCODER 4000 CTS./REV.
BALL SCREW (THK MBF 08) 2mm/REV.
DRIVE RAMP 1 IN 5 SLOPE (20%) ▲
MEANING: MOVE WEDGE 1mm AND SLIT
WIDTH OPENS 0.2mm PER JAW OR 0.4mm
TOTAL - REALLY 2 IN 5 SLOPE OR 40%
WAVES.

2mm/REV X 1 REV/4000CTS = 0.0005 mm/CT

0.0005 mm/CT X 0.2 SLOPE X 2 JAWS = 0.0002mm/CT

IF SERVO LOOP IS GOOD TO 2 COUNTS,
RESOLUTION IS 0.4 MICRONS

RESOLUTION AND REPEATABILITY HAVE BEEN TESTED TO 0.0005" MECHANICALLY.
BETTER TESTING WILL BE PERFORMED WITH THE GUIDER CCD TV.

PLATE SCALE AT SLIT IS 1.375 ARC-SEC/mm

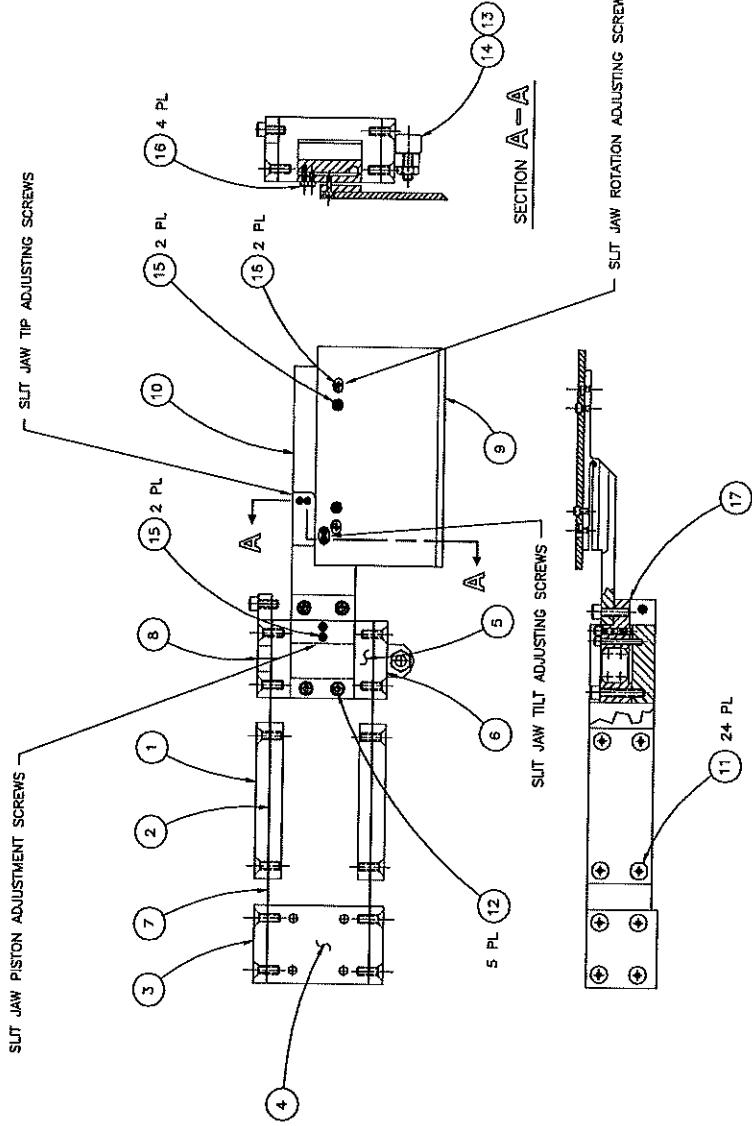
0.0004mm X 1.375 ARC-SEC/mm = 0.00055 ARC-SEC

SLIT JAWS:
SLIT JAW MATERIAL IS 420 STAINLESS STEEL
JAWS ARE GROUND FLAT MECHANICALLY.
JAWS ARE OPTICALLY POLISHED AND FLAT TO
5 WAVES.

420 STAINLESS STEEL:
CARPENTER TECHNOLOGY
2263 NATIONAL AVE. A
HAYWARD, CA. 94545-1715
800-522-5077

SEE H1424 FOR FULL ASSEMBLY
SEE H6758 FOR SLIT AREA MOUNTINGS
SEE H6760 FOR SLIT MOUNTINGS

▲ 1/2 size | 1/2 note
KECK/HIRES
SLIT ASSY
DATA AND GEAR REDUCTION
H1326.A
04/17/92



① SLIT JAW/SPRING ASSEMBLY

1 REQ'D

H1322-1	1	SCREW, 2-56 UNC
H1322-2	1	SLIT MOUNT BASE
-	1	SCREW, 4-40 UNC
-	1	SCREW, 10-32 UNC
-	1	ADJ. PLATE
-	1	FLAT CHAMFER
-	1	SCREW, 8-32 UNC
-	1	SCREW, 8-32 UNC
H1322-1	1	SLIT MOUNT
H1322-2	1	END PLATE
H1322-3	1	SPRING
H1322-4	1	FLAT CHAMFER PLATE
H1322-5	1	BASE SPRING
H1322-6	1	BASE PLATE
H1322-7	1	BASE PLATE SPRING
H1322-8	1	BASE PLATE SPRING
H1322-9	1	BASE PLATE SPRING
H1322-10	1	BASE PLATE SPRING
H1322-11	1	BASE PLATE SPRING
H1322-12	1	BASE PLATE SPRING
H1322-13	1	BASE PLATE SPRING
H1322-14	1	BASE PLATE SPRING
H1322-15	1	BASE PLATE SPRING
H1322-16	1	BASE PLATE SPRING

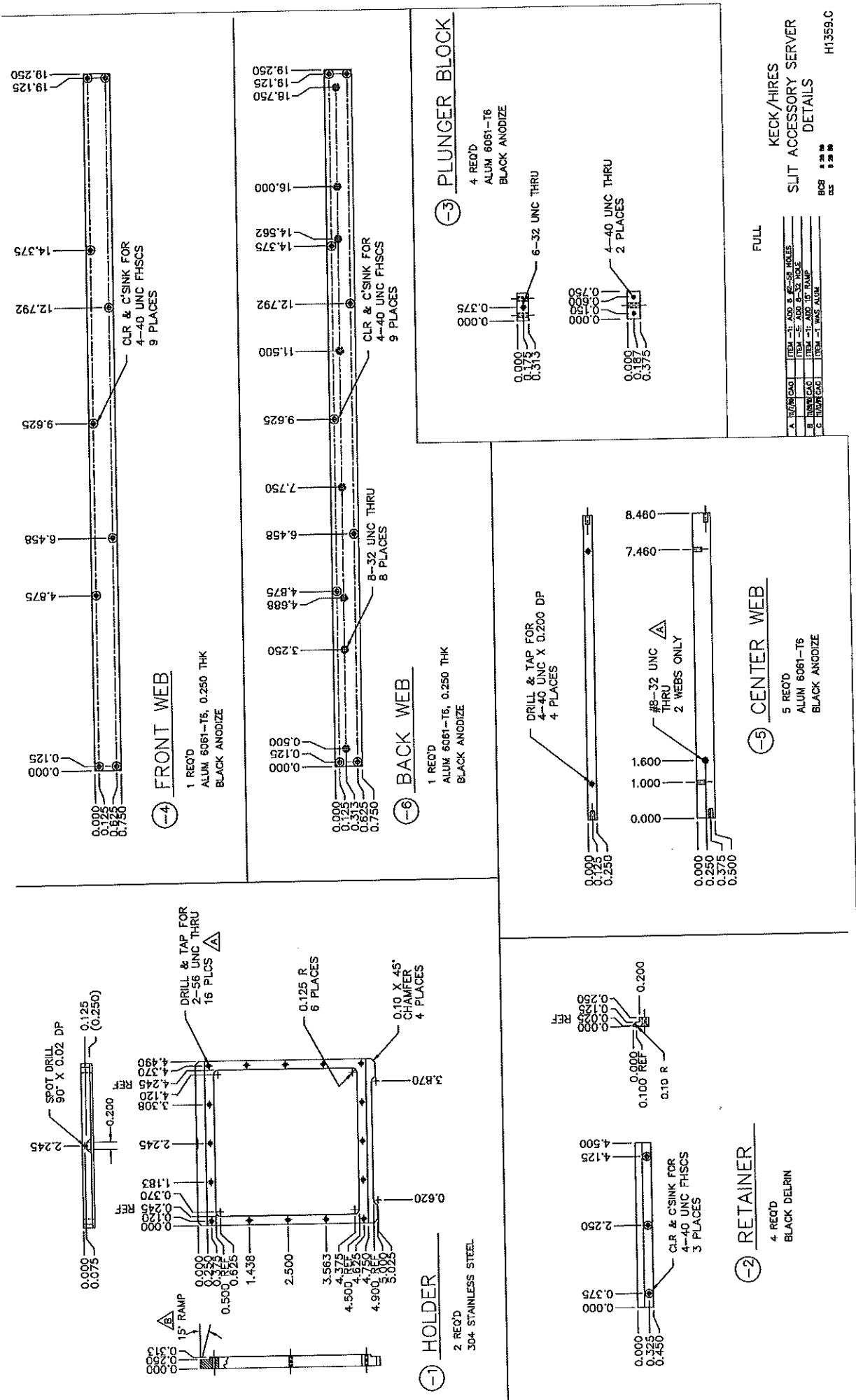
FULL

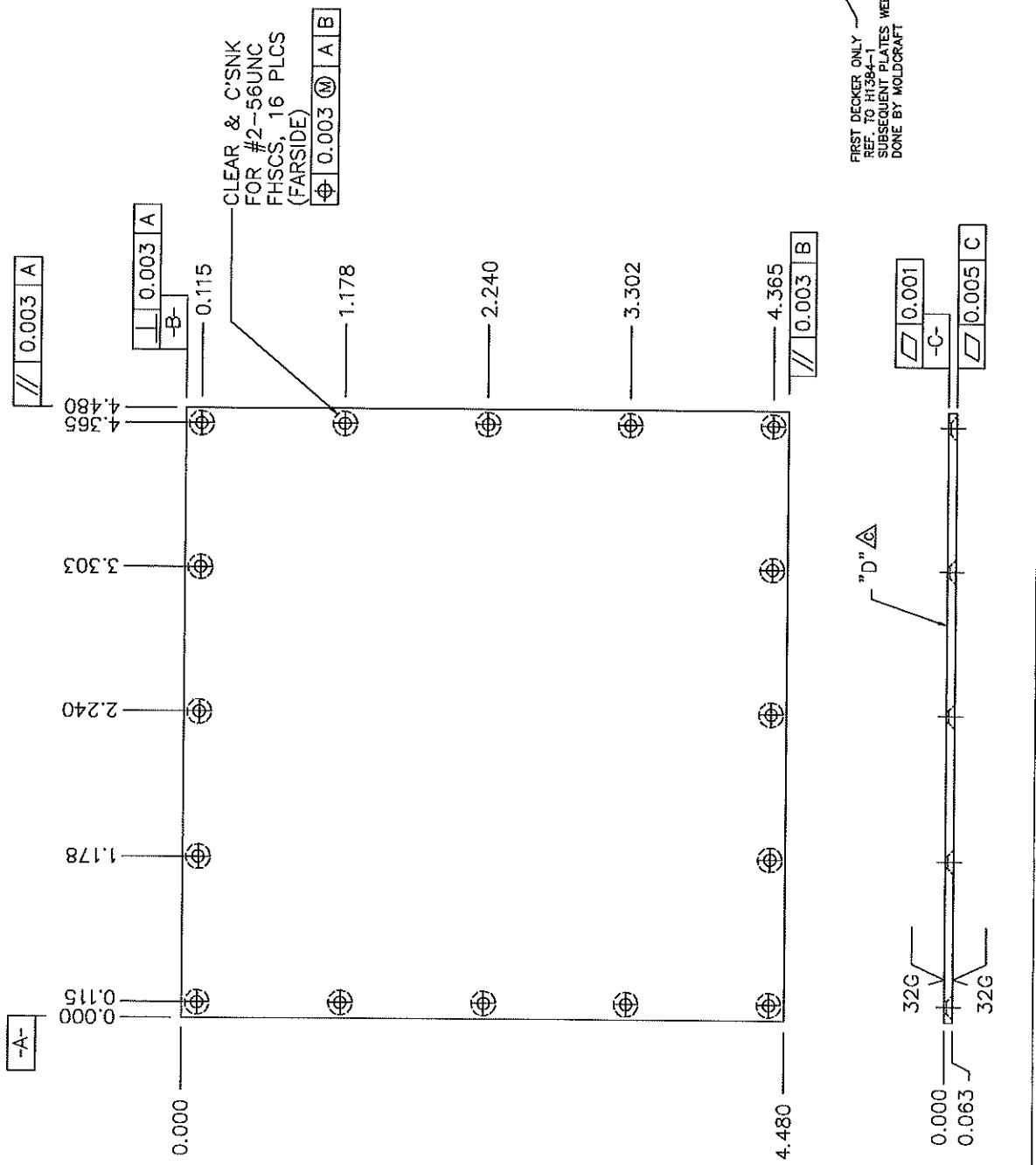
KECK/HIRES
SLIT JAW/
SPRING ASSEMBLY

BLW
DRAFTED
07/07/98
EAO
H1322.A

Appendix M List of Drawings — Slit Accessory Server

1. H1359 Details
2. H1388 Decker Plate Blank
3. H1384 Decker Plate #A
4. H1389 Decker Plate #B
5. H1390 Decker Plate #C
6. H1391 Decker Plate #D
7. H1378 Slit Accessory Server Assembly
8. H1387 Slit Accessory Server Drive Schematic
9. H6758 Slit Area Stage Mountings — Assembly
10. H6759 Slit Area Stage Mountings — Details
11. H6760 Slit Area Stage Mountings — Details





FABRICATION NOTES:

2. CUT TO 4.480 DIMENSIONS AND PUT IN HOLES (NO C'SINKS)
 3. BENDING HEAT TREAT TO RCS2 TO RCS5
 VENDOR: (408) 727-6630
 4. DOUBLE DISK GRIND TO 1/16" THICK
 VENDOR: PETERSON GRINDING IN REDWOOD CITY, CA
 (415) 365-4373
 5. COUNTERSINK HOLES
 6. LICK OPTICAL SHOP TO GRIND SIDE "D"
 WITH 5 MICROGRIT LOOSE ABRASIVE, WET.
 7. EDM AT MOLDRACT, 1643 S. MAIN ST., MILPITAS, CA
 VENDOR: (408) 945-1055
 8. LICK OPTICAL SHOP TO POLISH SIDE "D"
 9. S: OPTICAL DIMENSIONS TO BE $\pm 0.003"$
 S: ALL Y:XXXX" DIMENSIONS TO BE $\pm 0.0005"$
 AND ALL X:XXXX" DIMENSIONS TO BE $\pm 0.0003"$

(7. EDM AT MTE, INC. IN SAN JOSE, CA)
VENDOR: (408) 227-7040

FIRST DECKER ONLY
REF. TO H1384-1

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		KECK/HIRES
A	UNLTD CO	BROKEN DOME BEP AS BUILT
B	UNLTD CO	ADD MATT HARDON NEO
C	2-21 Ls.	ADD FABRICATION PROCEDURE

SUIT ACCESSORY SERVER
DECKER PLATE BLANK

ECS 15375-2
7/10/82

CDO

HI388.B.C

A 1/10/00 CDO REVERSED DMS PER AS BUILT
B 1/10/00 CDO ADD MATT HARDEN INFO

-A- SEE NOTES

4.480

4.370

2.245

1.183

0.000

0.120

0.000

0.003 A

-B-

0.003

A

CLEAR & C'SNK
FOR #2-56UNC
FHSCS, 16 PLCS
(FARSIDE)

Φ 0.003 (N) A B

(-1) DECKER PLATE #A

1 REQ'D
420 ST. STEEL, 0.063 THK.

NOTES:

1. SLOTS ARE CONCENTRIC @ C.
D 2. FABRICATION NOTES, SEE H1388



SEC B-B

3/1

KECK/HIRES
SLIT ACCESSORY SERVER
DECKER PLATE #A
B38
H1388.D
H1388.D
H1388.D
H1388.D

0.001

C

0.001

B

0.001

A

0.001

D

0.001

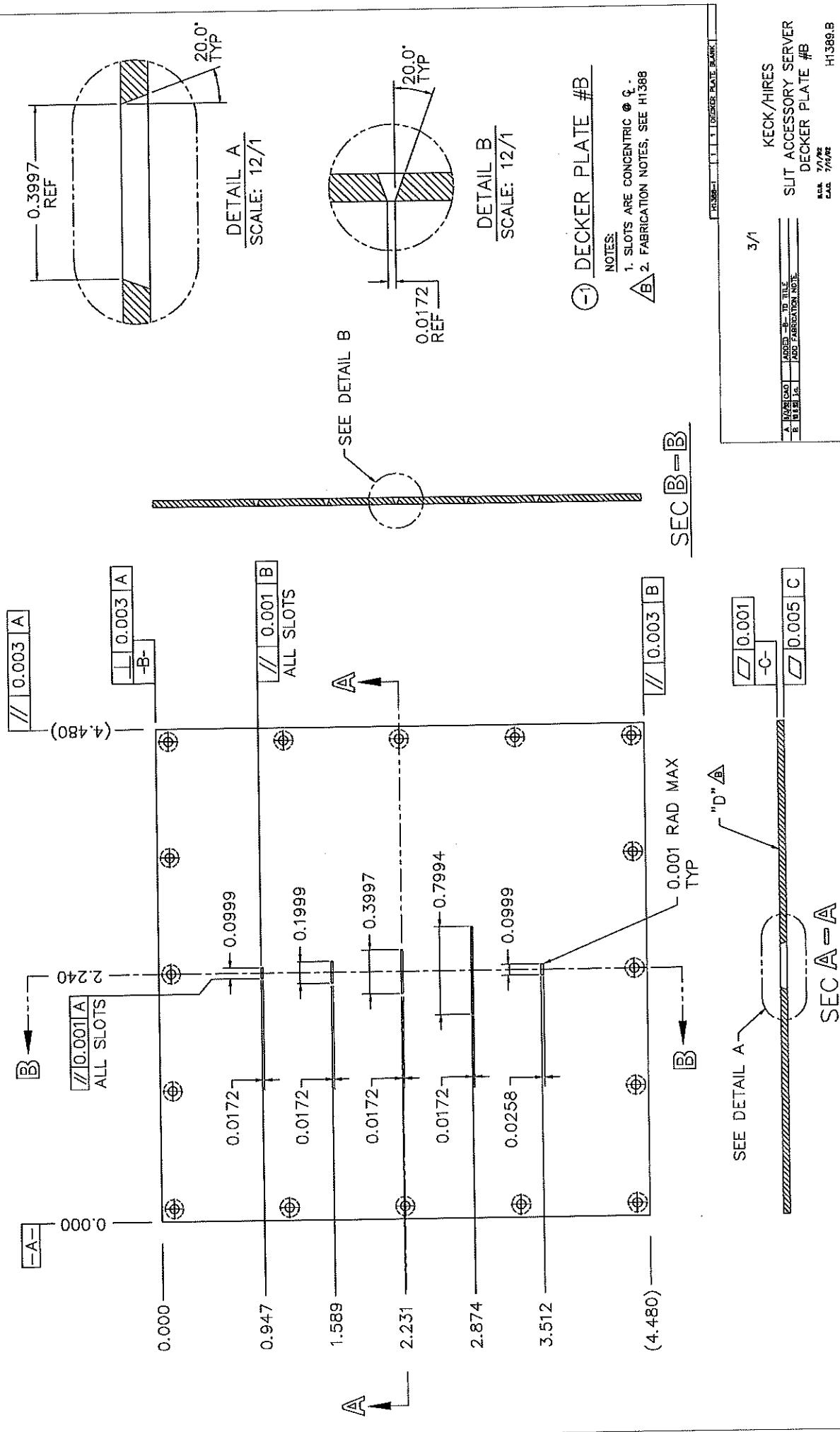
C

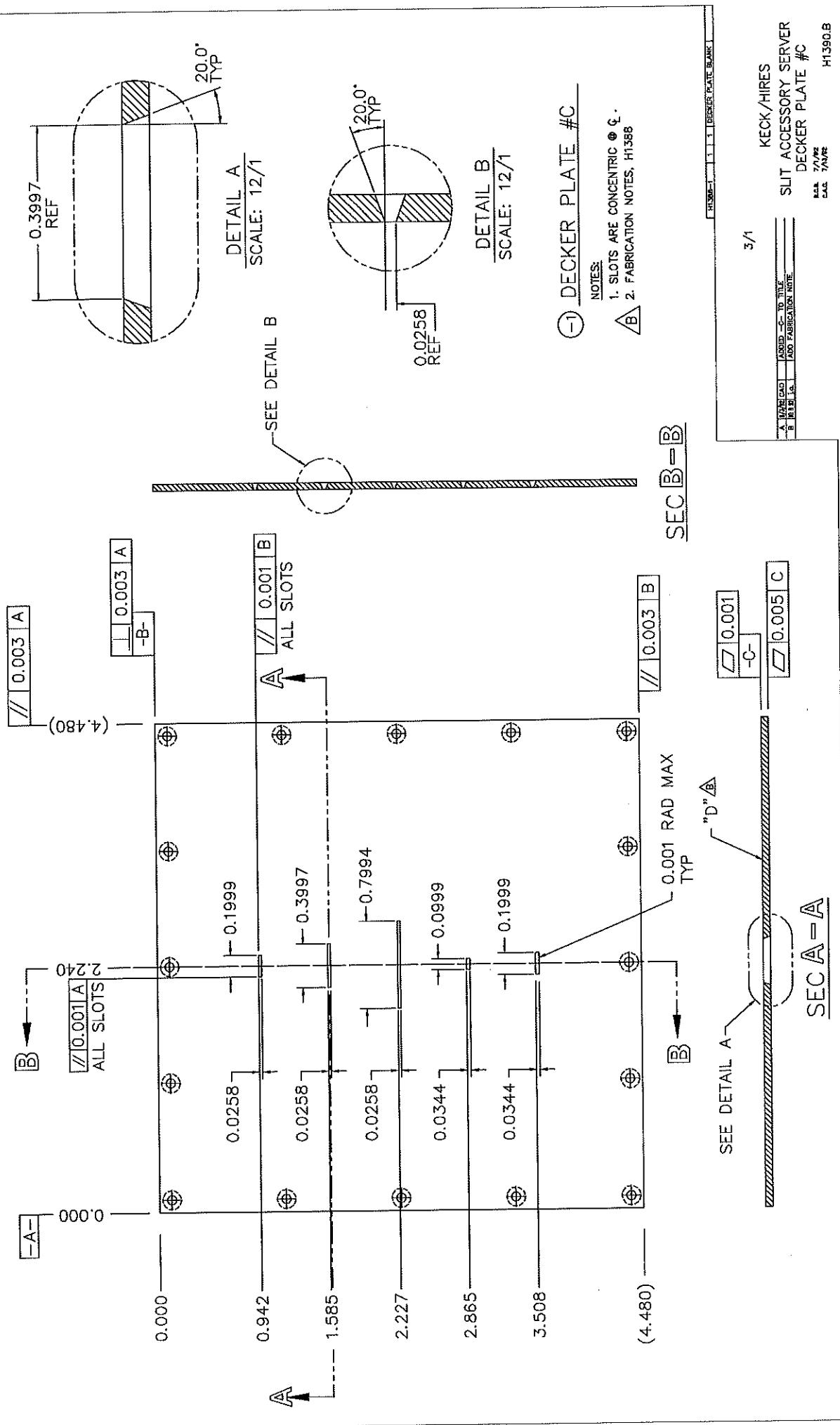
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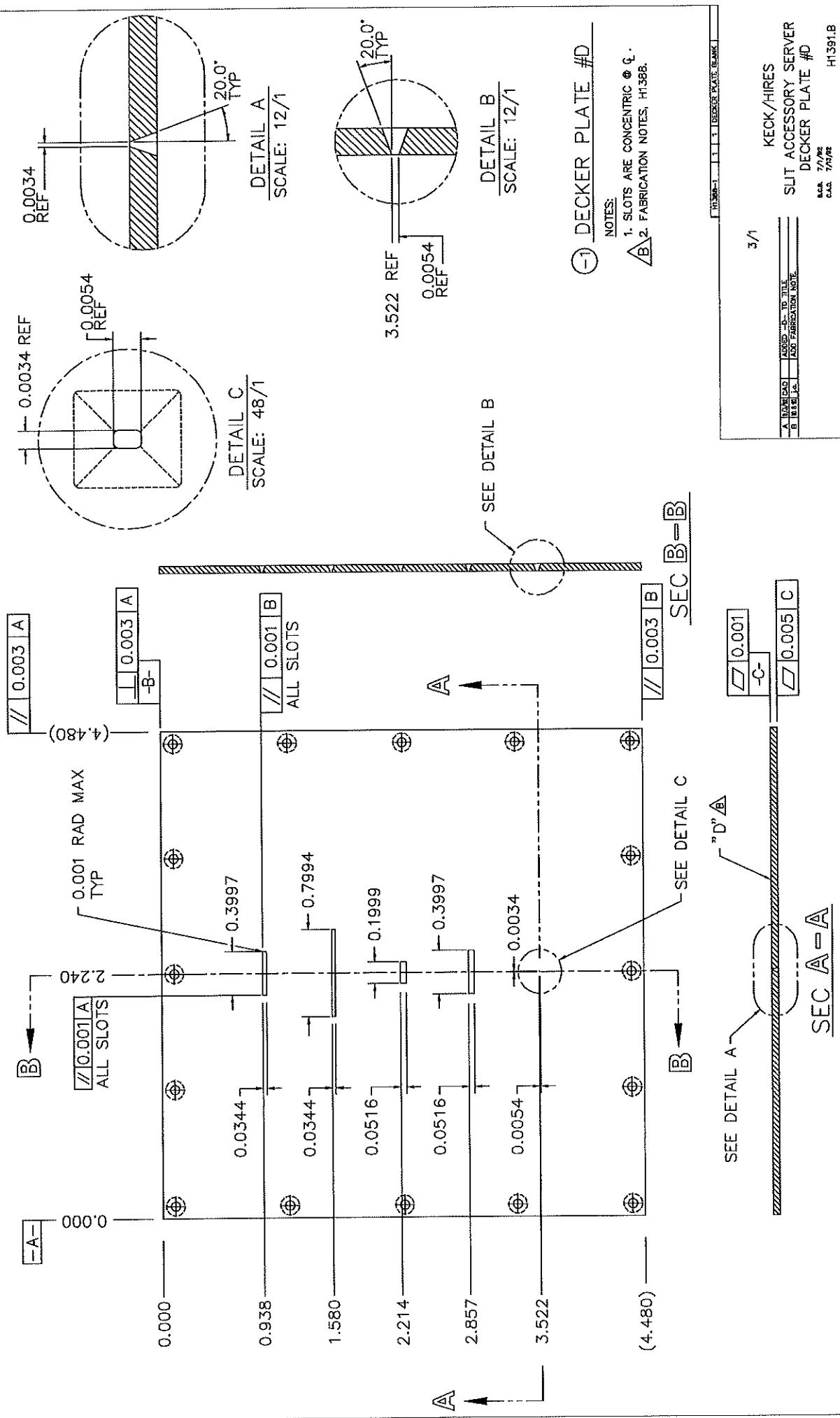
B

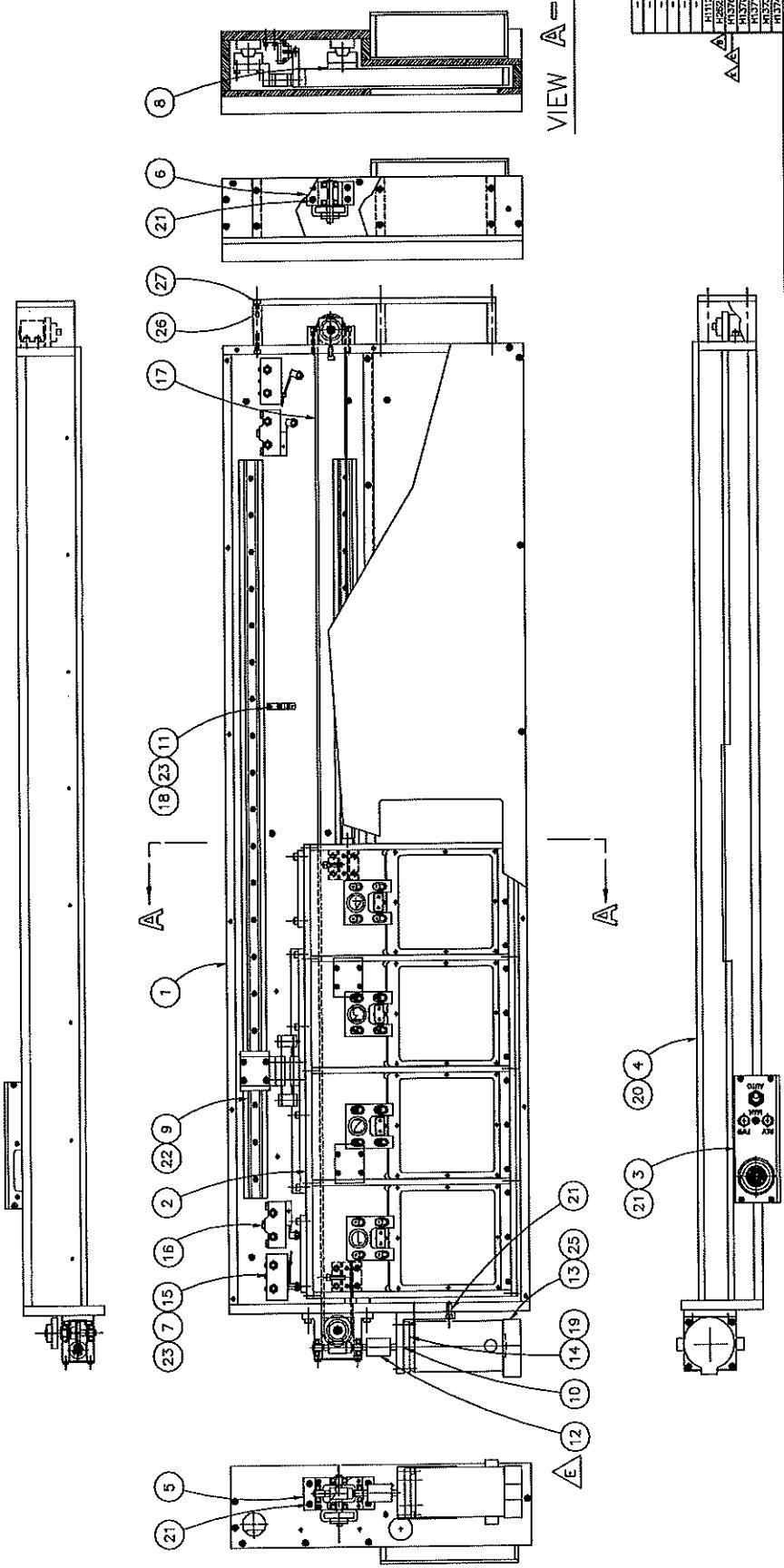
0.001

A









VIEW A-A

-	20	A/R SFR B-32 INC SHS
-	12	A/R SFR B-32 INC SHS
-	19	A/R SFR 4-40 INC SHS
-	7	LINK BELT SERO 2D1B-20
-	15	2 LIMIT SWITCH SPDT NOTE 1
-	15	2 LIMIT SWITCH SPDT NOTE 2
-	14	1 COMPTR NEPA 23
A-A	13	1 MOTOR NOTE 1
A-A	12	1 COUPLING SYSTEM
A-A	10	1 BEARING, TAPERED
A-A	9	2 SIDE SPACER
A-A	8	1 SIDE SPACER ALUM
H1362-2	27	1 COVER PLATE ALUM
H1362-1	25	3 SUPPORT ALUM
H1362-1	24	6 HEX NUT, 6-32 UNC
-	24	1 FEUDAL TENV OPR 920-175
-	23	1 HITCH-1
-	22	1 KNEE-1
-	21	1 A/R SFR B-32 INC SHS
	1	1 HOUSING ASSY

(-1) SLIT ACCESSORY SERVER

1 REQD

NOTES:

1. CALL 50/1000
2. MICRO, DT-2R/22-A7
3. MICRO, BZ-RWB622-A2
4. HEWLETT PACKARD HED 6000 SERIES

KECK/HIRES
SLIT ACCESSORY SERVER
ASSEMBLY

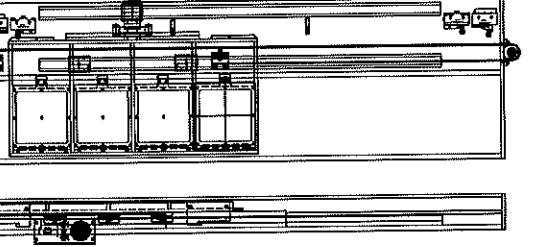
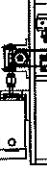
H1378.F

BLW/CW
C.A.D.
C.P.R./P

1/2

7.5° SENDS A 15°
BEAM TO TV GUIDER

7.5°

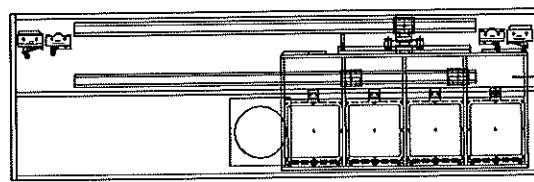
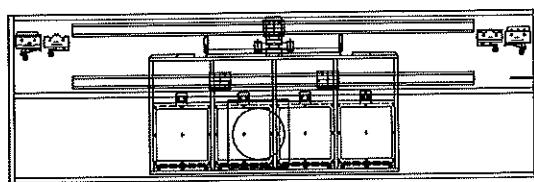
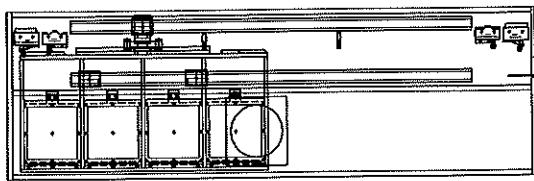


DECKER PLATES:
DECKER PLATE MATERIAL IS 420 STAINLESS STEEL.
PLATES ARE FINE GROUND, THEN EDM MACHINED,
AND OPTICALLY POLISHED TO A FEW WAVES.

420 STAINLESS:

CARPENTER TECHNOLOGY
2263 NATIONAL AVE "A"
HAYWARD, CA 94545-1715
800-522-5077

S.A.S. SHOWN 1/4 SCALE



DECKER CARRIAGE
IN STOWED POSITION
CENTER OF TRAVEL
AT UPPER LIMIT



1 CT X 1 REV/4000 CT X 1 REV OUT/60 REV IN X 2 (.6366)/REV OUT = 0.000016666 INCH

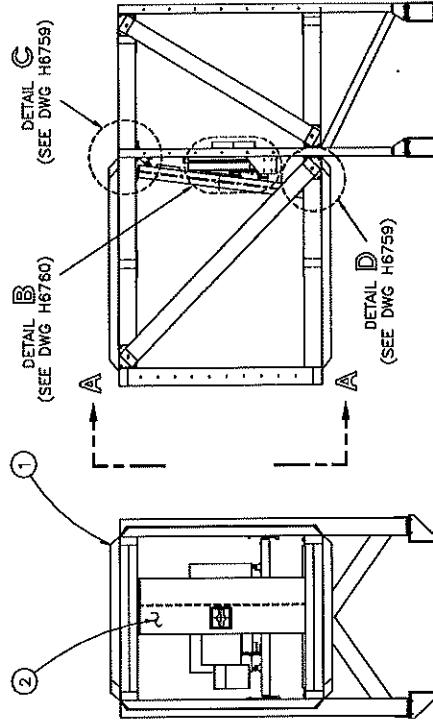
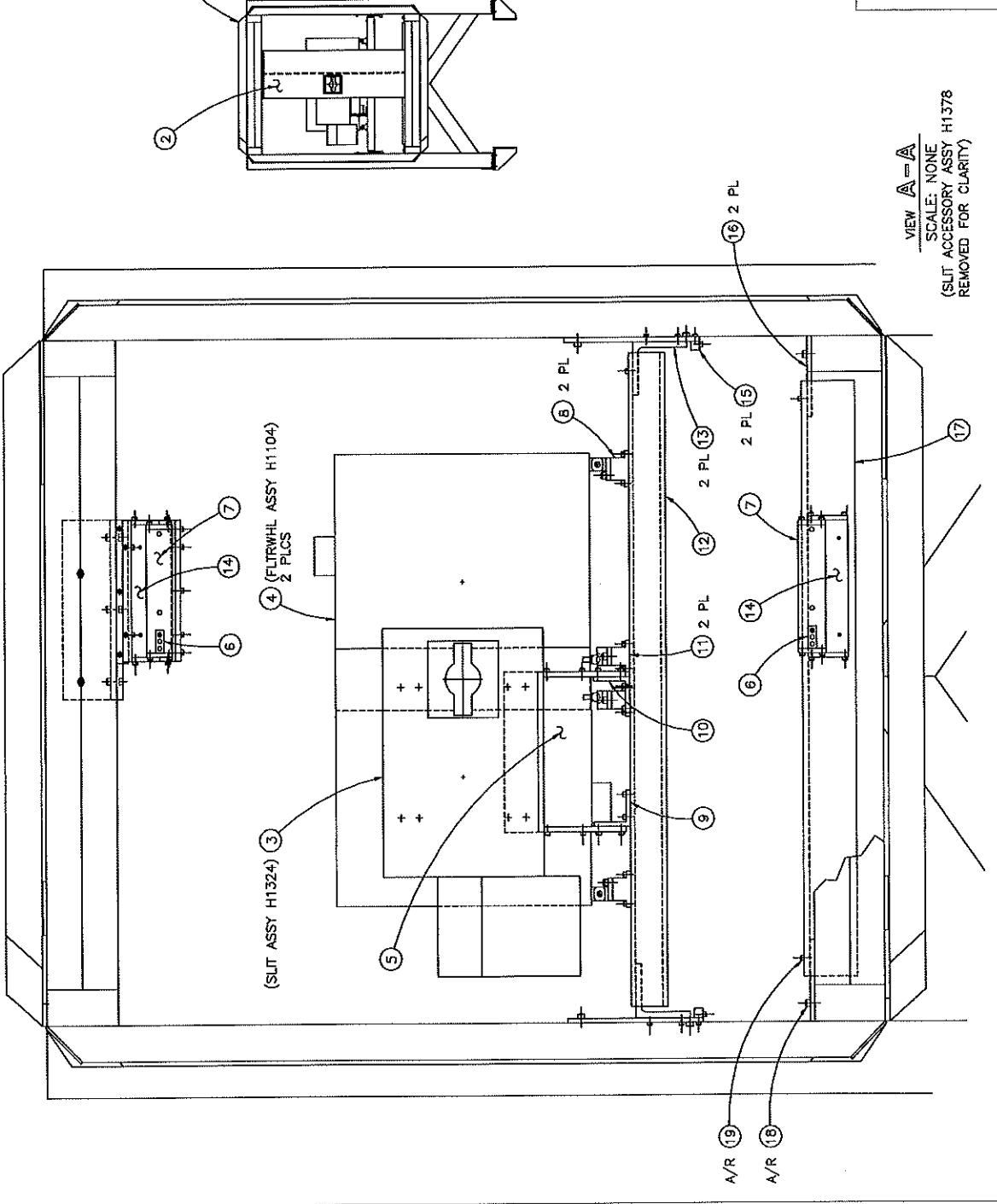
OR 0.000016666 INCH/CT.
OR @ 60,000 CTS/INCH

IF SERVO LOOP IS GOOD TO 2 COUNTS,
RESOLUTION IS 0.000033333 INCHES.

RESOLUTION AND REPEATABILITY HAVE BEEN TESTED MECHANICALLY TO 0.00005 INCHES.
BETTER TESTING WILL BE PERFORMED WITH THE GUIDER CCD TV.
PLATE SCALE AT SLIT ACCESSORY SERVER IS 1.375 ARC-SEC/mm.

TRAVEL OF CARRIAGE IS 18"
APERTURE PLATES ARE 4.5" X 4.5" X 0.063" THICK
USABLE APERTURE AREA IS 4" X 4"

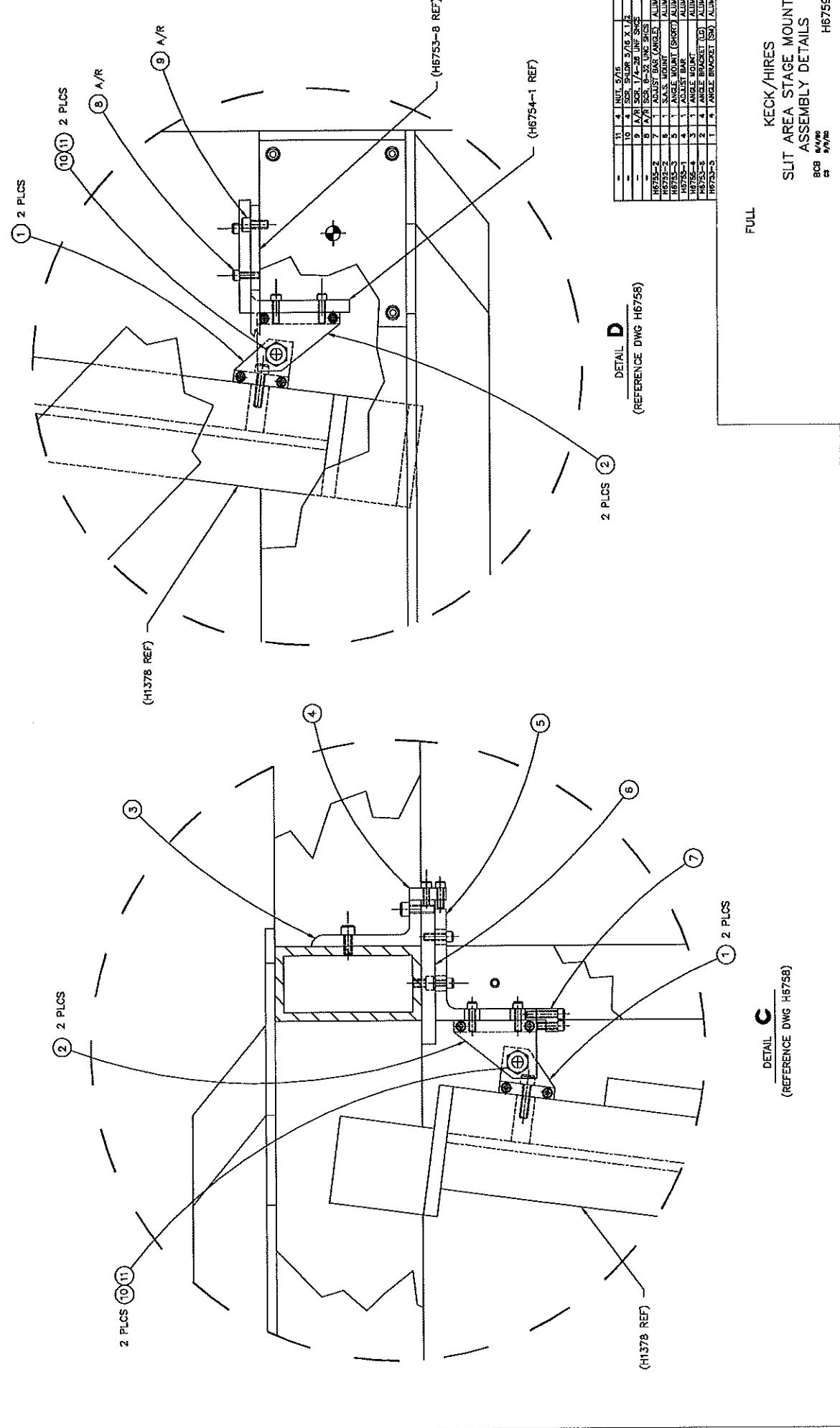
KECK/HIRES
SLIT ACCESSORY SERVER
DATA AND GEAR REDUCTIONS
H1387.A
04/20/98

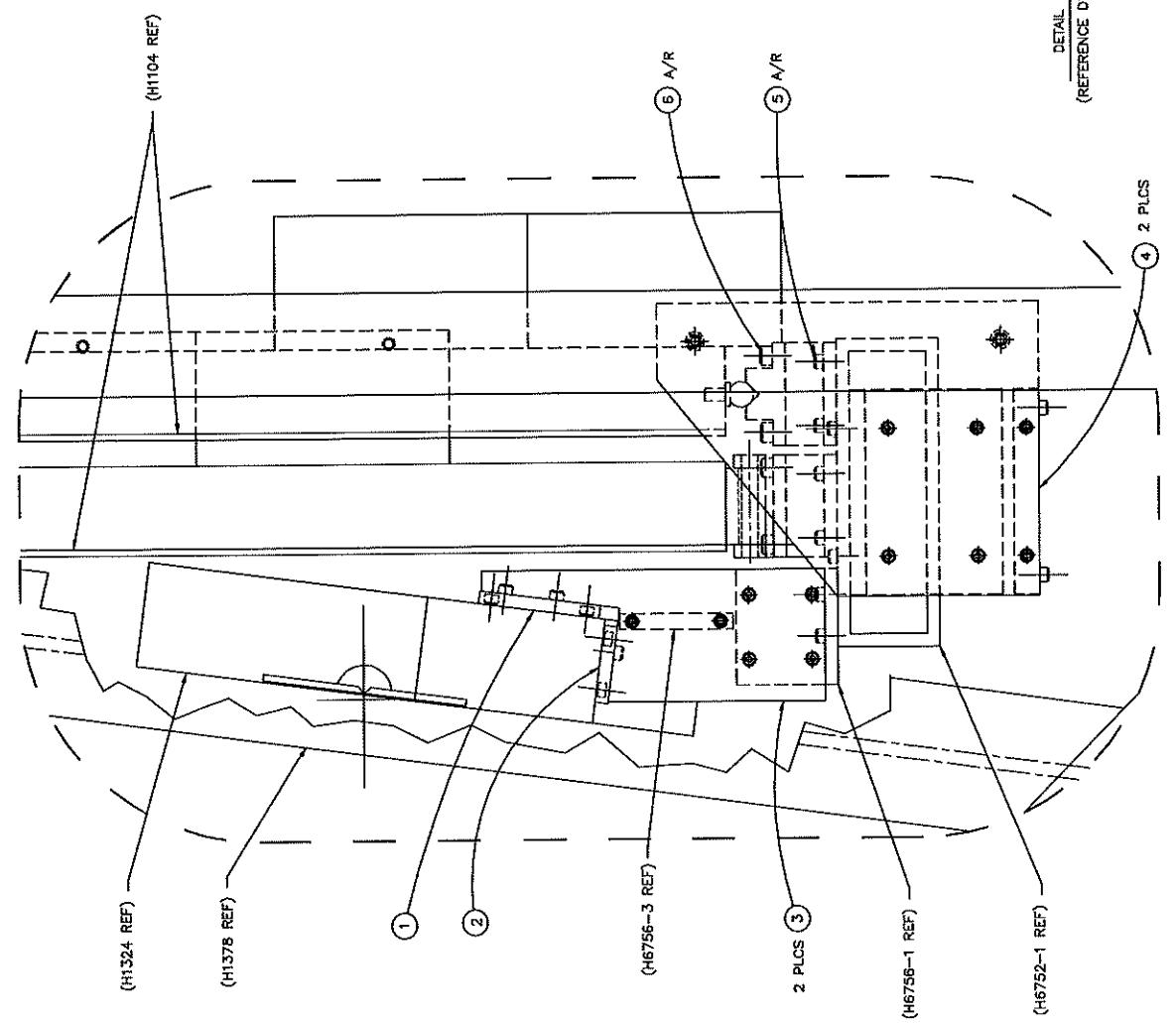


-	19	A/R	SOR. Ø-32 LINE SPOTS
-	15	A/R	SOR. 1/4-20 UNF SPC
H6754-1	17	1	ANGLE MOUNT
H6754-8	15	2	SUPPORT PLATE
H6754-2	15	2	ADJUST BAR
H6754-3	14	2	SIDE MOUNT
H6754-4	13	2	ANGLE SUPPORT
H6754-5	12	1	CROSS STND
H6754-6	10	1	ANGLE MOUNT (RIGHT)
H6754-2	9	1	ANGLE MOUNT (LEFT)
H6754-1	8	2	T-N. STAND
H6754-4	7	2	FLANGE (SMALL)
H6754-7	6	2	STANDOFF
H6754-3	5	1	CROSS SUPPORT
H6754-8	4	2	12 HOLE ULTR. WHEEL ASSY
H6754-9	3	1	SUIT ACCESSORY ASSY
H6754-10	2	1	SUIT ACCESSORY
H6754-11	1	1	SUIT AREA STRUCTURE ST.

NOTED

KECK/HIRES
SLIT AREA STAGE MOUNTS
ASSEMBLY
B/CB A/C/A/B
8/3/20
H6758





-	6	A/R	SCH. 1A-22 UNF SPGS.
-	5	A/R	SCH. 8-32 UNF SPGS.
H6754-3	4	2	SIDE SUPPORT
H6754-4	3	2	SIDE SUPPORT
H6755-5	2	1	MOUNT SUPPORT (SM)
H6755-6	1	1	MOUNT SUPPORT (LG)

DETAIL B
(REFERENCE DWG H675B)

FULL

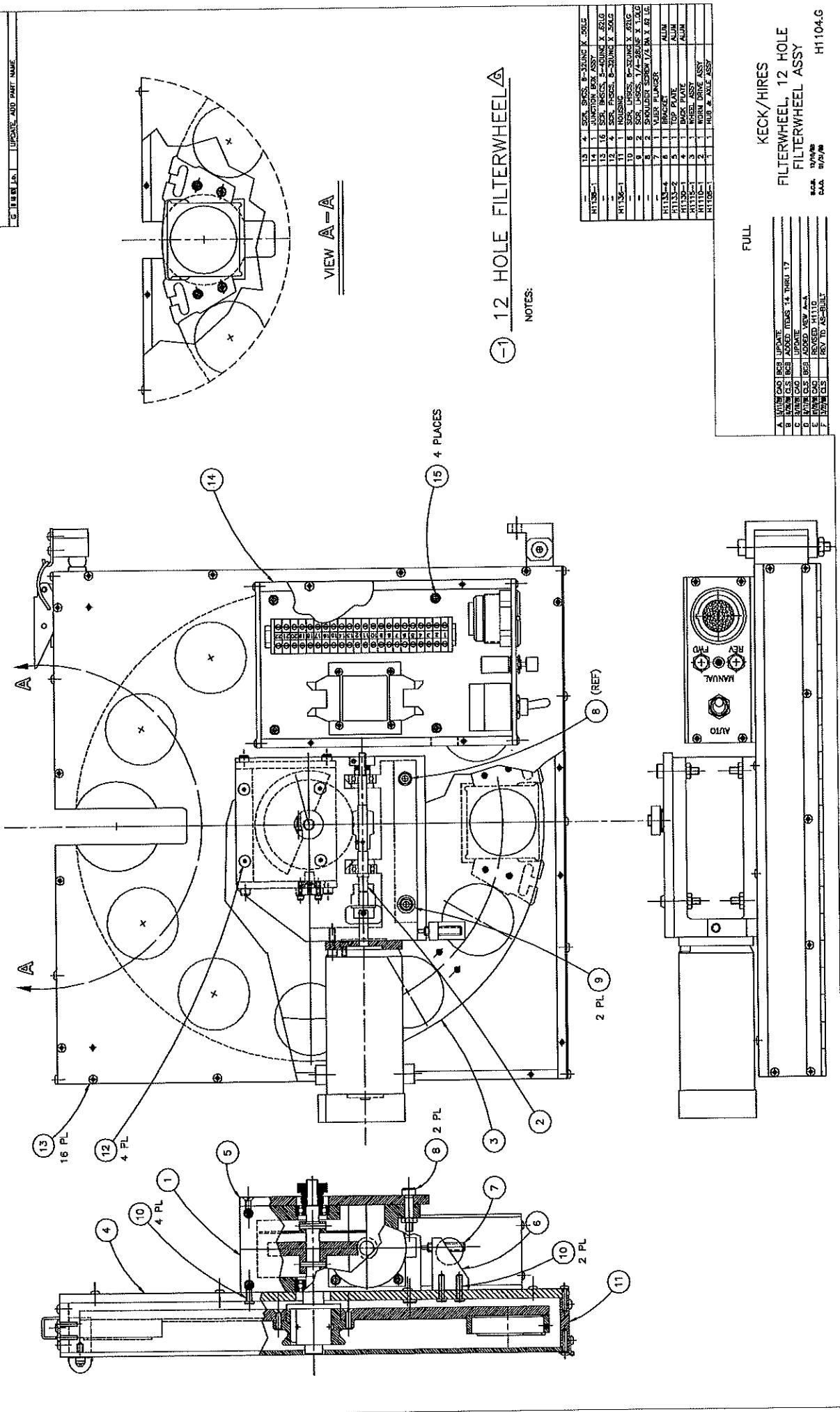
KECK/HIRES
SLIT AREA STAGE MOUNTS
DETAILS

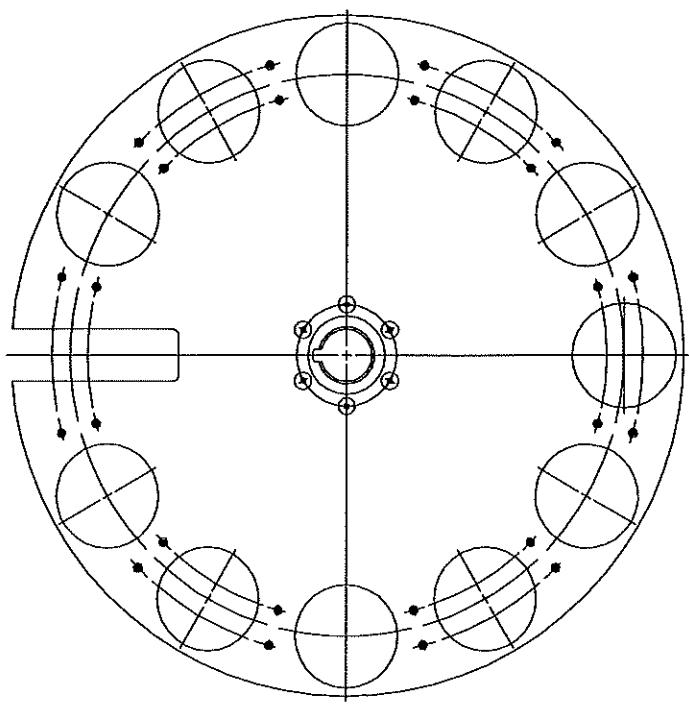
H6760
9/24/80
9/26/80

Appendix N List of Drawings — Filterwheels

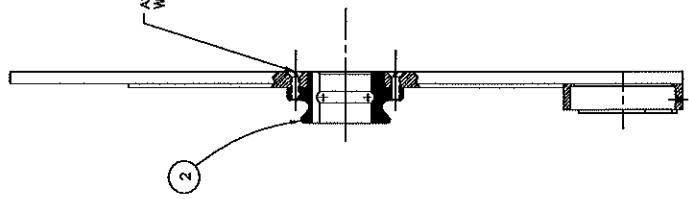
1. H1104 12-Position Filterwheel Assembly
2. H1115 Filterwheel
3. H1114 Filterwheel Drive Schematic
4. H1110 Filterwheel Drive Motor Assembly
5. H6758 Slit Area Stage Mountings — Assembly
6. H6760 Slit Area Stage Mountings — Details

G R E V. J.D. UPDATE AND PART NAME





ASSEMBLY HUB
WITH "NEVER SEIZE"



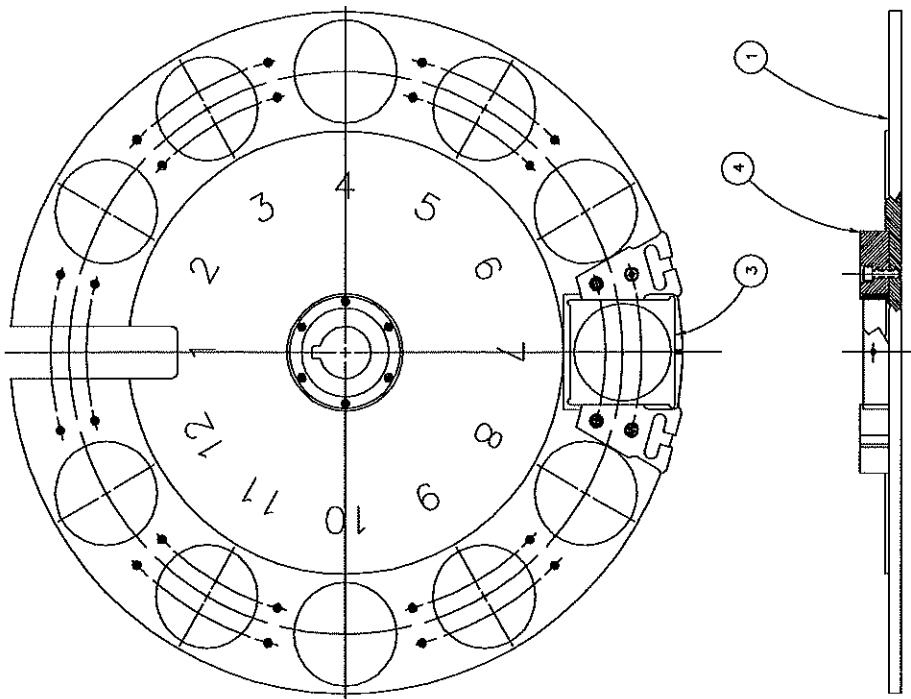
① FILTERWHEEL ASSY

1 REQ'D

H1120-1	4	A/R	KEEPER	ALUM
H1117-2	3	A/R	FILTER HOLDER	ALUM
H1117-1	2		HUB	ALUM
H1118-1	1		FILTERWHEEL 12 HOLES	ALUM

KECK/HIRES
FILTERWHEEL 12 HOLE
FILTERWHEEL ASSY

B.C.B. 12 05 89
C.A.S. 01 20 86
H1115.E



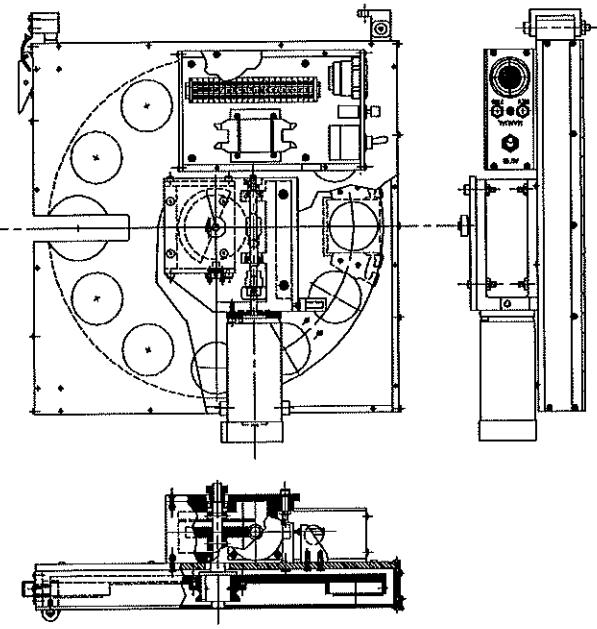
THEORETICAL POSITIONING RESOLUTION:
MOTOR ENCODER 4000 CTS./REV
GEAR REDUCTION 30:1
FILTER POSITIONS 12 (12/REV)

1 CΤ X 1 REV/4000 CTS X 1 REV OUT/30 REV IN X 360°/REV = 0.003°
OR 0.003°/CT
OR 333 CTS/DEGREE

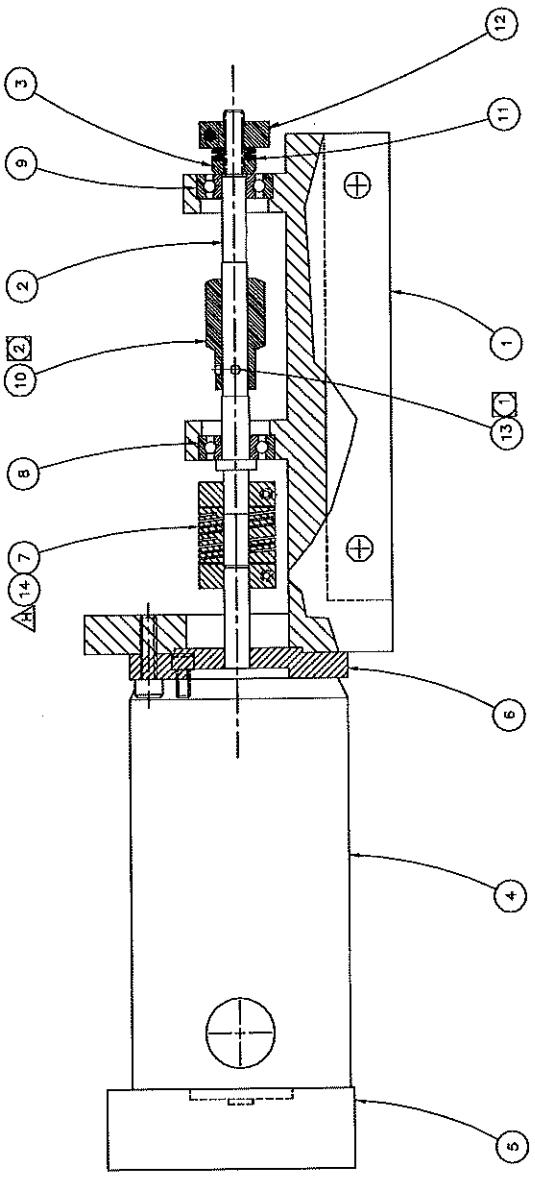
ACTUAL RESOLUTION IS DOMINATED BY FREE PLAY BETWEEN THE
HUB AND THE FILTERWHEEL.

FILTER HOLDER IS H1117-2, AND WILL HOLD A 2" DIA. OR 2" SQUARE FILTER.
FILTER THICKNESS CAN BE AT LEAST 0.400" IN THIS HOLDER.
THICKER FILTERS WILL REQUIRE A CUSTOM HOLDER.

FILTERWHEEL ASSY. SHOWN 1/2 SCALE



A. [REDACTED] GENERAL SECTION
KECK/HIRES
FILTERWHEEL, 12 HOLE
DATA AND GEAR REDUCTION
H1114.A
DRAFT



-1 WORM DRIVE ASSY

REO, D

NOTES: PIN GEAR IN PLACE AFTER ASSEMBLY AND ALIGNMENT.

2. LUBRICATE WITH NYGEL
HAGLUND #60 / 1000. SEE H2623-1
3. HEWLETT-PACKARD HEADS-B6000 SERIES
4. BEARING #CO415-1 WITH KEYWAY ADDED
5. BEARING #6017-0RS
6. SKF #626-2RS
7. BELLEVILLE SPRING WASHER #80437-016-S

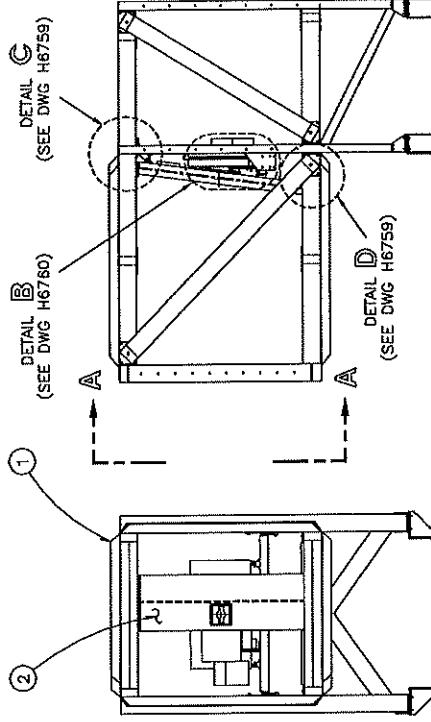
H. 3.5 SH. MOVE NOTES WERE IN PARTS LIST.
ADD KEYWAY TO COUPING. 1/16 KENS

2/1

KECK/HIRES

FILTERWHEEL, 12 HOLE
WORM DRIVE ASSY

三



-	19	A/R	SCH. 6-SZ UNI SPCE
-	18	A/R	SCH. 7-2B UNI SPCE
H6754-1	17	1	ANGLE MOUNT ALUM
H6754-2	16	2	SUPPORT PLATE ALUM
H6754-3	15	1	SUPPORT PLATE ALUM
H6754-4	14	2	SUPPORT PLATE ALUM
H6754-5	13	2	ANGLE SUPPORT ALUM
H6754-6	12	1	SUPPORT ALUM
H6754-7	11	2	F.W. STAND ALUM
H6754-8	10	1	ANGLE MOUNT (RIGHT) ALUM
H6754-9	9	1	ANGLE MOUNT (LEFT) ALUM
H6754-10	8	2	F.W. STAND ALUM
H6754-11	7	2	FLANGE (SMALL) ALUM
H6754-12	6	2	STANDOFF ALUM
H6754-13	5	1	CROSS SUPPORT ALUM
H6754-14	4	2	12-HOLE FLANGE ALUM
H6754-15	3	1	SUIT ACCESSORY ALUM
H6754-16	2	1	SUIT ACCESSORY ALUM
H6754-17	1	1	SUIT AREA STRUCTURE ST
			H6771

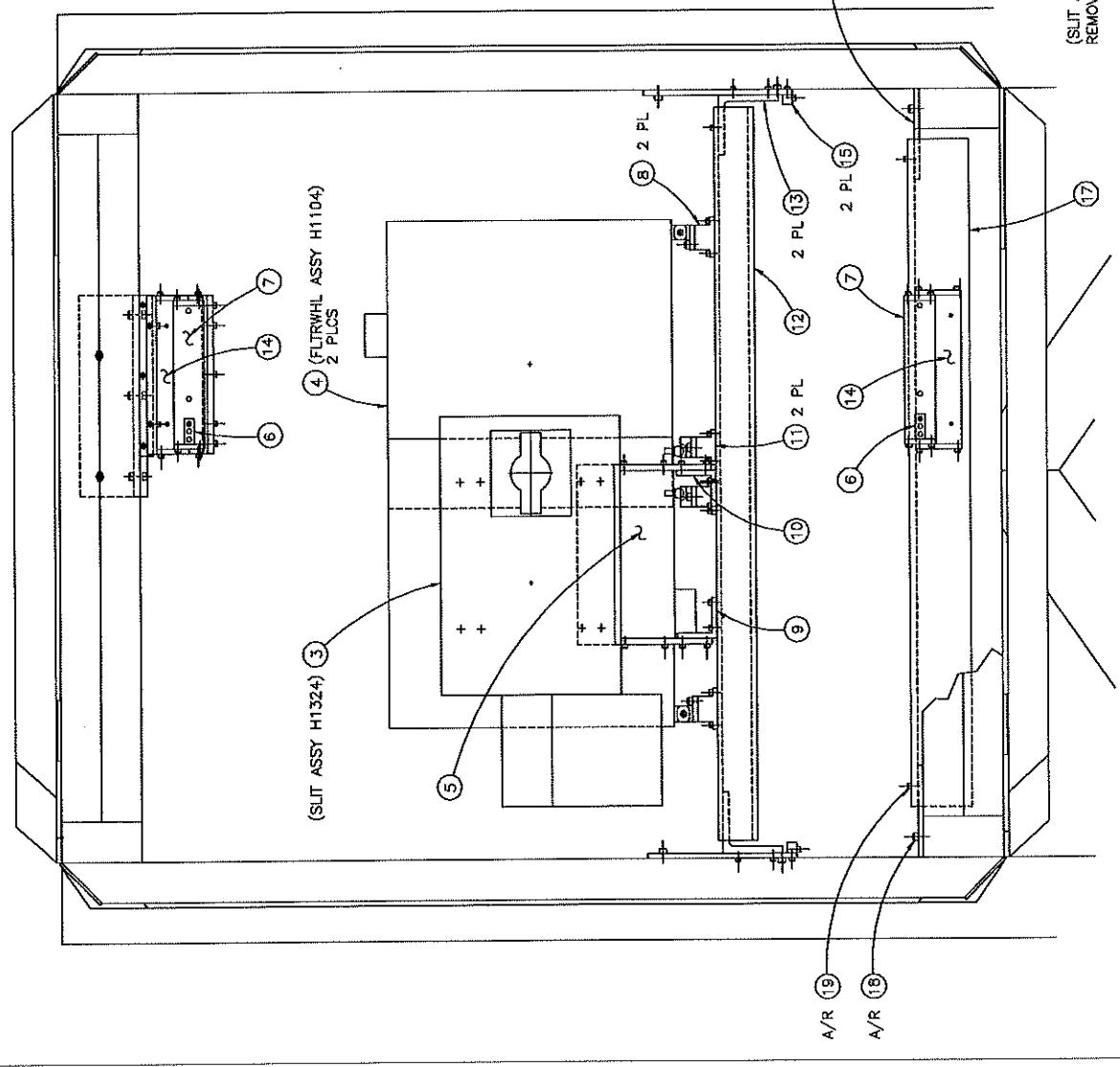
NOTED

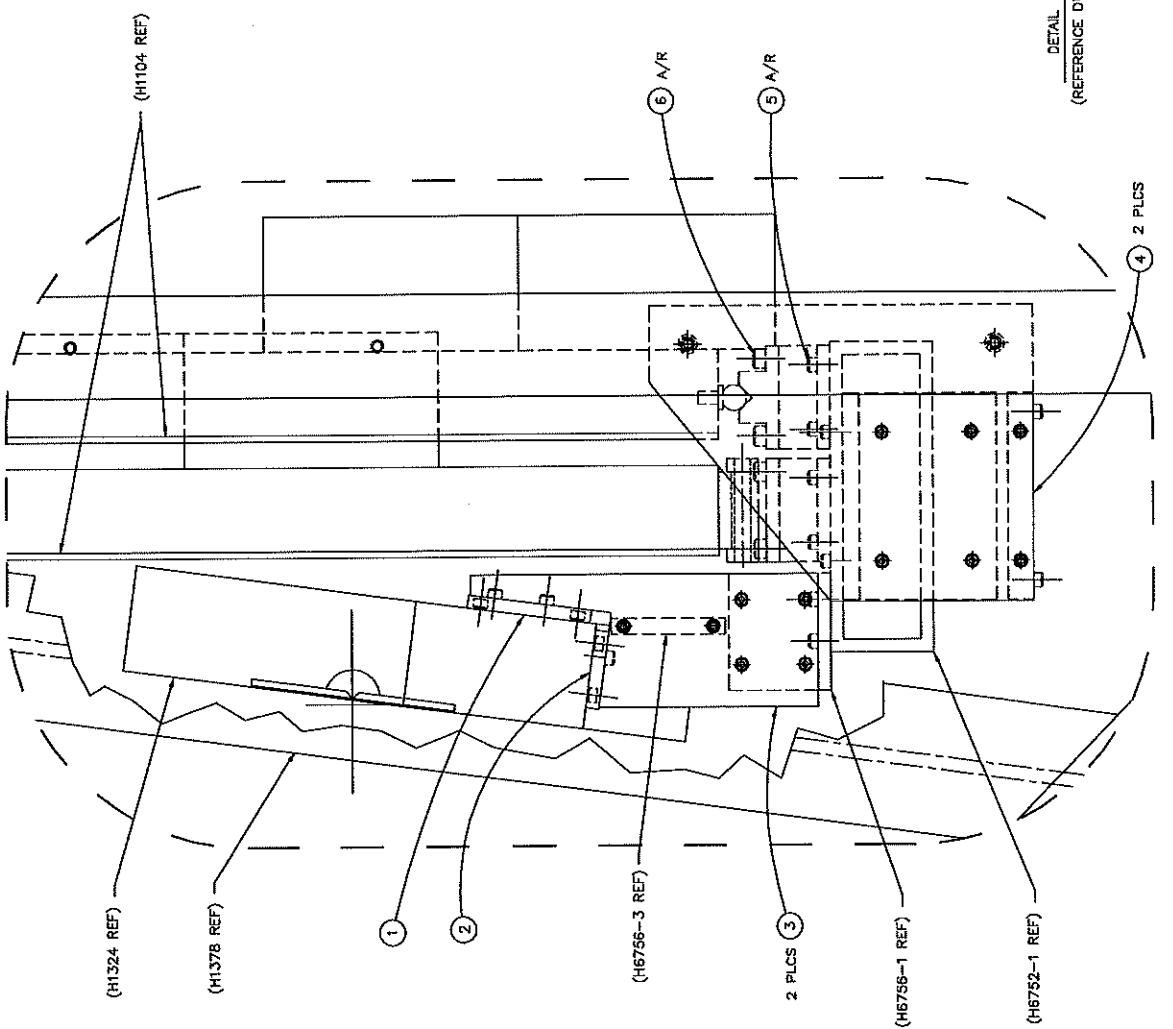
KECK/HIRES
SLIT AREA STAGE MOUNTS
ASSEMBLY

B6B 9/2/06
AS 9/3/06

H6758

VIEW A → A
SCALE: NONE
(SLIT ACCESSORY ASSY H1378
REMOVED FOR CLARITY)





-	6	A/R	SCH. 1/4-28 UNF SPLCS
-	5	A/R	SCH. 6-32 UNF SPLCS
H6754-3	4	2	SEE SUPPORT
H6755-4	3	2	SEE SUPPORT
H6756-5	2	1	MOUNT SUPPORT (S)
H6756-6	1	1	MOUNT SUPPORT (C)

FULL

KECK/HIRES

SPLIT AREA STAGE MOUNTS

DETAILS

B6760

B6760

9/2/80

Appendix O List of Drawings — Shutter

1. H1203 Shutter and Filterwheel

(-1) SHUTTER

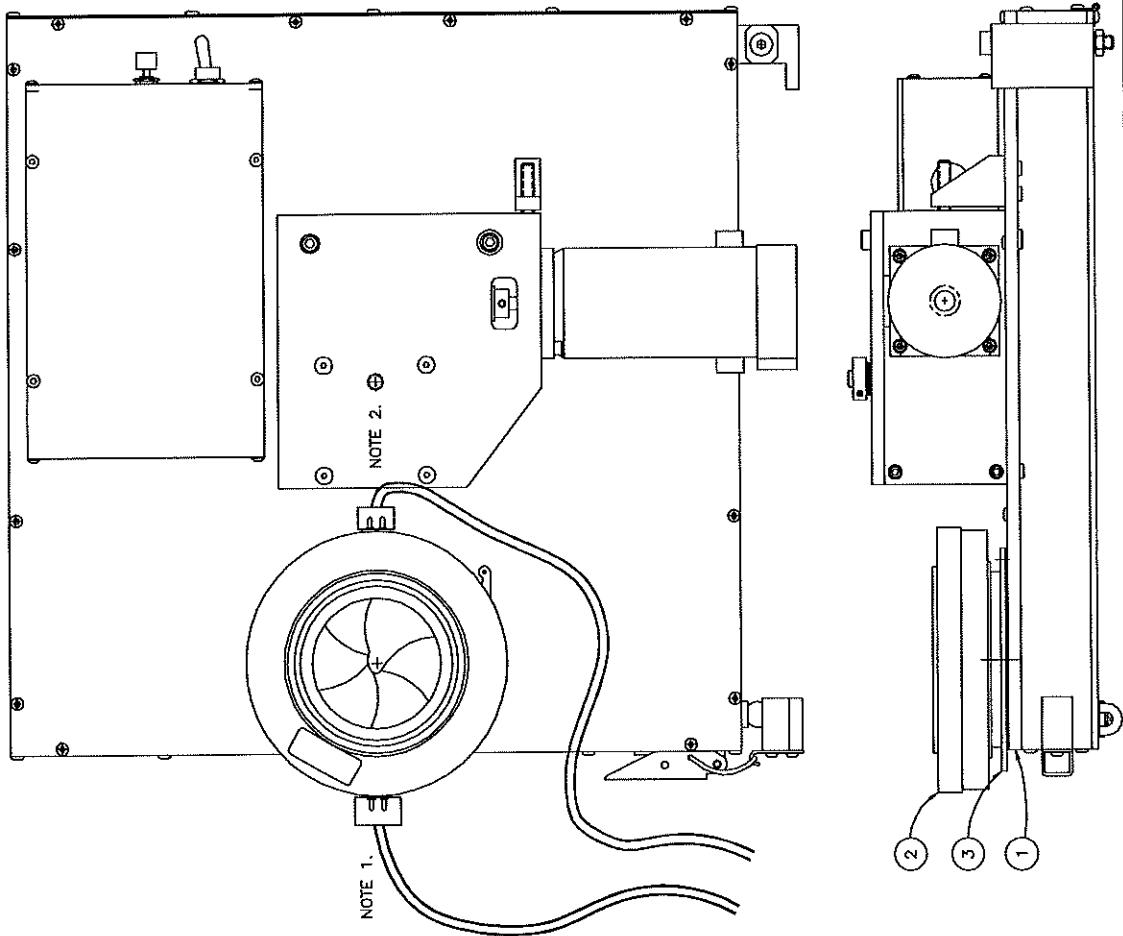
1 REQD
NOTES:
 △1. ACTUATE SOLENOID.
 △2. SENSE (REED SWITCH)

PART NO.		QTY	DESCRIPTION	ALIN.
H1203A-5	-	3	SOLVENT ACTUATING SOLENOID	
H1203A-2	-	2	1 TO 2 POSITION SWITCH	
H1203A-1	-	1	FLUTTERWHEEL ASSY	
H1203A-3	-	1		

FULL
KECK/HIRES
FILTERWHEEL W/ SHUTTER
ASSEMBLY

▲ [SUSP] [SUSP] AND STONE AND AERATION MEDIUM

SCALE: 12 10 80
C.A. 07 29 92
H1203A



Appendix P List of Drawings

— TV Autoguider

1. H1401 TV Cooling Assembly
2. H1406 TV Cooling Installation Details
3. P1003 HIRES GUIDER FORMAT
4. H8105 Lens Drive Assembly
5. H8119 Lens Drive Schematic
6. H8326 8-Position Filterwheel Assembly
7. H8325 Filterwheel
8. H1110 Filterwheel Drive Motor Assembly
9. H8001 TV Guider Assembly
10. H8002 TV Guider Mount Assembly

(-1) T.V. COOLING ASSEMBLY

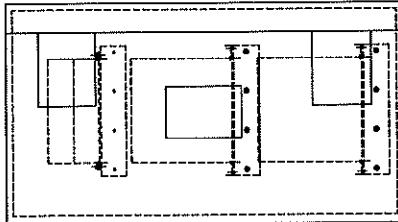
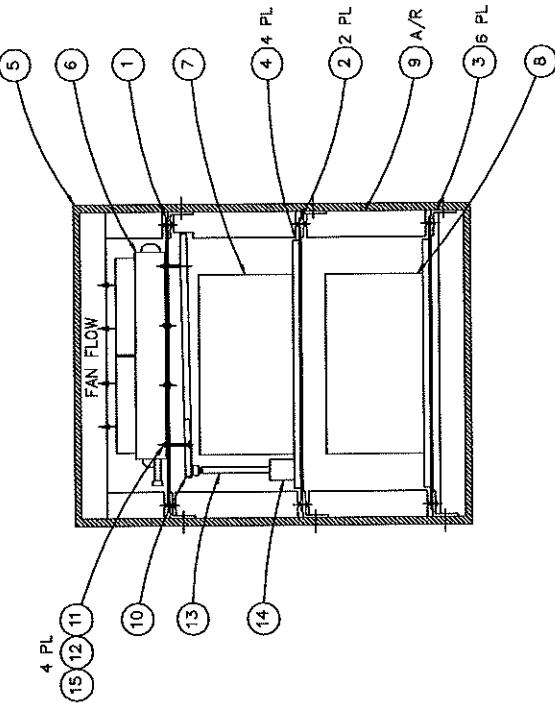
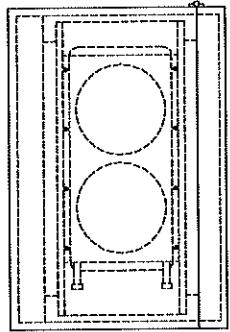
1 REQ'D

NOTE:

1. LYTRON INC.
DRAGON COURT
WOBBURN, MA 01801
617-933-7300
2. WARD-BAGBY PLASTICS
1310 PIPER DR.
MILPITAS, CA
409-262-2111

3. WIRING & PLUMBING HOLES
LOCATED AS REQ'D.

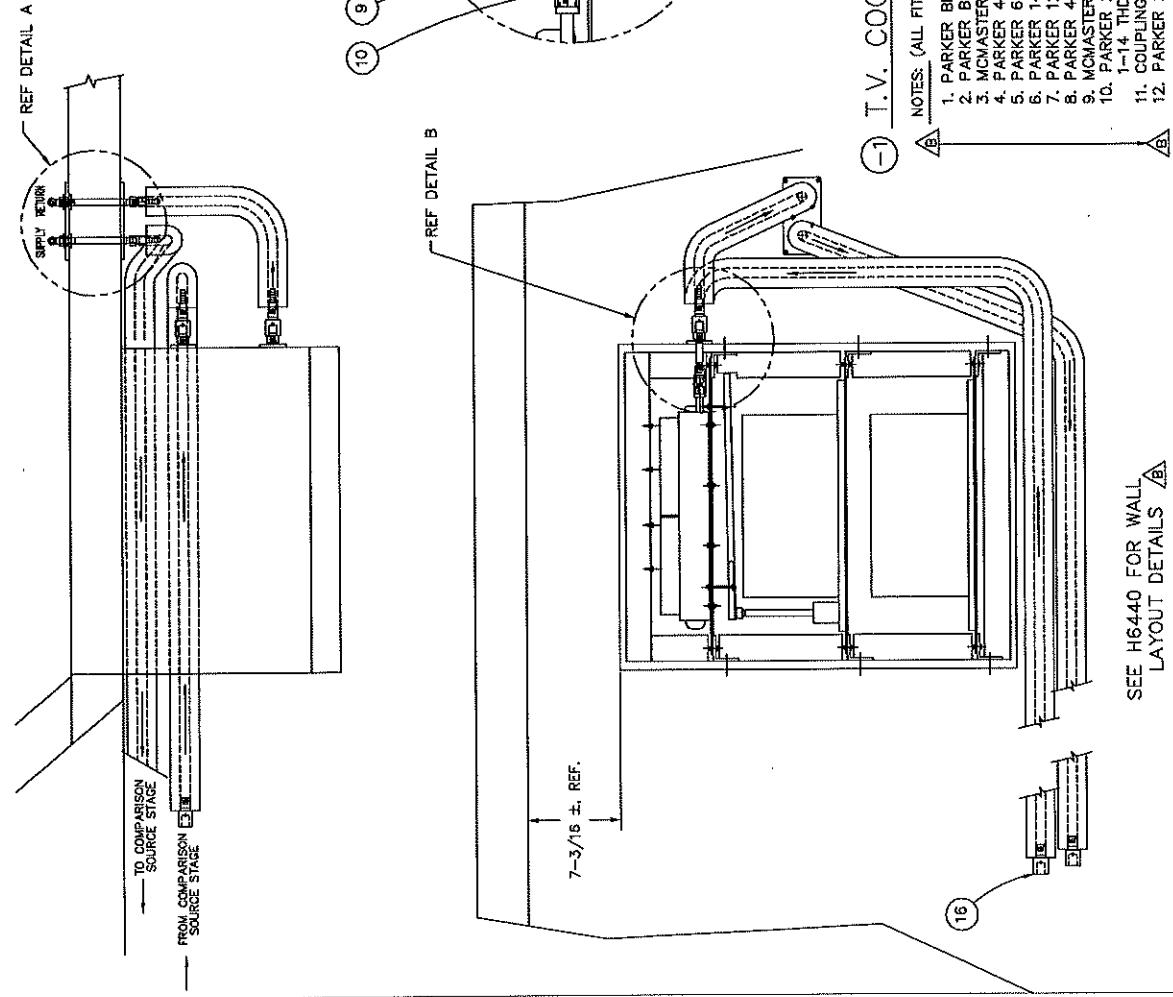
-	15	4	WIRE, #22
-	14	1	CONTAINER
-	13	2	SCREW, #22 RED
-	12	2	SCREW, #22 SILVER
H1401-A	11	1	CONDENSATION TRAP
-	10	2	INSULATION, ALUM
-	9	1	COOLANT, RADIATOR
-	8	1	CAMERA, ELECTRIC
-	7	1	ELECTRONICS
-	6	1	HEAT EXCHANGER
H1401-A	5	1	CONTAINER
H1401-A	4	4	UP
H1401-A	3	6	RIGHT
H1401-A	2	2	SELF
H1401-A	1	1	SHELF



1/4

KECK/HIRES
T.V. COOLING
ASSEMBLY

B.C.A. 12/0/91
C.A. 2/2/92
H1401.A



(-1) T.V. COOLING INSTALLATION

NOTES: (ALL FITTINGS ARE BRASS)

1. PARKER BH3-60, FEMALE 3/8 NPT COUPLER
2. PARKER BH3-61, FEMALE 3/8 NPT NIPPLE
3. MCMASTER-CARR 456BK155, X 3" LONG
4. PARKER 46F-8-6, 3/8 NPT FEMALE TO 1/2 TUBE
5. PARKER 660FH0-B1, 1/2 FLARE UNION
6. PARKER 46HBLSV-6-6, 3/8 HOSE BARB TO FEMALE SAE
7. PARKER 12SHBL-6-6, HOSE BARB
8. PARKER 46F-6-6 (3/8 NPT TO 3/8 SAE)
9. MCMASTER-CARR 456BK159, 3/8 NIPPLE X 5" LONG
10. PARKER 207AEE-6, 3/8 NPT TO 3/8 NPT,
- 11-14 THD O.D. x 1.31 LOA
11. COUPLINGS CO. 49-E6 (3/8 FLARE TO 3/8 NPT)
12. PARKER 2200P-6-6, 3/8 NPT 90° ELBOW

SEE H6440 FOR WALL
LAYOUT DETAILS

-	17	3/8 PIPE NIPPLE, BRASS
-	15	2 COUPLING, QUICK CONNECT
-	14	1 UNION
-	13	1 COUPLING
-	12	1 COUPLING, QUICK
-	11	1 COUPLING, QUICK
-	10	1 COUPLING, QUICK
-	9	1 COUPLING, QUICK
-	8	1 COUPLING, QUICK
-	7	1 COUPLING, QUICK
-	6	1 COUPLING, QUICK
-	5	1 COUPLING, QUICK
-	4	1 COUPLING, QUICK
-	3	1 COUPLING, QUICK
-	2	1 COUPLING, QUICK
-	1	1 COUPLING, QUICK
-	10	1 FLARE UNION, 1/2"
-	9	1 Female CONNECTOR
-	8	1 Female CONNECTOR
-	7	1 NIPPLE, 3/8
-	6	1 COUPLING, QUICK
-	5	1 COUPLING, QUICK
-	4	1 Female CONNECTOR
-	3	1 COUPLING, QUICK
-	2	1 COUPLING, QUICK
-	1	1 COUPLING, QUICK

SCALE: 1/2

DO NOT SCALE FITTINGS

SCALE: 1/2

DO NOT SCALE FITTINGS

SCALE: 1/2

DO NOT SCALE FITTINGS

1/4

KECK/HIRES

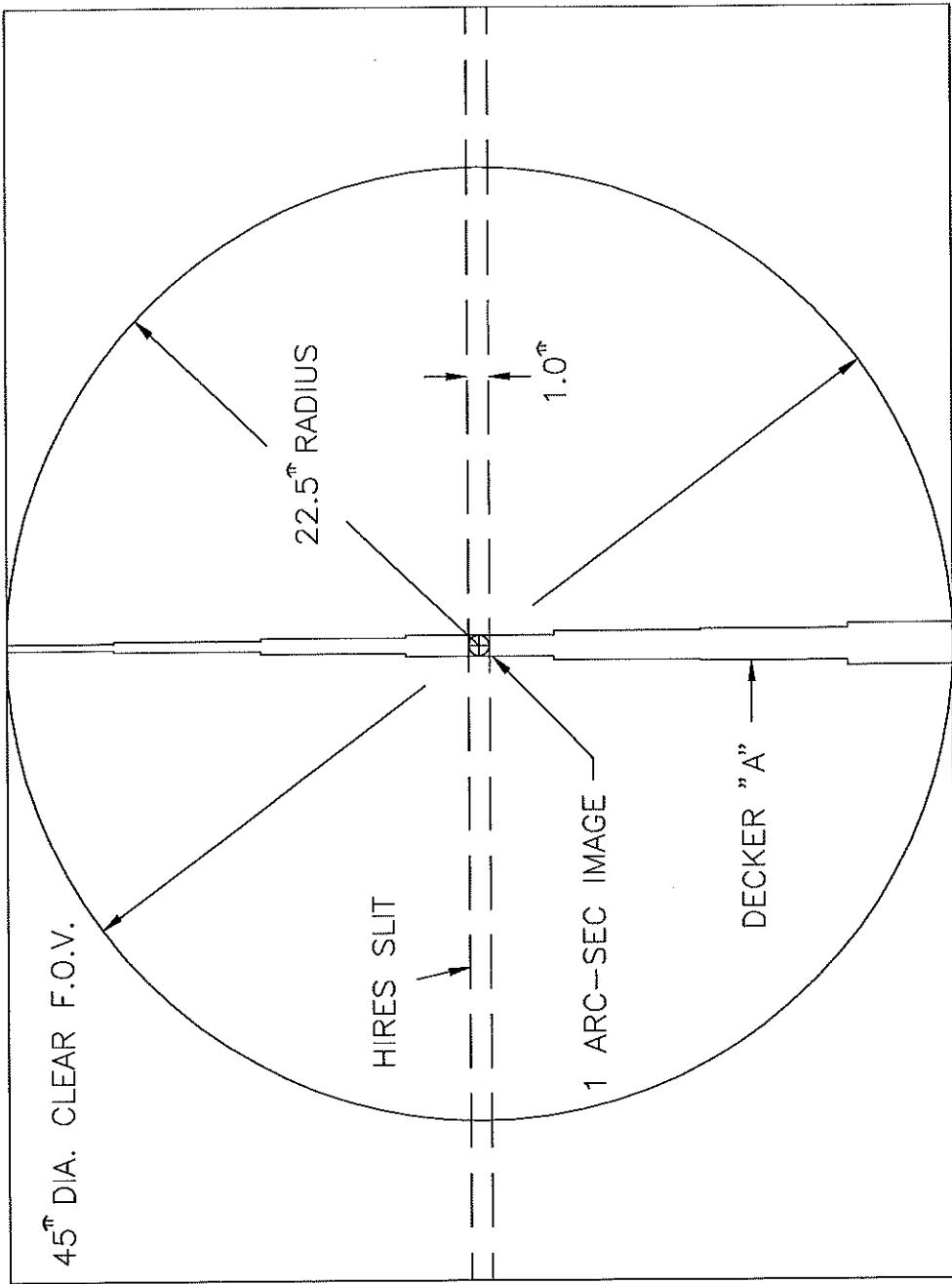
T.V. COOLING

INSTALLATION DETAILS

H1406.B

1/16
2/25/92

B.C.A.
2/25/92



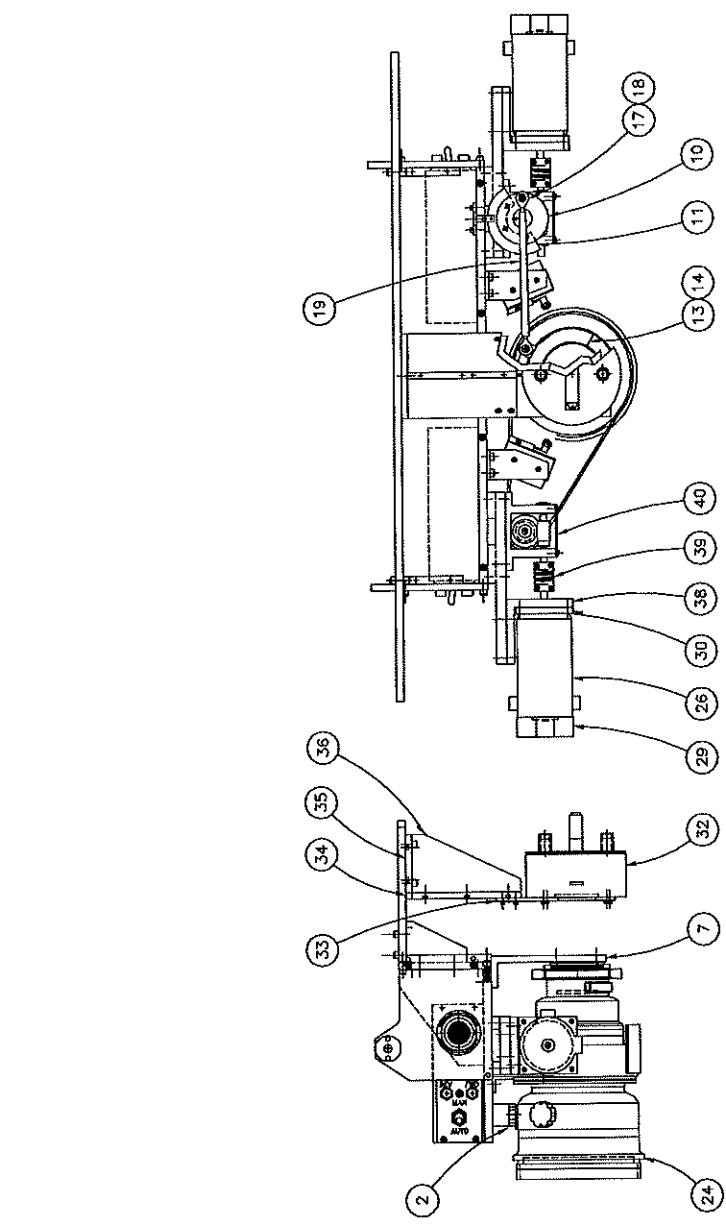
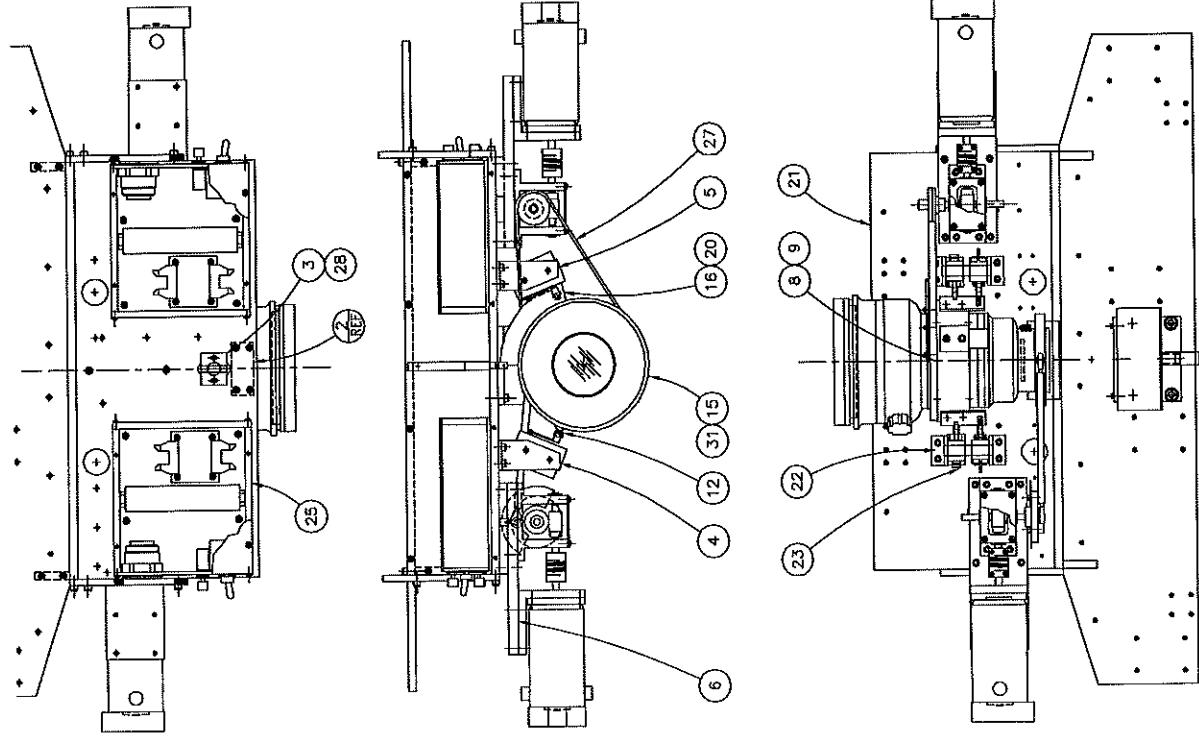
45° (288 PIXELS)

60° (384 PIXELS)

HIERES GUIDER FORMAT

PHOTOMETRICS CH250 LC CAMERA HEAD
THOMSON TH7883 CCD, 23μ PIXELS
1.379° /mm AT SLIT
6.78°+/- 0.26° /mm AT TV CCD
0.156°+/- 0.006° /PIXEL

/H/PSTUFF/P1003.DWG
4/16/92 BCB



H8105-1	40	2	GEAR, RED. ASSY.	H8115-5	10	3	SPACER	ALUM
H8105-1	39	2	GEAR, RED. DISC, ALUM	H8115-5	10	1	LINK SHAFT	ALUM
H8105-1	38	2	TRIPOD, MOTOR	H8115-5	18	1	NUT	SST
H8105-1	37	1	TRIPOT	H8115-5	17	2	BALL END, LINK	ALUM
H8110-2	36	1	WEED	H8115-5	16	1	FIDUCIAL, DISC	ALUM
H8110-1	35	1	PLATE	H8115-5	15	1	GEAR, MODIFIED, BEND	ALUM
H8117-2	34	1	PLATE	H8115-5	14	1	OPENING, RING CLAMP	ALUM
H8117-2	33	1	HE, PUL. - LC, CANDIDA, HEAD, ALUM	H8114-1	13	1	OPENING, RING CLAMP	ALUM
H8205-1	32	1	LC, CANDIDA, HEAD	H8114-1	12	1	SWEEP, LIMIT SWITCH	ALUM
H8205-1	31	1	TARING BELT, BERIC	H8113-2	11	1	FIDUCIAL, DISC	ALUM
H8112-1	30	2	ADAPTER, NEMA, 23	H8113-2	10	1	SWEEP, LIMIT SWITCH	ALUM
H8282-1	29	2	ENCODER, SERVOM, DK	H8113-2	9	1	SWEEP, LIMIT SWITCH	ALUM
H8110-1	28	3	SWING, SERVOM, DK	H8113-2	8	1	SWEEP, LIMIT SWITCH	ALUM
H8115-4	27	3	SWING, SERVOM, DK, MOD	H8113-2	7	1	LENS, BRACKET	ALUM
H8282-1	26	3	SWING, SERVOM, DK, MOD	H8111-1	6	2	SWEEP, LIMIT ADAPTER	ALUM
H8110-1	25	3	SWING, SERVOM, DK, MOD	H8110-5	5	2	SWEEP, LIMIT, RIGHT, ALUM	ALUM
H8110-1	24	1	LENS, CANTON, 200mm	H8110-5	4	2	SWEEP, LIMIT, LEFT, ALUM	ALUM
-	23	2	LIMIT SWITCH	H8110-5	3	1	FOCUS, BKT, FOOS, ALUM	ALUM
-	22	2	LIMIT SWITCH	H8110-5	2	1	LENS, ADAPTER, MOUNT	ALUM
H8110-1	21	1	MOUNTING PLATE, ASSY	H8110-5	1	1		

(-1) TV LENS DRIVE

1 REQD

1/2

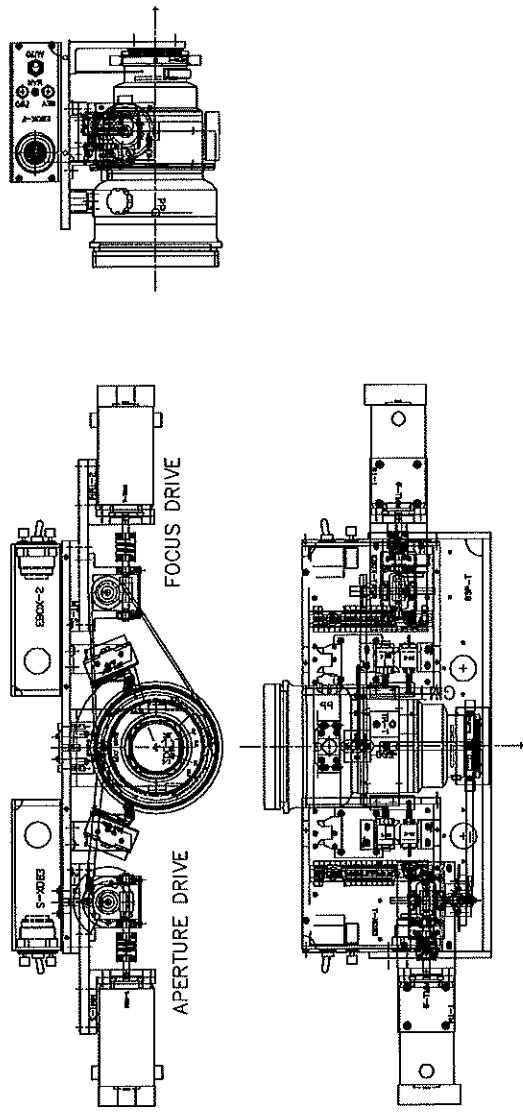
KECK/HIRES
LENS DRIVE
ASSEMBLY

H8105.B
1/16/90
1/15/91

A. F. HUYNH
REVISED & REDRAWN
B. T. BISHOP
GENERAL UPDATING

POSITIONING OF LENS APERTURE AND FOCUS:

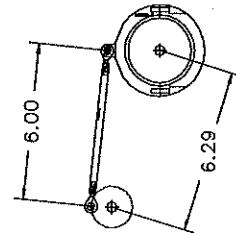
LENS DRIVES SHOWN 1/2 SCALE



APERTURE DRIVE:	4000 CTS/REV
MOTOR ENCODER	60:1 REDUCTION
GEAR BOX	
BELL CRANK RADIUS	0.800
LENS CRANK RADIUS	2.050
APERTURE RING TRAVEL	46°
CRANK ARM IS 6" LONG.	

ONE REVOLUTION OF THE BELL CRANK RING WILL COVER THE FULL APERTURE RANGE FROM F/1.8 TO F/22. THE RATIO IS NOT LINEAR AND SETTINGS WILL COME FROM A LOCK-UP TABLE.

THE BELL CRANK DRIVE ALLOWS THE FULL RANGE OF APERTURE SETTINGS WITHOUT THE POSSIBILITY OF OVER-RUNNING THE BUILT-IN TRAVEL LIMITS.

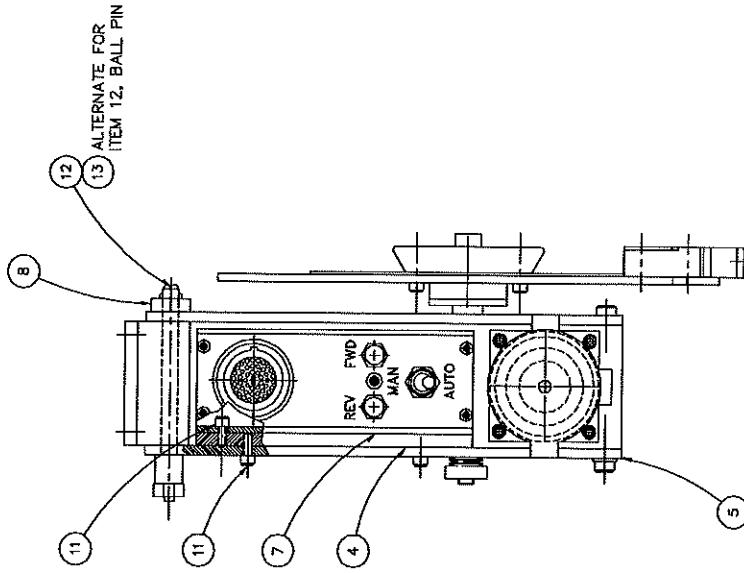


FOCUS DRIVE:	4000 CTS/REV
MOTOR ENCODER	60:1 REDUCTION
GEAR BOX	
TIMING BELT REDUCTION	72 T : 20 T (4.5883 P.D. TO 1.2732 P.D.) = 3.602:1
FOCUS RING TRAVEL	136°
1 CT X 1 REV/4000 CTS X 1 REV OUT/60 REV IN X 1/3.6 X 360°/REV = 0.0004166°	
OR 0.0004166/CT	
OR 2400 CTS/DEG.	

FOCUS RANGE IS XX TO INFINITY.
FOCUS TRAVEL IS LIMITED BY HARDWARE AND SOFTWARE LIMIT SWITCHES.

1/2

KECK/HIRES
LENS DRIVES
DATA AND GEAR REDUCTIONS
H-8119.A

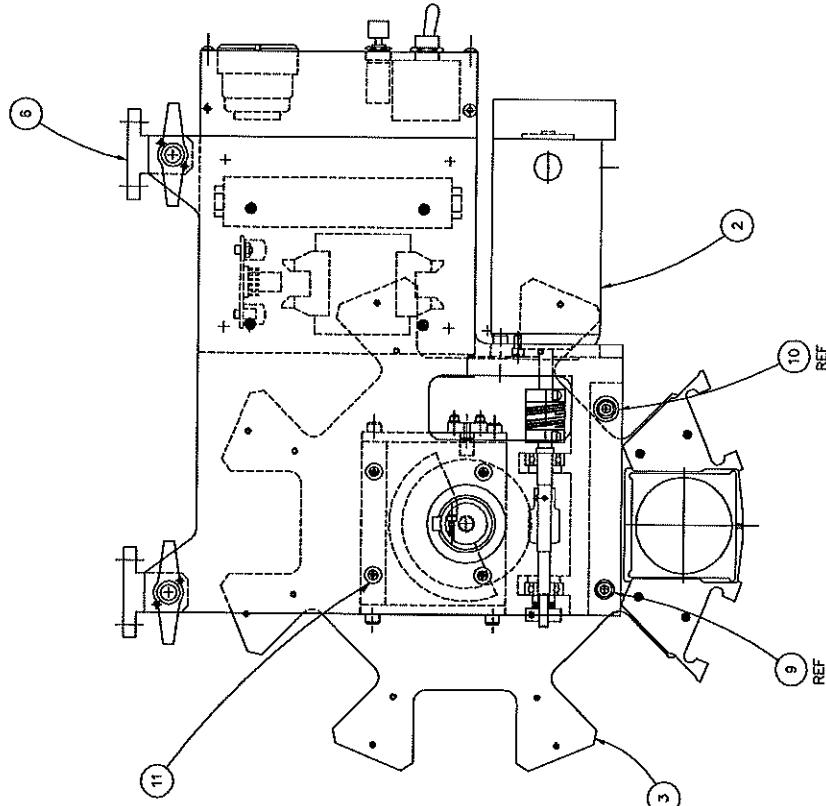


REF ID	QTY	2 IN
12	2	BLK LOCK PIN 3/32 INCH
11	16	SCREW SLEEVES 7/32 INCH
10	2	SHOULDER SCREW 7/32 X .35 IN
9	2	SPACER
H324-2	8	ALUM JUNCTION BOX ASSY
H324-1	7	ALUM MOUNTING BLOCK
H322-2	6	ALUM MOUNTING PLATE
H322-1	5	ALUM SCREWING PLATE
H320-1	2	ALUM SPACER
H320-2	1	ALUM SPACER
H320-3	1	ALUM SPACER
H320-4	1	ALUM SPACER

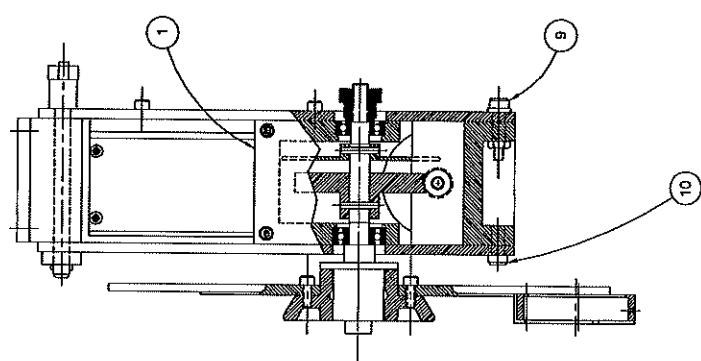
FULL

KECK/HIRES
FILTER WHEEL, 8 HOLE
ASSEMBLY

REV B
12/2000
H326.B



(-1) FILTER WHEEL ASSEMBLY
2 REQD

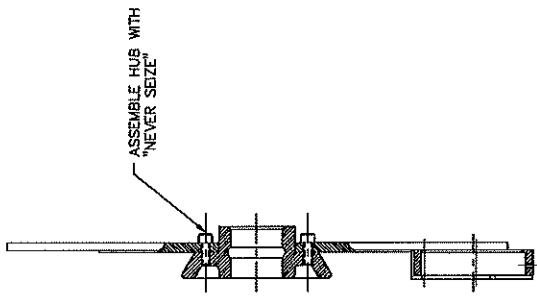
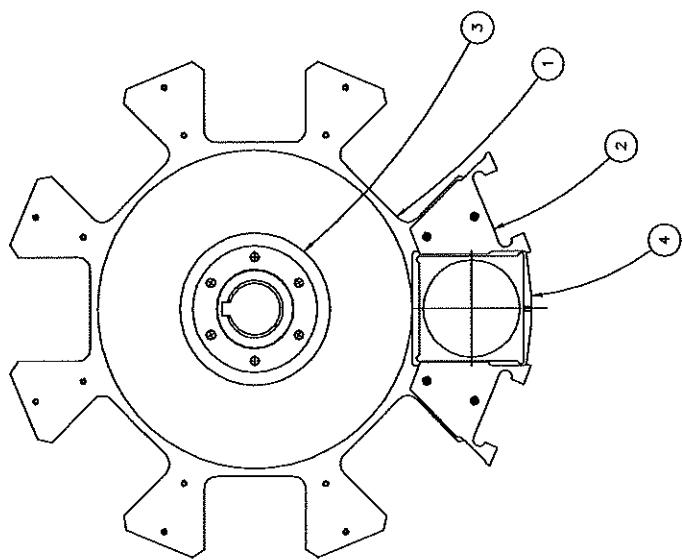


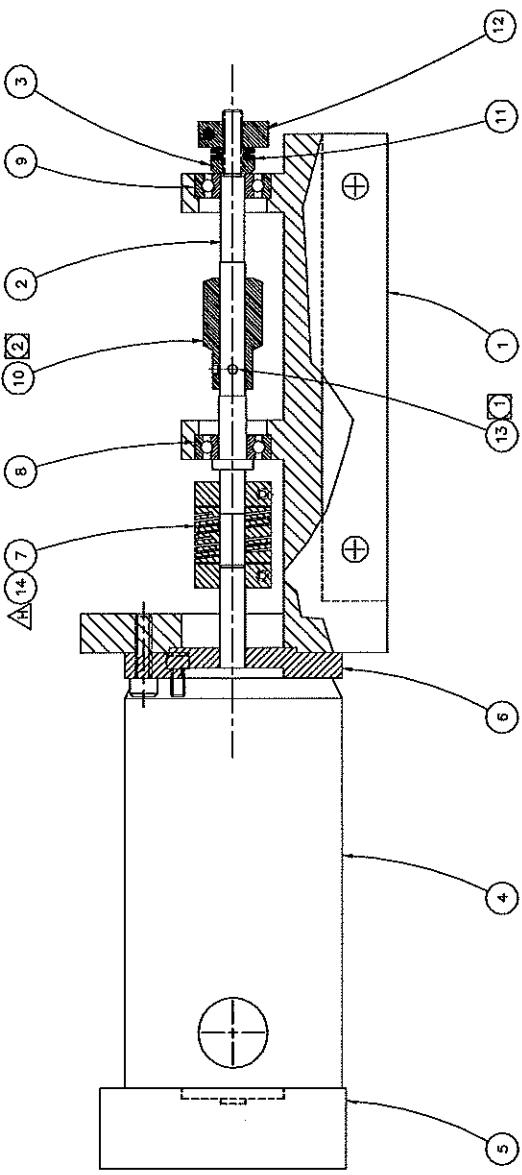
1	2	3	4	5	6	7	8	9	10	11	12
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1	2	3	4	5	6	7	8	9	10	11	12
1	2	3	4	5	6	7	8	9	10	11	12
1	2	3	4	5	6	7	8	9	10	11	12

FULL

KECK HIRE
FILTERWHEEL, 8 HOLE
SUB-ASSEMBLY
E.C.B. 10/20/76
C.A.O. 2/20/76
H3325

(-1) FILTER WHEEL SUB-ASSEMBLY
2 REQ'D



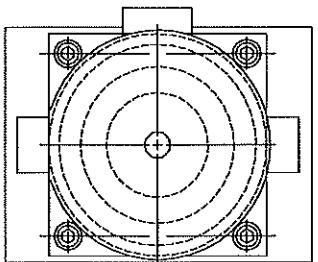


(-1) WORM DRIVE ASSY

1 RECDP

NOTES:

- ① PIN GEAR IN PLACE AFTER ASSEMBLY AND ALIGNMENT.
- ② LUBRICATE WITH NYTEGEL.
- ③ GAIL #50/1000. SEE H2623-1
- ④ 2. HEWLETT-PACKARD HEDS-6000 SERIES
- ⑤ BERG #C04/S-1 WITH KEYWAY ADDED
- ⑥ SKF #607-2RS
- ⑦ SKF #626-2RS
- ⑧ BERG #H24S-4D
- ⑨ BELLEVILLE SPRING WASHER #B0437-016-S



△	-	14	2	1/16" TEF	SST
△	-	15	1	SPRING PIN, 1/16" DIA X 1/16" L SST	
△	11112-2	12	1	SHIM, 1/16" X 1/16" X 1/16" SST	
△	11112-3	13	1	SHIM, 1/16" X 1/16" X 1/16" SST	
△	11111-1	14	3	SHIM, 1/16" X 1/16" X 1/16" SST	NOTE 7
△	11111-2	15	1	KEY, 1/2" X 1/16" X 1/16" SST	
△	11111-3	16	1	WORM, MOD ID	
△	11111-4	17	1	WORM BEARING	
△	11111-5	18	1	BALL BEARING	
△	11111-6	19	1	BALL BEARING	
△	11112-1	20	1	COPPLING, MOD ID	NOTE 4
△	11112-2	21	1	MOUNTING PLATE	
△	11112-3	22	1	ENCODER, SERVO, DM	NOTE 2
△	11112-4	23	1	MOTOR, SERVO DM	
△	11112-5	24	1	SPACER	SST
△	11112-6	25	1	SHAFT	SST
△	11112-7	26	1	WORM CARRIER	
△	11112-8	27	1	ALUM	

2/1

KECK/HIRES

FILTERWHEEL, 12 HOLE	
WORM DRIVE ASSY	
12/20/96	
BLA	
01/20/97	
C.A.	
H1110.H	

KECK/HIRES
T.V. GUIDER
ASSEMBLY

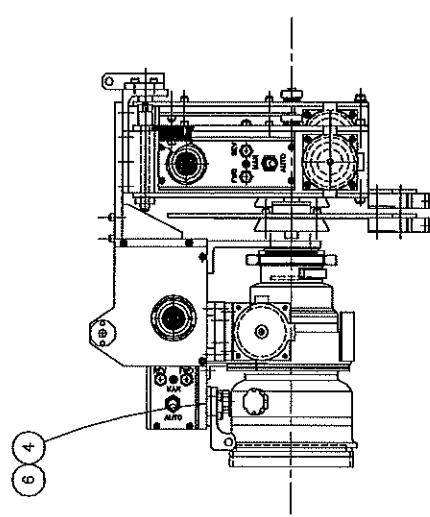
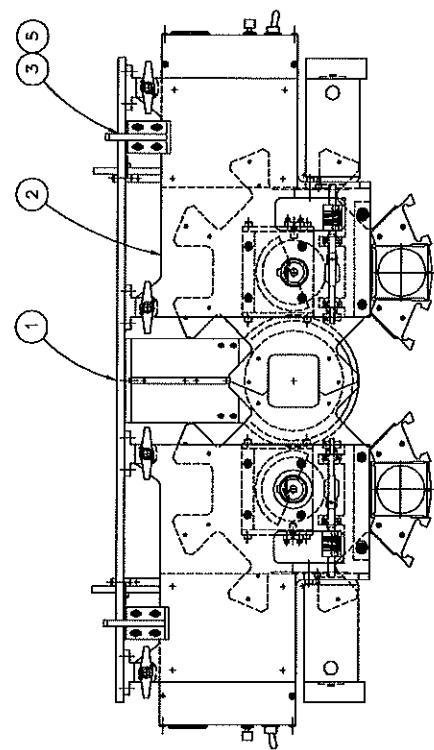
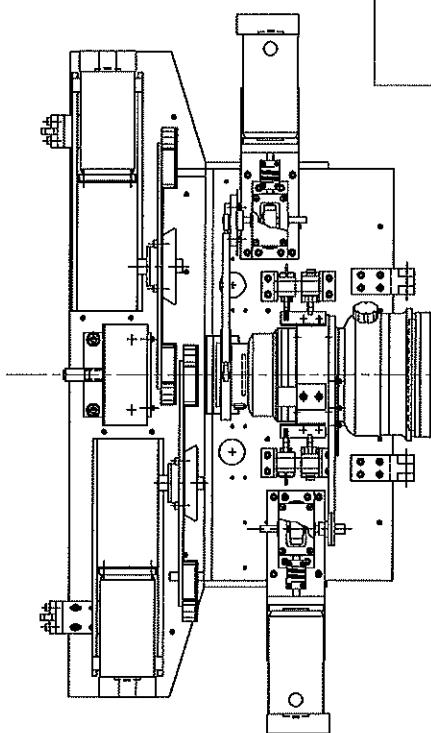
H601

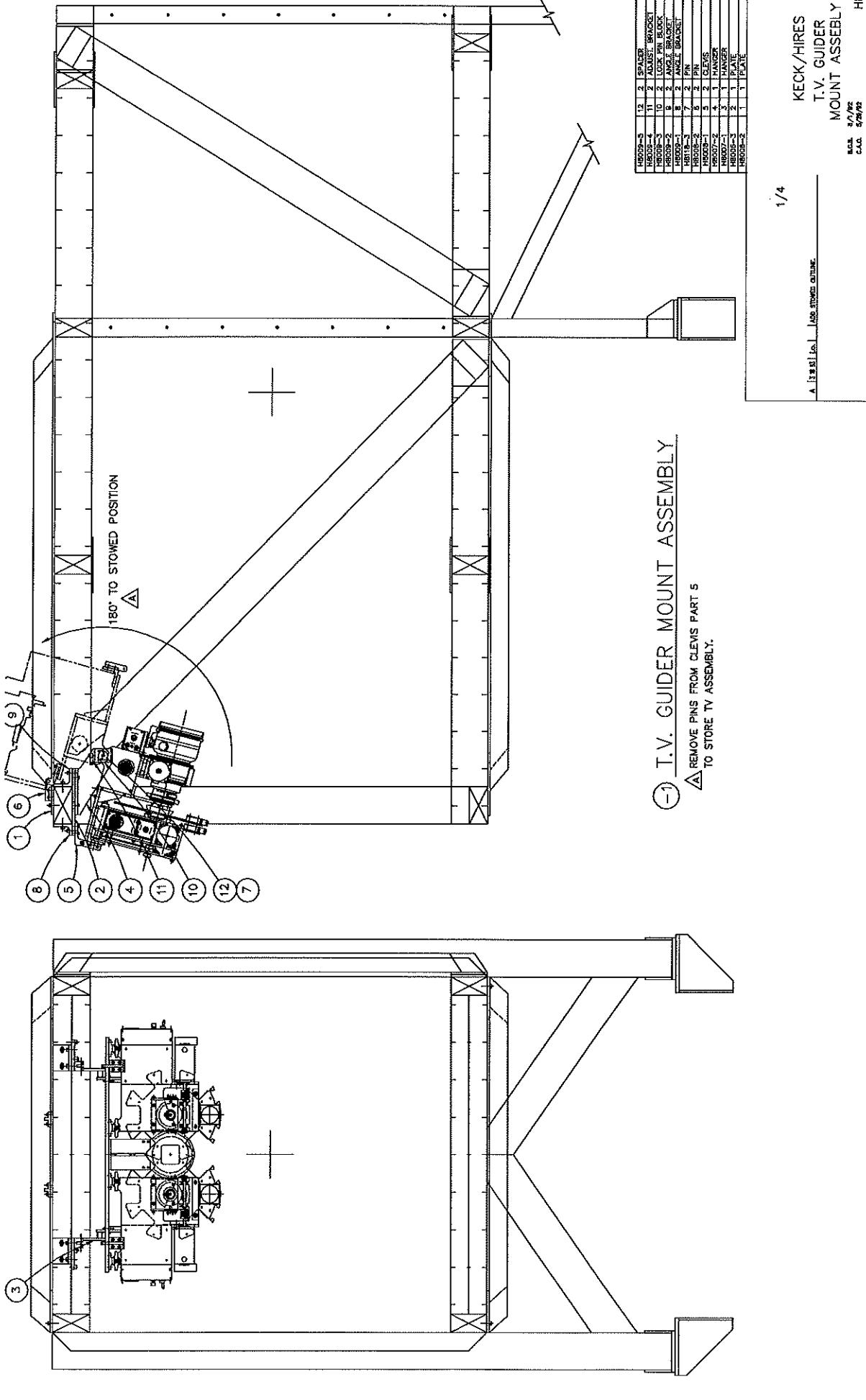
5/17/92
5/27/92

1/2

① T.V. GUIDER ASSEMBLY

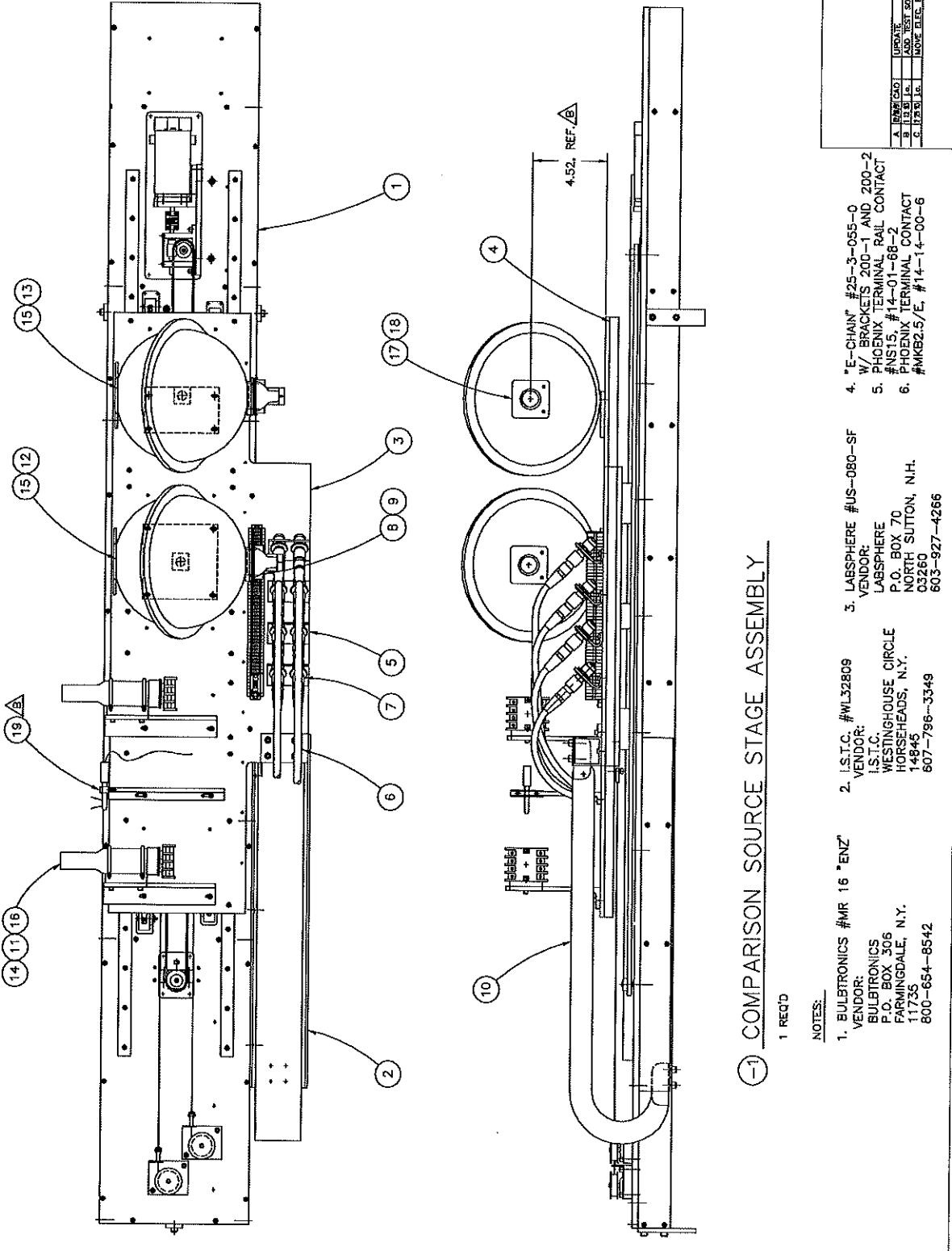
H6008-3	6	2	LENS SPACER	ALUM.
H6008-2	5	2	PIN	ALUM.
H6008-1	4	2	CIRCLIPS	ALUM.
H6008-1	3	2	ANGLE SPACER	ALUM.
H6008-1	2	2	3 MM FLANGE NUT	SS
H6008-1	1	1	LINE DRIVE ASSEMBLY	

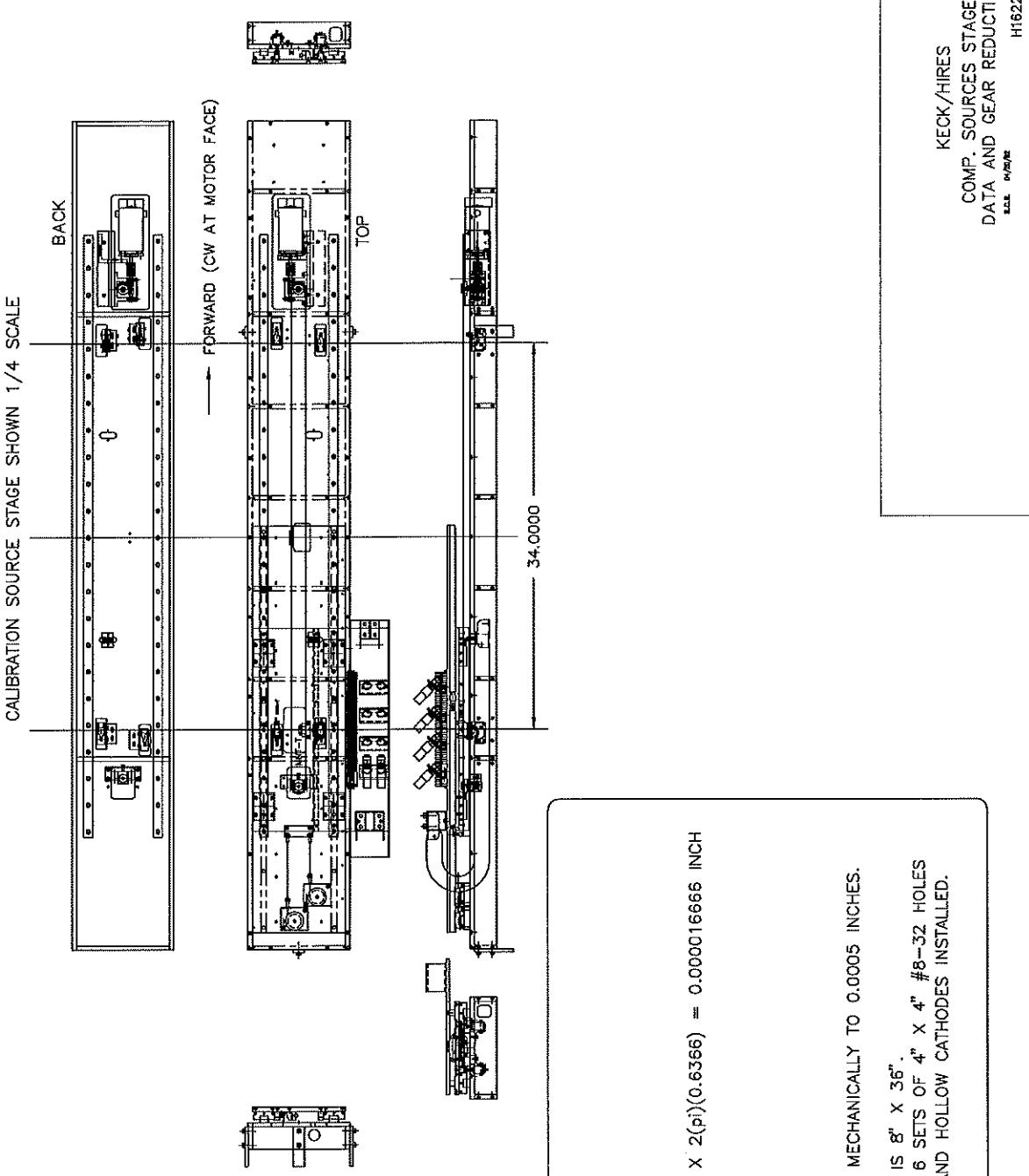




Appendix Q List of Drawings — Comparison Source System

1. H1620 Comparison Source Stage Assembly
2. H1622 Stage Schematic
3. H1618 Comp. Source Stage Mounting Surface Layout
4. H1688 Enclosure Assembly
5. H1701 Comp. Source Stage Filterwheel Assembly
6. H1706 Comp. Source Stage Filterwheel Mounting
7. H1655 Comp. Source Optics (Epps 3940)
8. H1657 Comp. Source Optics Stop Layout
9. H1812 Comp. Source Optics 2" Lens Cell Mount
10. H1816 Comp. Source Optics 6" Lens Cell Mount
11. H1814 Comp. Source Optics 6" Flat Mirror Mount





THEORETICAL POSITIONING RESOLUTION:
 MOTOR ENCODER 4000 CTS/REV
 GEAR BOX 60:1 REDUCTION
 DRIVE GEAR PITCH RADIUS 0.6366

$$1 \text{ CT.} \times 1 \text{ REV/4000 CT} \times 1 \text{ REV OUT/60 REV IN} \times 2(\pi)(0.6366) = 0.000016666 \text{ INCH}$$

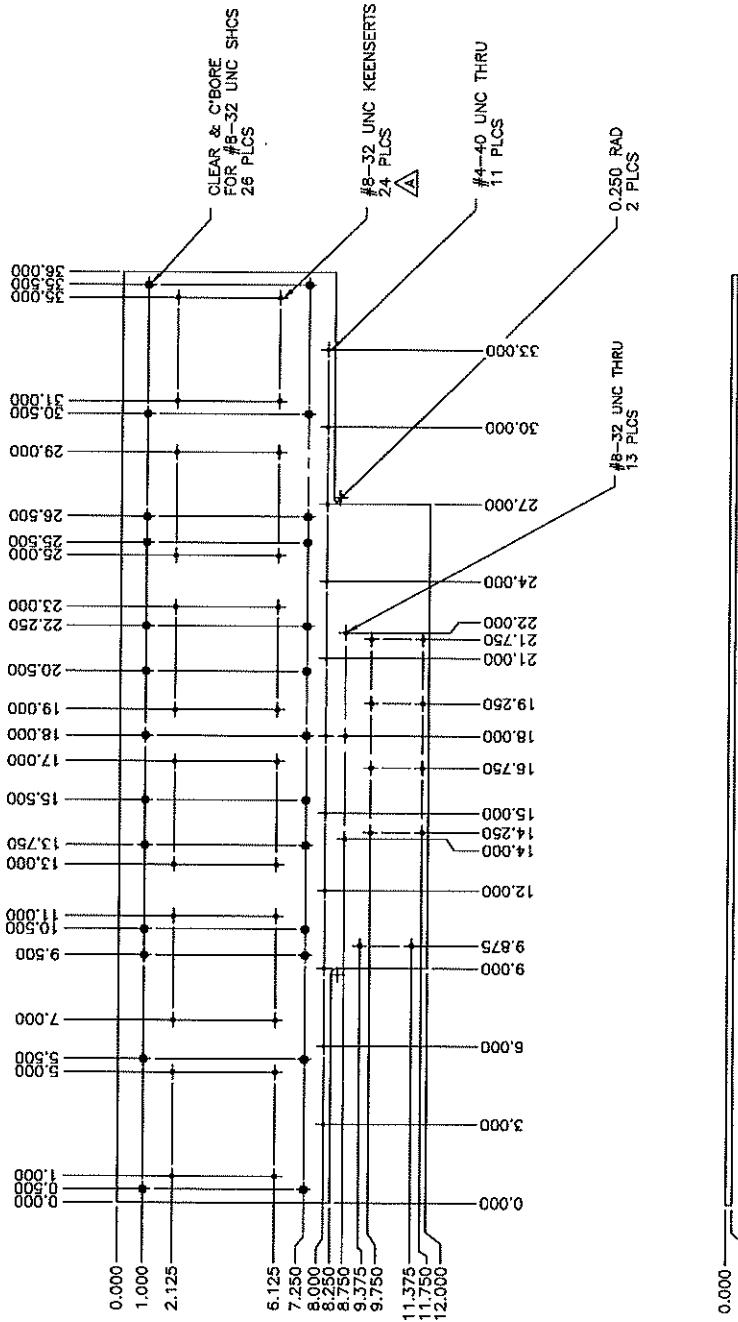
OR 0.000016666 INCH/CT
 OR @ 60,000 CT/INCH

IF SERVO LOOP IS GOOD TO 2 COUNTS,
 RESOLUTION IS 0.00003333 INCHES.

RESOLUTION AND REPEATABILITY HAVE BEEN TESTED MECHANICALLY TO 0.0005 INCHES.

TOTAL TRAVEL IS 34 INCHES. MOUNTING PLATFORM IS 8" X 36".
 MOUNTING PLATFORM HOLE PATTERN IS ON H1618, 6 SETS OF 4" X 4" #8-32 HOLES
 H1620 SHOWS STAGE WITH INTEGRATING SPHERES AND HOLLOW CATHODES INSTALLED.

KECK/HIRES
 COMP. SOURCES STAGE
 DATA AND GEAR REDUCTION
 H1622



(-1) MOUNTING PLATE

1 RECD
ALUM., 6061-T651 TOOLING PLATE
BLACK ANODIZE
FINISH: $\pm \frac{3}{32}$

1	RECD	NO Holes	NO Holes
2	RECD	NO Holes	NO Holes
3	RECD	NO Holes	NO Holes

COMP. SOURCE STAGE
SOURCE MOUNT L/O
H1518.B

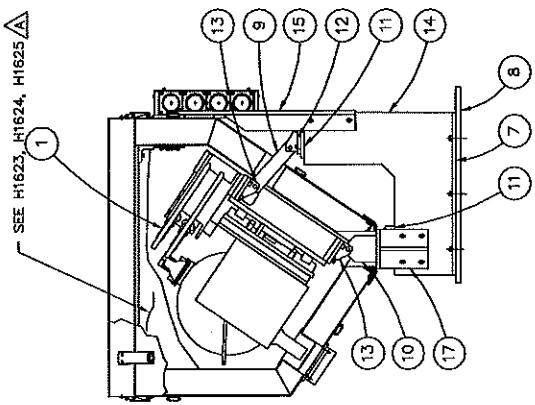
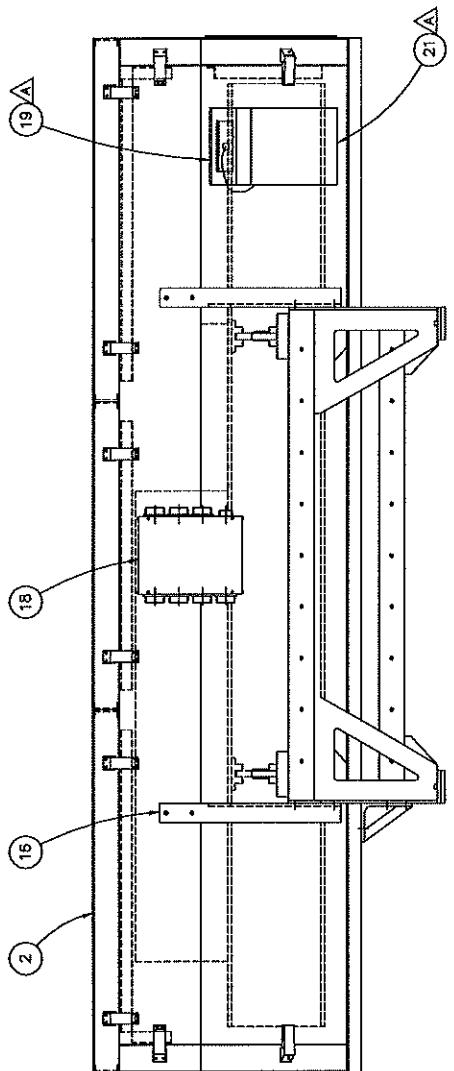
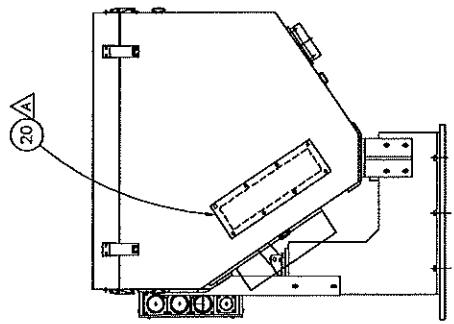
R.D.
C.G.
C.A.

KECK/HIRES
COMPARISON SOURCE STAGE
ENCLOSURE ASSY
E.D.A. 1/17/92
C.A.D. 1/17/92
H1688.A

1/4

16768-1	21	1	JUNCTION BOX ASSY
16768-1	20	1	END PLATE 100 ALUM
16768-1	19	1	MOUNT PLATE
16768-1	18	1	DECAL ASSY
16768-1	17	1	ALUM
16768-1	16	2	BRACE
16768-1	15	1	BRACE
16768-1	14	1	MOUNTING MOUNT
16768-1	13	4	BRACKET
16768-1	12	2	BRACKET
16768-1	11	4	MOUNT
16768-1	10	2	MOUNT
16768-1	9	2	LINK
16768-1	8	2	BASE PLATE STEEL
16768-1	7	2	BASE PLATE
16768-1	2	1	CAP SIDE COVER ALUM
16768-1	1	1	COMPARISON SOURCE STAGE

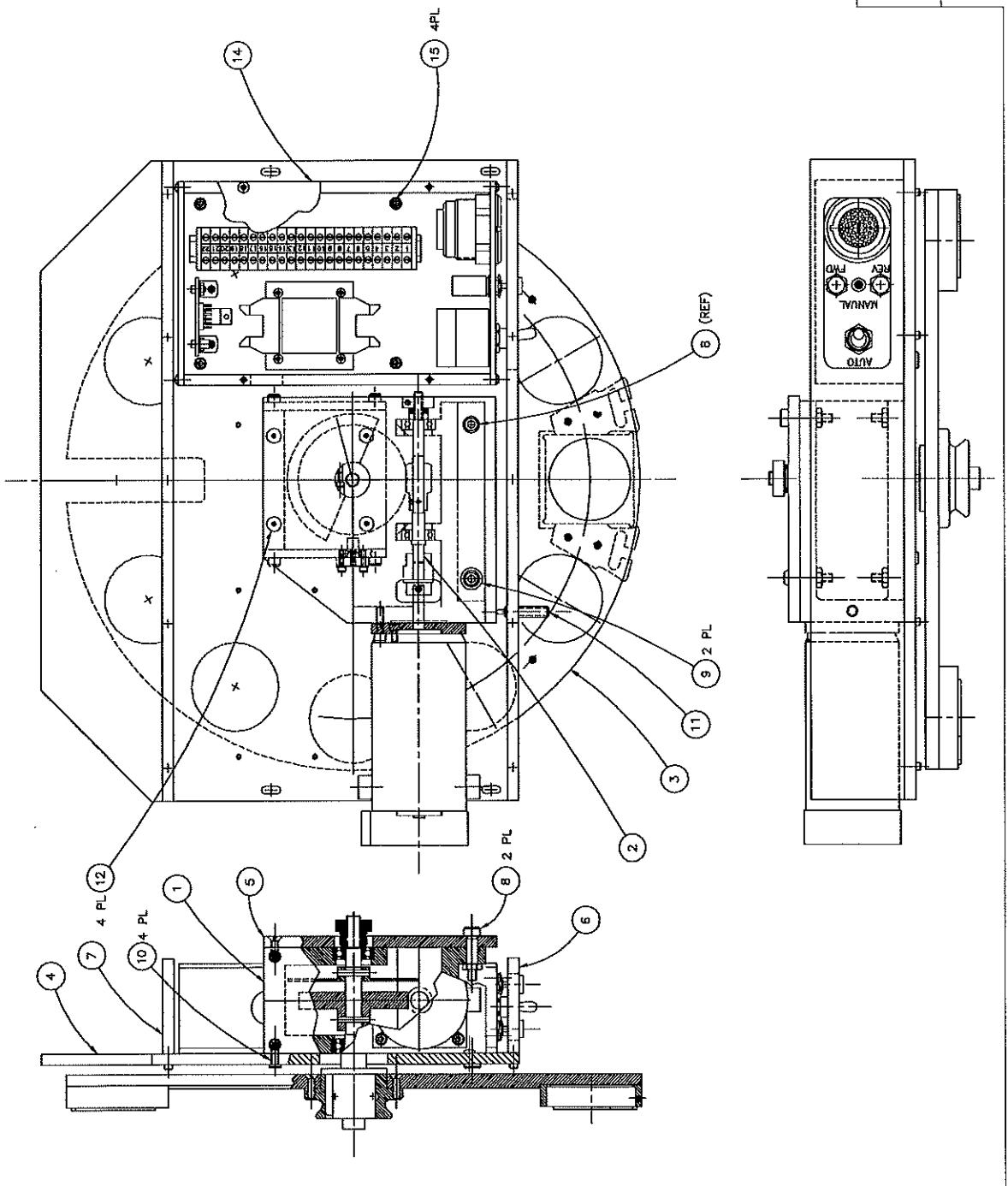
① COMPARISON SOURCE STAGE / ENCLOSURE ASSEMBLY



SEE H1623, H1624, H1625

△

1

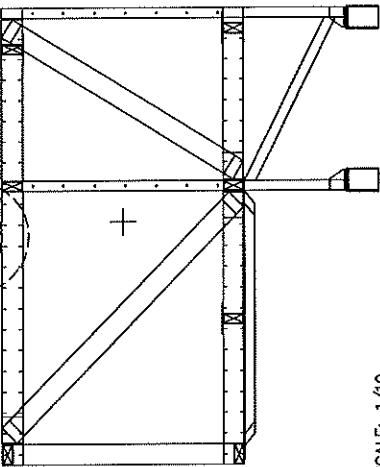


-	15	4	SCH. SHEET. B-2410C X-3006
-	13	10	SCH. SHEET. C-4010C X-3016
-	12	4	SCH. SHEET. C-3010C X-3016
-	11	1	FLIER PLUNGER
-	10	5	SOC. PLUNGER
-	9	2	SOC. PLUNGER
-	8	2	SOC. PLUNGER
-	7	1	SIDE PLATE
-	6	1	SIDE PLATE
-	5	1	SIDE PLATE
-	4	1	MEASURING PLATE
-	3	1	WHEEL ASSY
-	2	1	WORM DRIVE ASSY
-	1	1	WHEEL & AXLE ASSY

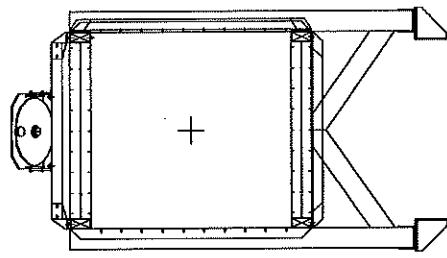
FULL

KECK/HIRES
COMPARISON SOURCE STAGE
FILTERWHEEL ASSY
REV. 8/1/91
E.D.A.
H1701A

SEE DETAIL A



SCALE: 1/10



(1) FILTERWHEEL MOUNT ASSY

1 REQ'D

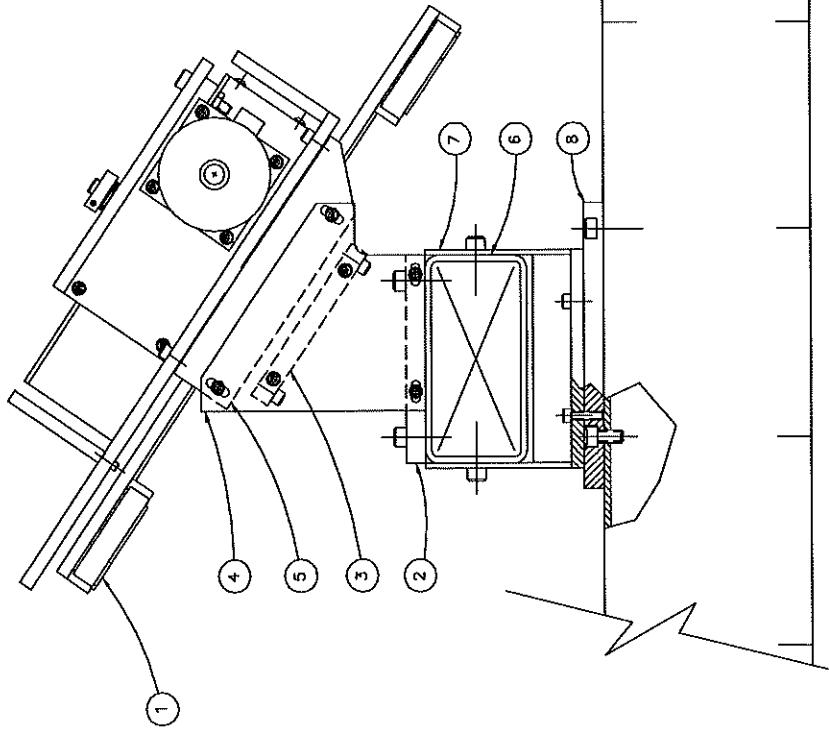
H1776-3	6	2	BASE	STL
H1777-1	7	2	BRACKET MOUNT ASSY	STL
H1776-1	8	1	HORIZONTAL	STL
F1704-4	5	2	STAGE MOUNT	ALUM
F1704-3	4	2	SPACER	ALUM
F1704-2	3	2	ADJUSTMENT PLATE	ALUM
F1704-1	2	2	SCREW BASE	ALUM
F1703-1	1	1	FILTERWHEEL ASSY	ALUM

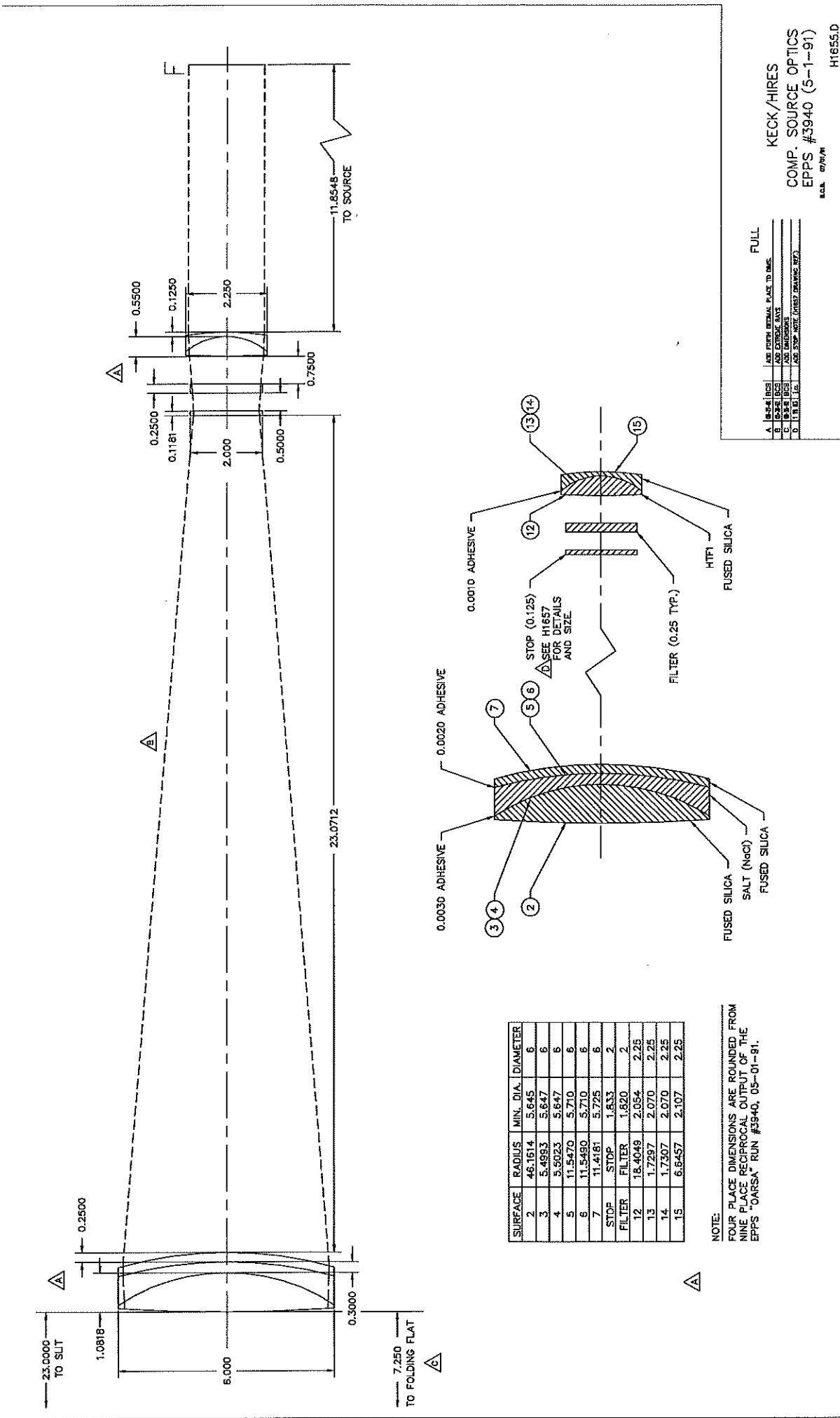
FULL

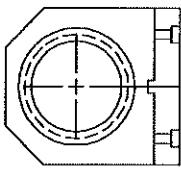
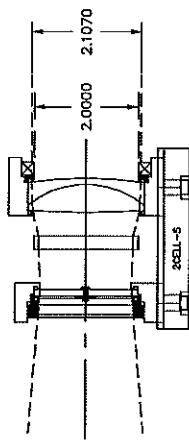
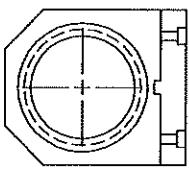
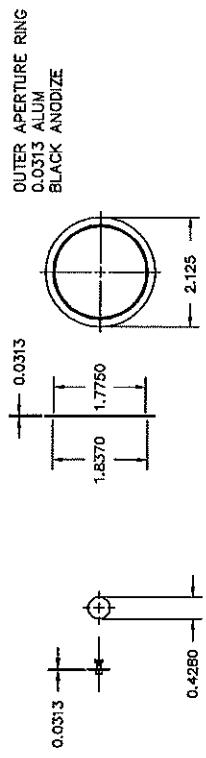
KECK/HIRES

COMPARISON SOURCE STAGE
FILTERWHEEL MOUNT ASSY
R.C.A. 7/1/92
D.A.C. 7/2/92

H1706.A

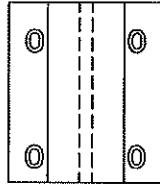






(-1) DETAIL OF STOP

1 RECD



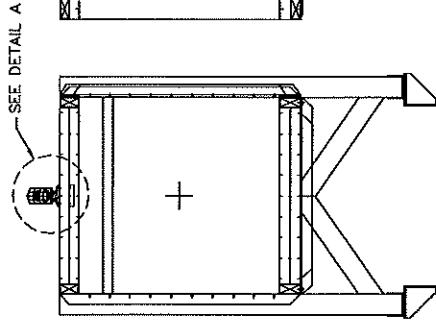
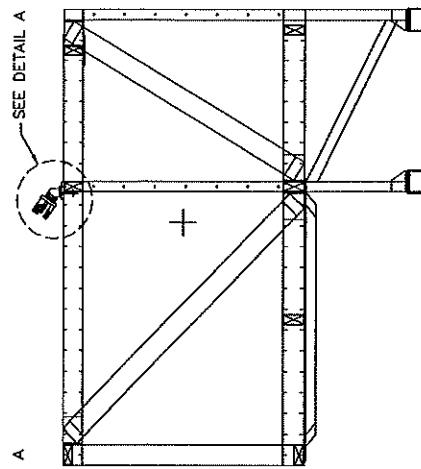
FULL

S. 11451 I.e.	LOCATE VS REF FROM PRED. OPTICS
L.C.A.	LAWSON, BRUCE, AND CO., INC. DRAWING
CHANGE A IS UNKNOWN	

COMP. SOURCE OPTICS
STOP L/O

B.C.A. 02/27/74

H1657.B



SCALE: 1/10

SEE DETAIL A

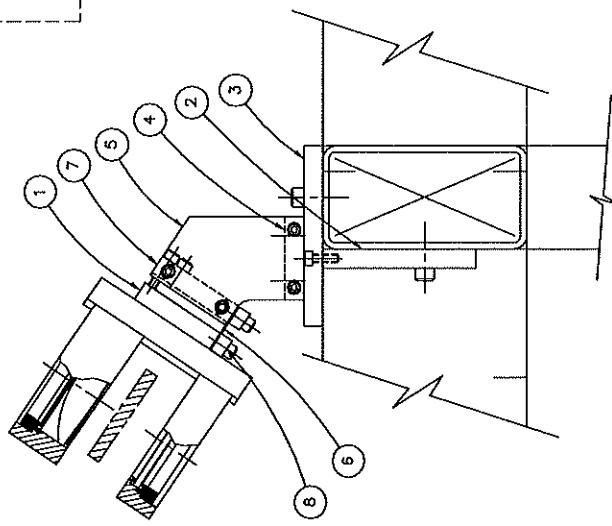
(-1) 2" LENS CELL MOUNT ASSY

1 REQD

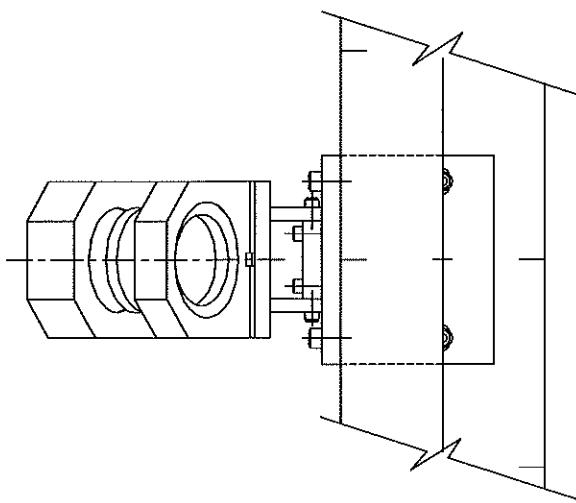
N1802-3	8	2	CAMP	ALUM
N1802-2	7	1	MOUNT LOWER	ALUM
N1802-1	6	1	SPRING	STEEL
N1802-4	5	4	SUPPORT	ALUM
N1802-3	4	1	MB	ALUM
N1802-2	3	1	MOUNTING PLATE	ALUM
N1802-1	2	1	MOUNTING PLATE	ALUM
N1801-1	1	1	LENS CELL ASSY	ALUM

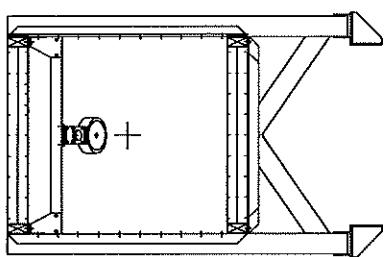
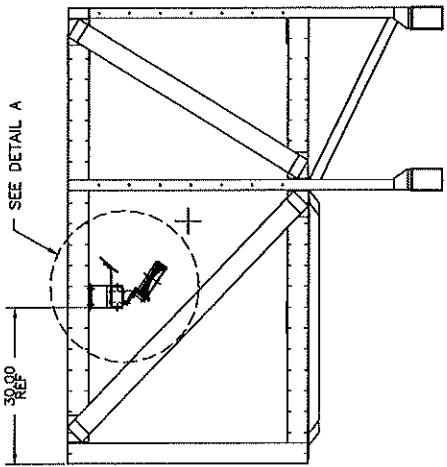
NOTED

KECK/HIRES
CALIBRATION OPTICS
2" LENS CELL MOUNT ASSY
A.D.A. 4/7/91
C.A.A. 5/7/92
H1812



DETAIL A
SCALE: FULL



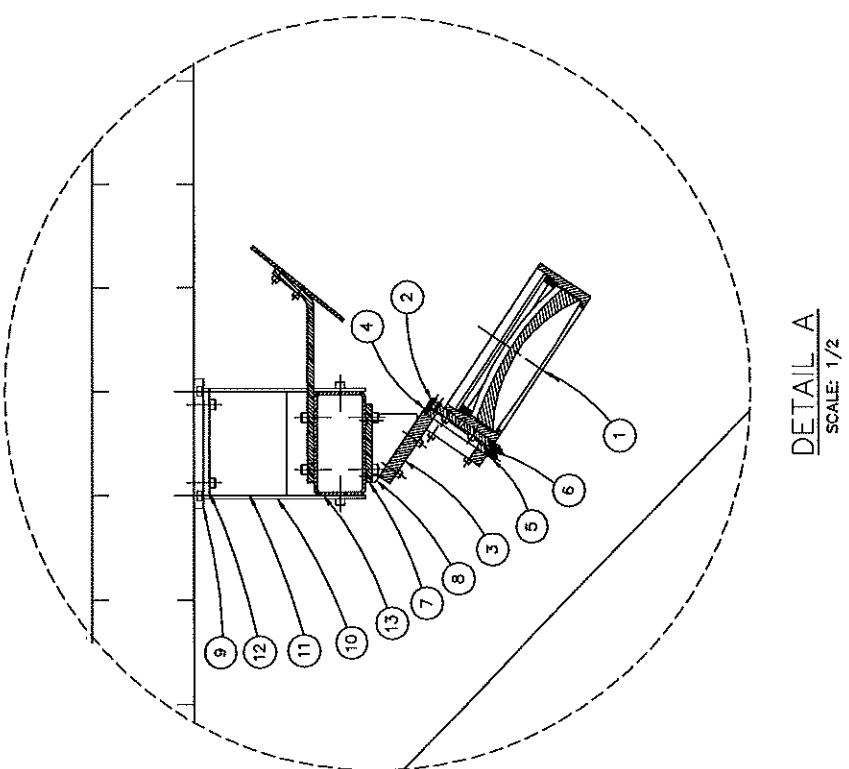


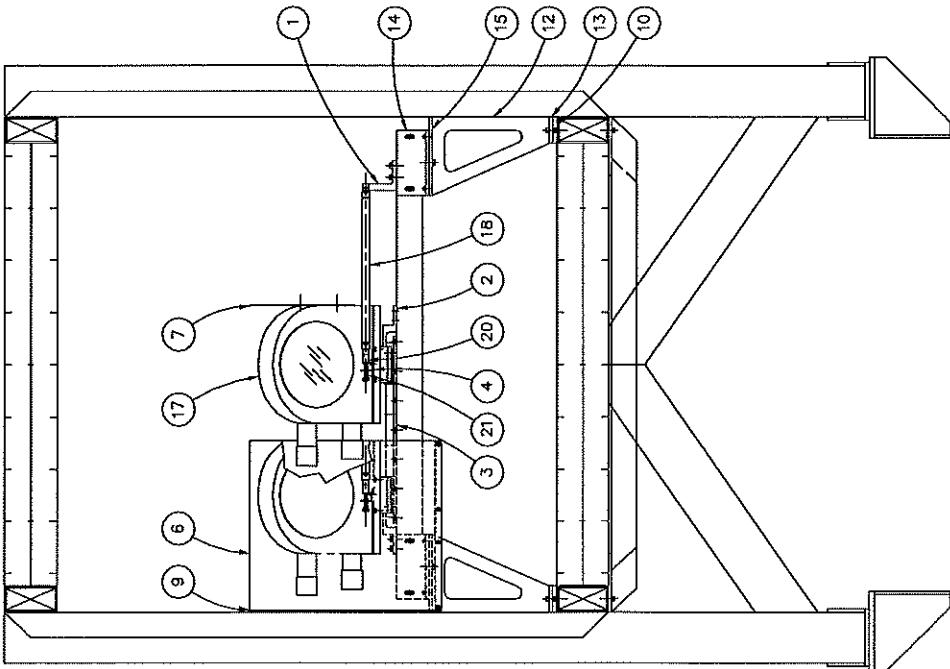
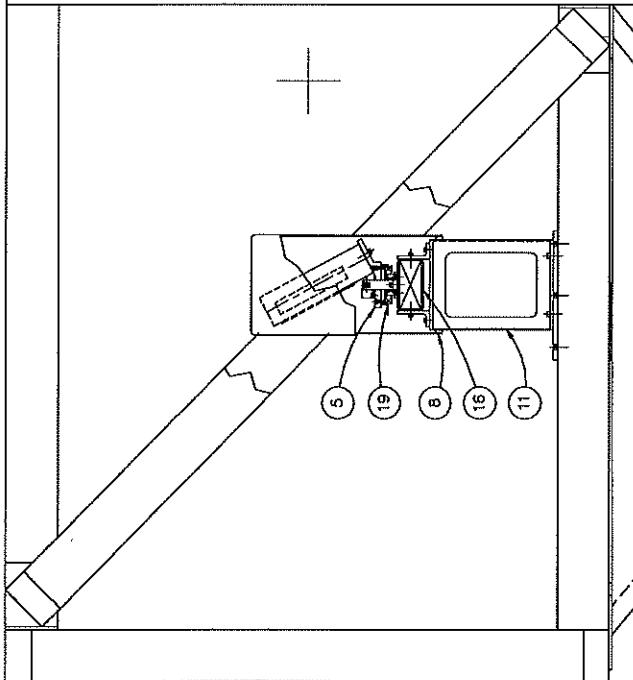
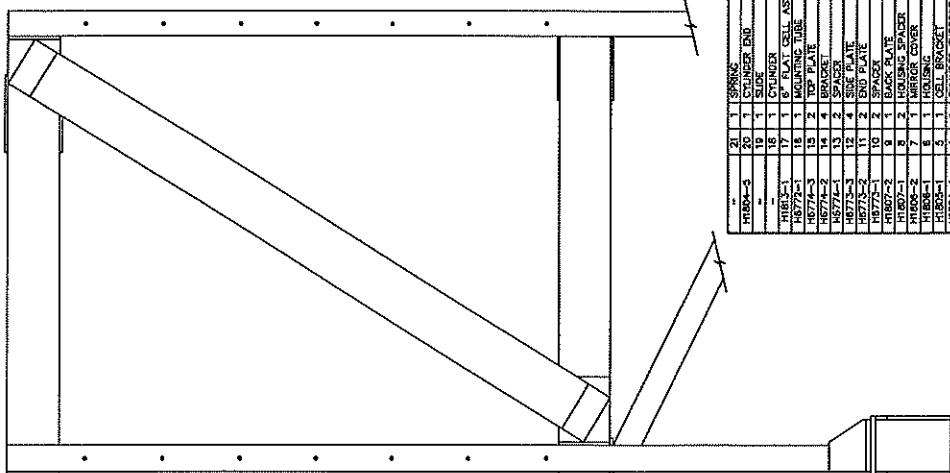
(-1) 6" LENS CELL MOUNT ASSY
1 RECD

ITEM	DESCRIPTION	QUANTITY	UNIT
H6770-1	1 MOUNTING TUBE	1	INCHES
H6770-2	2 MOUNTING PLATE	1	STEEL
H6770-3	4 SIDE PLATE	1	STEEL
H6770-4	10 BACK PLATE	1	STEEL
H6770-5	2 MOUNTING PLATE	1	STEEL
H6770-6	2 MOUNTING PLATE	1	ALUM
H6770-7	2 MOUNTING PLATE	1	ALUM
H6770-8	2 MOUNTING PLATE	1	ALUM
H6770-9	2 MOUNTING PLATE	1	ALUM
H6770-10	2 MOUNTING PLATE	1	ALUM
H6770-11	2 MOUNTING PLATE	1	ALUM
H6770-12	2 MOUNTING PLATE	1	ALUM
H6770-13	1 MOUNTING TUBE	1	ALUM
H6770-14	1 SPRING	1	STEEL
H6770-15	5 CLAMP	2	ALUM
H6770-16	4 HORN, UPPER	1	STEEL
H6770-17	3 HORN, LOWER	1	STEEL
H6770-18	2 LENS, CELL ASSY	1	ALUM
H6770-19	1 LENS, CELL ASSY	1	ALUM

1/4

KECK/HIRES
CALIBRATION OPTICS
6" LENS CELL MOUNT ASSY
E.D.E. 9/1/81
C.D.C. \$71/81
H1816





	21	1	SPRING	STEEL
HRS00-3	20	1	CYLINDER END	SST
-	19	1	SIDE	SST
-	18	1	CYLINDER	SST
HRS00-1	17	1	6" FLAT CELL ASSY	STEEL
HRS00-2	16	1	MOUNTING TUBE	STEEL
HRS00-3	15	2	TOP PLATE	STEEL
HRS00-4	14	4	SCREWS	ALUM
HRS00-5	13	2	SPACER	STEEL
HRS00-6	12	2	SPACER	STEEL
HRS00-7	11	2	SPACER	STEEL
HRS00-8	10	2	SPACER	STEEL
HRS00-9	9	1	BLACK PLATE	ALUM
HRS00-10	8	2	HOUSING SPACER	ALUM
HRS00-11	7	1	MIRROR COVER	ALUM
HRS00-12	6	1	HOUSING	ALUM
HRS00-13	5	1	CELL BRACKET	ALUM
HRS00-14	4	1	CYLINDER SUPPORT	STEEL
HRS00-15	3	1	SPACER	STEEL
HRS00-16	2	2	STOP	ALUM
HRS00-17	1	1	CYLINDER BRACKET	ALUM

1/4

KECK/HIRES
CALIBRATION OPTICS
6" FLAT CELL MOUNT ASSY
B.M. 8/1/91
C.A.C. 5/4/92
H1614.A

A. D. LEE (EAO) JAMES R. COLE

Appendix R List of Drawings

— Iodine Cell Stage

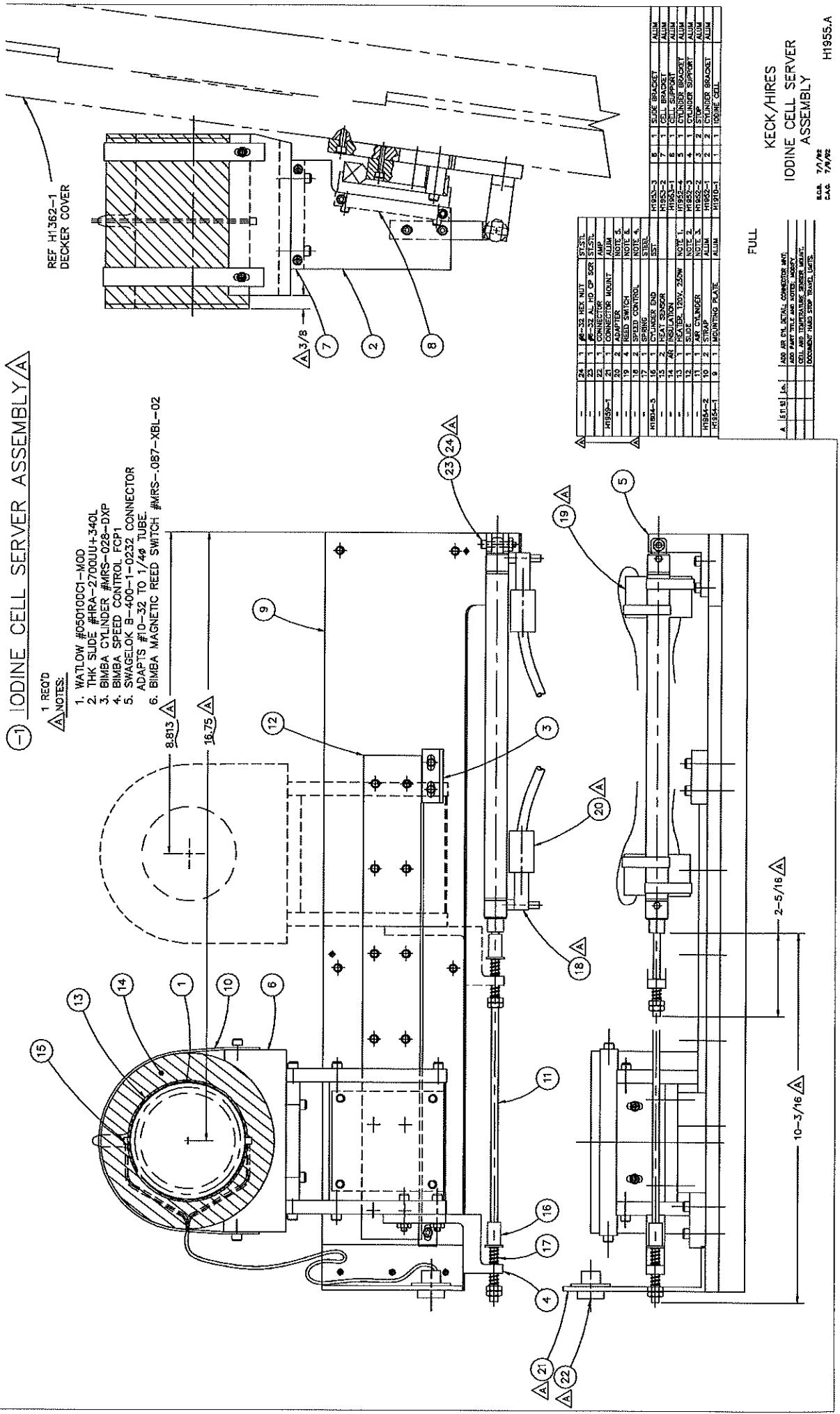
1. H1955 Iodine Cell Server Assembly

(-1) IODINE CELL SERVER ASSEMBLY A

1 REOD

△ NOTES:

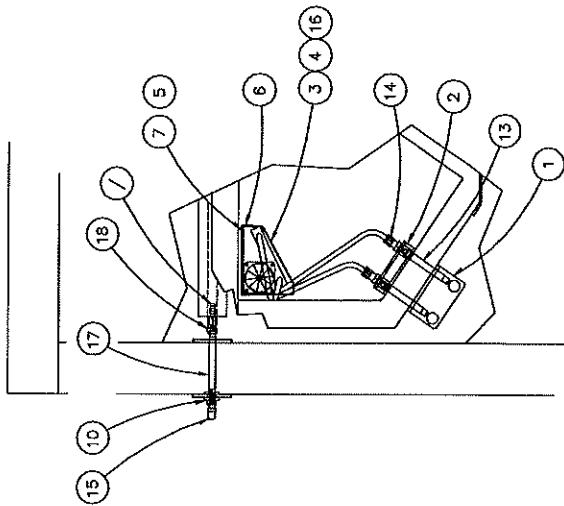
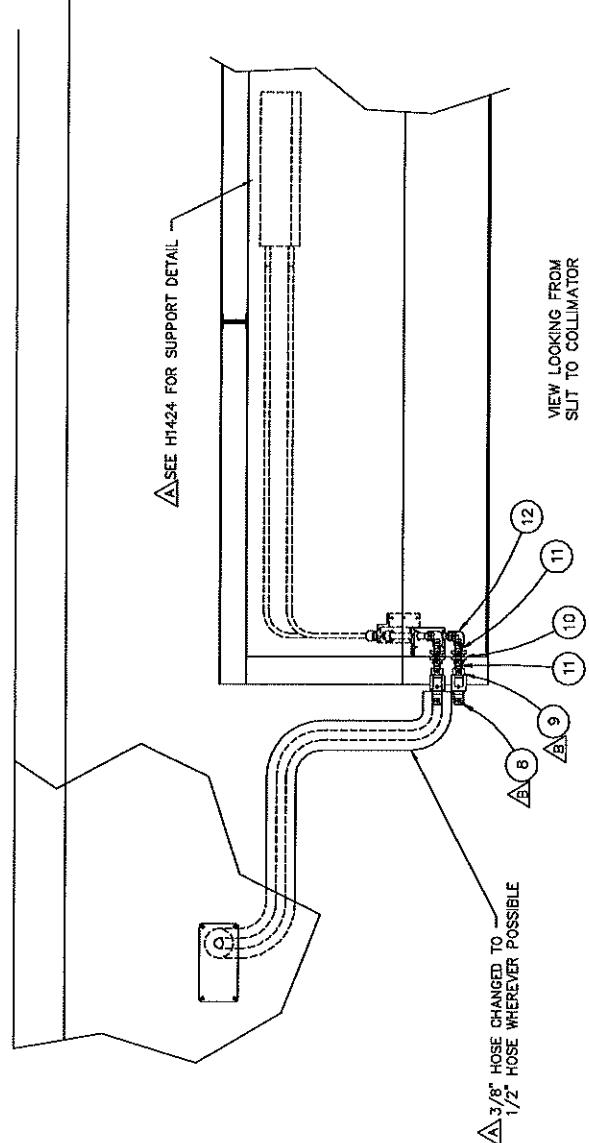
1. WATLOW #50501001-MOD
2. THK SLIDE #HRA-2700U+34QL
3. BIMBA CYLINDER #MPS-028-DXP
4. BIMBA SPEED CONTROL FCP1
5. SWAGELOK B-400-1-0232 CONNECTOR ADAPTS #D-32 TO 1/4" TUBE
6. BIMBA MAGNETIC REED SWITCH #MRS-.087-XBL-02



Appendix S List of Drawings — Slit Area Collimation

Appendix T List of Drawings — Instrument Cooling System

1. H1423 Comparison Source Stage Cooling Details
2. H1406 TV Cooling Details
3. H1411 Vault Cooling Details
4. H1432 Cooling System Control Panel
5. H1450 Cooling System Installation
6. H1462 CCD Electronics Cooling
7. H1463 CCD Cooling Installation Details



-	1	HOSE BARB
-	18	1 FEMALE CONN
-	17	1 NIPPLE 5/8" LG
-	16	2 FAN
-	15	1 MALE COUPLING
-	14	2 BULGE CONN, FEMALE
-	13	2 BULGE CONN, FEMALE
-	12	2 BULGE CONN, FEMALE
-	11	2 BULGE CONN, FEMALE
-	10	2 BULGE CONN, FEMALE
-	9	2 BULGE CONN, FEMALE
-	8	2 BULGE CONN, FEMALE
-	7	2 BULGE CONN, FEMALE
-	6	2 BULGE CONN, FEMALE
-	5	2 BULGE CONN, FEMALE
-	4	2 BULGE CONN, FEMALE
-	3	2 BULGE CONN, FEMALE
-	2	2 BULGE CONN, FEMALE
-	1	2 BULGE CONN, FEMALE

(-1) COOLING SYSTEM

1 REQ'D

NOTES:

- 1. PARKER 46F-6-4
- 2. MCMASTER-CARR 4568K139
- 3. MCMASTER-CARR FS24AB3/03J157
- 4. PARKER 46F-6-4
- 5. SWAGELOC SS-800-71-4
- 6. MCMASTER-CARR 4568K137
- 7. PARKER 2200P-4-4
- 8. MCMASTER-CARR 4568K131
- 9. PARKER 20TACHH5-4
- 10. PARKER BH3-81
- 11. PARKER BH3-80

1/4

KECK/HIRES

COMPARISON SOURCE STAGE
COOLING INSTALLATION DETAILS

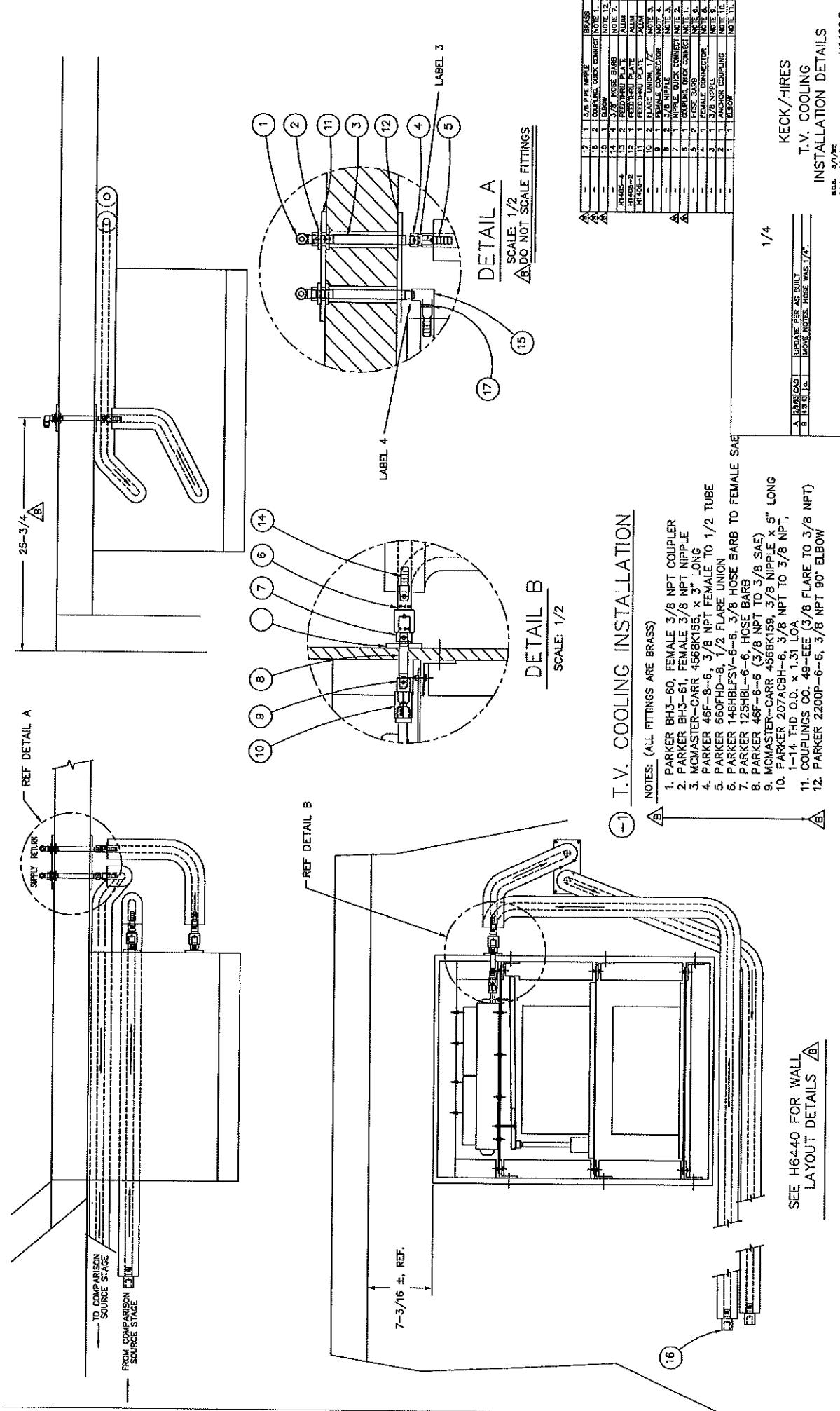
H1423.8

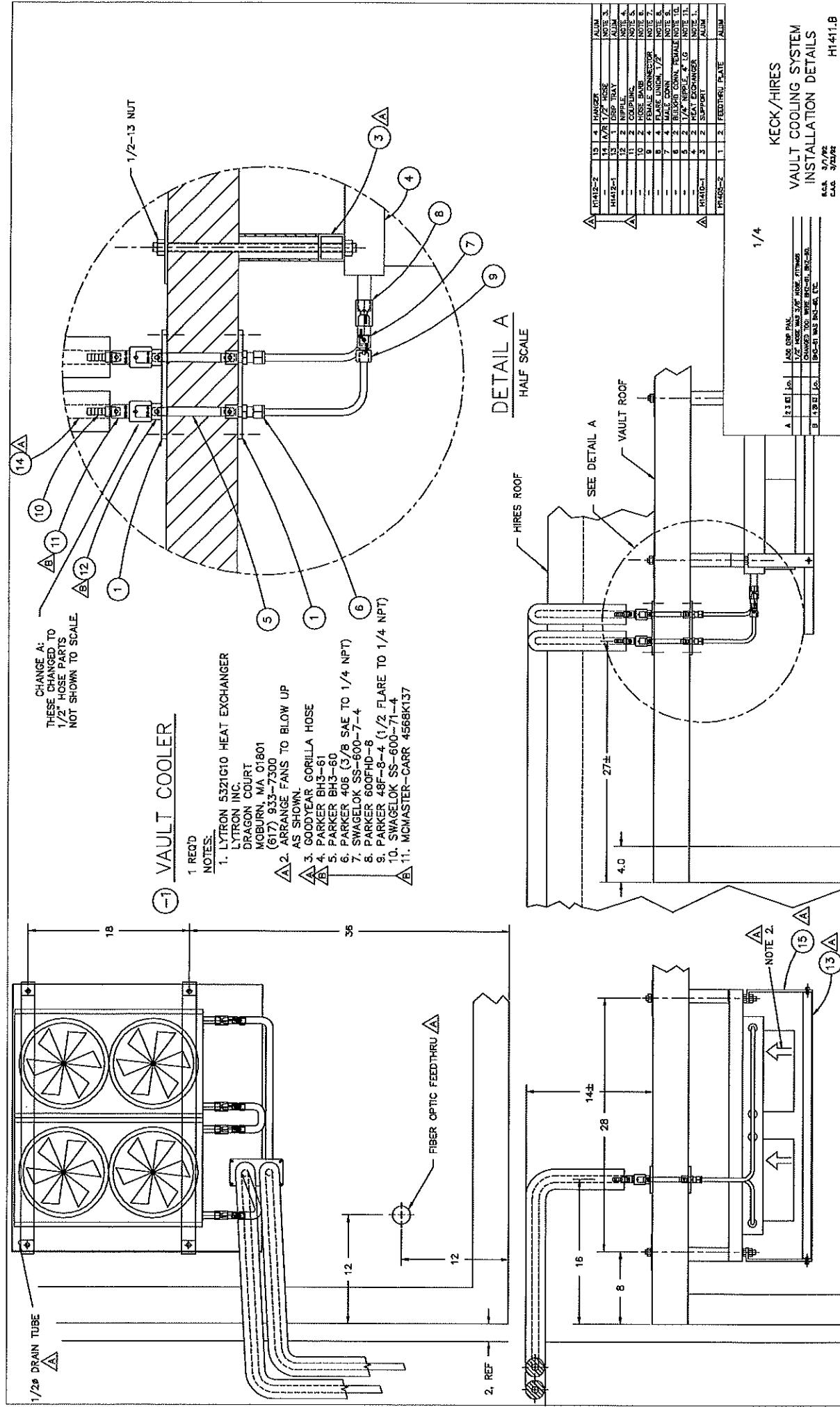
REV. 3/1/92
C.A.D. 3/25/92

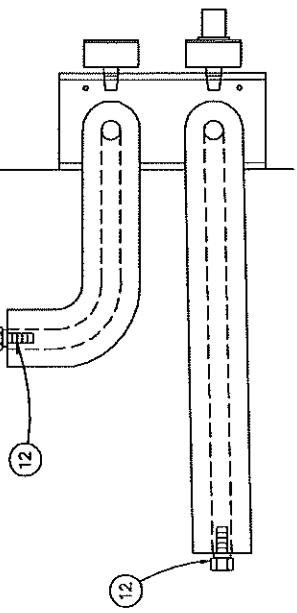
A 105 lbs.
B 438 lbs.

NOTE: CROSS-REF. NOTES

SEE H1424 FOR SUPPORT DETAILS.





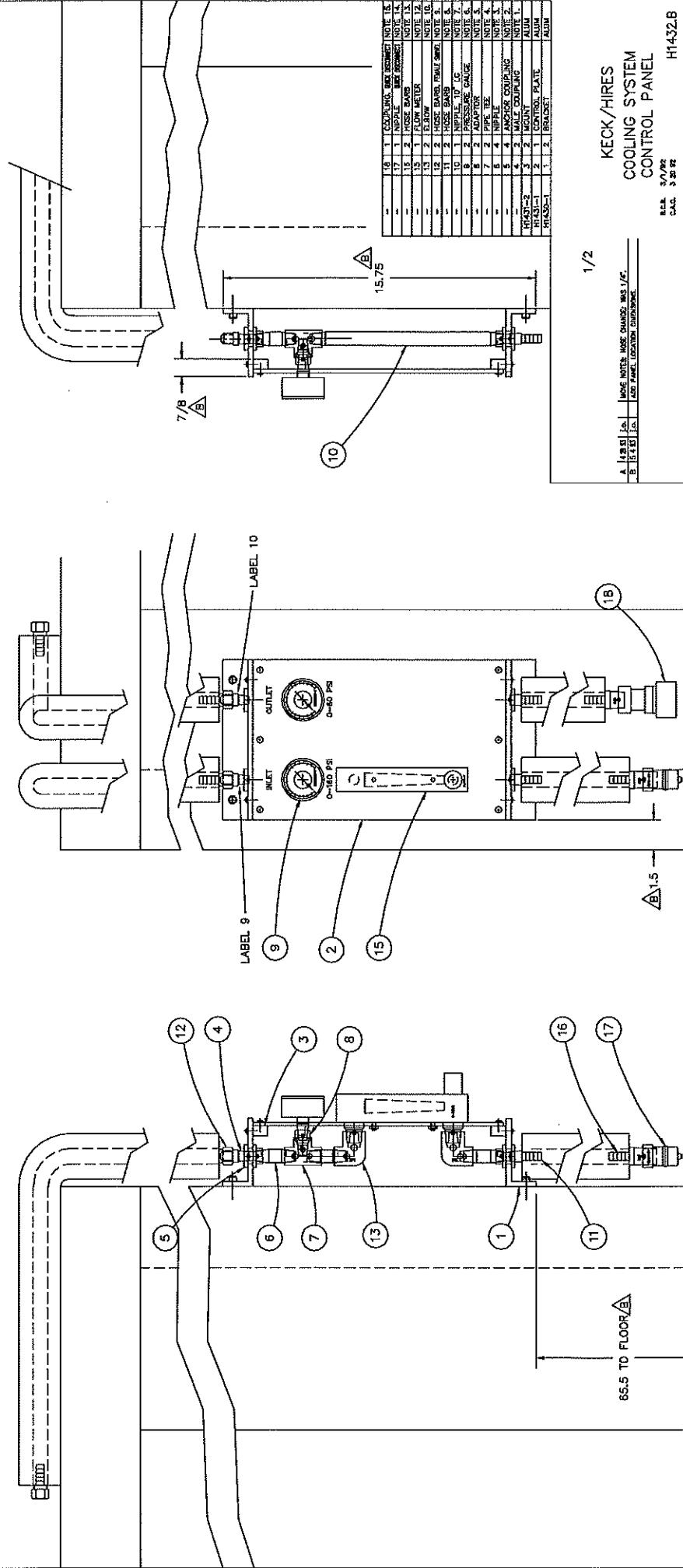


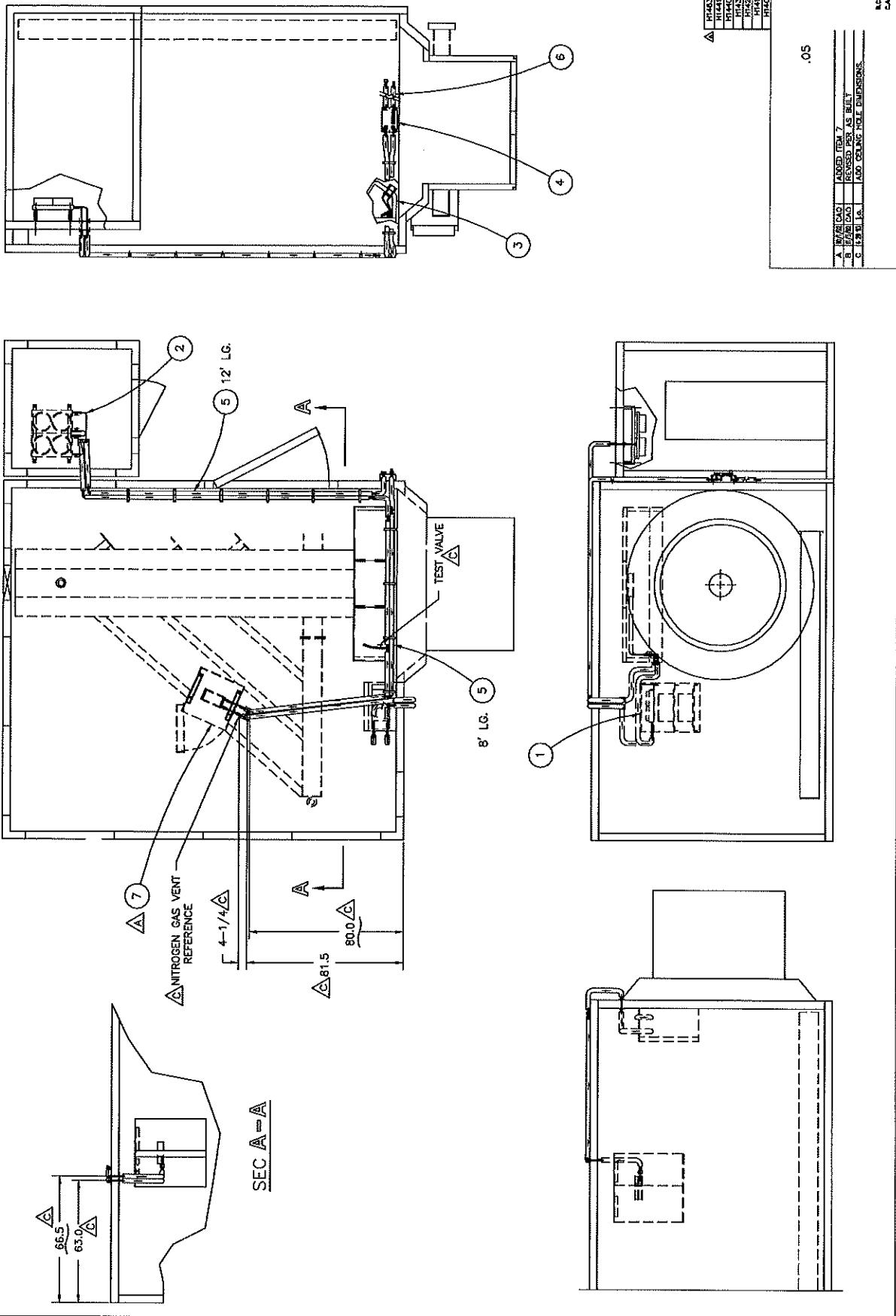
(1) CONTROL PANEL COOLING SYSTEM

△ NOTES:

1. PARKER 48F-6-6, 3/8 MALE NPT TO 45° FLARE
2. PARKER 207ACBH-6, 3/8 NPT TO 3/8 NPT
1-1/4 THD O.D. x 1.31 LOA
3. MCMASTER-CARR 436BK152, 3/8 NPT NIPPLE x 1-1/2 LG
4. PARKER 120SP-6, 3/8 NPT "T"
5. PARKER BUSHING, 20SP-4-6
6. MCMASTER-CARR 4000K54, 1/4 NPT PRESSURE GAGE
7. MCMASTER-CARR 436BK165, 3/8 NPT NIPPLE, 10" LOA
MODIFY TO FIT.

DO NOT SCALE FITTINGS.

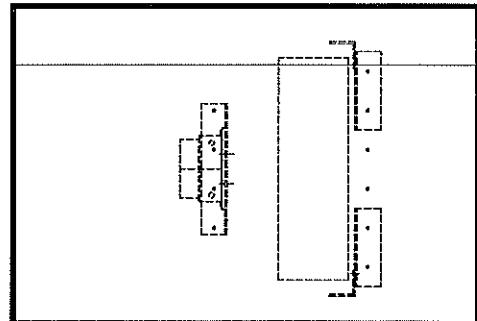
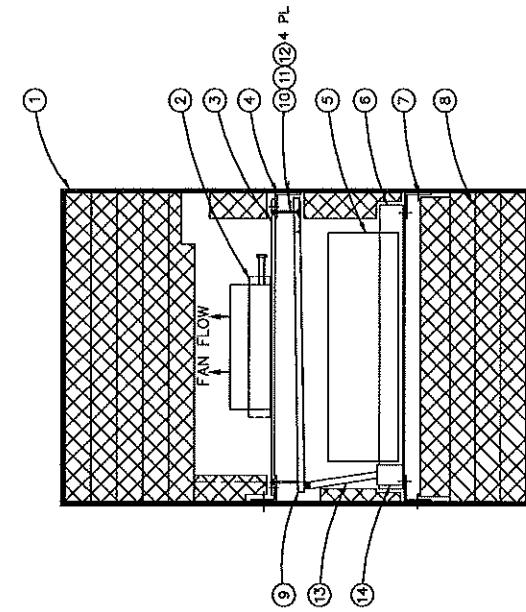
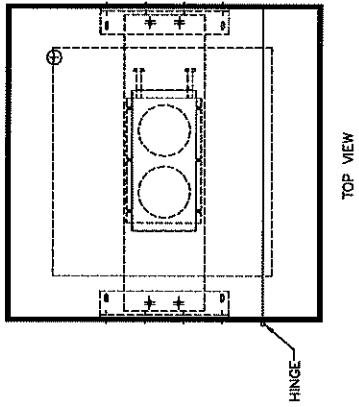




(-1) CCD ELECTRONICS COOLING ASSY

NOTES:

1. LYTRON INC.
DRAGON CT
WOBURN, MA 01801
617/933/7300
2. WARD-BRASBY PLASTICS
1510 PIFER DR.
MILPITAS, CA
408/262/2111
3. WIRING AND PLUMBING HOLES
LOCATED AS RECD.
4. INSULATION THICKNESS, 2-1/4".



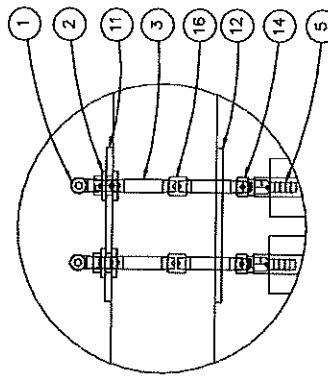
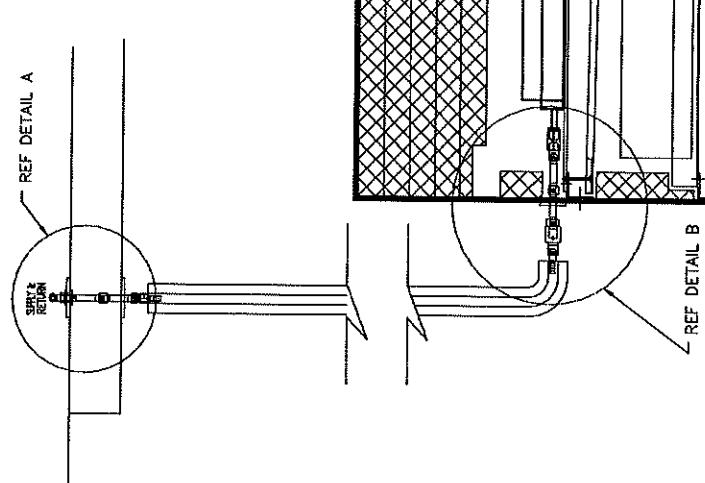
-	14	1	CONTAINER
-	13	1/8" TUBE, 7/8" RED	SILICON
-	12	4	MFL. 8-32
-	11	2	CORNS
-	10	3	SCREW, 1/4-20, 1/2" LONG
REF ID:	9	1	SCREW, 1/4-20, 1/2" LONG
-	8	1/4"	SCREW, 1/4-20, 1/2" LONG
H1451-1	7	4	BRACKET
H1451-2	5	2	BRACKET
-	5	1	CCD ELECTRONICS
H1451-3	4	2	BRACKET
H1451-4	3	1	SHLF
LDT S21C	2	1	HEAT EXCHANGER
-	1	1	CONTAINER

1/4

KECK/HIRES
CCD ELECTRONICS COOLING
ASSMBLY

9/20/92
H1462.C

4	1/2" O.D.	REMOVED FOR ASSEMBLY
3	1/2" I.D.	ADDED TO ADD FIBER OPTIC
2	1/2" I.D.	FEED THRU HOLE



DETAIL A

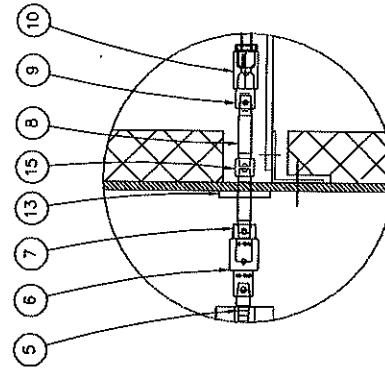
SCALE: 1/2

VIEW IS 90° FROM ORIGINAL

(-1) CCD COOLING INSTALLATION

NOTES:

1. ITEM 14, 4 EA, USED TO CONNECT TO EXISTING COOLING SYSTEM, SEE H1450.
2. NOT TO SCALE.



DETAIL B

SCALE: 1/2

NTS

KECK/HIRES	
A. PER CAO	REVISED PER AS BUILT
B. PER CAO	UPDATE PER AS BUILT
C. 12/91	REPLACE FIBER OPTIC SKT. AND NOTE

CCD COOLING
INSTALLATION DETAILS
H1463.C
REV. 9/22/92

-	15	2	1/2 COUPLING
-	15	2	1/2 COUPLING
-	15	3	1/2 COUPLING, 1/2 IN. 1/2 OUT
H1463.C	4	1	1/2 COUPLING, 1/2 IN. 1/2 OUT
H1463.C	2	1	1/2 COUPLING, 1/2 IN. 1/2 OUT
H1463.C	1	1	1/2 COUPLING, 1/2 IN. 1/2 OUT
-	9	2	FLARE UNION, 3/8
-	9	2	1/2" MALE UNION, 3/8
-	8	4	1/2" NIPPLE, 2.0 LC
-	7	2	1/2" FEMALE UNION, 2.0 LC
-	8	2	1/2" COUPLING, 1/2 IN. 1/2 OUT
-	5	2	1/2" COUPLING, 1/2 IN. 1/2 OUT
-	4	2	1/2" COUPLING, 1/2 IN. 1/2 OUT
-	3	4	1/2" NIPPLE, 2.0 LC
-	2	2	1/2" ANGLED COUPLING
-	1	2	1/2" ANGLE, 90°, 1/2 IN. 1/2 OUT

CCD COOLING
INSTALLATION DETAILS
H1463.C
REV. 9/22/92

Appendix U List of Drawings

— Camera Mirror

1. H5000 Locating Tree
2. H5560 Camera Mirror Cell Assembly
3. H5530 Camera Mirror Cell, Radial Support Sub-Assembly
4. H5550 Door Assembly
5. H5380 Pneumatic Control Panel
6. H5514 Radial Support Load Testing
7. H5537 Axial Support Details
8. H5536 Axial Support Details
9. H5539 Torsional Link Assembly
10. H5824 Horizontal Mirror Testing
11. H5825 10° Mirror Testing (As Used in HIRES)
12. Figure 1: Half Model Showing 3 Axial Supports
13. Figure 2: Half Model — ANSYS Model
14. Figure 3: Deflection Contours and Clear Aperture

H5000	THIS DRAWING		
H5016	CELL DOOR VS. DEWAR CLEARANCE		
H5100	OPEN	H5700	OPEN
H5200	CORRECTOR TOOLING H5210 CORRECTOR TOOLS H5214 CAMERA MIRROR TOOLS	H5800	FABRICATION AND TEST TOOLING H5807 OPTICS PALET L/O
H5300	CORRECTOR CELLS AND SUPPORT STRUCTURE H5303 CORRECTOR BASE ASSY (ISOMETRIC) H5314 CORRECTOR CELL TOOLING L/O H5324 MENISCUS CORRECTOR CELL ASSY H5328 CORRECTOR CELL AXIAL SUPPORT ASSY'S H5324 MEN. CORR. DOOR ASSY H5349 BICONVEX CORRECTOR CELL ASSY H5363 CORRECTOR CELL ASSEMBLY FIXTURE H5370 BI-FOLDING DOOR ASSEMBLY	H5900	FEA MODELS AND TEST (H4532 CROSS DISPERSER DOORS VS. BIFOLDING DOOR CLEARANCE)
H5400	CAMERA MIRROR AND TOOLING H5401 CAMERA MIRROR L/O H5409 CAMERA MIRROR SUPPORT ASSY (FOR STRASBAUGH) H5411 CAMERA MIRROR 25" LAP (STRASBAUGH) H5412 CAM. MIRROR 30" LAP (STRASBAUGH) H5415 CAM. MIRROR (HEXTEK DRAWING) H5420 BEAM1 RAYTRACE AT CAMERA PLANE H5421 BEAM1 RAYTRACE OF TELESCOPE AND HIRES		
H5500	CAMERA MIRROR CELL H5530 CAM. MIRROR RADIAL SUPPORT ASSY. H5536 CAM. MIRROR AXIAL SUPPORT ASSY. H5539 CAM. MIRROR TANGENT LINK ASSY. H5541 CAM. MIRROR CELL SPIDER ASSY. H550 CAM. MIRROR DOOR ASSY.		
H5600	FIELD FLATTENER H56XX		KECK/HIRES SUPERCAMERA LOCATING TREE H5000

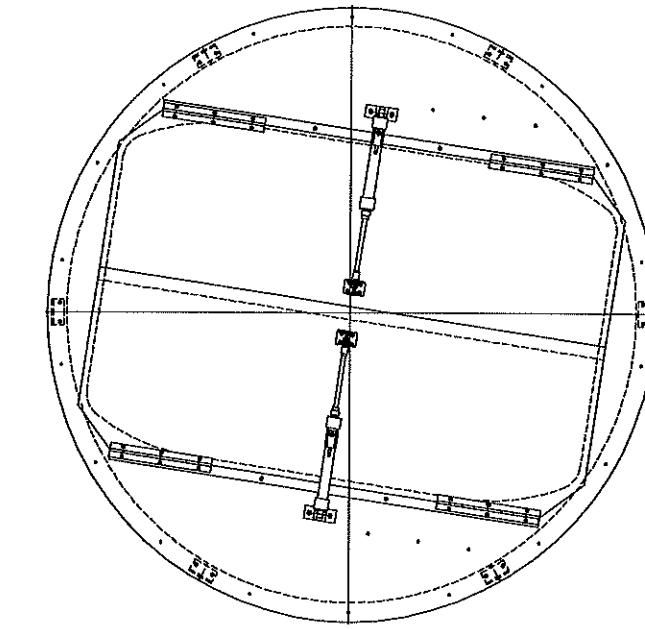
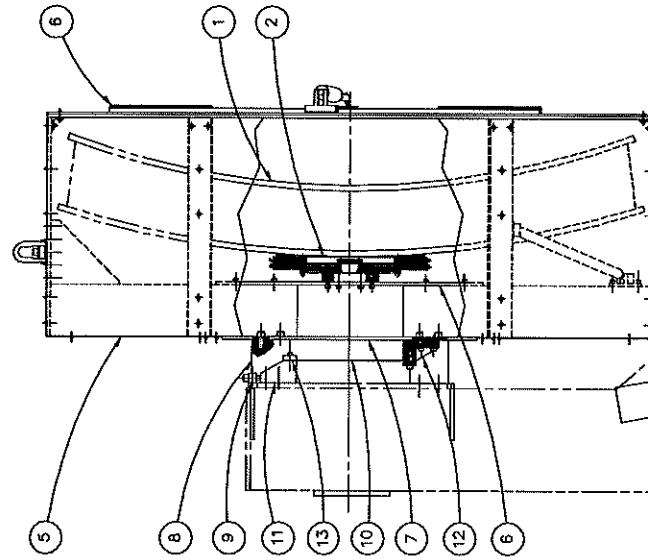
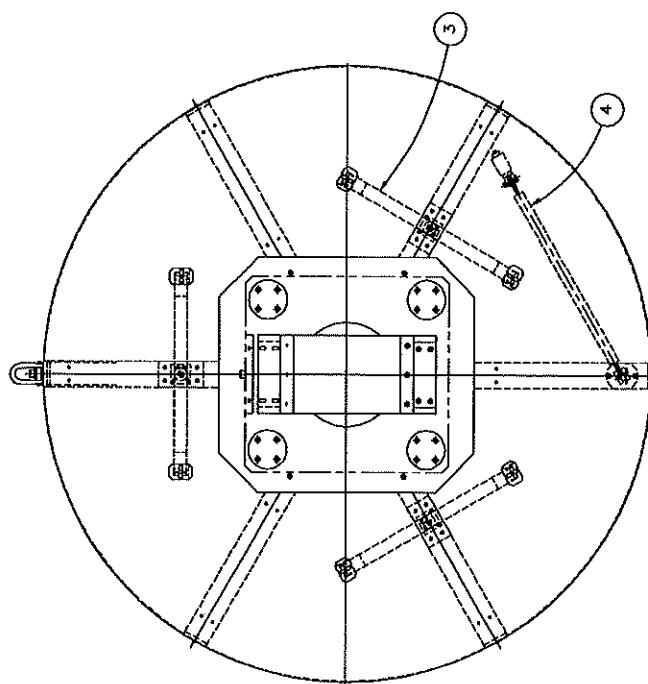
(-1) CAMERA MIRROR CELL ASSEMBLY

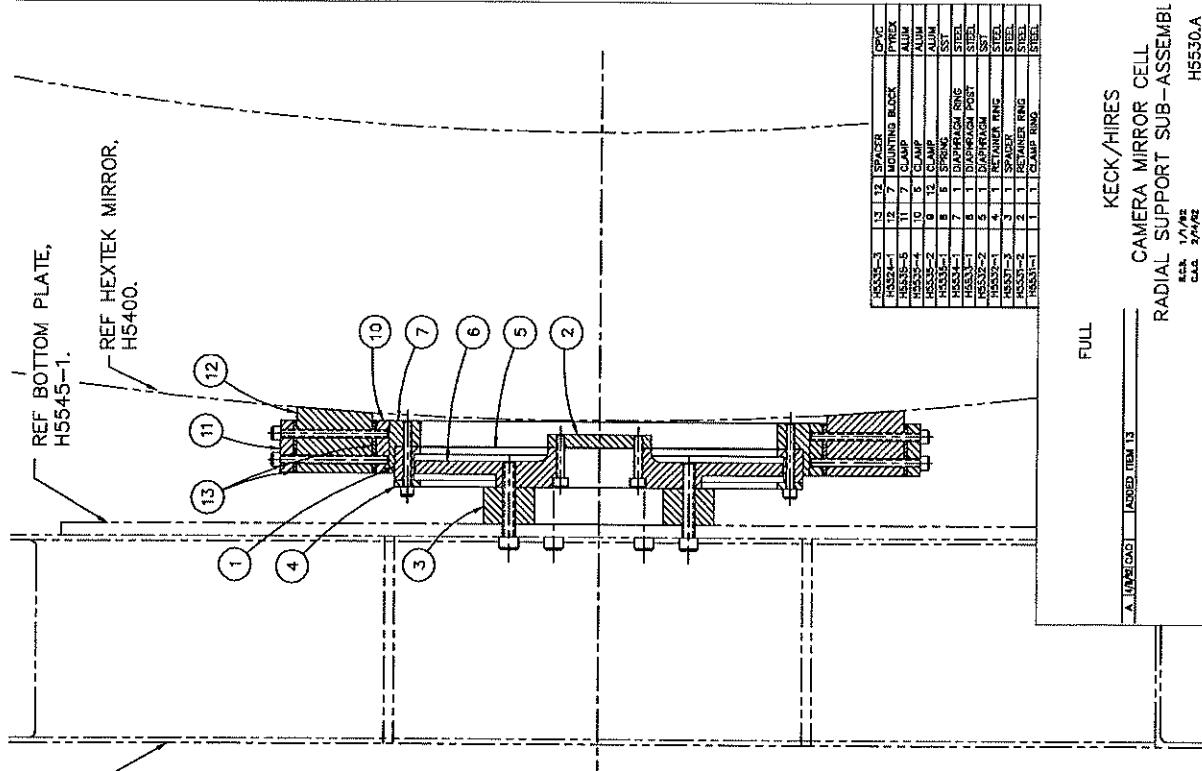
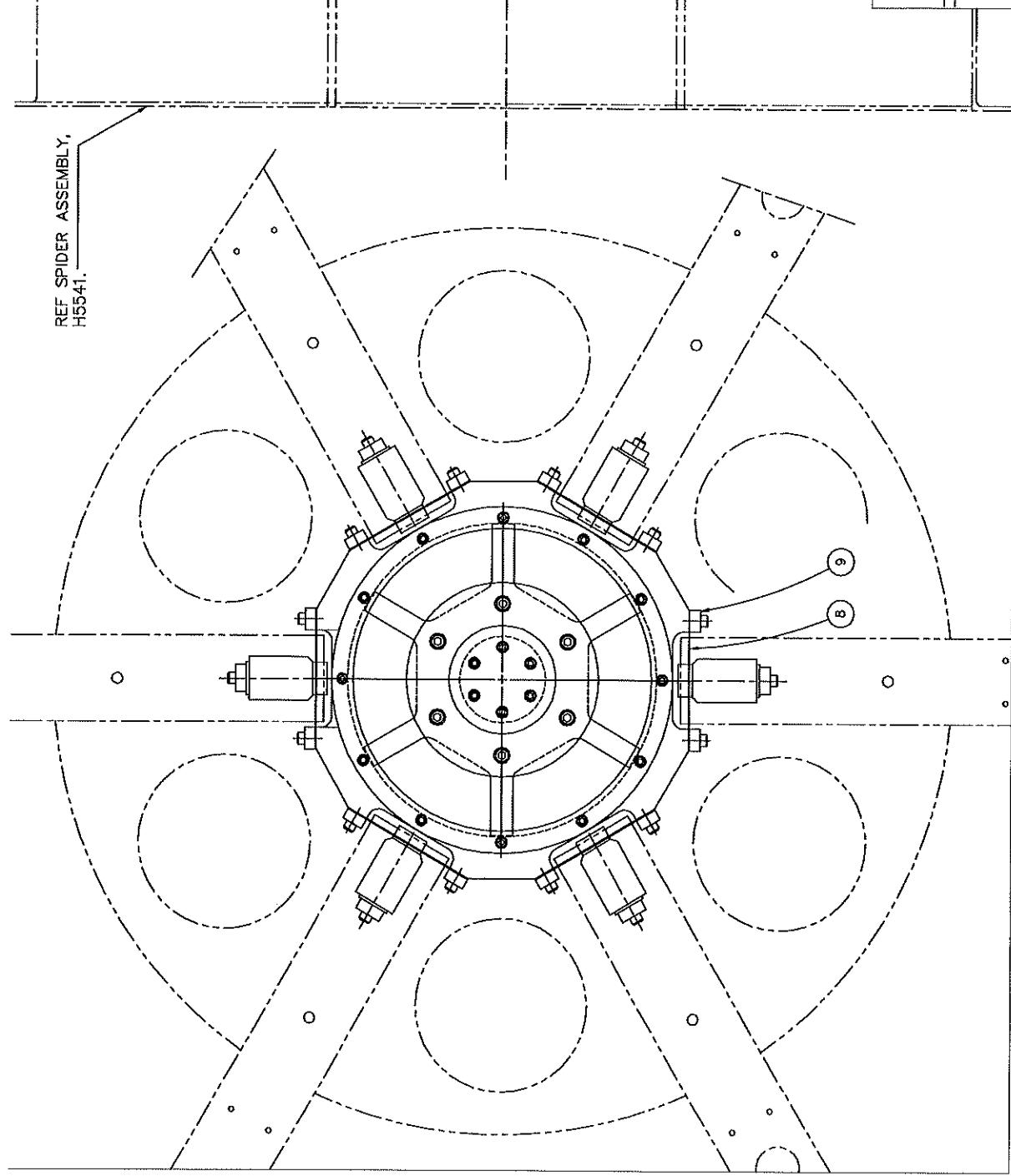
1/4

KECK/HIRES
CAMERA MIRROR
CELL ASSEMBLY
S.R.N. 1A/2/2
C.A.C. 4/20/92
H5560

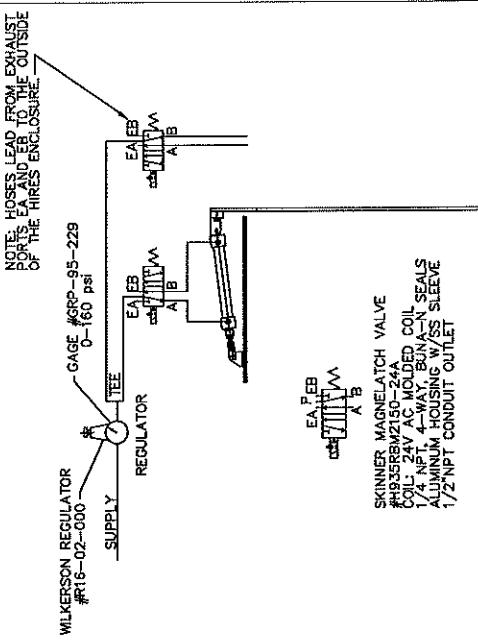
REF ID	DESCRIPTION
H5560-1	SPIDER ASSY
H5560-2	TOP PLATE
H5560-3	DOOR ASSY
H5560-4	SPIDER ASSY
H5560-5	TANGENT LINE ASSY
H5560-6	AXIAL SUPPORT ASSY
H5560-7	RADIAL SUPPORT ASSY
H5560-8	HEATER MIRROR

REF WELDMENT ASSY
H6925





FULL
KECK/HIRES
CAMERA MIRROR CELL
RADIAL SUPPORT SUB-ASSEMBLY
S.C.N. 1/1/92
C.L.A. 2/2/92
H5530.A

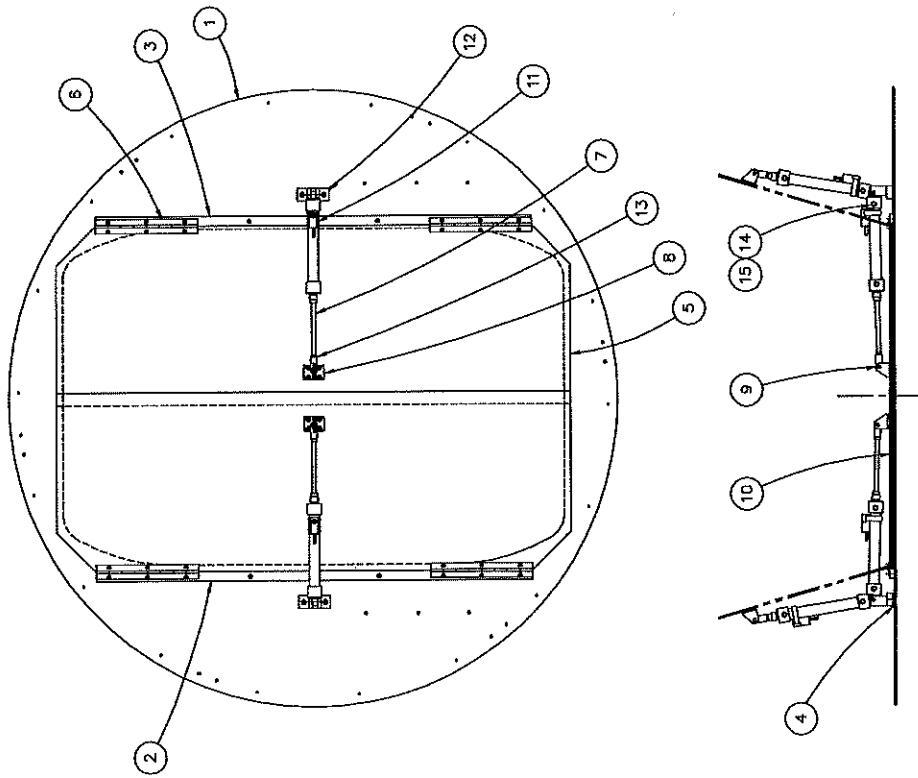


(-1) DOOR ASSEMBLY

NOTES

1. CELLULAR URETHANE, #4701-01-20125-1604
WITH ADCHEM #256 ADHESIVE.

1. BIMBA #MRS-044-DXF.
WITH ADCHEM #256 ADHESIVE.
BOTW CORP.
2209 FAIRVIEW DR.
CERES, CA 95307
 2. BIMBA #MRS-044-DXF.
 3. BIMBA #D-166-3.
 4. BIMBA #MRS-087-YBL-04.
144° LEADS, FITS 3/4" CYLINDER.
 5. BIMBA #D-167.
 6. BIMBA #CP2, 1/8 NPT.
 7. SWAGELOK FITTING, #B-400-1-2 (1/8 NPT).
 8. SWAGELOK FITTING, #B-400-1-4.
 9. REFERENCE DRAWING H4750 FOR PNEUMATIC
DIAGRAMS.

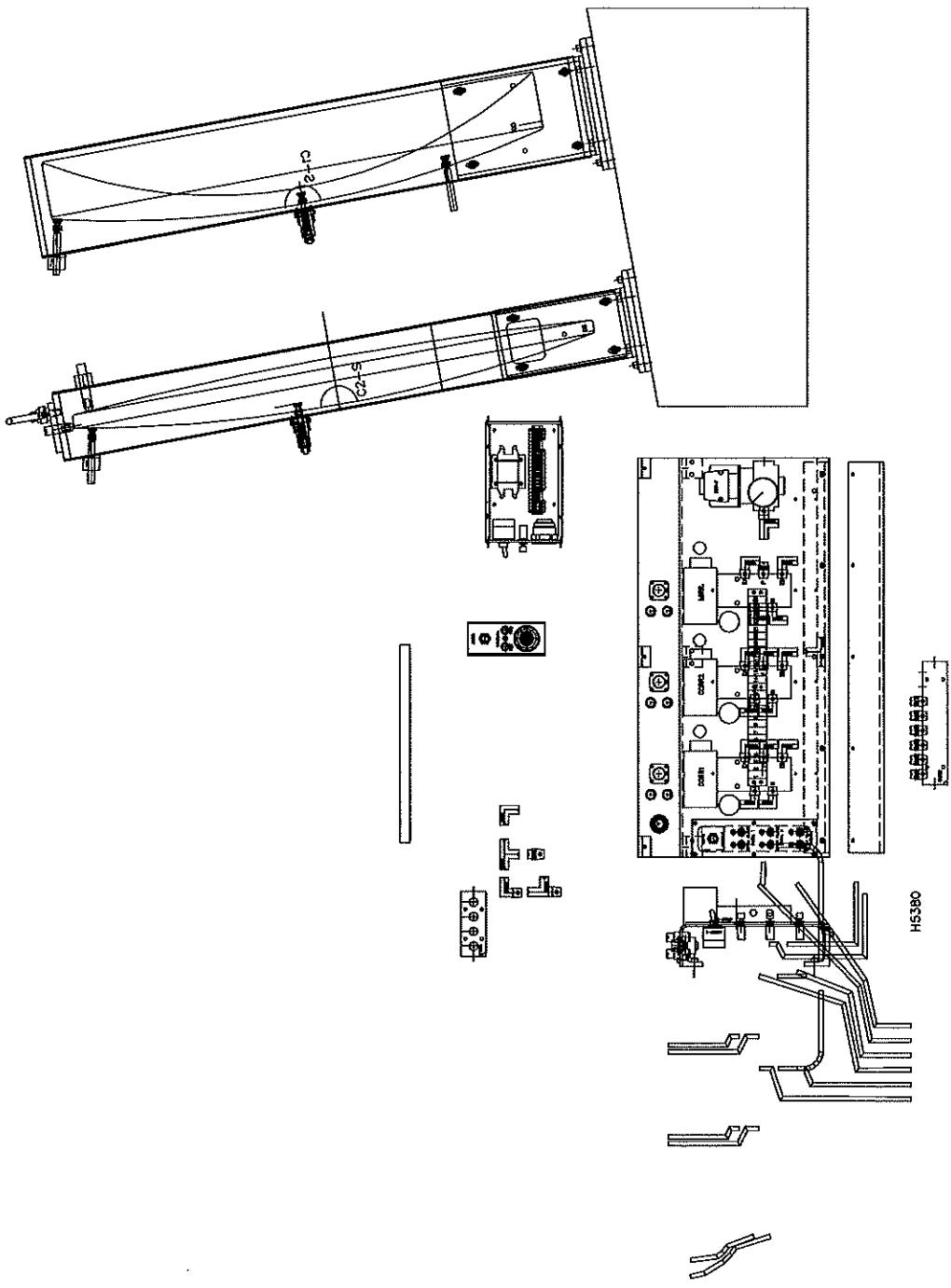


1 / 4

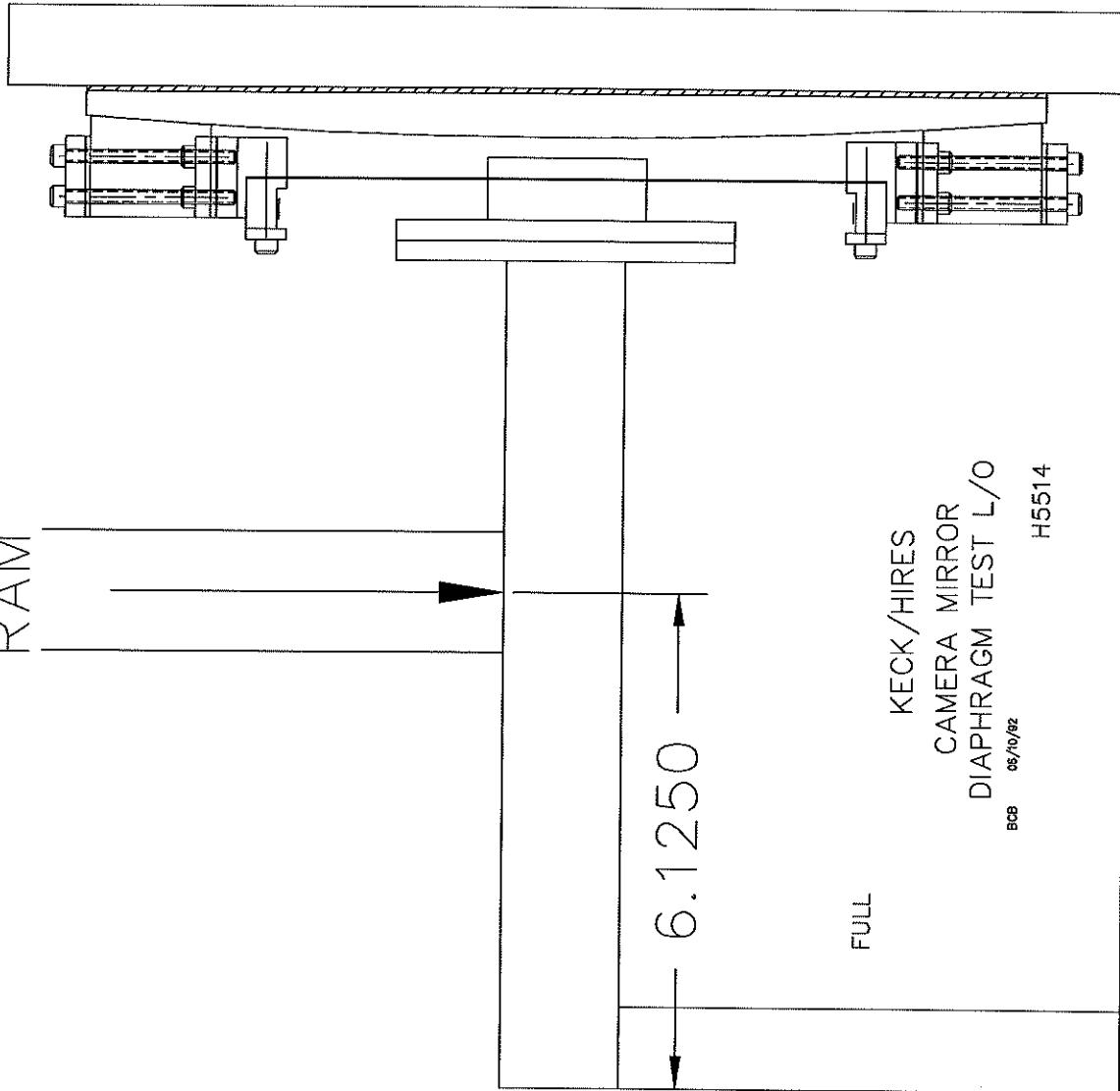
KECK/HIRES

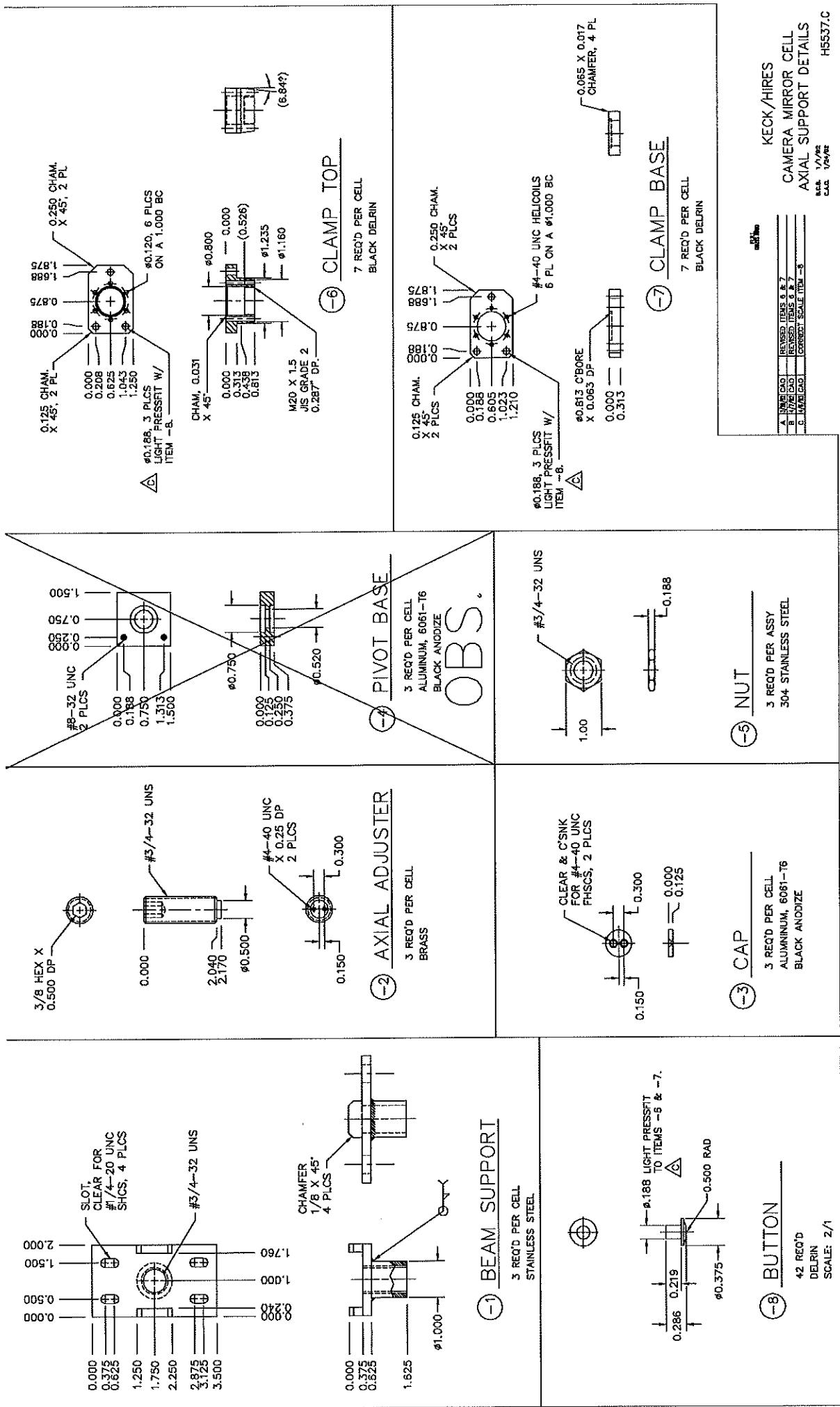
CAMERA MIRROR CELL DOOR ASSEMBLY

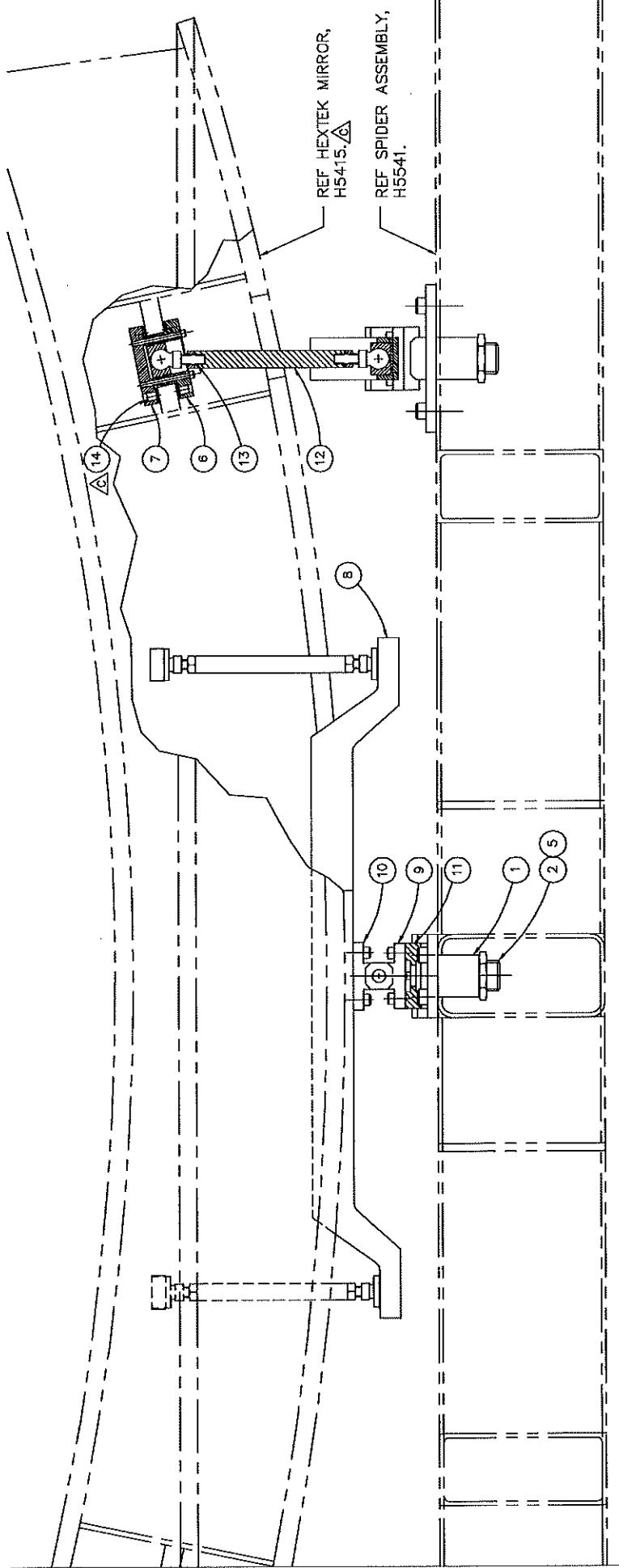
N/A
KECK/HIRES
CORRECTOR CELLS
AIR SYSTEM LAYOUT
BCB 2 18 42
H5380.A



RAM







(1) AXIAL SUPPORT SUB-ASSEMBLY

REF	QTY	DESCRIPTION
H5537-6	14	BUTTON
-	13	LINK BALL, TFE TEFLON
H5537-5	12	SPRING
H5537-4	10	SPRING, STAINLESS
H5537-3	8	SPRING, STAINLESS
H5537-2	6	SPRING, STAINLESS
H5537-1	5	SPRING, STAINLESS
H5537-7	4	ANL CLAMP
H5537-6	7	ANL CLAMP BASE
H5537-5	6	CLAMP
H5537-4	5	CLAMP BASE
H5537-3	4	SPRING, BASE
H5537-2	3	CAP
TINY-	2	AXIAL ADJUSTER
TINY-	1	BEAM SUPPORT

FULL

KECK/HIRES

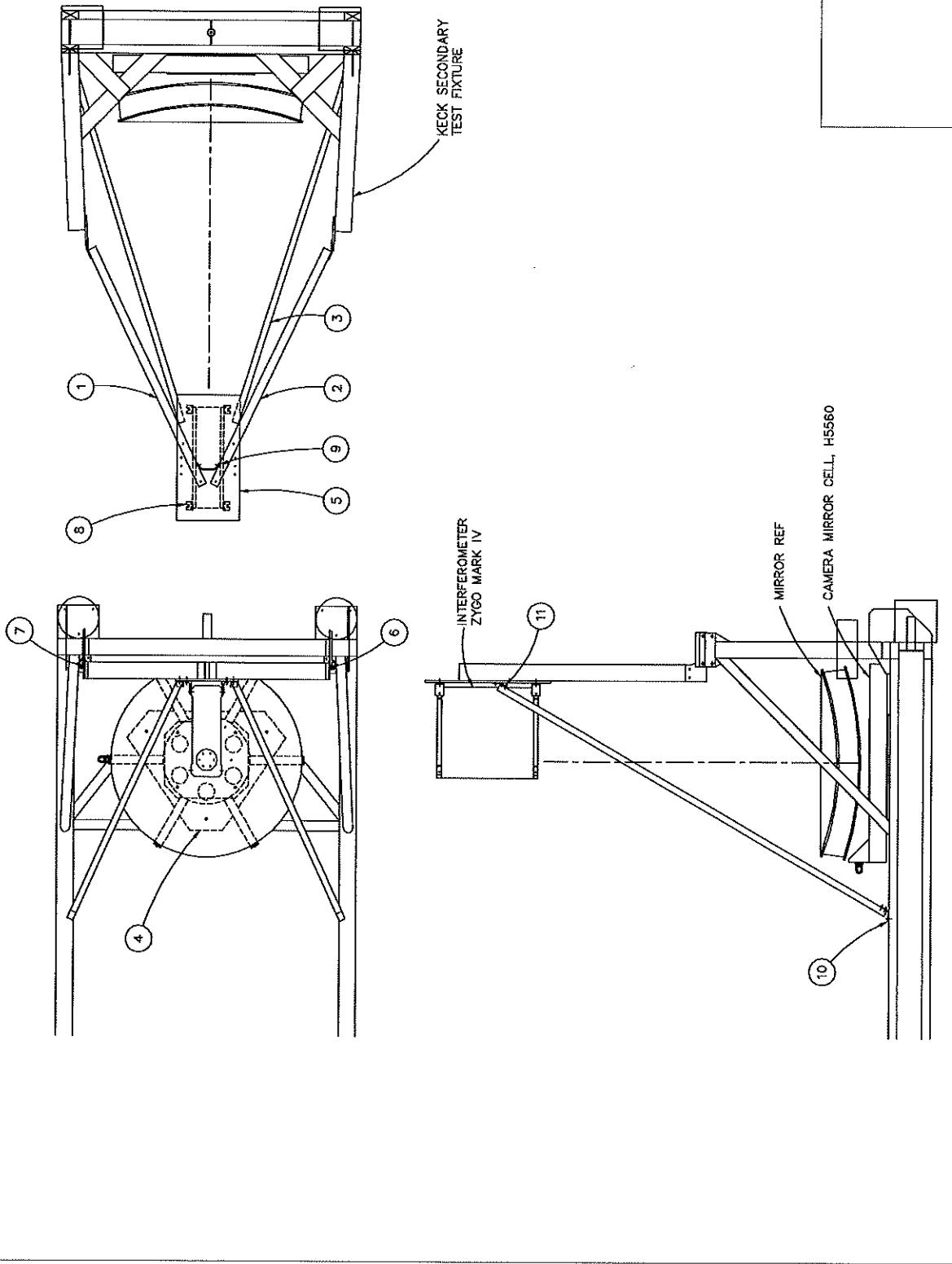
CAMERA MIRROR CELL
AXIAL SUPPORT ASSY

H5536.C

N.O. 1/1/92
C.A. 1/26/92

REF HEXTEK MIRROR,
H5415.

REF SPIDER ASSEMBLY,
H5541.



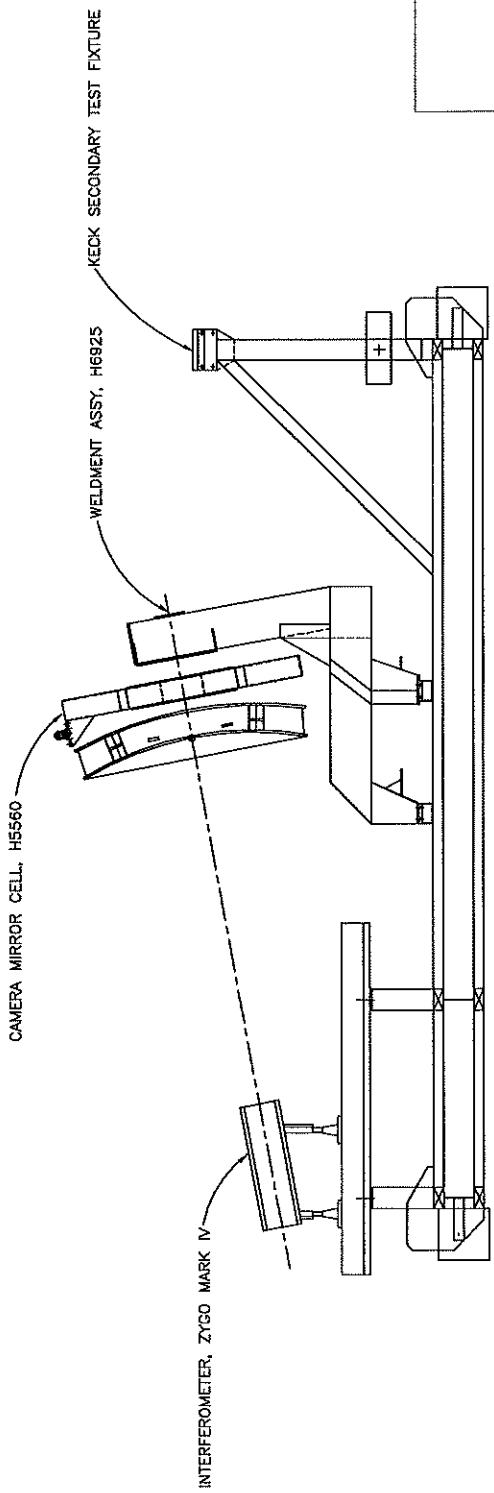
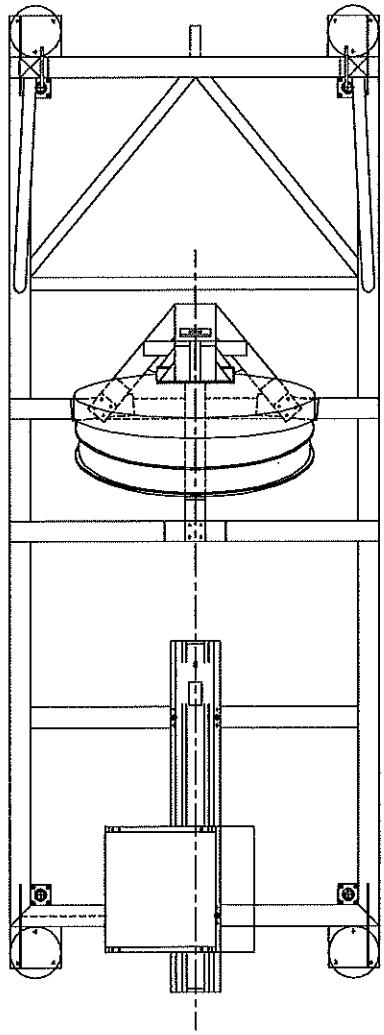
H5560-4	11	2	BRACKET	ALUM
H5560-3	10	2	BRACKET	ALUM
H5560-2	9	2	BRACKET	ALUM
H5560-1	8	4	BRACKET	ALUM
H5560-3	7	1	BRACKET	ALUM
H5560-2	6	1	BRACKET	ALUM
H5560-1	5	1	MOUNTING PLATE	ALUM
H5560-2	4	1	MOUNTING PLATE	ALUM
H5560-1	3	2	SURF	STEEL
H5560-2	2	1	SURF	STEEL
H5560-1	1	1	SURF	STEEL

.1

KECK/HIRES
CAMERA MIRROR
HORIZONTAL TEST ASSY
B.C.B. 4/7/92
G.L.D. 4/7/92
H5824

E.D.M. 7/1/92
C.A.O. 8/2/92
H6925

.1
KECK/HIRES
CAMERA MIRROR
10° TEST ASSY



ANSYS 4.4A
JUL 22 1992
15:44:32
PLOT NO. 2
POST1 NODES

ZV =-1
DIST=24.962
XF =11.346
ZF =65.418
FACE HIDDEN

POST1 NODES
TDIS

ZV =-1
DIST=24.962
XF =11.346
ZF =65.418
FACE HIDDEN

POST1 ELEMENTS
TYPE NUM
TDIS

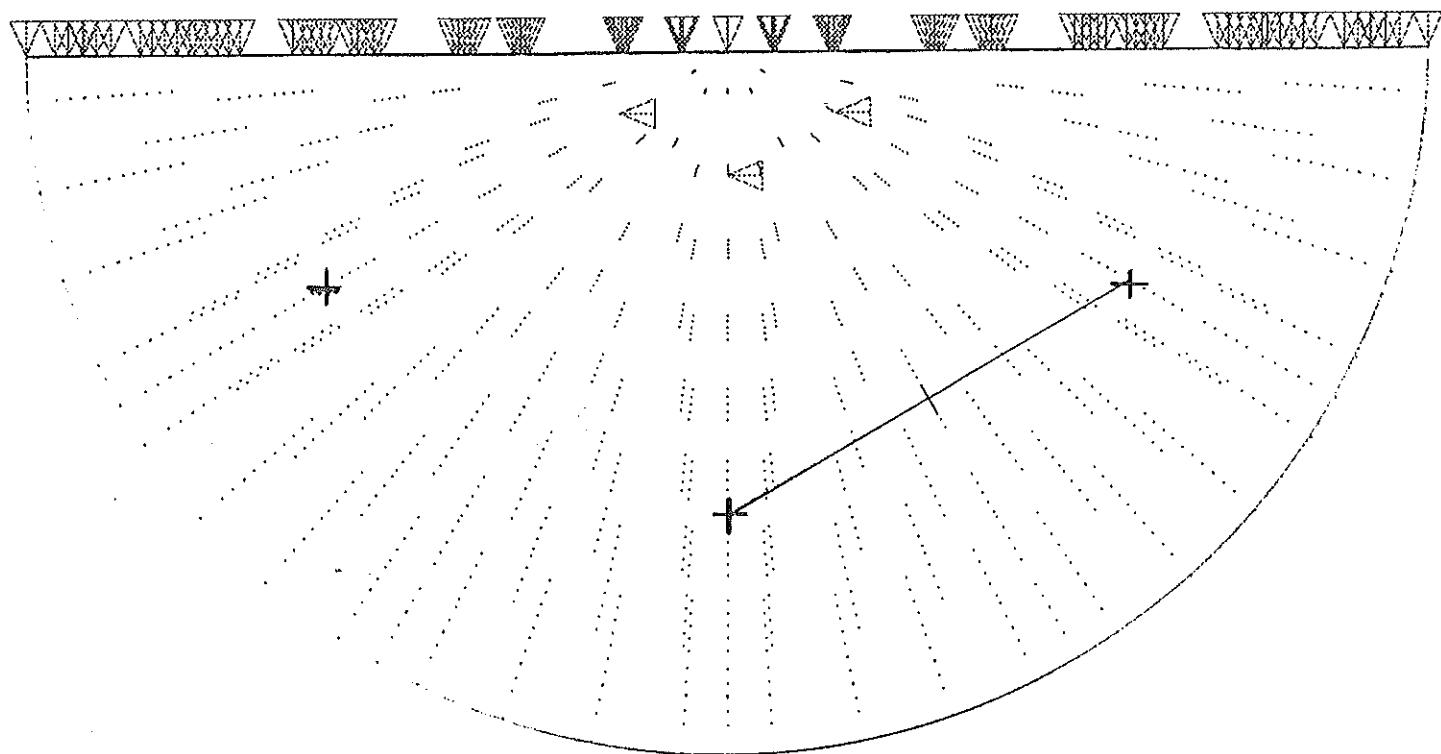


FIGURE 1

ANSYS 4.4A
AUG 10 1992
13:31:36
PLOT NO. 1
POST1 ELEMENTS
TYPE NUM

XV = -1
YV = 0.3
ZV = -1
DIST=26.997
XF = 11.346
ZF = 65.418
FACE HIDDEN

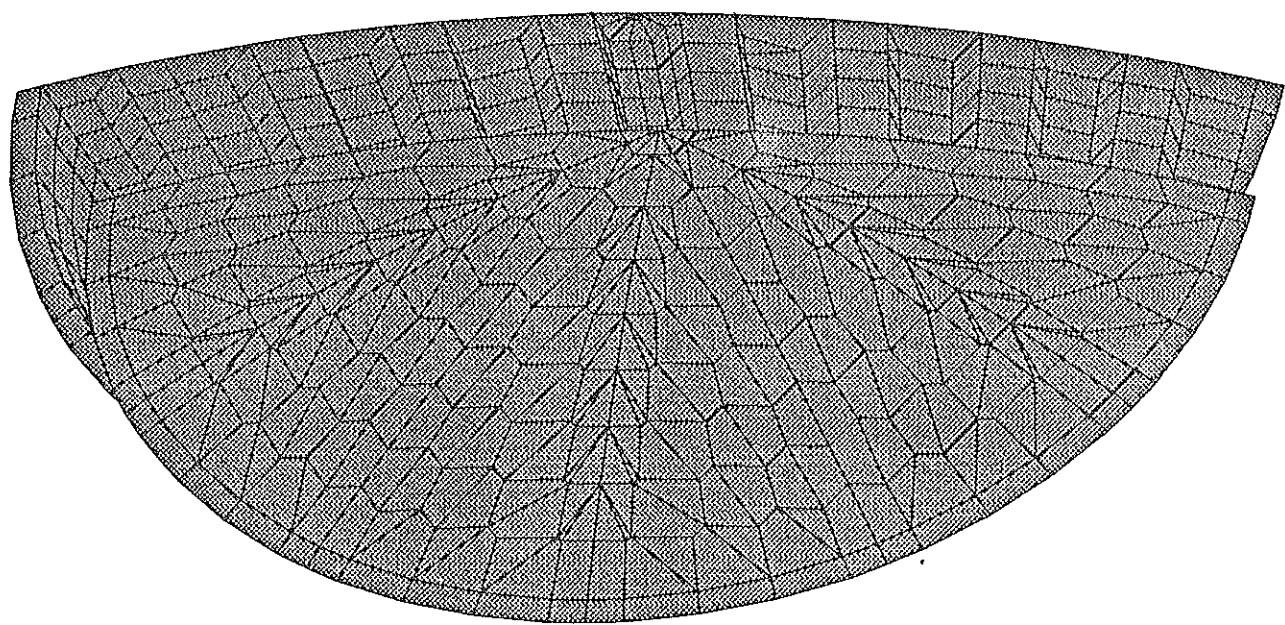


FIGURE 2

ANSYS 4.4A

JUL 22 1992

15:43:27

PLOT NO. 1
POST1 STRESS

STEP=1
ITER=1
UZ

D GLOBAL
DMX = 0.232E-04
SMN = -0.169E-04
SMX = 0.194E-05

ZV = -1
DIST=24.962
XF = 11.346
ZF = 65.418
FACE HIDDEN
0.169E-04
-0.148E-04
-0.127E-04
-0.106E-04
-0.851E-05
-0.642E-05
-0.433E-05
-0.224E-05
-0.154E-06
0.194E-05

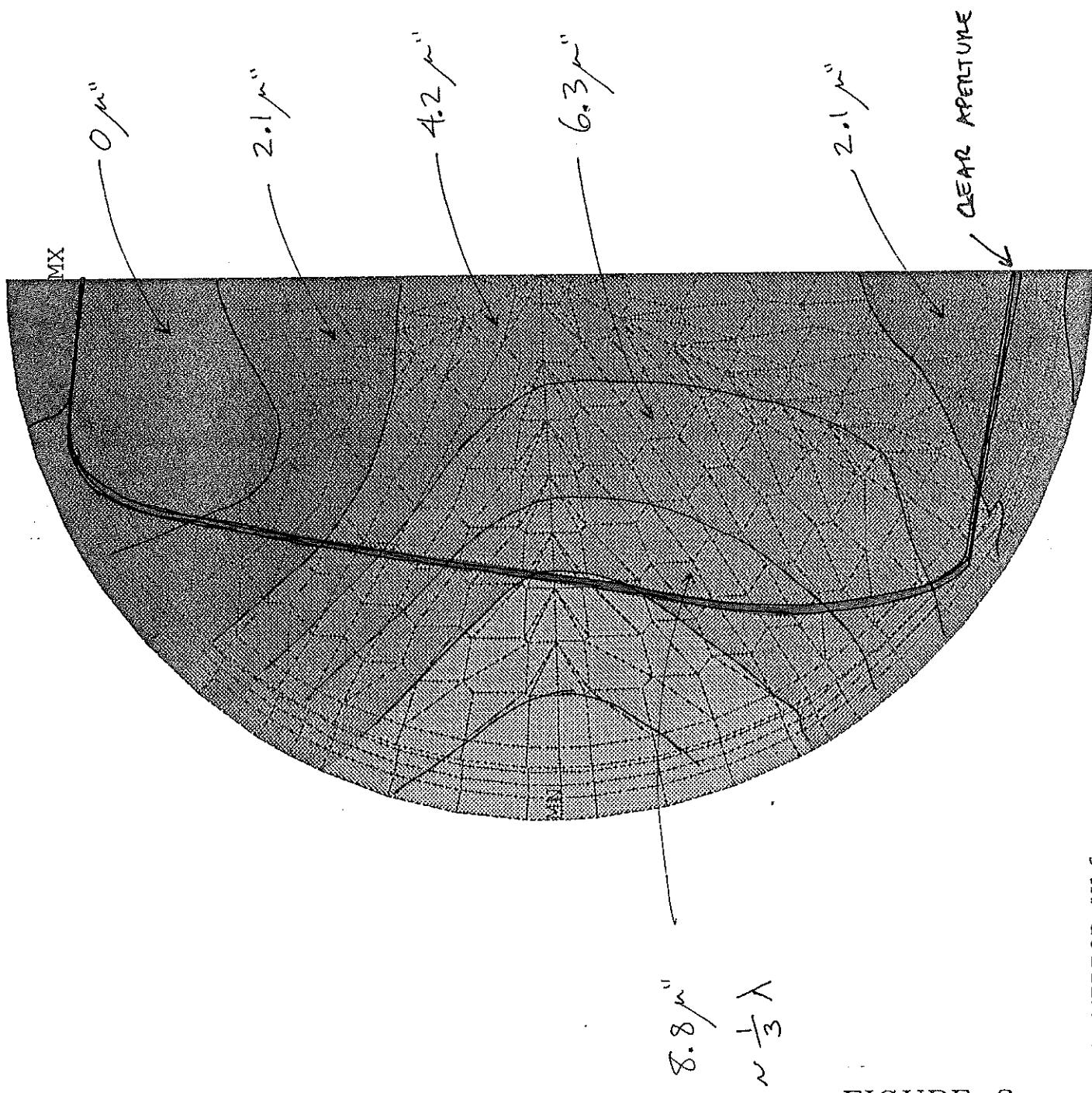


FIGURE 3

Appendix V List of Drawings

— Corrector Lenses

1. Figure 1: Finite Element Half Model Showing the Application of Radial Forces
2. Figure 2: Finite Element Contour Plot
3. Figure 3: FEA Contour Plot Showing Clear Aperture of Lens
4. Figure 4: Finite Element Contour Plot
5. Figure 5: FEA Contour Plot Showing Clear Aperture of Lens
6. Figure 9: Interferometer Output, No Forces
7. Figure 11: WISP Output, No Forces
8. Figure 17: Interferometer Output, Optimum Support
9. Figure 18: WISP Output, Optimum Support
10. H0148 Super-Duper III Camera Layout
11. H5000 Locating Tree
12. H5266 Meniscus Lens Sling Assembly
13. H5267 Biconvex Lens Sling Assembly
14. H5324 Meniscus Cell Assembly
15. H5328 Axial Support Details
16. H5349 Bi-Convex Cell Assembly
17. H5363 Lens Installation/Removal Tool
18. H5370 Bi-Fold Door Sub-Assembly
19. H5375 Bi-Fold Door Air Logic
20. H5376 Bi-Fold Door Assembly
21. H5380 Air System Layout
22. H5384 Bi-Convex Lens Installation/Removal Procedure
23. H5385 Meniscus Lens Installation/Removal Procedure
24. H5811 Testing Meniscus Lens
25. H5826 Testing Meniscus Lens at 10°

2

ANSYS 4.4A
 JUL 22 1992
 15:37:25
 PLOT NO. 1
 PREP7 NODES
 TDIS
 FORC

YV =-1
 DIST=16.678
 YF =1.68
 ZF =-7.581
 VUP =-X

AXIAL
DEFINING
POINT

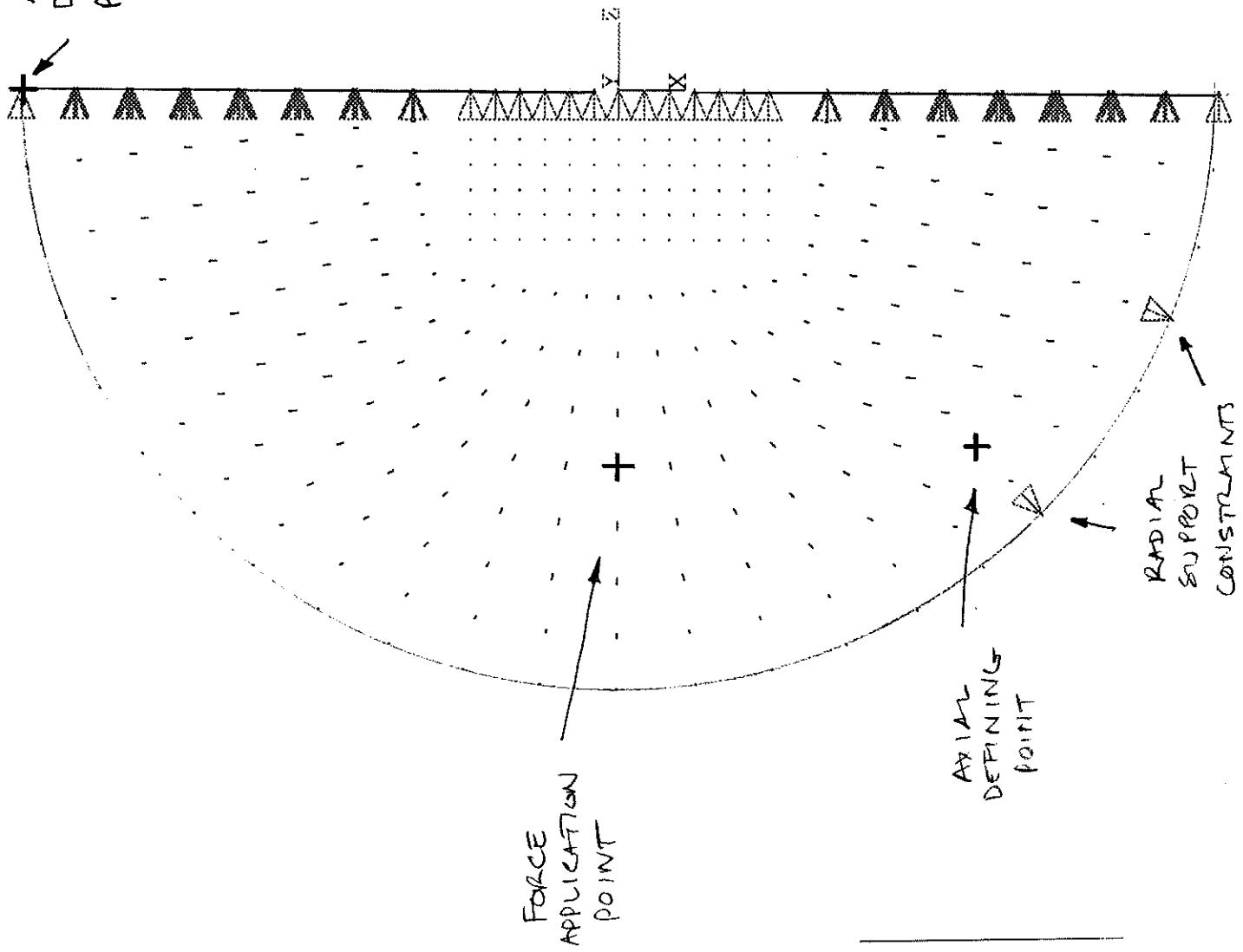


FIGURE 1

ANSYS 4.4A
JUL 22 1992
16:10:57
PLOT NO. 1
POST1 STRESS
STEP=2
ITER=1

UY
D GLOBAL
SMN =-0.358E-04
SMX =0.182E-04
TDIS

YV =-1
DIST=16.678
YF =1.68
ZF =-7.581
VUP =-X
FACE HIDDEN
-0.358E-04
-0.298E-04
-0.238E-04
-0.178E-04
-0.118E-04
-0.580E-05
0.194E-06
0.619E-05
0.122E-04
0.182E-04

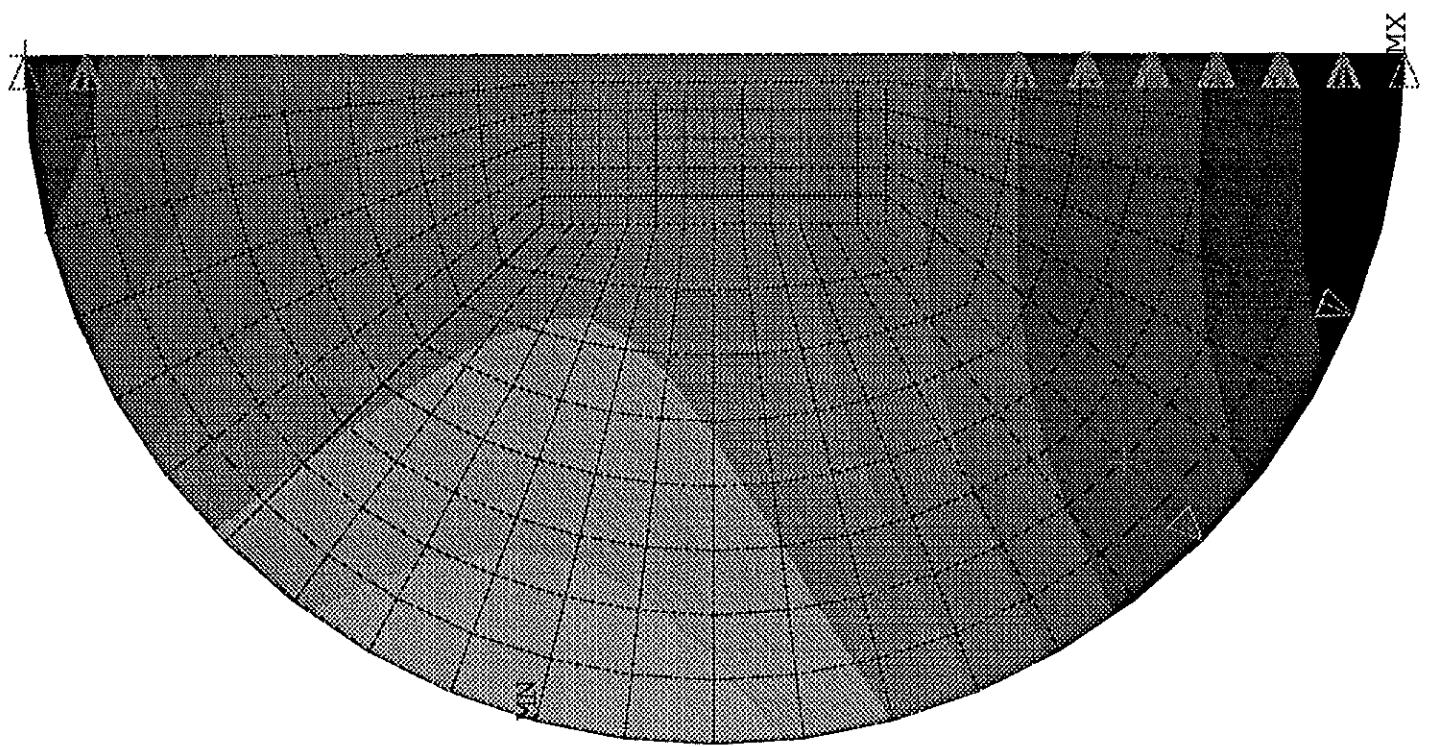


FIGURE 2

ANSYS 4.4A
JUL 22 1992
15:39:08
PLOT NO. 1
POST1 STRESS
STEP=3
ITER=1
UY

D GLOBAL
SMN =-0.558E-05
SMX =0.247E-05
TDIS

YV =-1
DIST=16.678
YF =1.68
ZF =-7.581
VUP =-X
FACE HIDDEN
-0.558E-05
-0.469E-05
-0.379E-05
-0.290E-05
-0.200E-05
-0.111E-05
-0.216E-06
0.678E-06
0.157E-05
0.247E-05

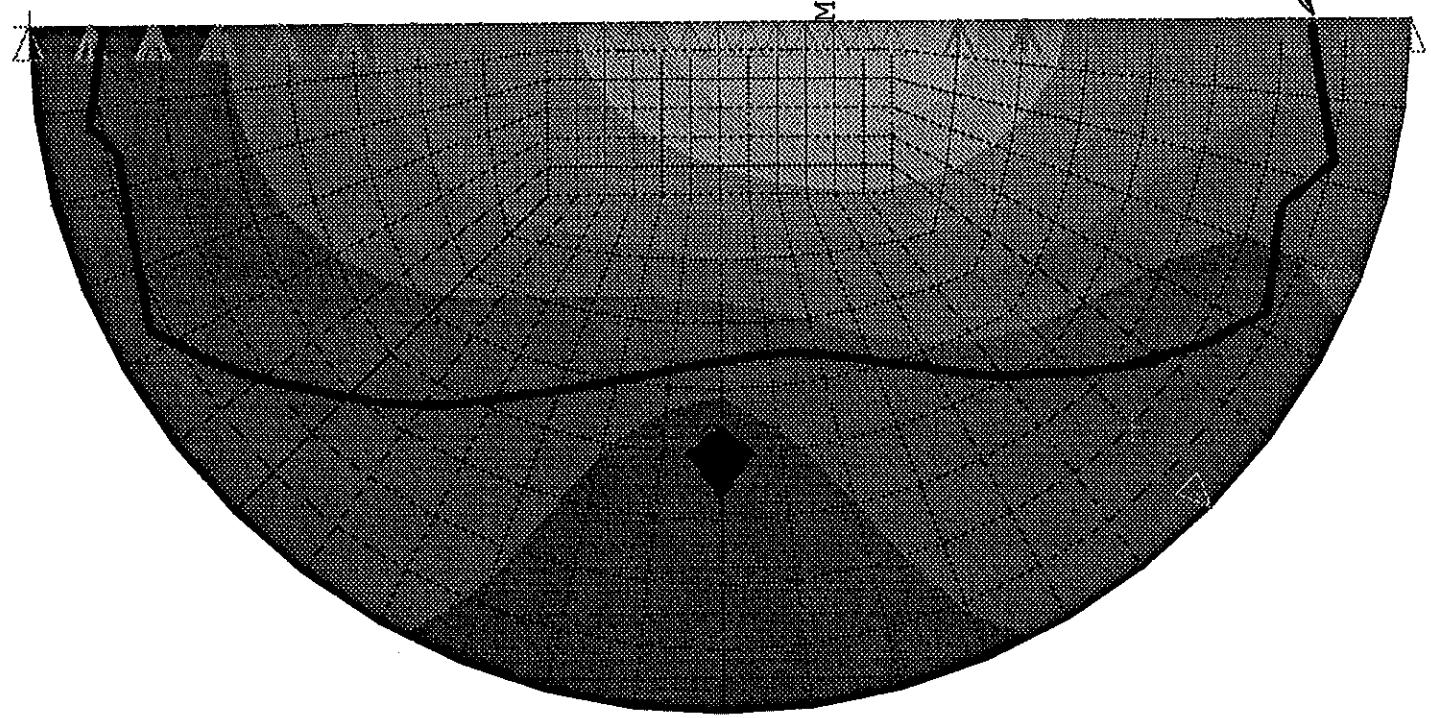


FIGURE 3

ANSYS 4.4A
AUG 18 1992
13:48:02
PLOT NO. 2
POST1 STRESS
STEP=2
ITER=1
UY
D GLOBAL
SMN =-0.298E-04
SMX =0.590E-05
FDIS
FORC

YY ==-1
DIST=17.677
YF ==-1.437
ZF ==-8.035
VUP ==-X
FACE HIDDEN
-0.298E-04
-0.258E-04
-0.219E-04
-0.179E-04
-0.139E-04
-0.997E-05
-0.600E-05
-0.203E-05
0.193E-05
0.590E-05

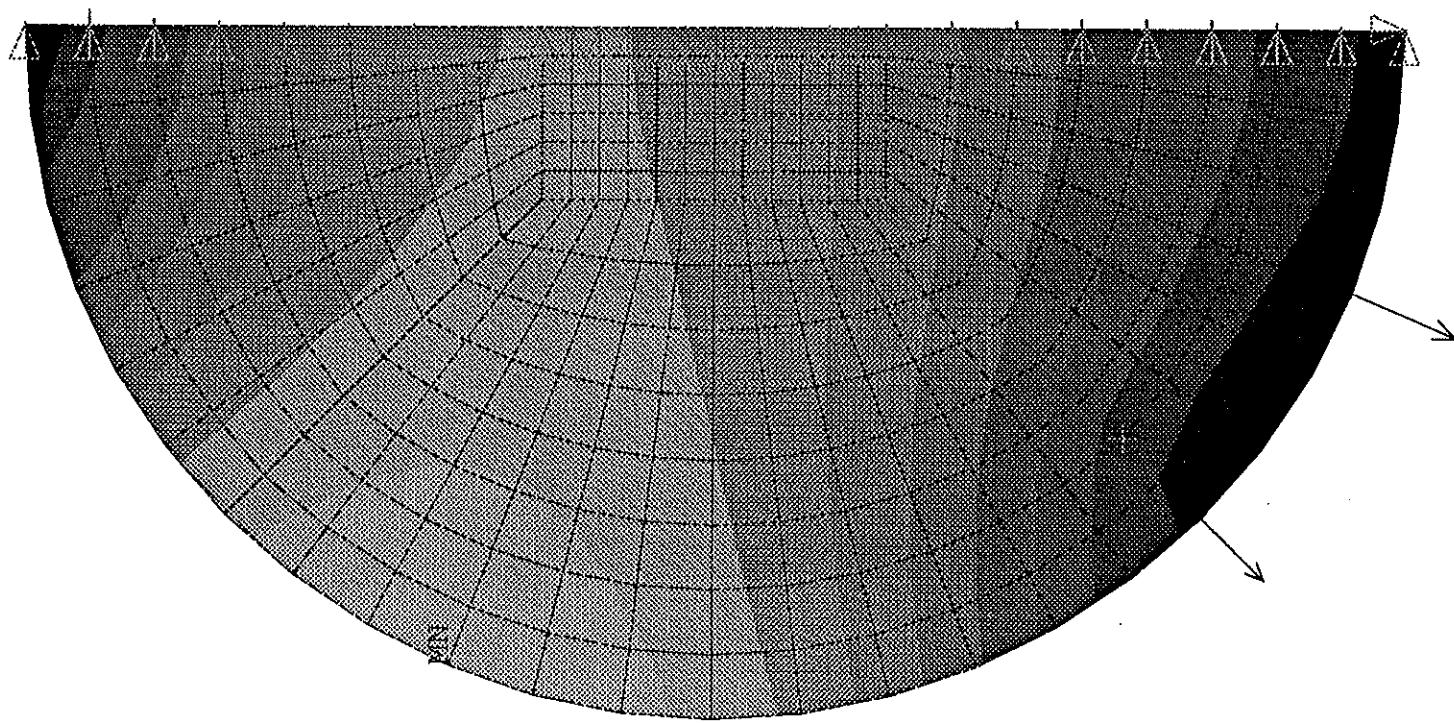


FIGURE 4

ANSYS 4.4A
AUG 18 1992

13:46:51
PLOT NO. 1
POST1 STRESS
STEP=1
ITER=1

UY

D GLOBAL

SMN = -0.686E-05

SMX = 0.130E-05

TDIS

FORC

YV = -1

DIST=17.677

YF = -1.437

ZF = -8.035

VUP = -X

FACE HIDDEN

-0.686E-05

-0.595E-05

-0.504E-05

-0.414E-05

-0.323E-05

-0.232E-05

-0.142E-05

-0.510E-06

0.396E-06

0.130E-05

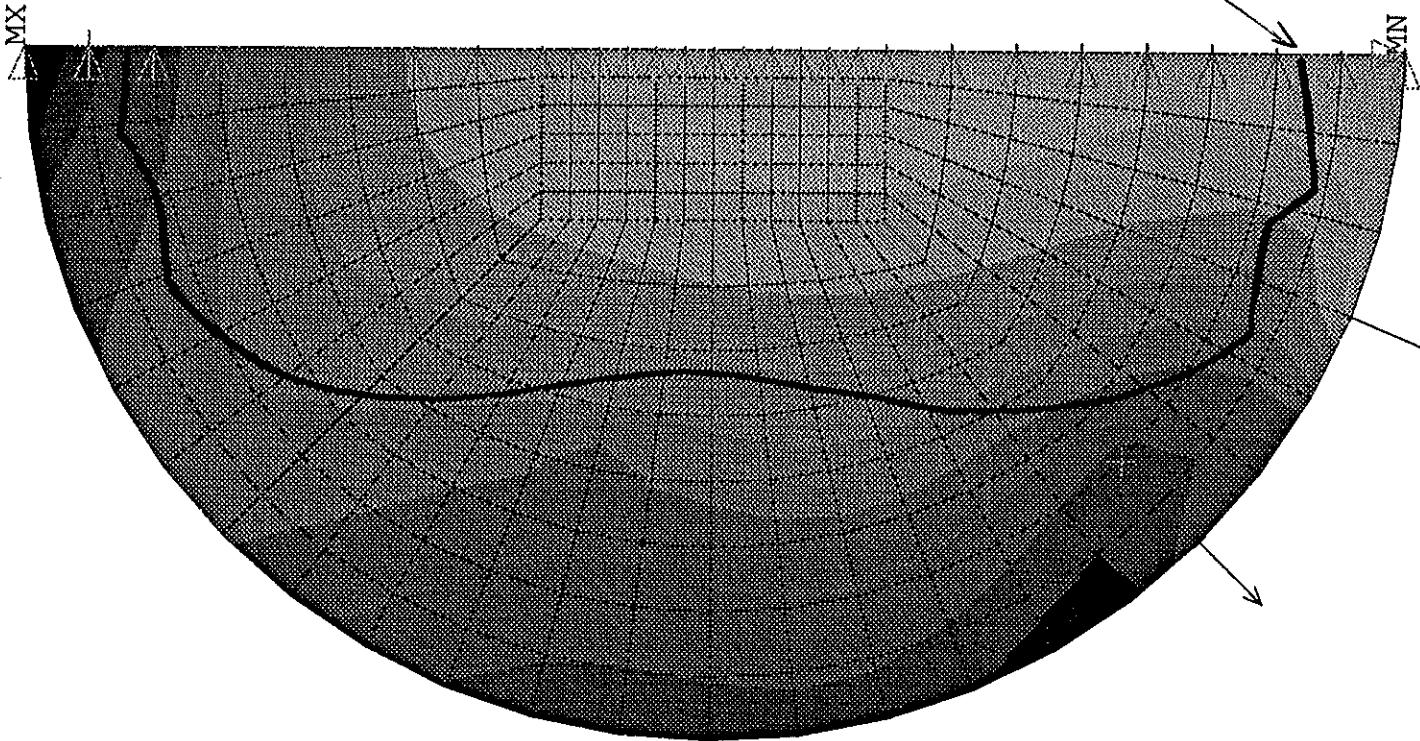


FIGURE 5

Corrector #2 T-67-92

10° Lean

No Tweeting / 3 pt.

File: ~~CR2~~ CR2NF#1
No Forces

#1

Support

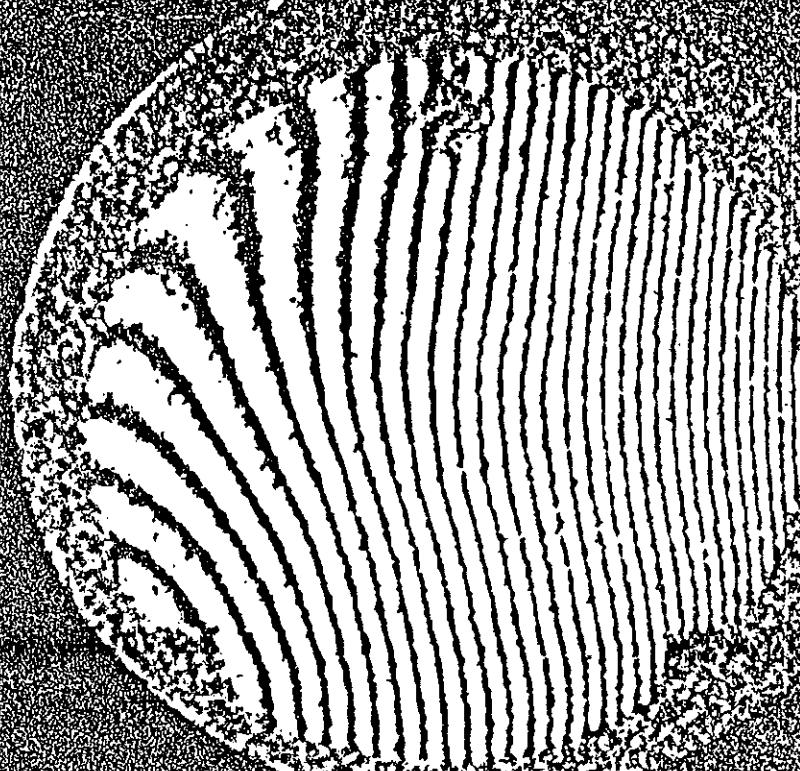


FIGURE 9

Corr #2 No Forces 08.29.44 08-02-92 TF

Rms: 0.747 P-V: 4.347

■ 1.218
■ 0.928
■ 0.638
■ 0.348
■ 0.058
■ -0.232
■ -0.522
■ -0.813
■ -1.103
■ -1.393
■ -1.683
■ -1.973
■ -2.263
■ -2.553
■ -2.843

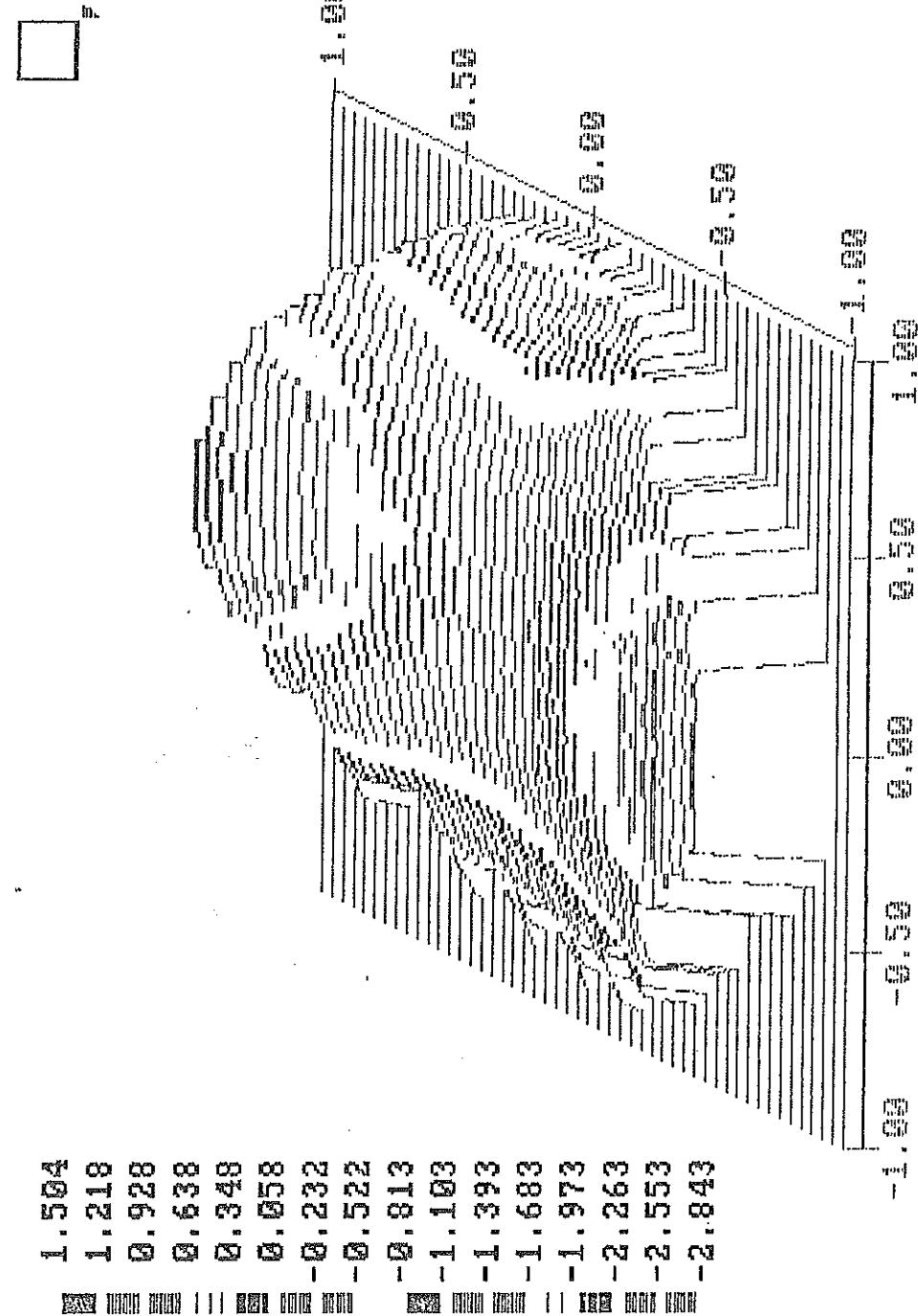
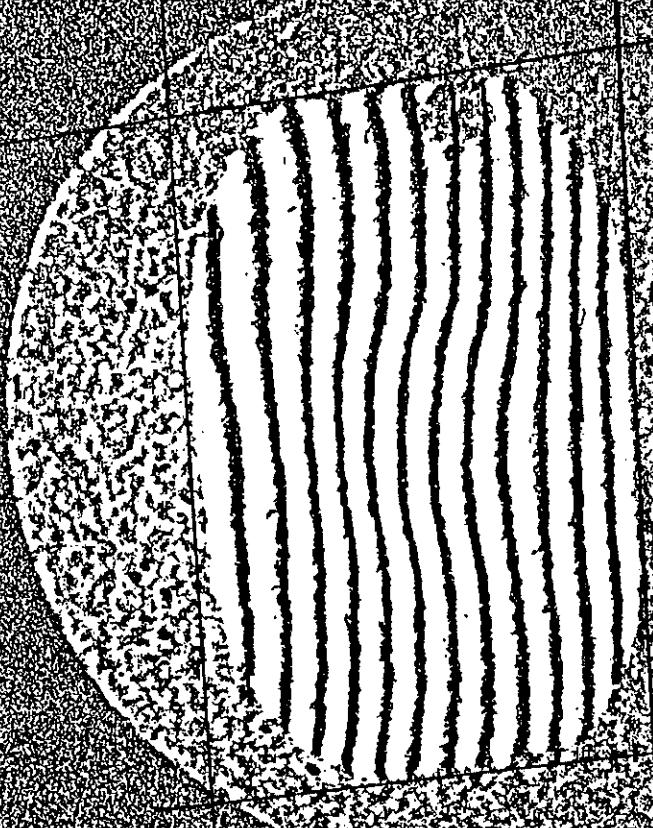


FIGURE 11



#1 Forces Applied 7-20-92
Masked C.A.

Corr. #2

File:
CR2FACAI

Final 10° Lean Test

Best Representative

Forces applied.

P-V = .657

FIGURE 17

Burr2 Cell Forces CA 06:42:40 07-19-92
OPD data

TERM	RMS FIT	COEFFICIENTS			
TILT	0.103	-4.9845	-0.0879		
FOCUS	0.097	-4.9897	-0.0828	0.0653	
SEIDEL	0.060	-4.8561	-0.0994	-0.1466	-0.3611
		0.0887	0.1285	-0.0331	-0.1432

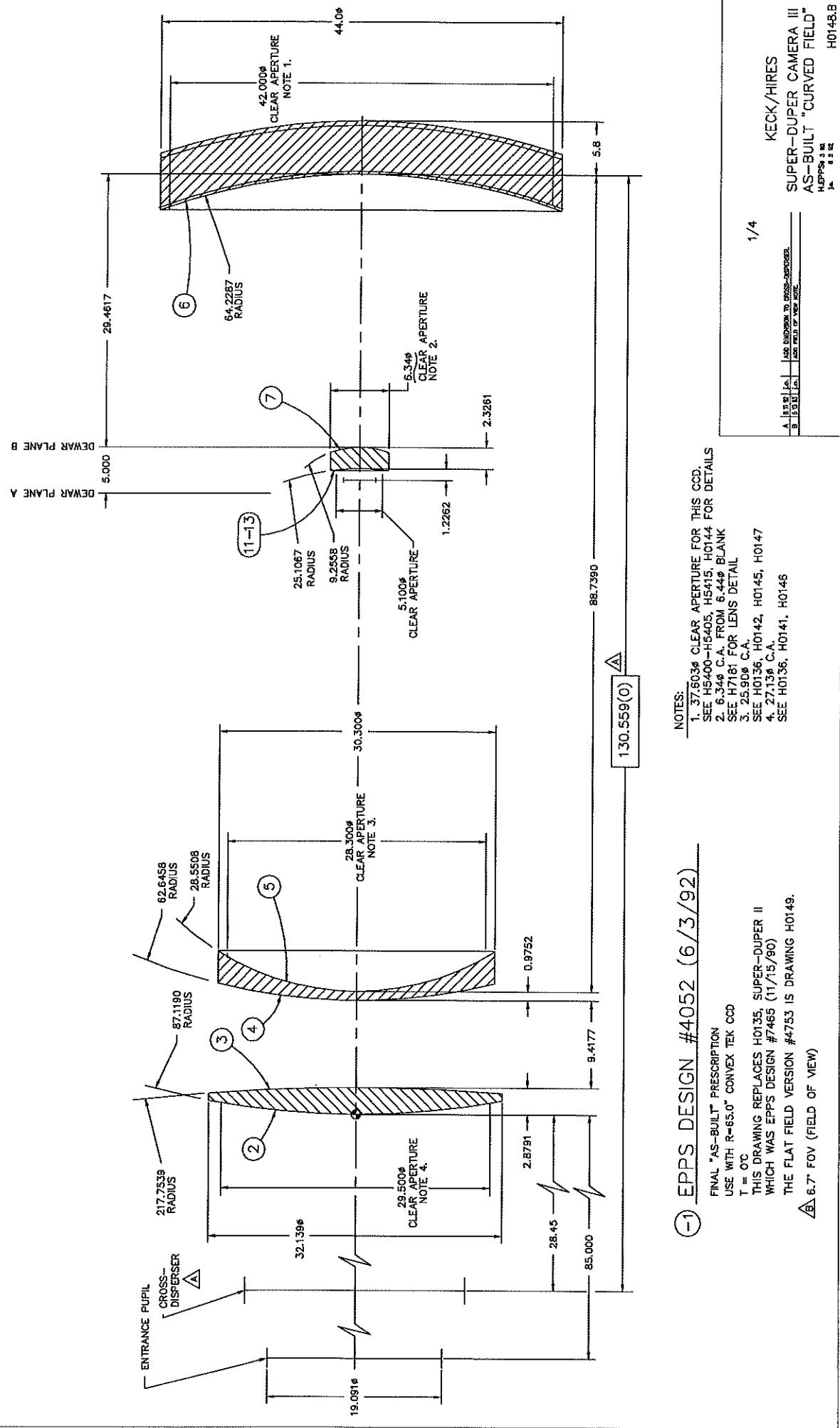
	AMT	ANGLE
TILT	5.113	180.4
FOCUS	0.194	
ASTIG	0.744	83.1
COMA	0.398	-14.5
SAG	-0.859	

TERMS REMOVED: TILT FOCUS

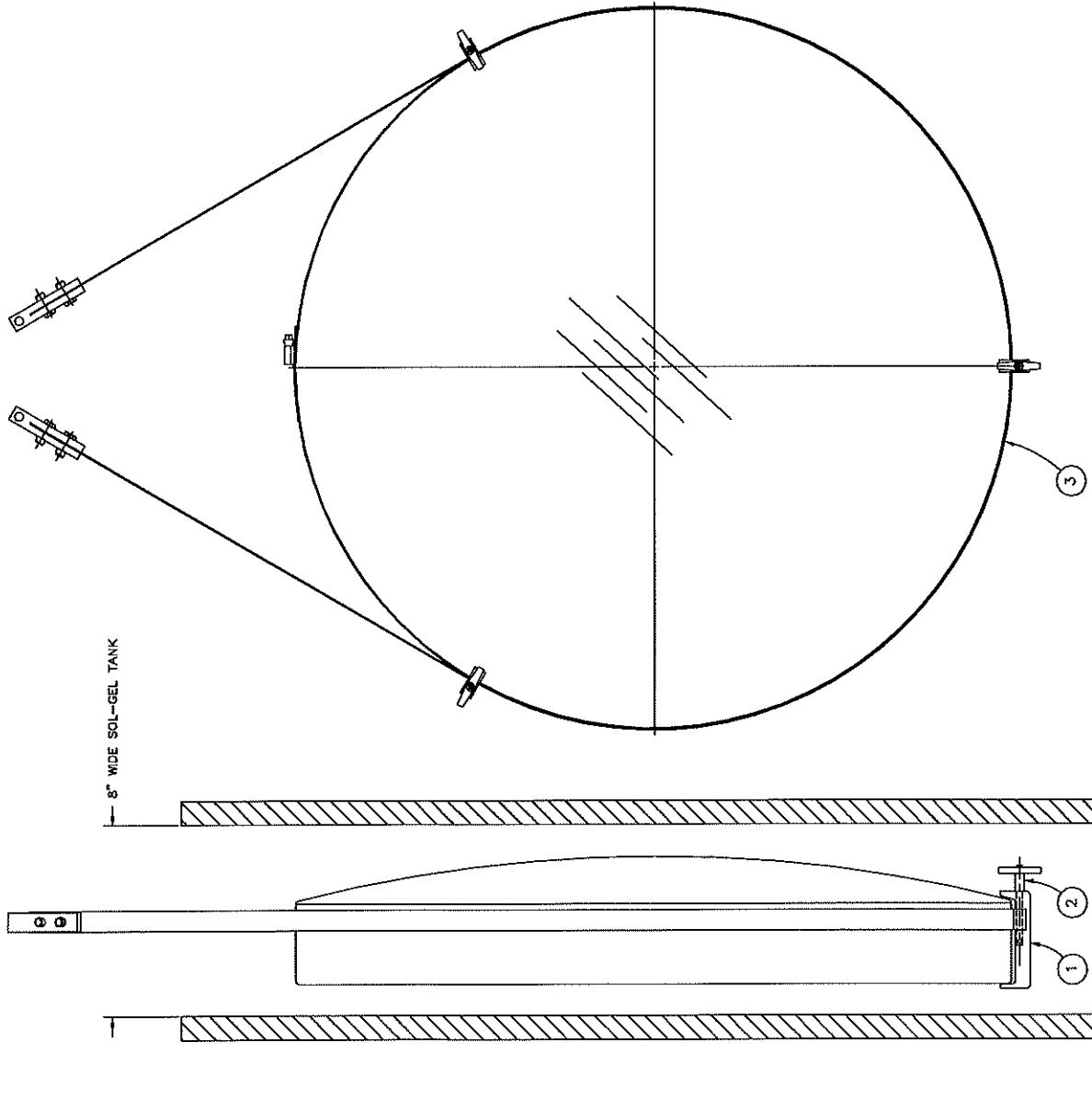
x center y center radius
50.00 50.00 39.81

DATA PTS	WEDGE	PEAK	VALLEY	P-V	RMS	STREHL RATIO
3096	0.50	0.222	-0.427	0.648	0.103	0.656

FIGURE 18



H5000	THIS DRAWING			
H5016	CELL DOOR VS. DEWAR CLEARANCE			
H5100	OPEN			
H5200	CORRECTOR TOOLING H5210 CORRECTOR TOOLS H5214 CAMERA MIRROR TOOLS			
H5300	CORRECTOR CELLS AND SUPPORT STRUCTURE H5303 CORRECTOR BASE ASSY (ISOMETRIC) H5314 CORRECTOR CELL TOOLING L/O H5324 MENISCUS CORRECTOR CELL ASSY H5328 CORRECTOR CELL AXIAL SUPPORT ASSY'S H5324 MEN. CORR. DOOR ASSY. H5349 BICONVEX CORRECTOR CELL ASSY H5363 CORRECTOR CELL ASSEMBLY FIXTURE H5370 BI-FOLDING DOOR ASSEMBLY	(H4532 CROSS DISPERSER DOORS VS BIFOLDING DOOR CLEARANCE)		
H5400	CAMERA MIRROR AND TOOLING H5401 CAMERA MIRROR L/O H5409 CAMERA MIRROR SUPPORT ASSY (FOR STRASBAUGH) H5411 CAMERA MIRROR 25" LAP (STRASBAUGH) H5412 CAM. MIRROR 30" LAP (STRASBAUGH) H5415 CAM. MIRROR (HEXTEK DRAWING) H5420 BEAM1 RAYTRACE AT CAMERA PLANE H5421 BEAM1 RAYTRACE OF TELESCOPE AND HIRES			
H5500	CAMERA MIRROR CELL H5530 CAM. MIRROR RADIAL SUPPORT ASSY. H5536 CAM. MIRROR AXIAL SUPPORT ASSY. H5539 CAM. MIRROR TANGENT LINK ASSY. H5541 CAM. MIRROR CELL SPIDER ASSY. H550 CAM. MIRROR DOOR ASSY.			
H5600	FIELD FLATTENER H56XX	KECK/HIRES SUPERCAMERA LOCATING TREE H5000		



1/2

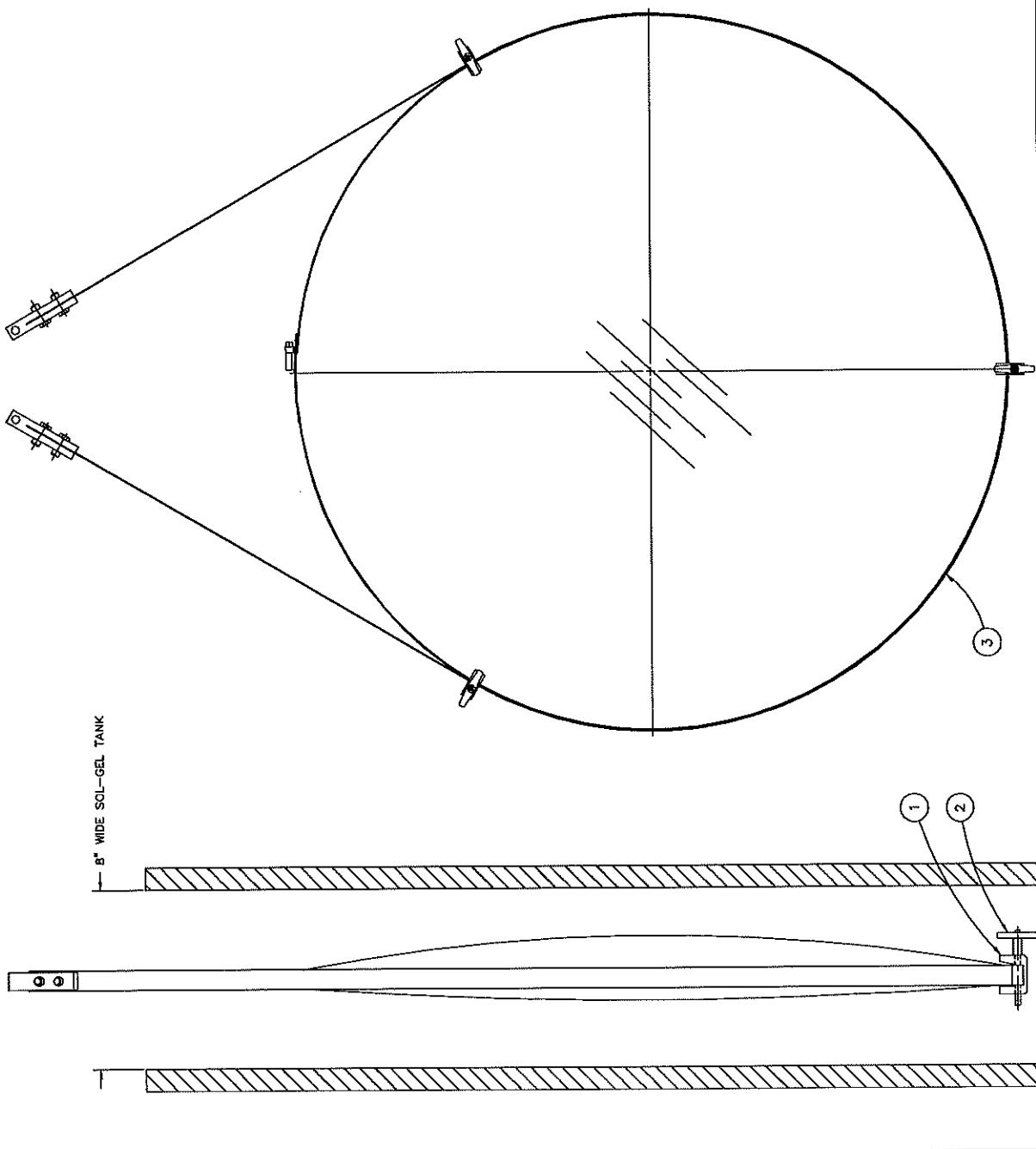
KECK/HIRES
MENISCUS LENS
SLING ASSEMBLY
H5266
B.L.A. 6/17/92
C.A.C. 6/20/92

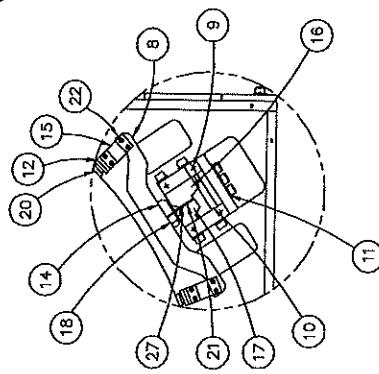
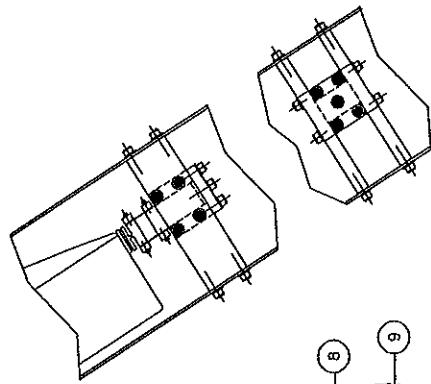
H5266-1	3	1	SLING MEDIUM	ISS
	2	3	FIN	ISS
H5266-2	1	3	LOCKING CLIP	ISS

H5266

H5657
 KECK/HIRES
 BICONVEX LENS
 SLING ASSEMBLY
 8/23/92
 D.A.C.
 E.C.M.
 S.S.T.
 S.S.T.
 ALUM

1/2





SCALE 1/2
DOOR & BACK PLATE REMOVED
FOR CLARITY

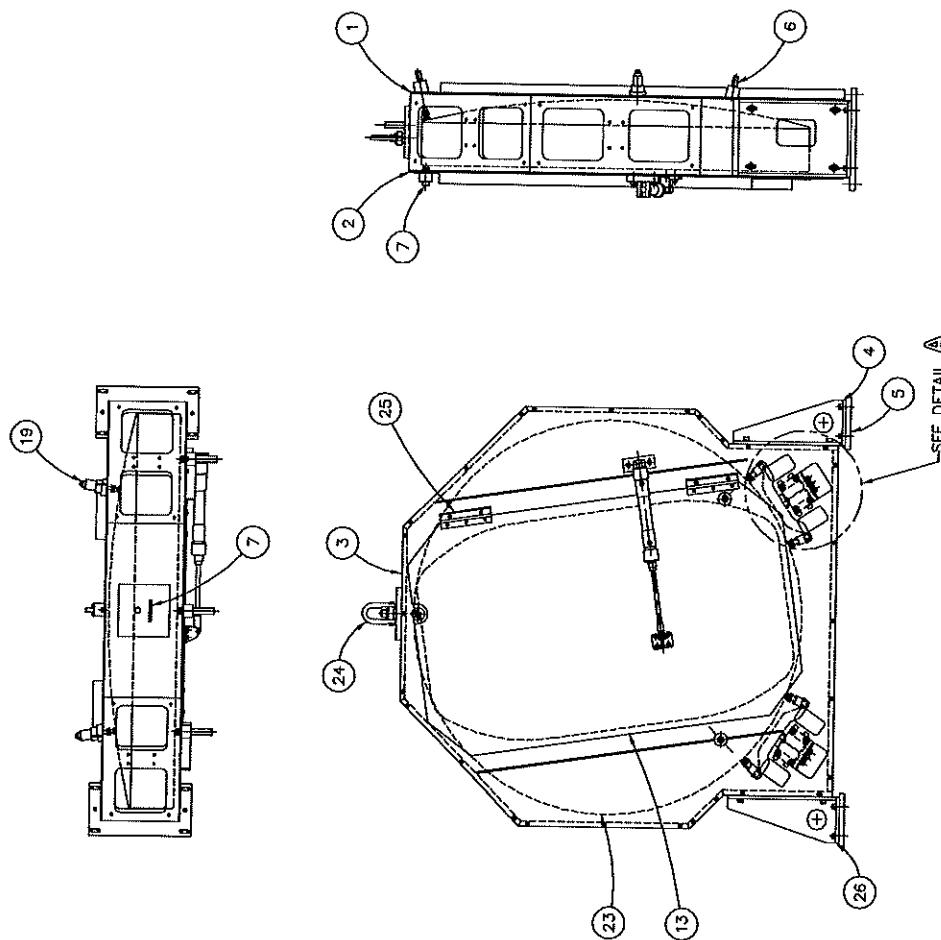
DETAIL A

REF.	QTY	DESCRIPTION	STL.
H5335-9	27	2 SPACER	STL.
H5334-2	26	1 CELL MOUNT WELD	STL.
-	25	2 SPACER	STL.
-	24	1 HINGE RING	STL.
H50152-1	23	1 MENISCUS CORRECT. LENS	STL.
H5335-1	22	4 CLAMP	STL.
H5335-5	21	3 SPRING	STL.
H5335-6	20	4 TENS CONTACT PAD	STL.
H5335-7	19	3 TENS POINT	STL.
H5335-8	18	3 CLAMP	STL.
H5335-9	16	2 CLAMP	STL.
H5335-2	15	3 SPRING	STL.
H5335-5	14	2 CLAMP	STL.
		1 BACK PLATE WELD.	STL.
		1 FRONT PLATE WELD.	STL.
		1 DOOR PLATE	STL.
		1 DOOR PLATE WELD.	STL.
		1 SIDE PLATE	STL.
		1 SIDE PLATE WELD.	STL.
		1 REAR PLATE	STL.
		1 REAR PLATE WELD.	STL.
		1 RETAINING POINT	STL.
		1 DEFINING POINT	STL.
		5 MOUNTING PLATE	STL.
		4 CELL MOUNT WELD.	STL.
		3 SIDE PLATE CIRCLIP	STL.
		2 BACK PLATE CIRCLIP	STL.
		1 FRONT PLATE CIRCLIP	STL.

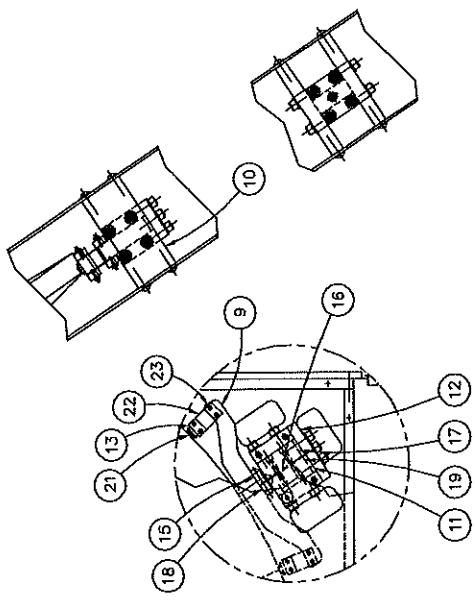
1/4

KECK/HIRES
MENISCUS CORRECTOR CELL
ASSEMBLY

S.L.D. 10/1/91
C.A.D. 11/4/91
H5324.B



<p>(-4) FORCE POINT</p> <p>2 REQ'D - BICONVEX CORRECTOR CELL △ SPRING COMPRESSION = 0.540" DESIGN FORCE = 6.0 LBS.</p>
<p>(-3) FORCE POINT</p> <p>2 REQ'D - MENISCUS CORRECTOR CELL SPRING COMPRESSION = 0.542" DESIGN FORCE = 11.5 LBS. △</p>
<p>(-2) RETAINING POINT</p> <p>4 REQ'D - MENISCUS CORRECTOR CELL 4 REQ'D - BICONVEX CORRECTOR CELL △</p>
<p>(-1) DEFINING POINT</p> <p>3 REQ'D - MENISCUS CORRECTOR CELL 3 REQ'D - BICONVEX CORRECTOR CELL △</p>
<p>2/1</p> <p>KECK/HIRES CORRECTOR CELL AXIAL SUPPORT DETAILS S.D.C. 8/17/81 C.A.S. 8/17/81 H5328.C</p>



DETAIL A
SCALE 1/2

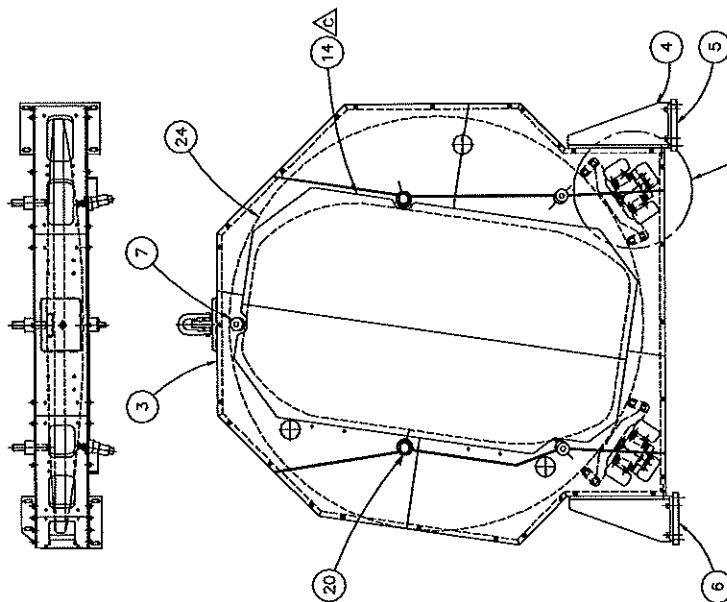
△ H5370-1	125	1	B-FOLDING DOOR
△ H5370-2	25	1	HINGE RING
△ H5370-3	24	1	LENS CORRECT. LENS
△ H5370-4	23	4	CALM
△ H5370-5	22	2	SPRING
△ H5370-6	21	3	CONTACT POINT
△ H5370-7	20	2	SPRING POINT
△ H5370-8	19	2	CALM
△ H5370-9	18	2	SPRING
△ H5370-10	17	2	CALM
△ H5370-11	16	2	SPRING
△ H5370-12	15	2	CALM
△ H5370-13	14	2	CONTACT SUPPORT
△ H5370-14	12	2	BASE PLATE
△ H5370-15	11	4	SIDE PLATE
△ H5370-16	10	2	SCREW, M6
△ H5370-17	9	2	SCREW, M6
△ H5370-18	8	2	RETAINING POINT
△ H5370-19	7	3	DEFINING POINT
△ H5370-20	6	1	MOUNTING PLATE
△ H5370-21	5	1	MOUNTING PLATE
△ H5370-22	4	2	CELL MOUNT WELD.
△ H5370-23	3	1	SCREW, DIRECT CELL WELD.
△ H5370-24	2	1	BACK PLATE, WELD.
△ H5370-25	1	1	FRONT PLATE, WELD.

1/4

KECK/HIRES

BICONVEX CORRECTOR CELL
ASSEMBLY

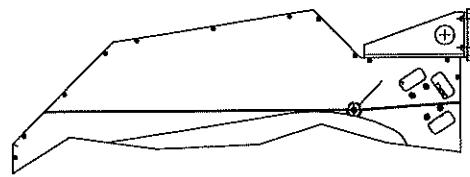
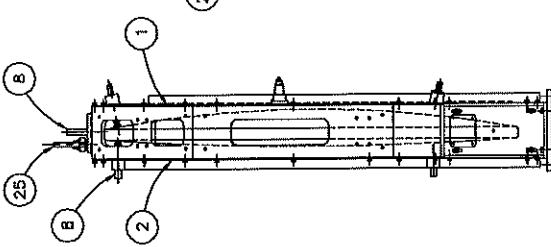
B.C.B. 8 1st
C.A.O. 10 23 Rev
H5349-C

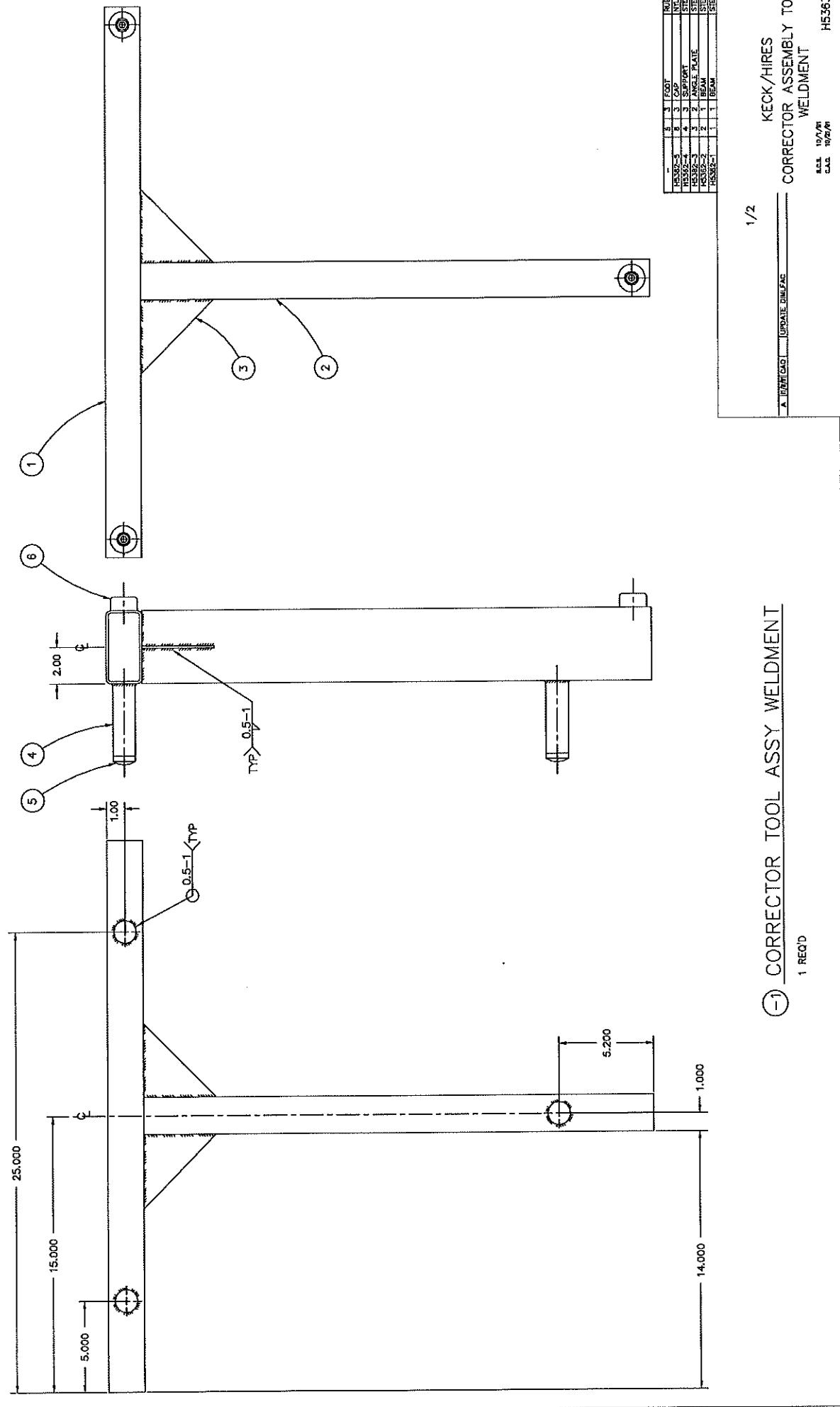


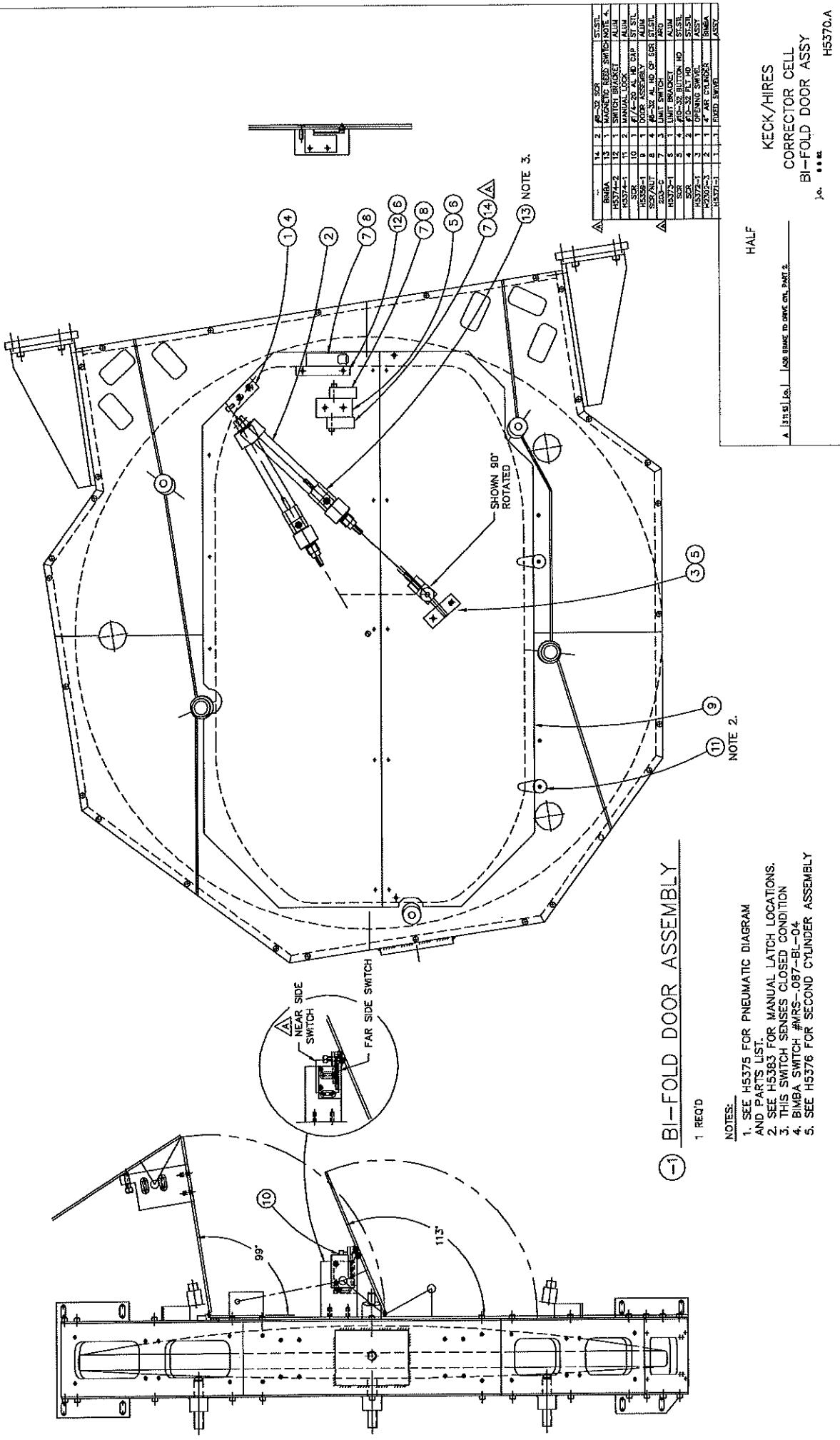
(-1) BICONVEX CORRECTOR LENS CELL

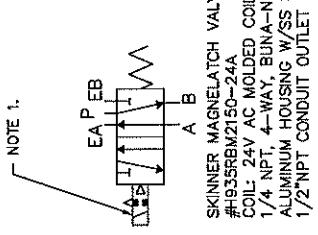
1 REQ'D

- NOTES:
 △ 1. SEE H5370 AND H5376 FOR OPENER
 AND CLOSER DETAIL.
 △ 2. SEE H5383 FOR MANUAL DOOR LATCHES.



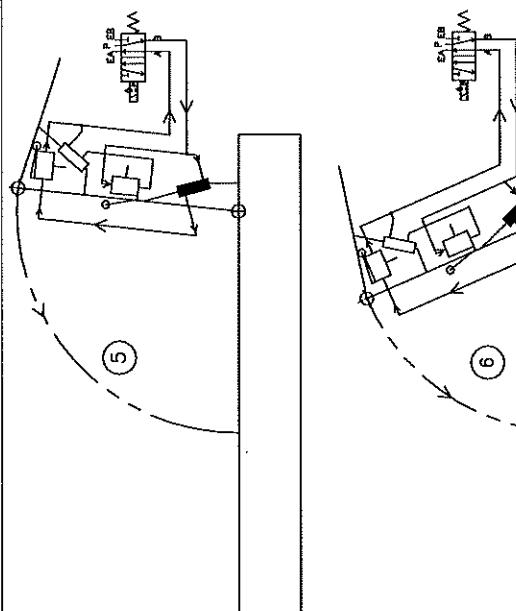






NOTE 1:

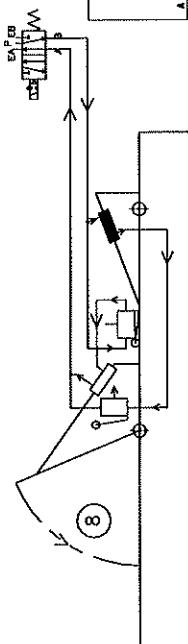
SKINNER MAGNELATCH VALVE
#H35RBM2150-024A
COIL 24V AC MOLDED COIL
1/4 NPT, 4-WAY, BUNA-N SEALS
ALUMINUM HOUSING W/SS SLEEVE
1/2" NPT CONDUIT OUTLET



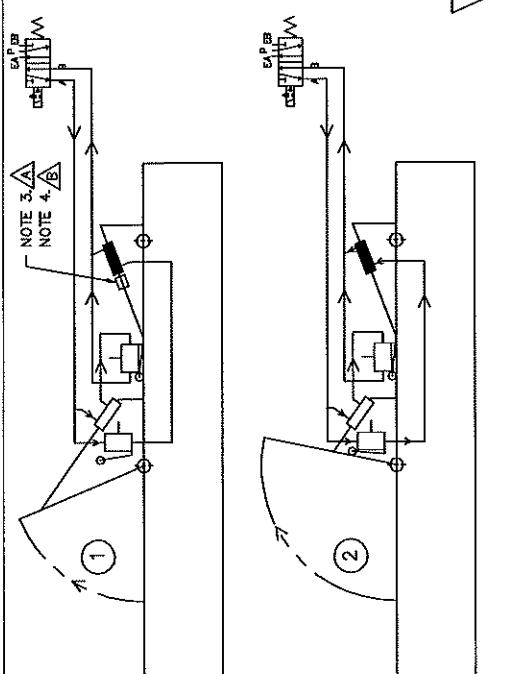
(-1) PNEUMATIC PLAN

OPENING AND CLOSING SEQUENCES
NOTES

1. LATCHING SOLENOID #CV5-LAAM-F24, SKINNER
ELECT-AIR
245 SINCLAIR FRONTAGE RD
MILITAS, CA 95035
(408) 262-2252
- △ 2. ADD ARO SWITCH TO THIS
SWITCH POSITION.
- △ 3. SPRING-OPERATED BRAKE, AIR-RELEASED.
- △ 4. MAIN DRIVE CYLINDER ADDED. BRAKE WAS
NOT NEEDED AND SO WAS REMOVED.

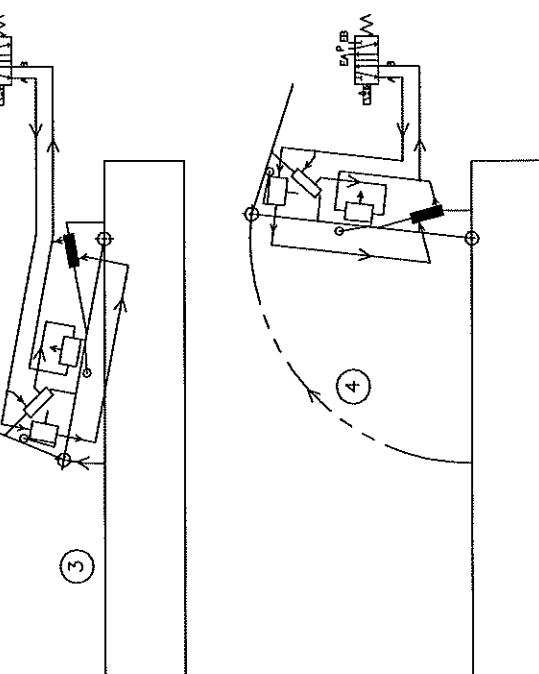


CLOSING SEQUENCE



NOTE 3.△

NOTE 4.△

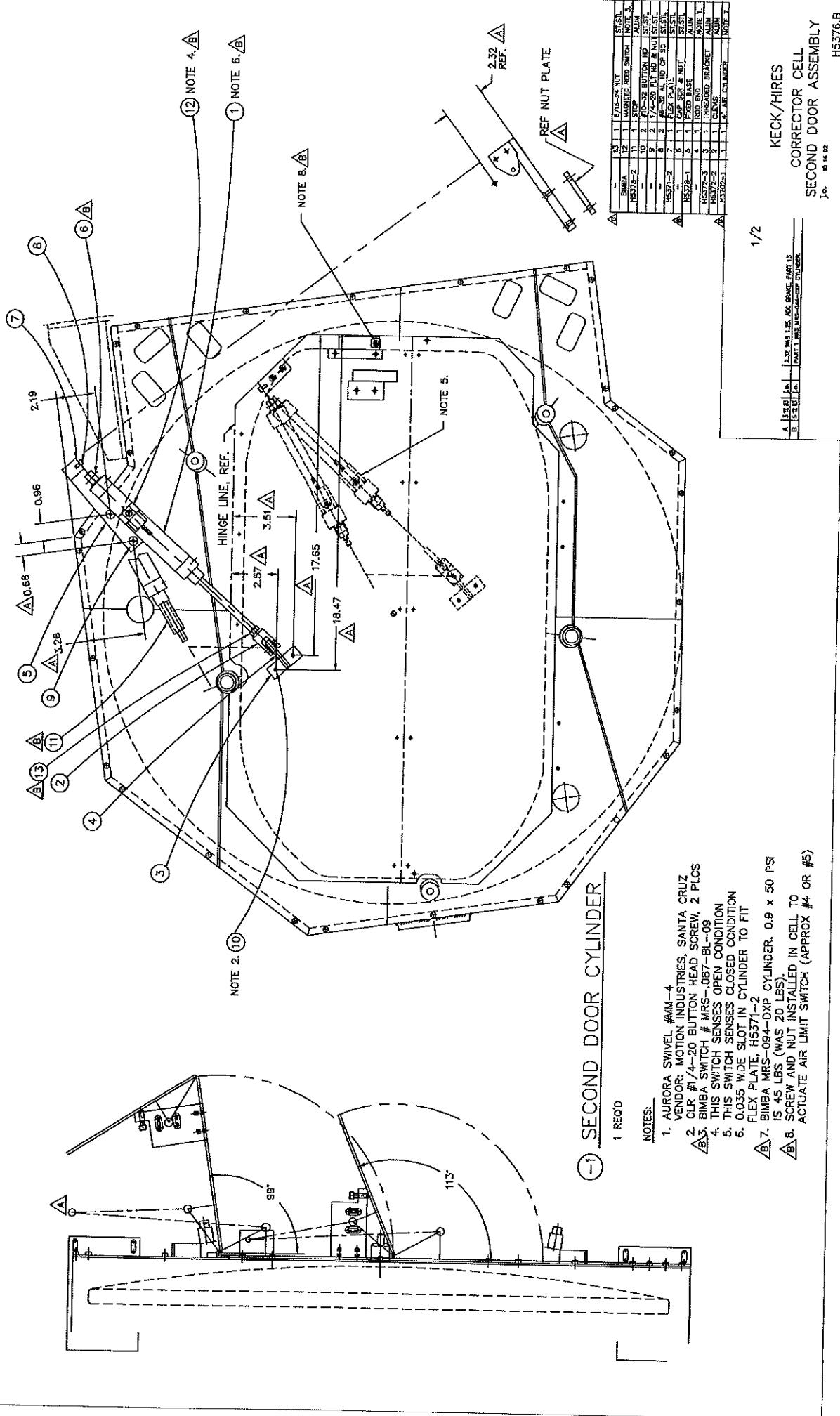


OPENING SEQUENCE

N/A

KECK/HRES
BI-FOLD DOOR
PNEUMATIC PLAN
10/13/92
See Ref 4

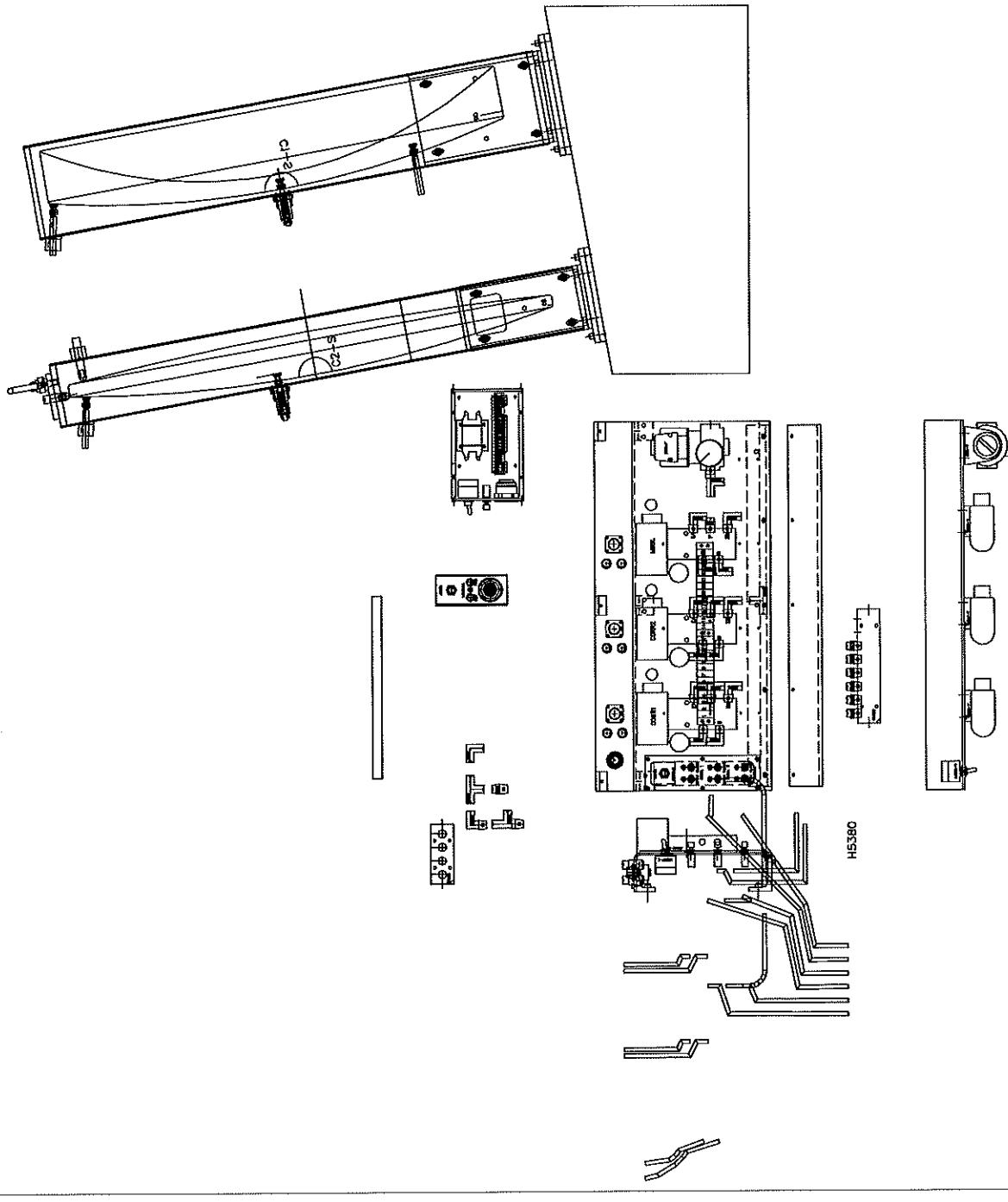
H5375.B

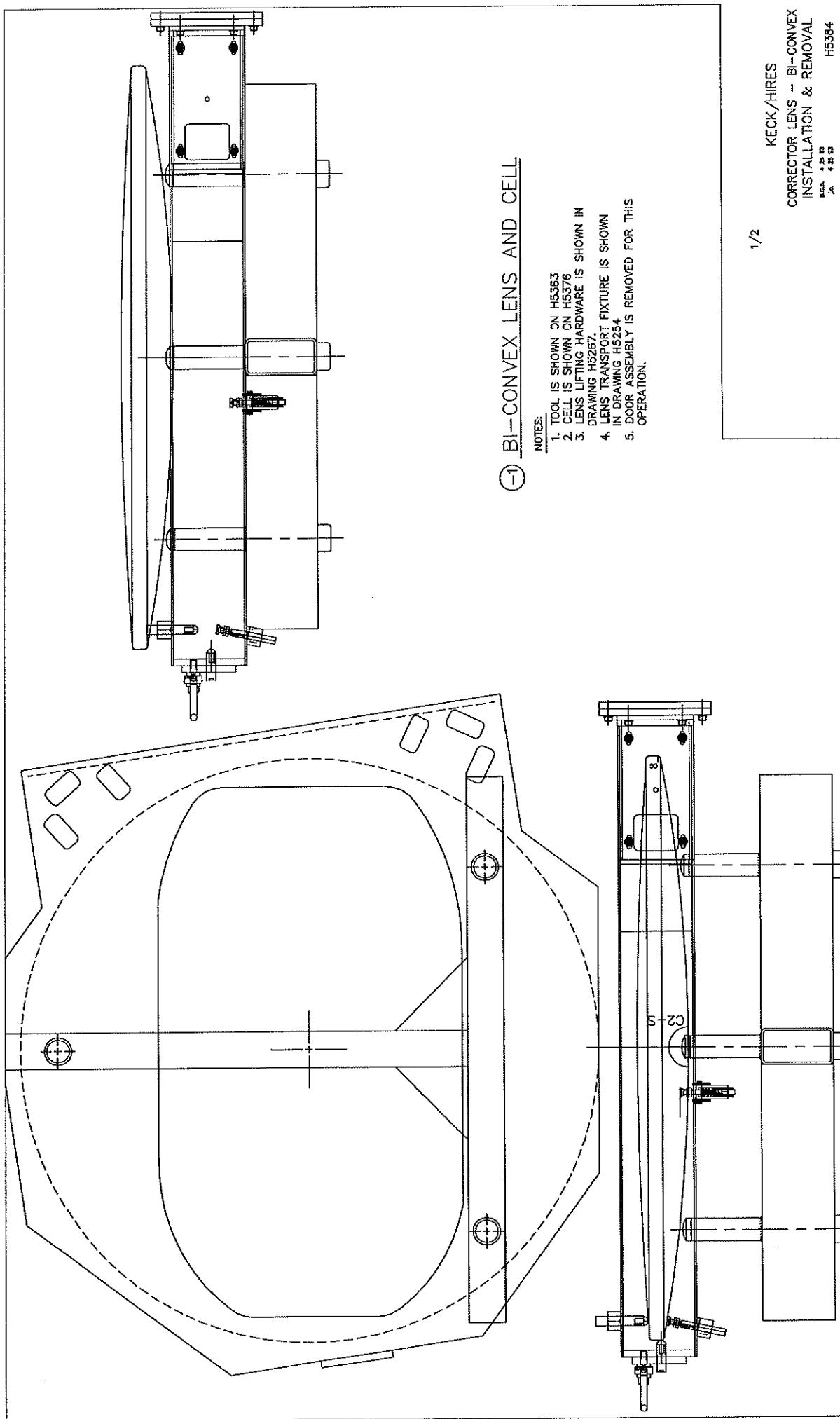


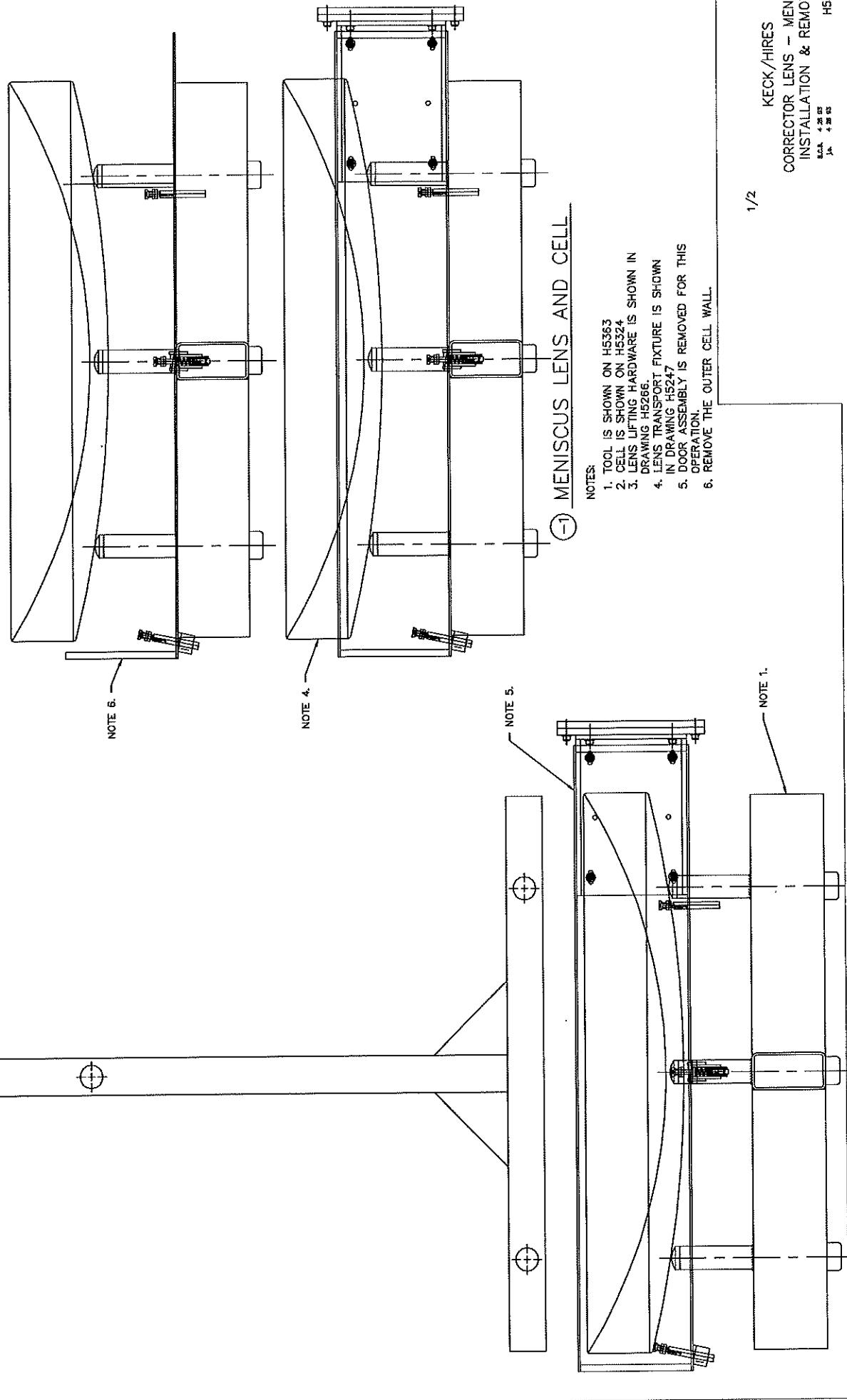
N/A
KECK/HIRES
CORRECTOR CELLS
AIR SYSTEM LAYOUT
H5380.A
BCB 2 18 82

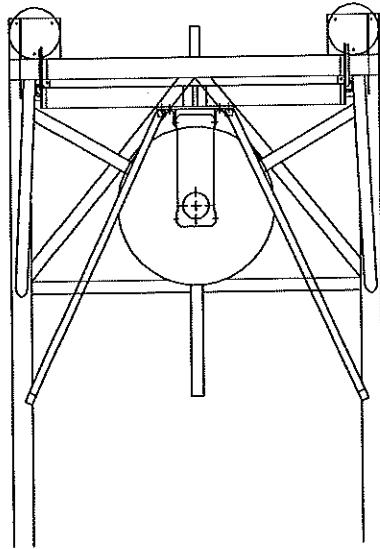
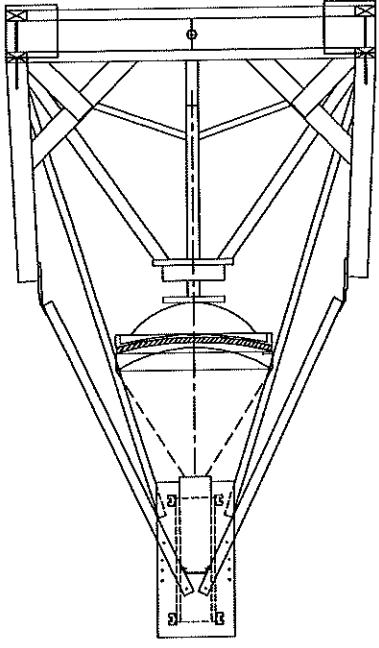
N/A

A [] B [] C [] D [] E [] F [] G [] H [] I [] J [] K [] L [] M [] N [] O [] P [] Q [] R [] S [] T [] U [] V [] W [] X [] Y [] Z []
RECOMMENDED FOR PRINTING



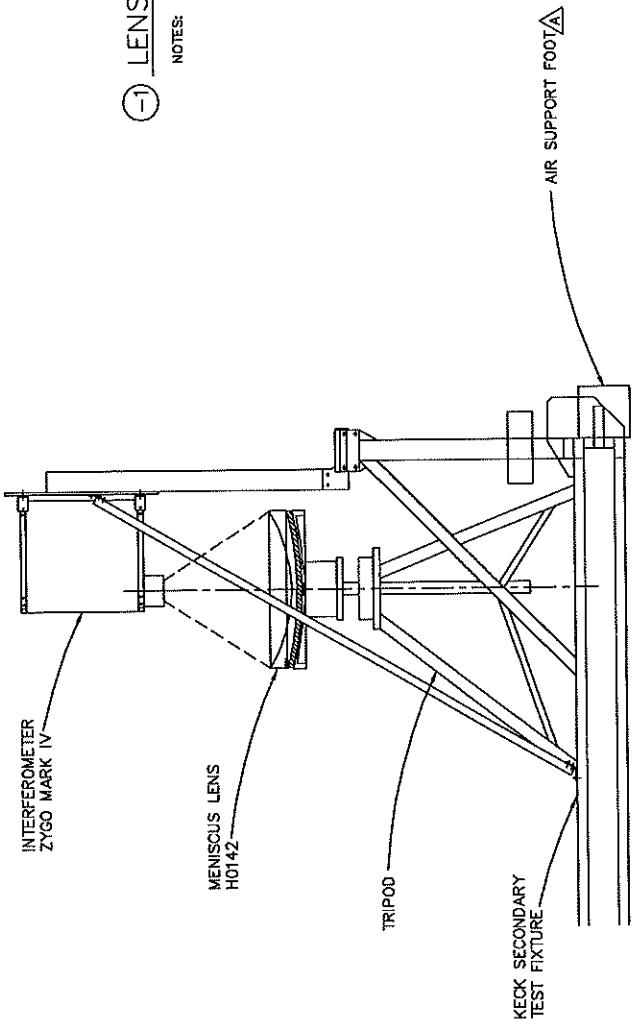




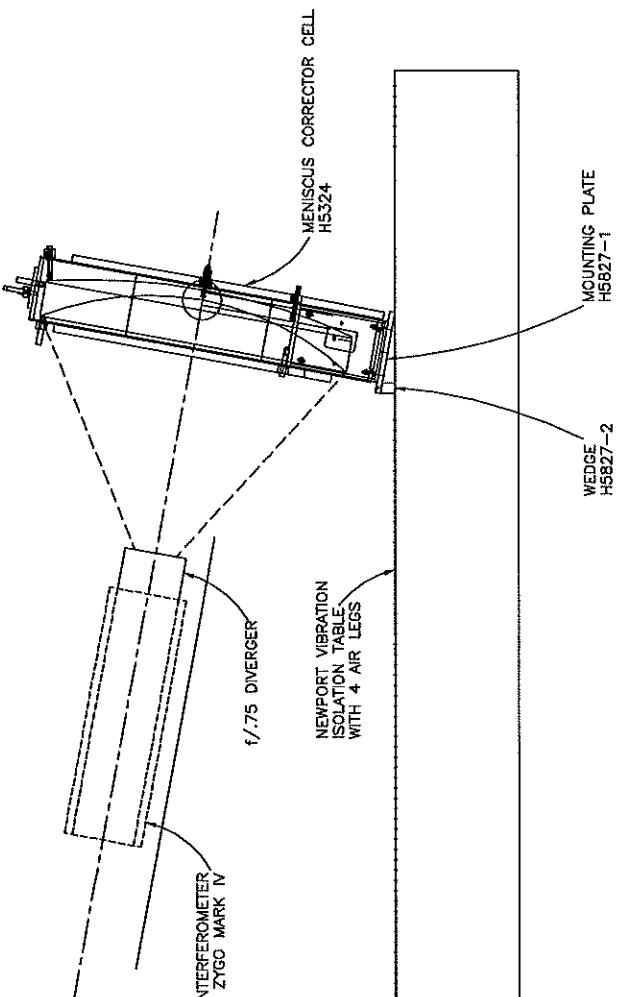


① LENS TEST SETUP

NOTES:



.1
KECK/HIRES
MENISCUS LENS
HORIZONTAL TEST ASSY
B.C.B. 10/24/92
G.A. 8/24/92
H5811.A



.2

KECK/HIRES
MENISCUS LENS
10° TEST SET-UP
E.C.R. 4/1/92
E.D.B. 2/21/92

A [] B [] C [] D [] E [] F [] G [] H [] I [] J [] K [] L [] M [] N [] O [] P [] Q [] R [] S [] T [] U [] V [] W [] X [] Y [] Z []

H5826.A

Appendix W HIRES Control Functions

August 1, 1991

HIRES Control Functions

prepared by Jack Osborne

see /u/jack/text/hires17.cf

1. ECHELLE MOSAIC

- a. 1 DC motor: 50 in-oz minimum, 1000 RPM.
- b. Rotation limits: (2 primary, 2 secondary)
- c. Encoder, linear, to measure 1 micron over 100 mm travel. This represents 0.32 arc-sec least count over about 9° total travel.
- d. An air-operated brake will be required.
- e. An air-operated cover will be used. (2 limits)
- f. Dual fiducials using an interrupter in conjunction with stage encoder index track.
- g. 'Brake off' is a special privilege requiring a password.

2. CROSS-DISPERSER (Uses a Heidenhain rotary incremental encoder with 3 arc-sec resolution: 10,800 lines with x40 logic = 432,000)

- a. 1 DC motor: 50 in-oz min, 1000 RPM.
- b. Encoder interface to Heidenhain electronics driver box is required.
- c. Limits for about 30 degrees rotation. (2 primary and 2 secondary)
- d. An air-operated, *user-optional* brake will be required.
- e. An air-operated cover will be used. (2 limits)
- f. A dual fiducial system like "ECHELLE MOSAIC" above.

3. CAMERA FOCUS (aka DEWAR FOCUS or DETECTOR FOCUS)

- a. 1 DC motor: 50 in-oz min, 1000 RPM.
- b. Linear encoder: position to 5 microns with 140 mm travel.
- c. Limits: 2 primary and 2 secondary.
- d. Electric-operated fail-safe brakes will be required.
- e. A dual fiducial system like "ECHELLE MOSAIC" above.

4. SHUTTER
 - a. Opening solenoid (spring return, so that shutter closes on power failure).
 - b. Magnetic reed switch sensing open condition.
5. DARK SLIDE
 - a. A slow-moving dark cover for the detector. Air cylinder, 2 limits.
6. FILTERS (2 wheels of 12 positions right behind the slit)
 - a. 2 DC motors: 50 in-oz min, 1000 RPM.
 - b. No limits.
 - c. Encode 12 positions each wheel using motor encoder.
 - d. A fiducial will be used for indexing each wheel.
7. TV Guiding
 - a. Focus: DC motor: 50 in-oz min, 1000 RPM.
 - b. Encode using motor encoder.
 - c. Limits: 2 primary and 2 secondary.
 - d. Two filter wheels of 8 positions each. See "FILTERS" above.
 - e. Aperture adjuster: DC motor and f-stop ring; 50 in-oz min, 1000 RPM.
 - f. Shutter (Ilex or equivalent)
8. COMPARISON LIGHTS
 - a. 6-position linear stage; 6" each, moves parallel to slit jaws.
 - b. 1 DC motor: 50 in-oz min, 1000 RPM.
 - c. Limits: 2 primary and 2 secondary.
 - d. Motor encoder is used for each of 6 positions.
 - e. A mirror is positioned by an air cylinder. (2 limits)
9. DECKER SLIDE (aka SLIT ACCESSORY SERVER)
 - a. 4 positions, linear motion.
 - b. DC motor is 50 in-oz min., 1000 RPM.
 - c. Encoding is done by using motor encoder.
 - d. Limits: 2 primary and 2 secondary
10. SLIT
 - a. A bilateral slit will be used. (both sides move, keeping the center fixed)
 - b. DC motor: 50 in-oz min, 1000 RPM.
 - c. Limits: 2 primary and 2 secondary.
 - d. Encoding will measure 0.001" (0.03 arc-sec on the sky), over 0.25" travel (each jaw). (Use the motor encoder to do this.)
11. CAMERA MIRROR (44" diameter) will be stationary.
 - a. Air-operated cover. (2 limits)

12. COLLIMATOR MIRRORS (17" diameter) Red and Blue.

- a. 2-position stage uses 2 air cylinders. Either Red or Blue mirror is "in".
- b. Limits: 2.
- c. Air-operated clamp will be required.
- d. Air-operated covers. (2 limits per cover = 4 limits total) Covers are interlocked, in software, with stage limits so that a cover may be opened only for the "in" position mirror.
- e. User with special password may open mirror cover which is in "out" position.
- f. Focus will be required. (1" travel)
 - DC motor: 50 in-oz min, 1000 RPM.
 - Encoding is via the motor encoder.
 - Limits: 2 primary and 2 secondary.
 - Electric-operated fail-safe brakes will be required.

13. ACCESS PORT TO SEAL HIRES (Near the slit)

- a. Air-operated hinged door. (2 limits)

14. FILTER WHEEL / COMPARISON LIGHTS

- a. 1 DC motor: 50 in-oz min, 1000 RPM.
- b. No limits.
- c. Encode 12 positions with motor encoder.
- d. A fiducial will be used for indexing.

15. LOCAL LOCKOUT

- a. This feature prevents injury to a worker inside the instrument. Lockout status will be visible at all control stations.
 - b. The lockout switch is located at the spectrograph and its status should be obvious.
-

NOTES:

1. Dual fiducials: All moving stations have at least 1 fiducial (an optical interrupter) At power-up this indicates which side of center you are. Coarse center-locating is done with this fiducial. A fine fiducial comes with the encoders on the three stages which use separate encoders. Position information then comes from the incremental encoder.
2. The "core" HIRES has no field rotator and so has no need for an X-Y stage for the TV.
3. The TV zoom function will be by "pixel-binning".

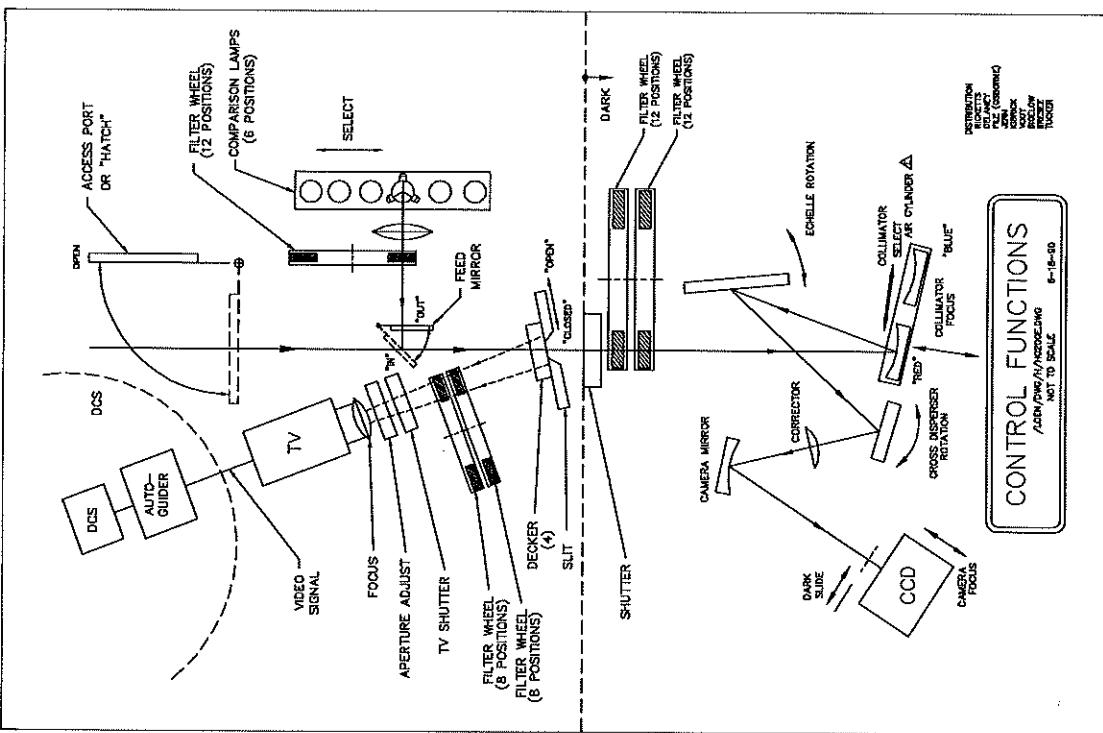
4. The 6 comparison source positions could each have up to 3 separate lamps (and lamp controllers). The stage will scan a single source along the slit or be driven to one of 6 pre-selected positions.
5. Limits: Secondary (heavy duty) will be the Microswitch DTE6 enclosed style. All others will be the low current Microswitch BZ style (we usually call these "logic" limits).

= modified or new items since last change.

See also drawing # H0200.E .

Distribution: HIRES file (Osborne)

Vogt
Bigelow
Jern
Kibrick
Ricketts
Delaney
Tucker
Bresee



NONE

KECK/HIRES CONTROL FUNCTIONS SCHEMATIC

HO200.E

UPGRADE SLOW SPECTROGRAPH INTERNAL,
MEMORY FOR SURFACE DESIGN REVIEW
MICROTEK SURFACE DESIGN REVIEW
REDUCE CAMERA CORRECTOR CHANGES
OPEN TV FILTERS MORE <10%
OPEN TV FILTERS WAS 1/2 SP. ACTIVE

1a

0 15 30

Appendix X Glossary

1. TPI is threads per inch, referring to screws and nuts. 5 TPI means that each revolution advances the nut 1/5 inch or 0.20" per rev.
2. CCD is charge-coupled device referring to the main HIRES Super-Duper Camera detector or the TV guider photo-sensitive detector
3. FEA is Finite Element Analysis. ANSYS is the software used.
4. OPD is Optical Path Difference.
5. Sol-gel is an anti-reflection lens coating..

